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EUROPEAN CONVENTION FOR CONSTRUCTIONAL STEELWORK
CONVENTION EUROPEENNE DE LA CONSTRUCTION METALLIQUE
EUROPAISCHE KONVENTION FÜR STAHLBAU

ECCS - Advisory Committee 5
Application of Eurocode 3

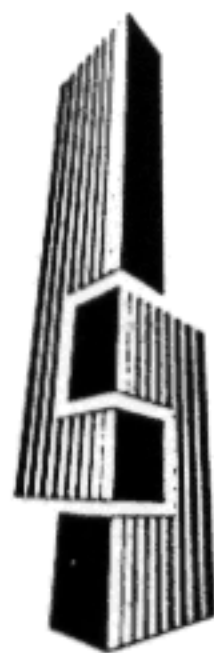
Examples to Eurocode 3

FIRST EDITION

1993

N°71

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Introduction

The European Convention for Structural Steelwork (ECCS) has as one of its primary objectives the promotion of the safe and economical use of steel in structures. ECCS believes that the introduction of the harmonized Eurocodes has a great value in achieving this objective; accordingly ECCS has set up an Advisory Committee, AC 5, charged with the task of promoting the introduction and adoption of the Eurocodes.

The AC 5 Committee have considered how this process could be best achieved and concluded that a three stage approach was desirable. The first stage was to produce a concise version of the Eurocodes which can be used for normal every day design; this part has already been issued as ECCS Publication No. 65.

The second stage is the production of this document which gives design examples to EC 3/1 and E-EC 3 and has been prepared as a design aid to facilitate the use of EC 3/1 for the design of steel buildings during the ENV period. These examples concentrate on those aspects which are likely to be needed for daily practical design work.

The third and final stage will be the production of a series of "Design Aids" which will enable the design process to be made more quickly by using tabulated or graphical values for the various design formula contained in EC 3/1.

The combination of these three documents will enable practising engineers to more easily adopt to the use of the new Eurocodes and should have a beneficial help in their speedy introduction.

Scope

These Design Examples to EC 3/1 and E-EC 3 have been prepared by the ECCS - Advisory Committee AC 5 as a design aid in supplement to the complete EC 3/1 to facilitate the use of EC 3/1 for the design of steel buildings in the ENV-period.

The Design Examples only contains examples to EC 3/1 and E-EC3 that are likely to be needed for daily practical design work.

The γ values used in this document are the values recommended in EC 3 main document. These values may deviate from the values recommended in the National Application Documents (NAD) of the member states.

The ECCS - Advisory Committee 5 is at present composed of the following members:

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Arda, T.S.	Turkey
Bock, H.	United Kingdom
Danieli, S.	Italy
Dowling, P.J.	United Kingdom
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Grotmann, D.	Germany
Taylor, J.C.	United Kingdom

Also particular thanks are given to the ECCS Technical Committees TC 8 and TC 10 who have contributed to the work.

References

- (1) EC 3/1: ENV 1993-1-1 Eurocode 3: Part 1.1
- (2) E-EC 3: Essentials of Eurocode 3 - Design Manual for Steel Structures in Buildings, ECCS-Publication No. 65
- (3) References to EC 3/1 are given in brackets [...]
- (4) References to E-EC 3 are given without brackets

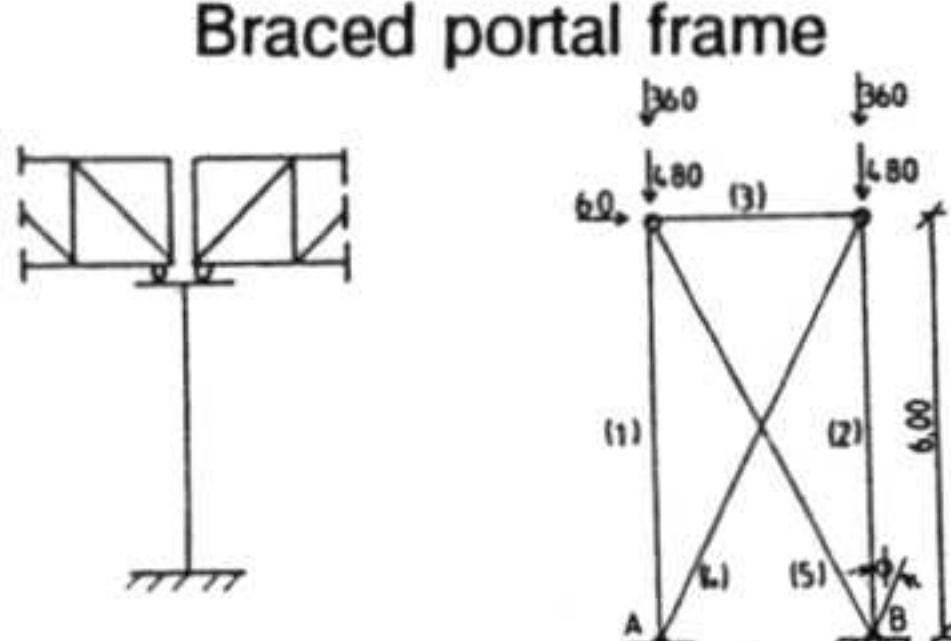
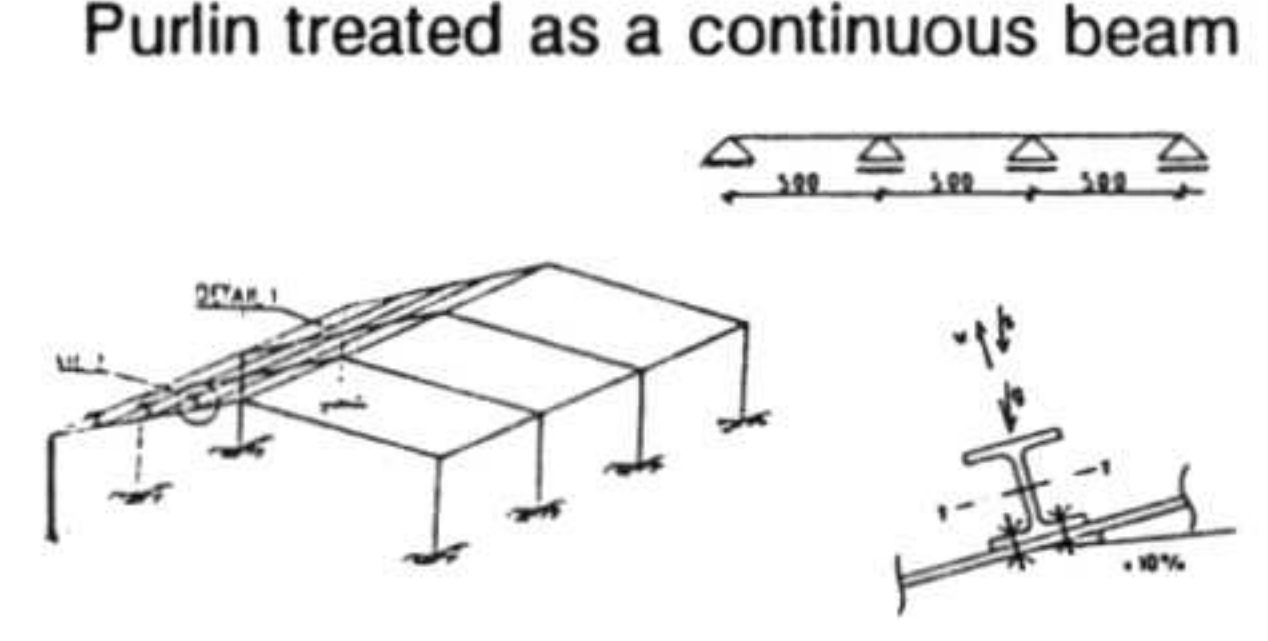
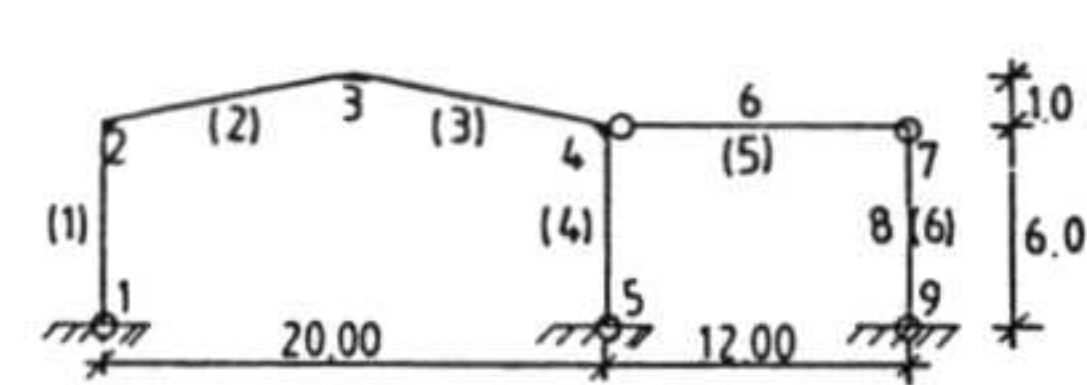
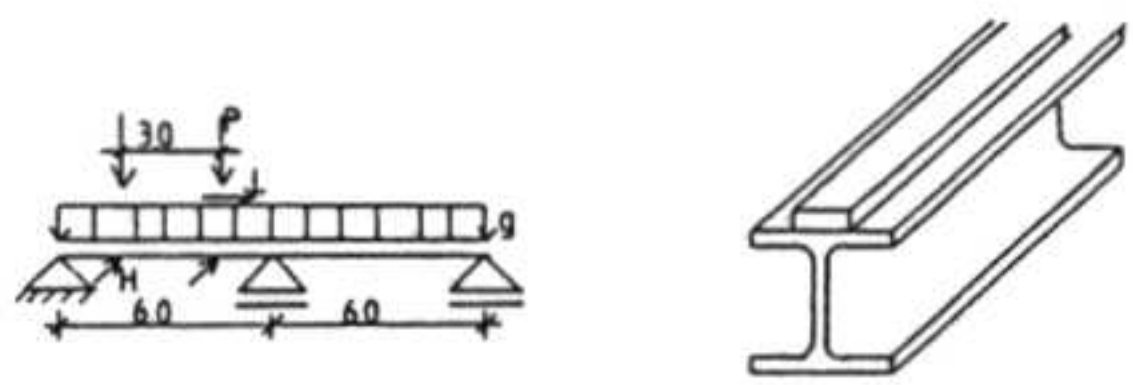
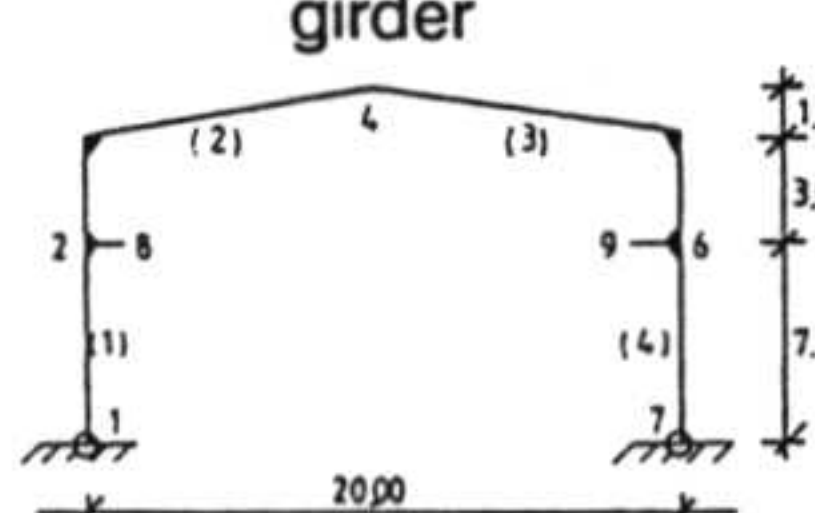
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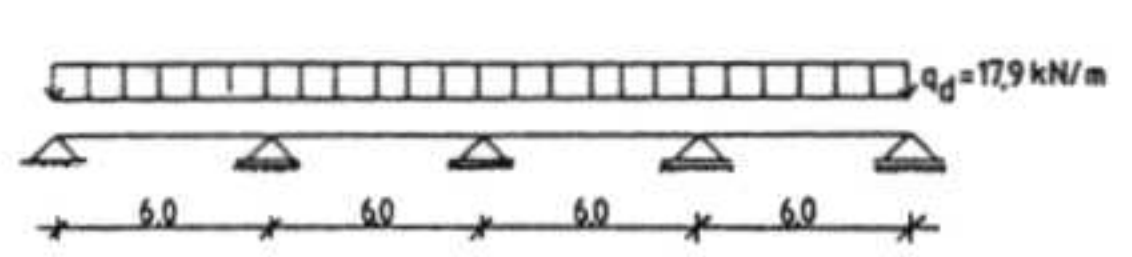
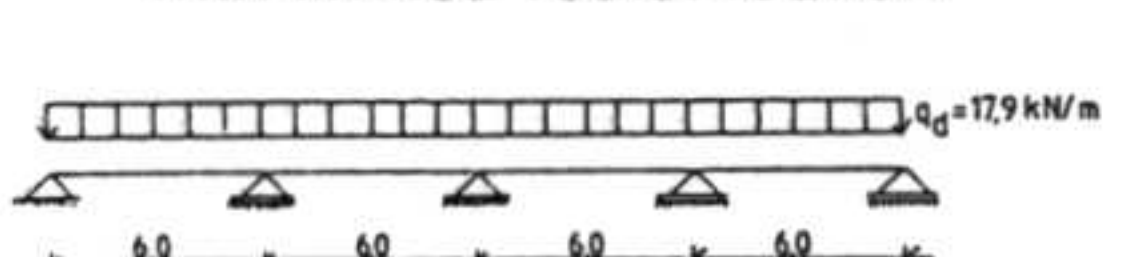
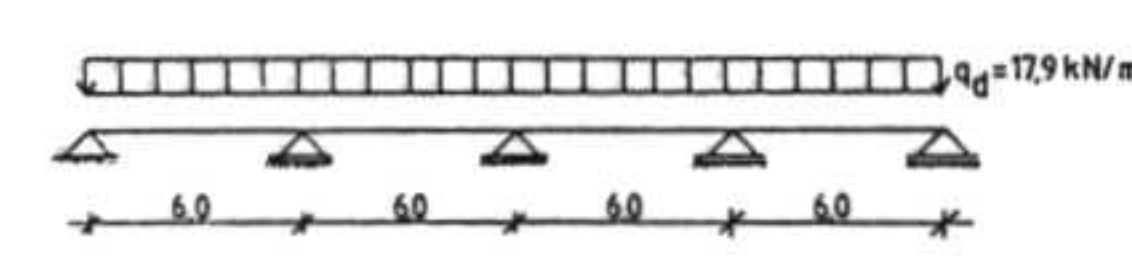
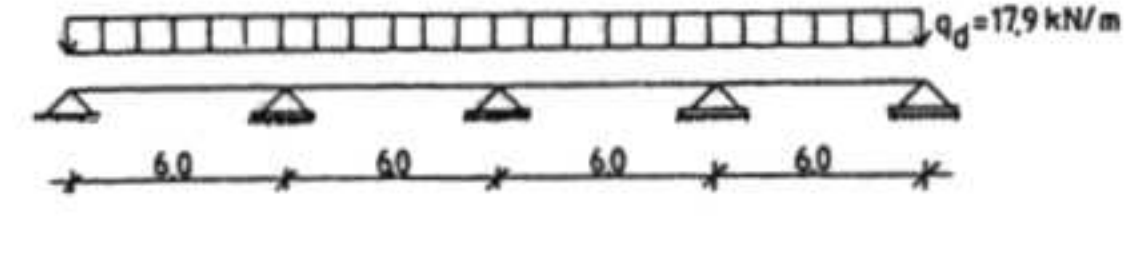
Scope

Part 1 1

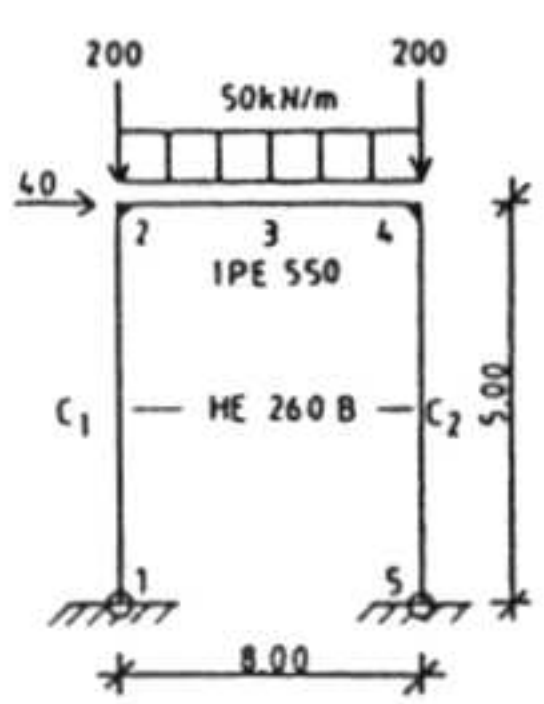
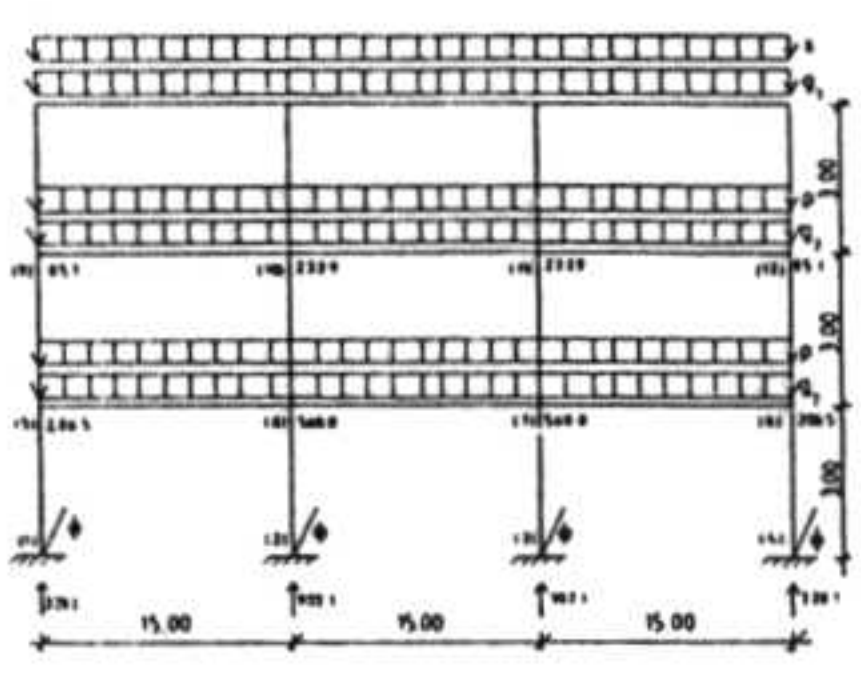
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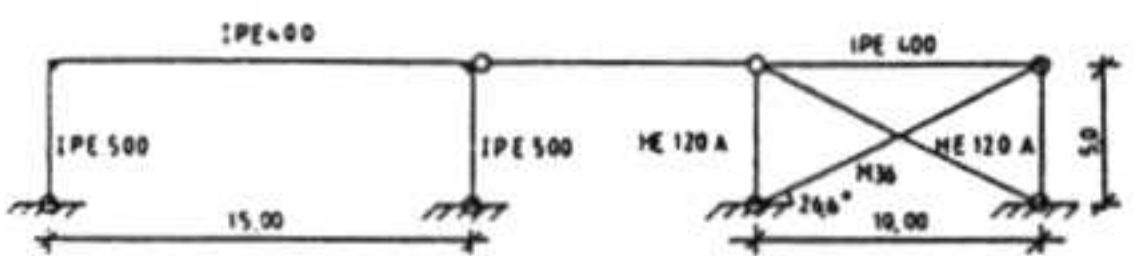
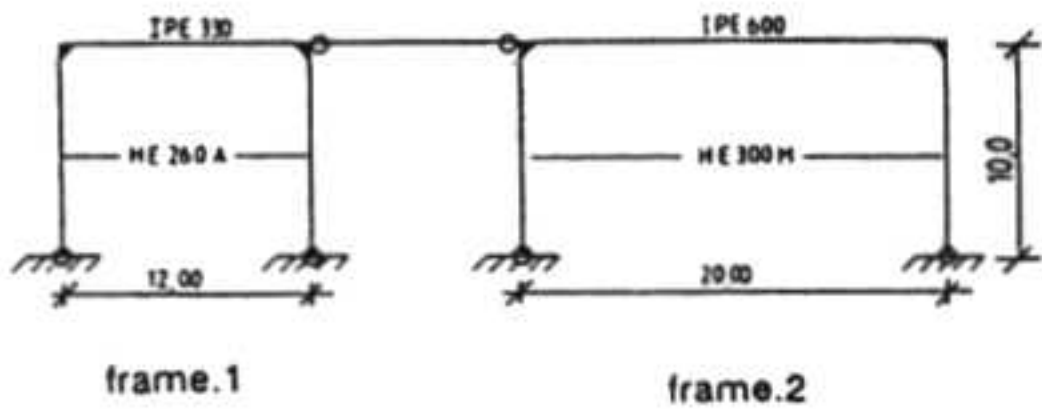
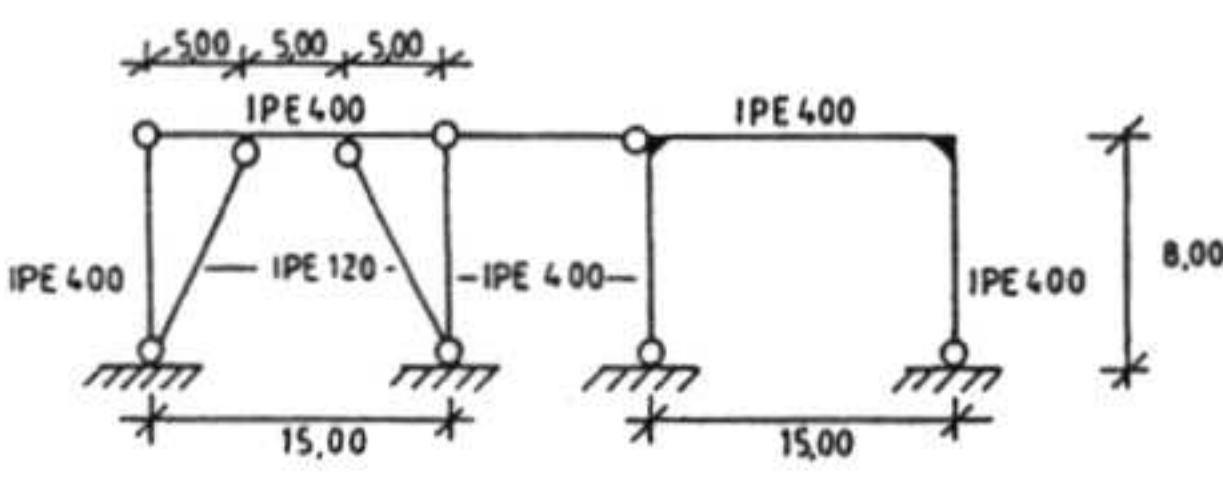
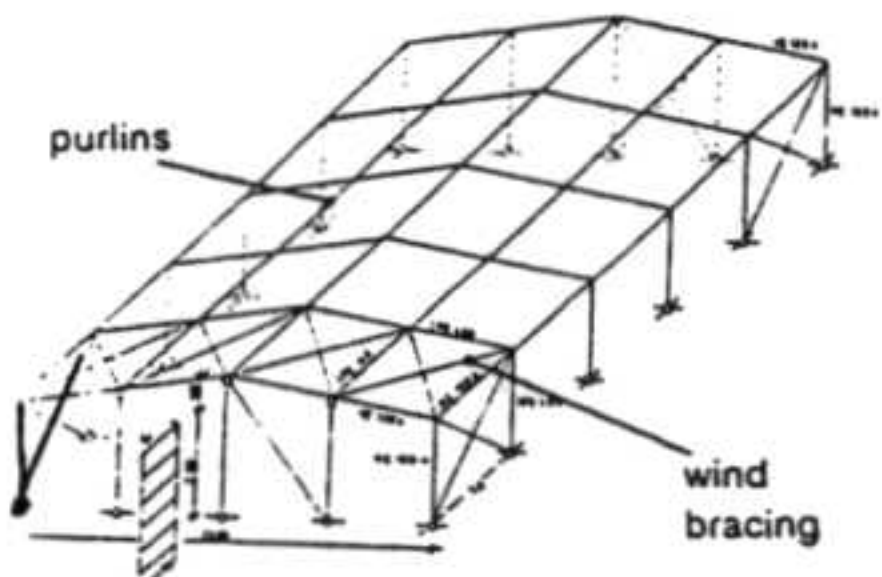
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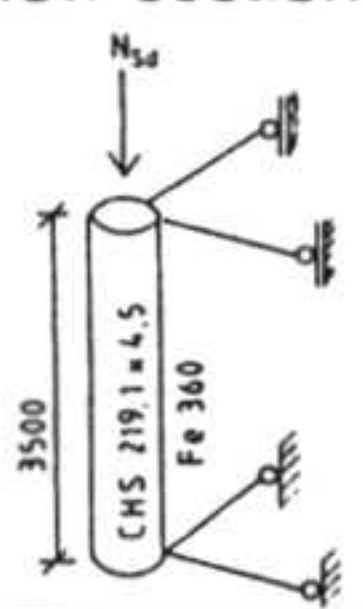
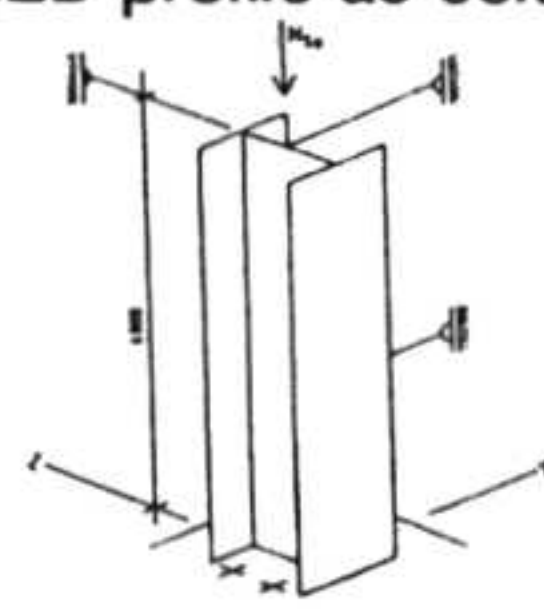
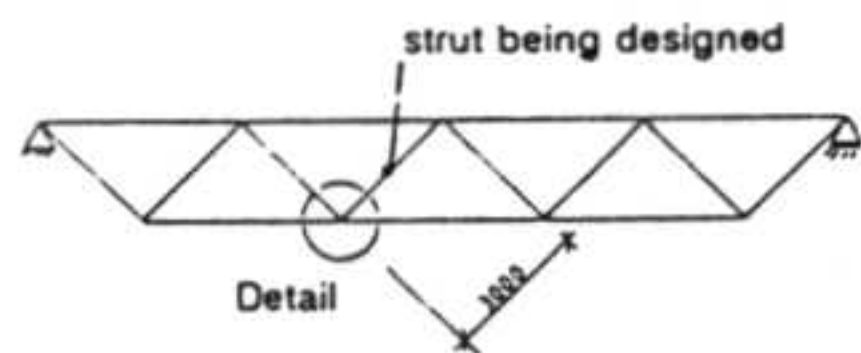
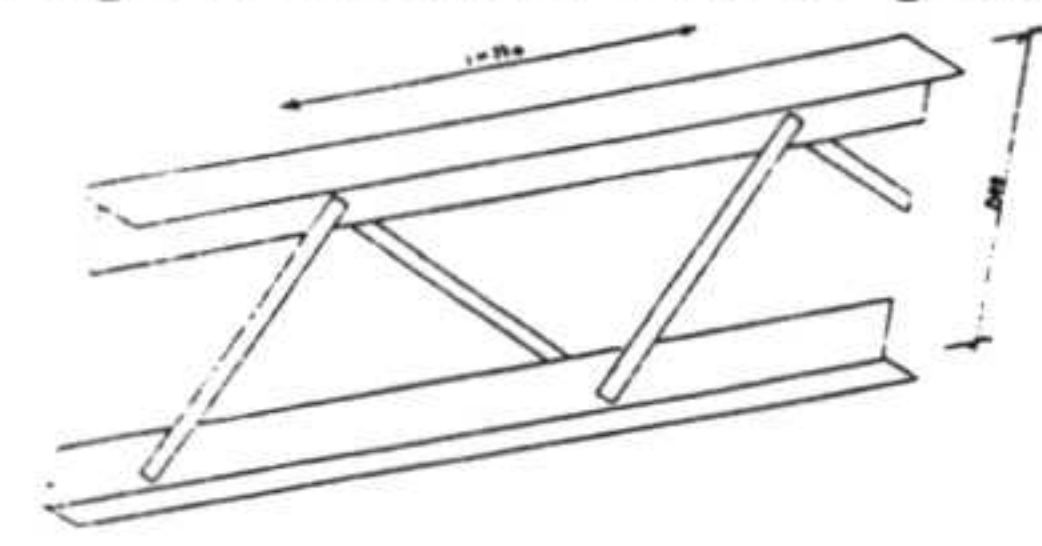
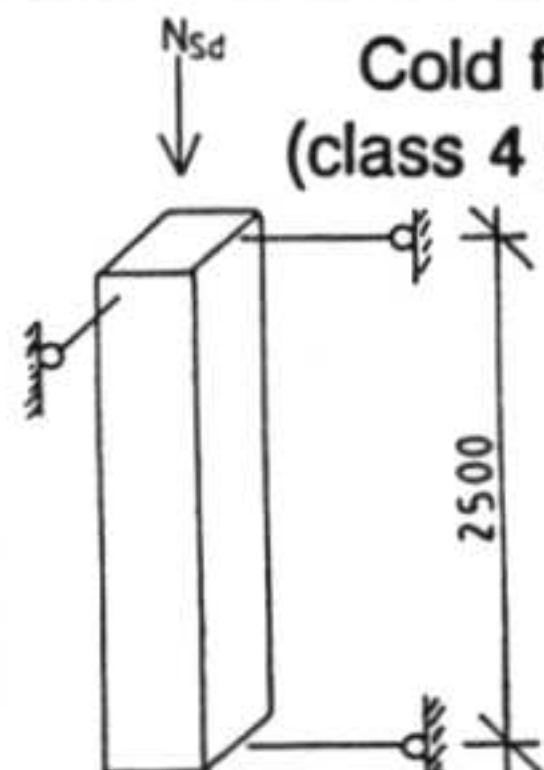
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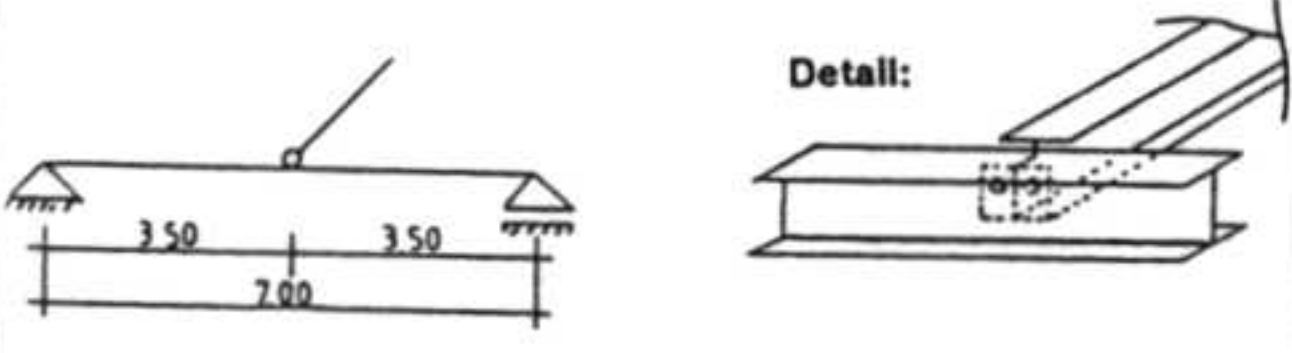
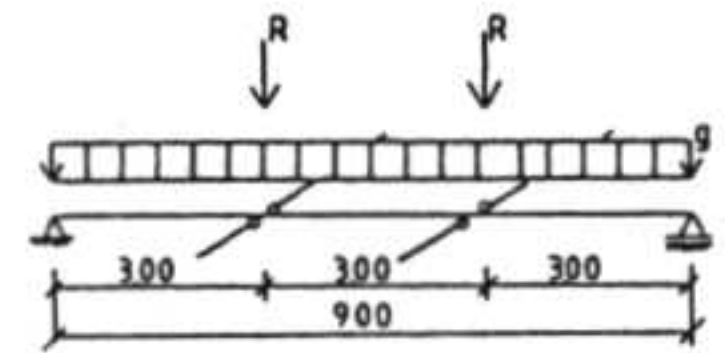
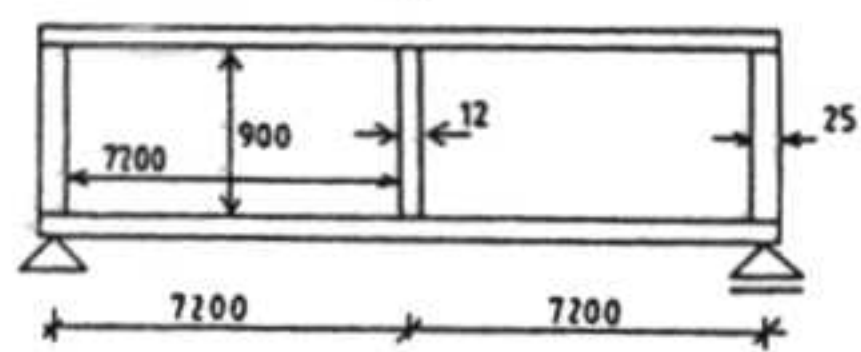
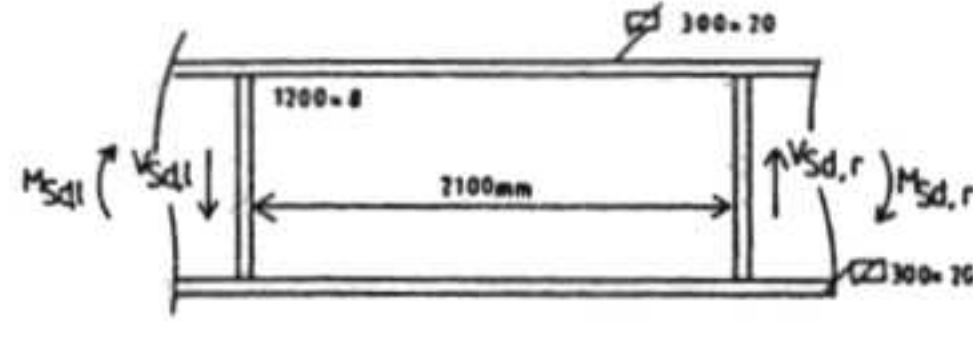
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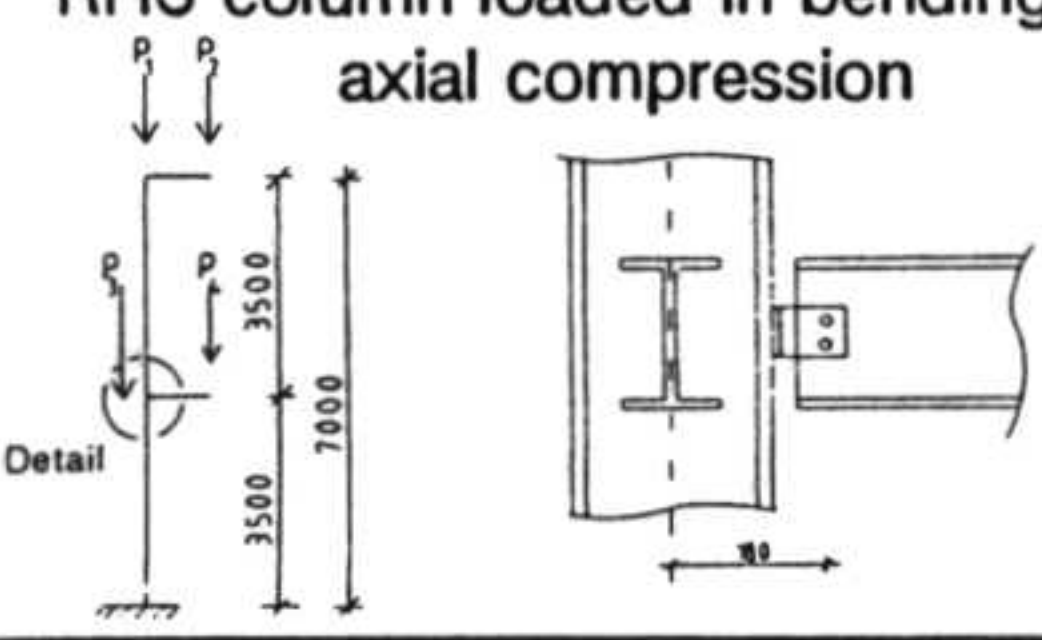
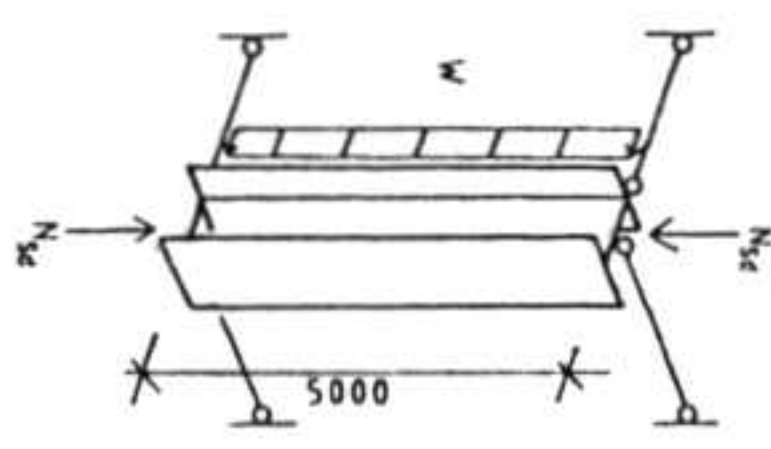
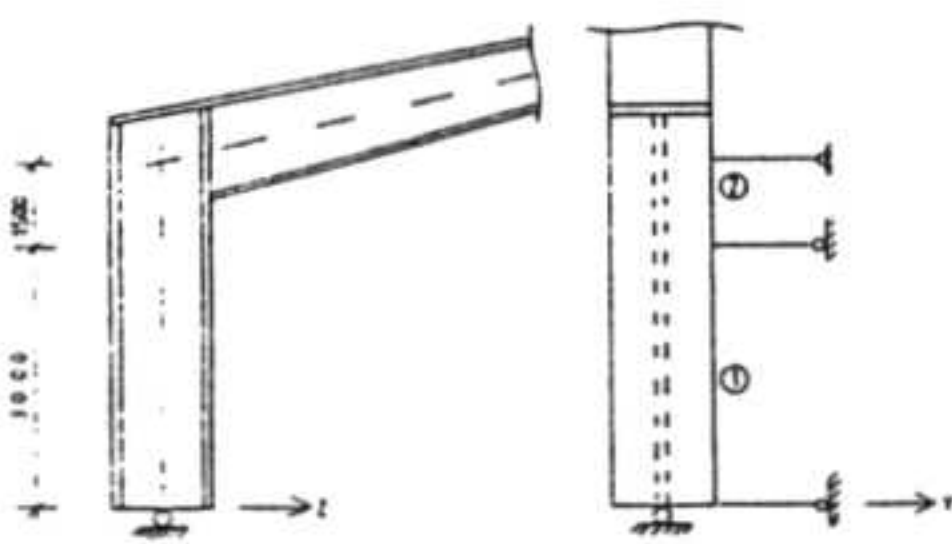
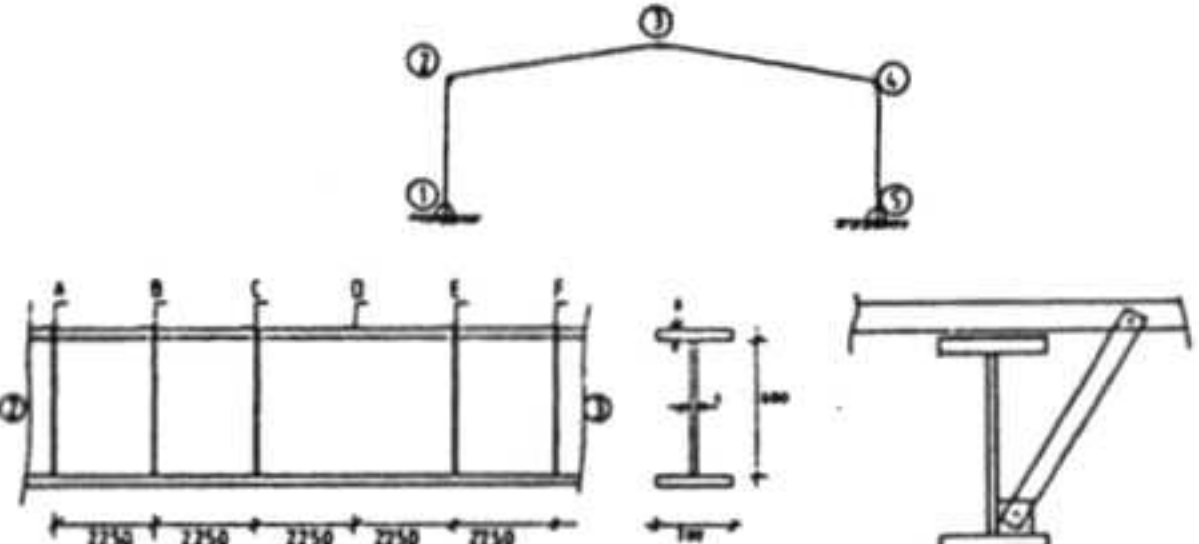
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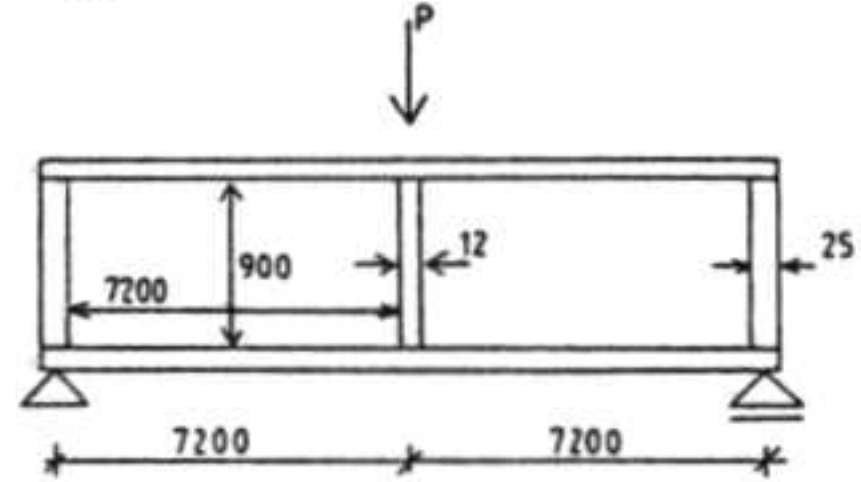
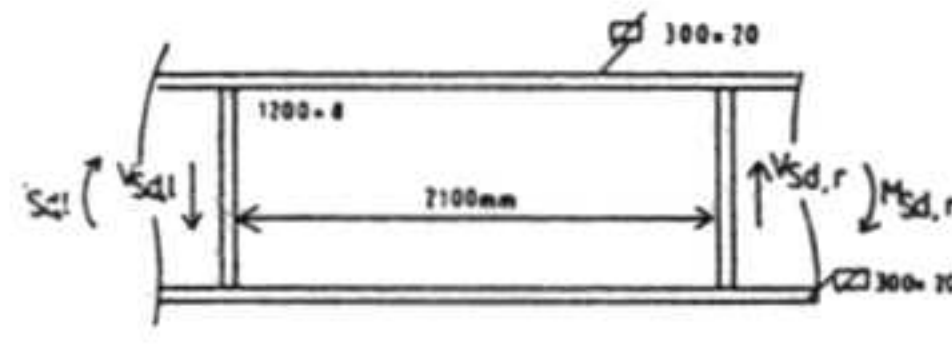
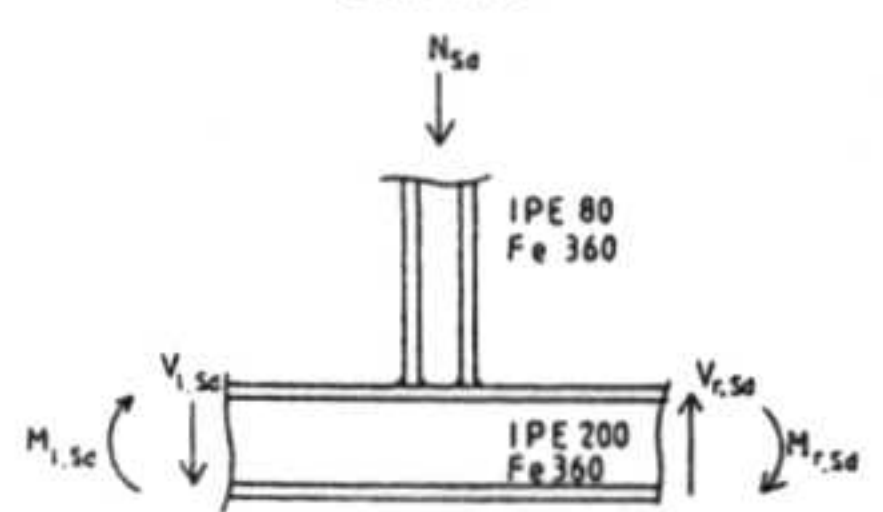
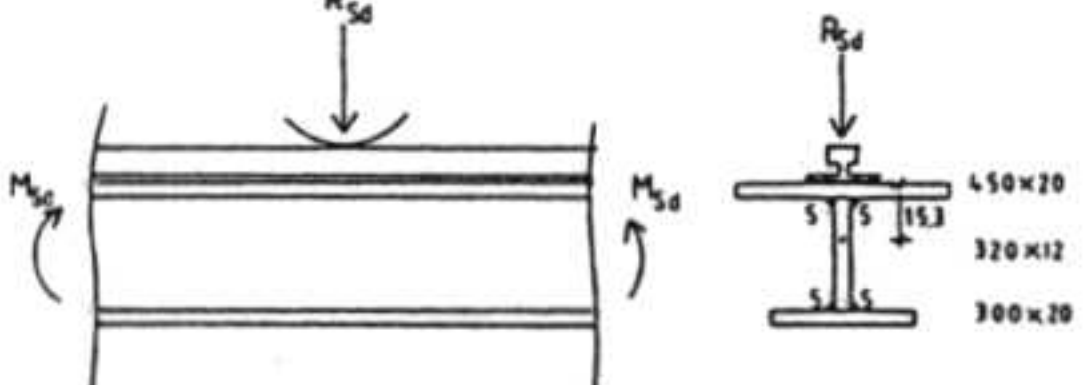
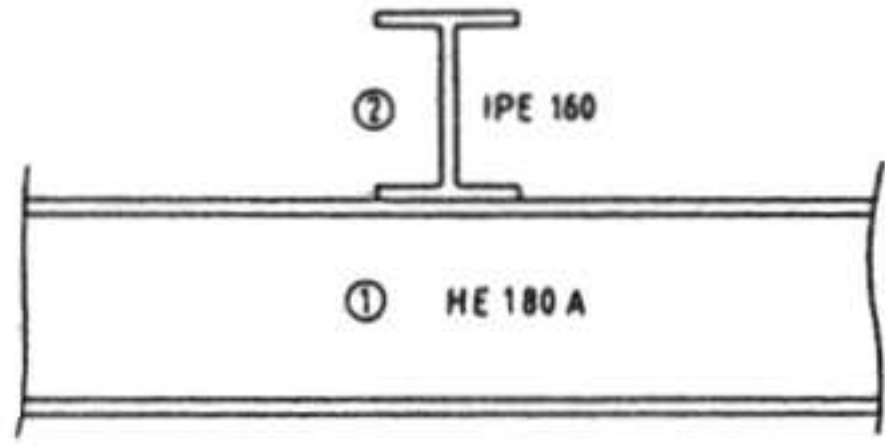
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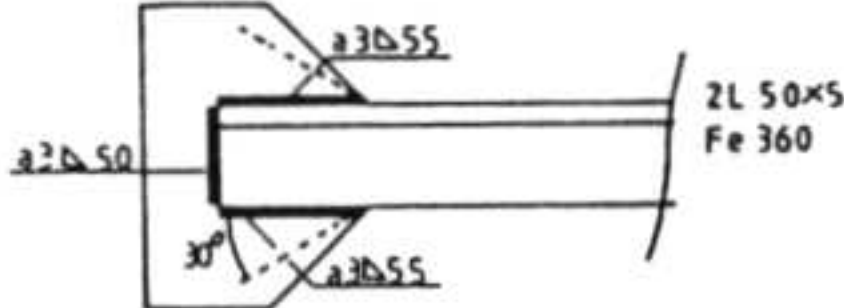
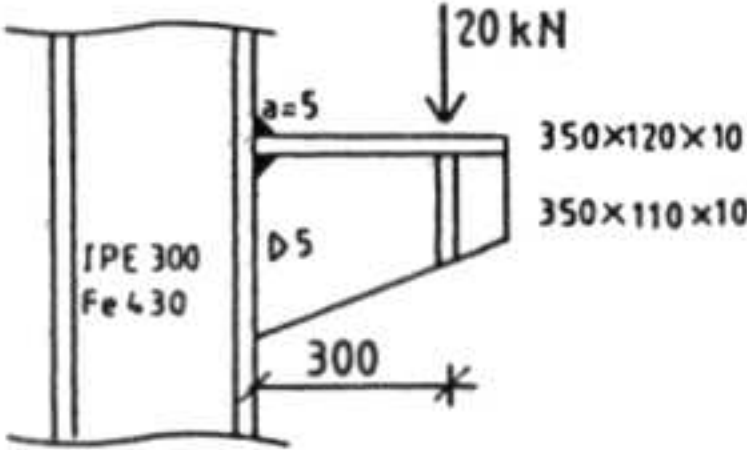
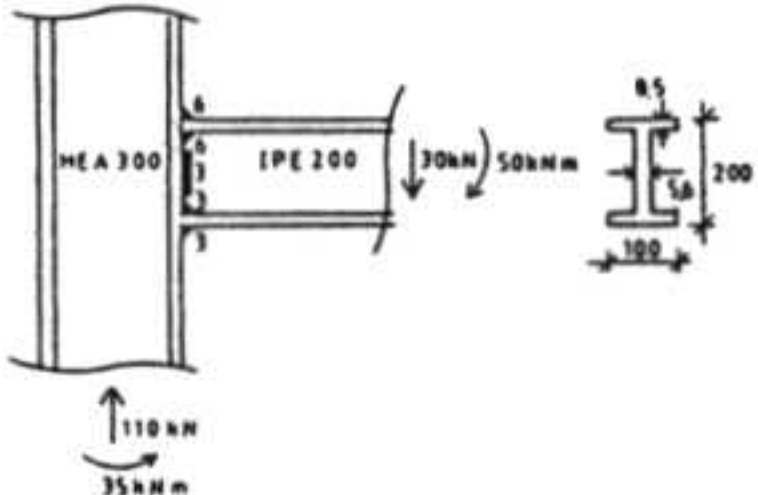
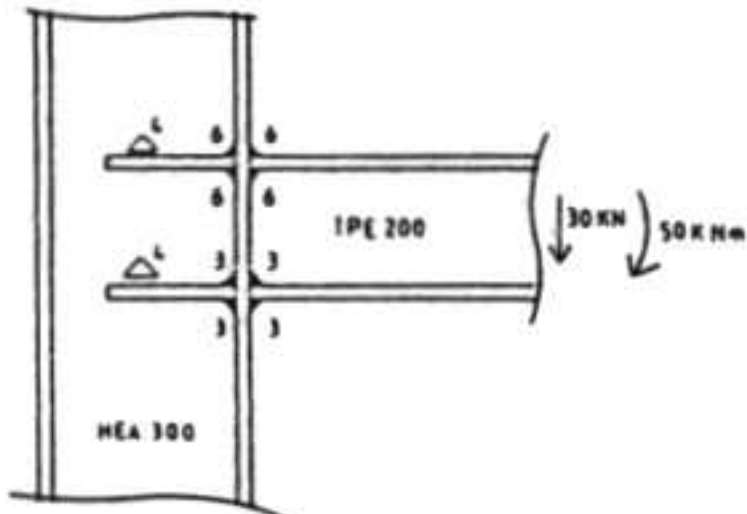
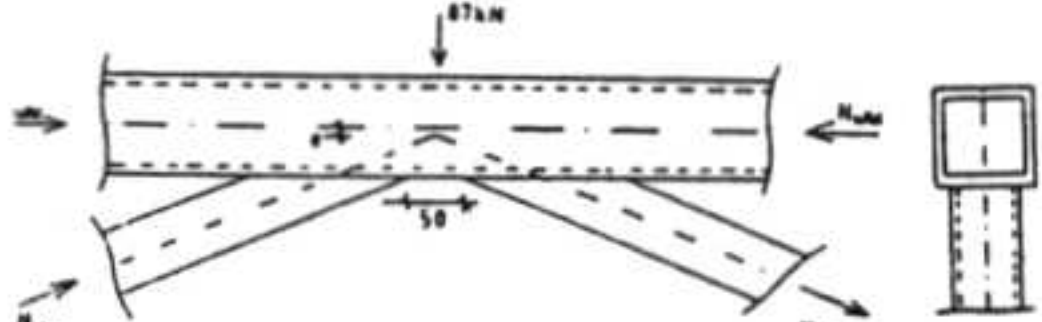
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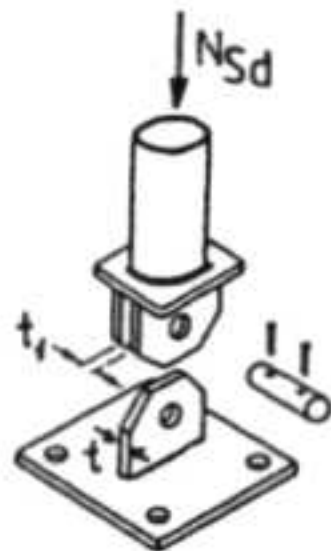
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Tabled reduction factors for buckling curves a₀, a, b, c, d

Part 1

Load combination
Methods of analysis
Frame analysis
Bracing system analysis

1.1 Load combination

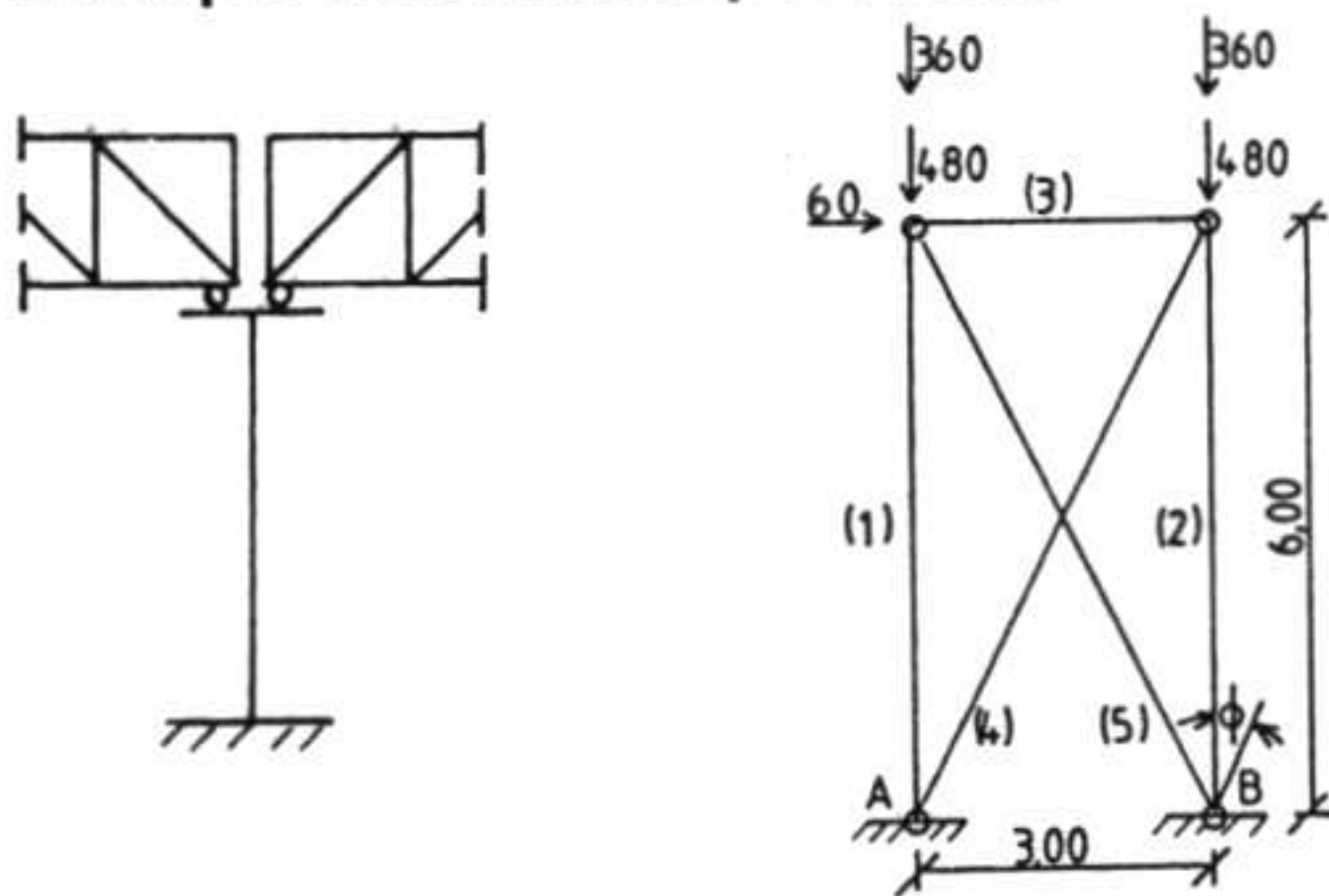
These examples demonstrate how the design values of action effects (N_{sd} , V_{sd} , M_{sd} , etc) are determined from the load assumptions. The further steps of design are not treated in this part. An action is a force (load) applied to the structure or an imposed deformation (e.g. temperature effects or settlements). Characteristic (unfactored) values of these actions are specified in ENV 1991 Eurocode 1 or other relevant loadings codes. These values of actions shall be multiplied by relevant safety factors and combination factors, see chapter "Combinations of actions" in ENV 1993 Eurocode 3 or Table 2.1 in the Essentials of Eurocode 3 to determine the design values of the effects of actions. The following examples show the method of determination of the maximum effects of actions. Not all possible combinations of actions are presented nor are relevant combinations worked out. In practice, one will collect experience to easily find out which load combination is decisive for verification of the structures.

The following examples are included in this chapter:

- | | |
|-----------------------|--|
| Example 1.1.1: | Braced portal frame |
| Example 1.1.2: | Purlin treated as continuous beam |
| Example 1.1.3: | Single storey frame without crane girder |
| Example 1.1.4: | Crane girder |
| Example 1.1.5: | Single storey frame with a crane girder |

Problem, Sketch, Calculation

Example 1.1.1: Braced portal frame



1. System and load configuration

Load assumptions:

$$G = 480 \text{ kN}$$

$$P = 360 \text{ kN}$$

$$W = 60 \text{ kN}$$

Frame imperfections $\Phi = 1/200$

Steel grade: Fe 360

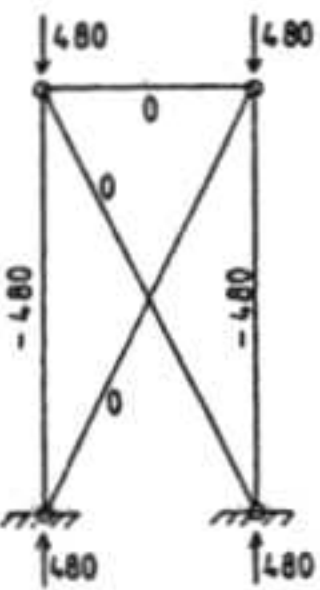
Member (1) and (2): 2 I-profiles with welded flange plates, $A = 104 \cdot 10^2 \text{ mm}^2$

Member (3): IPE 400, $A = 84,5 \cdot 10^2 \text{ mm}^2$

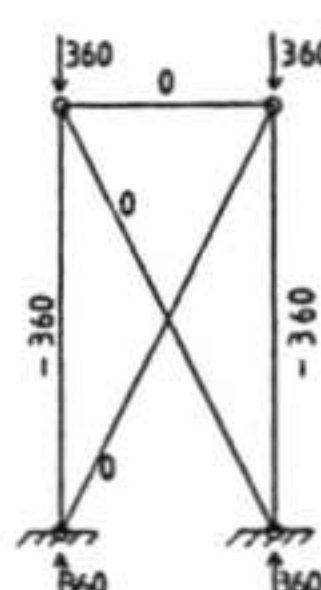
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2. Distribution of internal forces

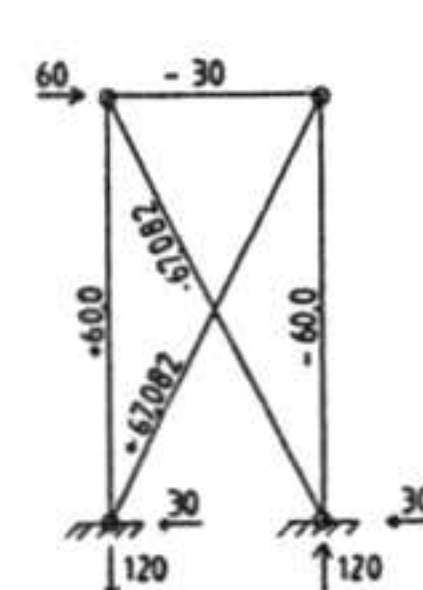
due to G



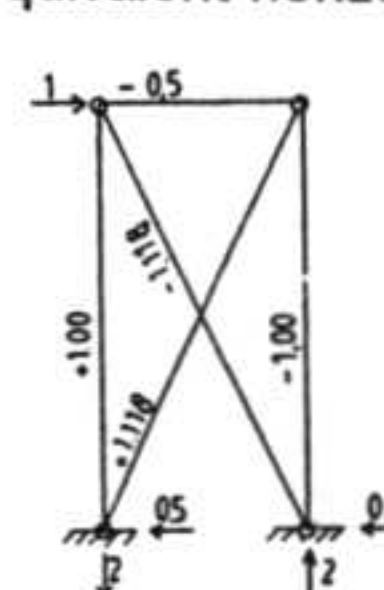
due to P



due to W



due to imperfections
equivalent horizontal unit force $F_i = 1$



3. Relevant load combinations

3.1 Relevant load combination for max compression force in the column (permanent + imposed load (1,35G + 1,5P))

Imperfection:

$$F_i = 2 \cdot (1,35 \cdot 480 + 1,5 \cdot 360) / 200 = 11,88 \text{ kN}$$

$$\text{maximum axial force in column (2): } N_{sd} = 1,35 \cdot 480 + 1,5 \cdot 360 + 11,88 \cdot 1,0 = 1199,9 \text{ kN}$$

3.2 Relevant load combination for max compression/tension in the diagonals (permanent + imposed load + wind (1,35 · (G + P + W)))

Imperfection:

$$F_i = 2 \cdot (1,35 \cdot 480 + 1,35 \cdot 360) / 200 = 11,34 \text{ kN}$$

$$\text{maximum compression force in diagonal (5): } N_{sd} = 1,35 \cdot 67,082 + 11,34 \cdot 1,118 = 103,2 \text{ kN}$$

$$\text{maximum tension force in diagonal (4): } N_{sd} = 1,35 \cdot 67,082 + 11,34 \cdot 1,118 = 103,2 \text{ kN}$$

$$\text{maximum reaction force at (B): } N_{sd} = 1,35 \cdot (480 + 360 + 120) + 11,34 \cdot 2,0 = 1318,7 \text{ kN}$$

3.3 Relevant load combination for uplift (permanent - wind (1,0 · G - 1,5 · W))

uplift reaction force at support (A):

$$N_{sd} = 480 - 1,5 \cdot 60,0 = 390,0 \text{ kN}$$

---> no uplift effect

Reference

Table 5.6

Table 2.1
[2.3.3.1.(1) and (5)]

Table 2.1
[2.3.3.1.(1) and (5)]

Table 2.1
[2.3.3.1.(1) and (5)]

Commentary

Characteristic (unfactored) values of actions are specified in ENV 1991 Eurocode 1 or other relevant loading codes.

The effects of imperfections shall always be taken into account in both first and second order theory.
Here: Frame imperfections are considered

Internal forces and moments (effects of actions) are responses of the structure to the loading (actions). Design values of the internal forces and moments are determined from the design (factored) values of loading using elastic or plastic global analysis.

Here: Using elastic global analysis, first order theory with unfactored loading by separating in single actions. This simplifies the decision of which load combination gives the worst effect.

The effect of loading is oriented in the same direction.

The effect of loading is oriented in the same direction.
The maximum reaction force is needed for base plate design. For foundation design characteristic values are needed.

The effect of dead load counteracts the effect of wind loading in column (1).

Problem, Sketch, Calculation		Reference	Commentary
3.4	Relevant load combination for max compression force in the beam (permanent + wind load (1,35G + 1,5W)) Imperfection: $F_l = 2 \cdot (1,35 \cdot 480) / 200 = 6,48 \text{ kN}$ maximum compression force in beam (3): $N_{sd} = 1,5 \cdot 30,0 + 6,48 \cdot 0,5 = 48,2 \text{ kN}$	Table 2.1 [2.3.3.1.(1) and (5)]	The effect of loading is oriented in the same direction.

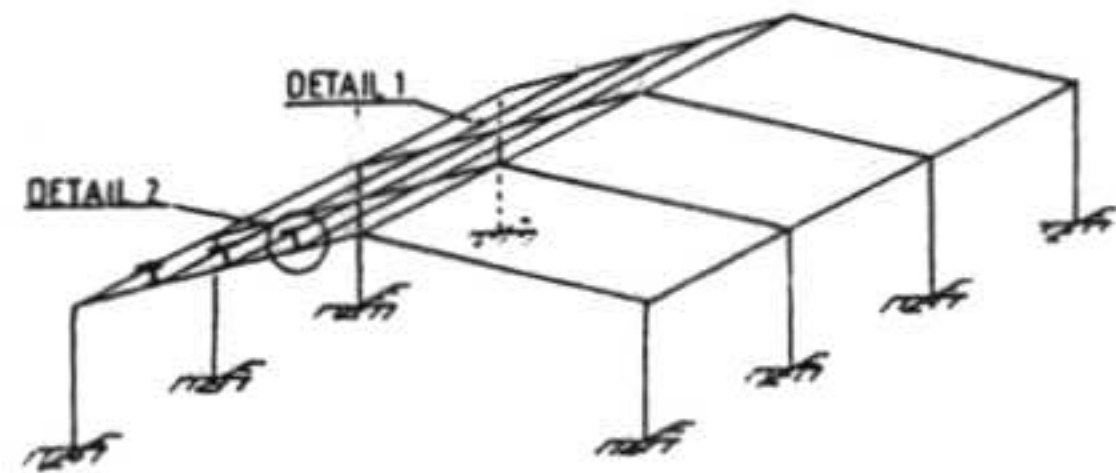
Problem, Sketch, Calculation

Reference

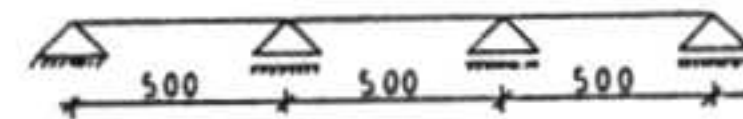
Commentary

Example 1.1.2: Purlin treated as a continuous beam

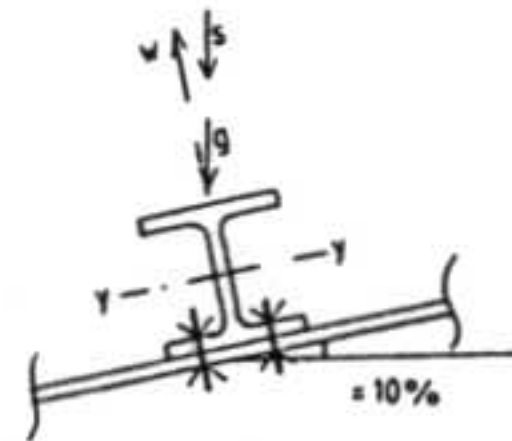
1. System



Detail 1: Purlin

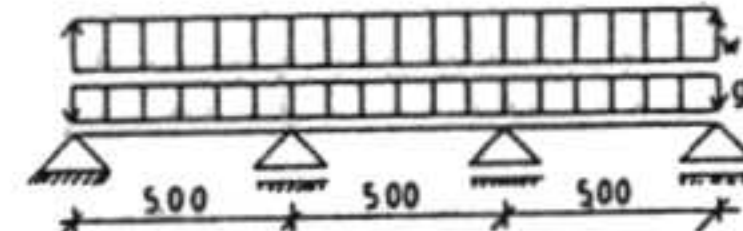
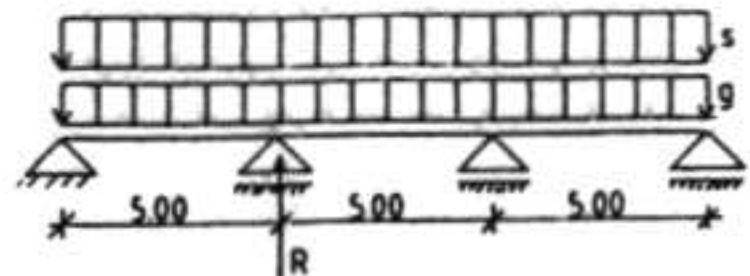


Detail 2: Fixing of purlin



Purlin (3-span-beam): IPE 140
Steel grade: Fe 430
Each span length: $L = 5,00$ m

2. Load configuration



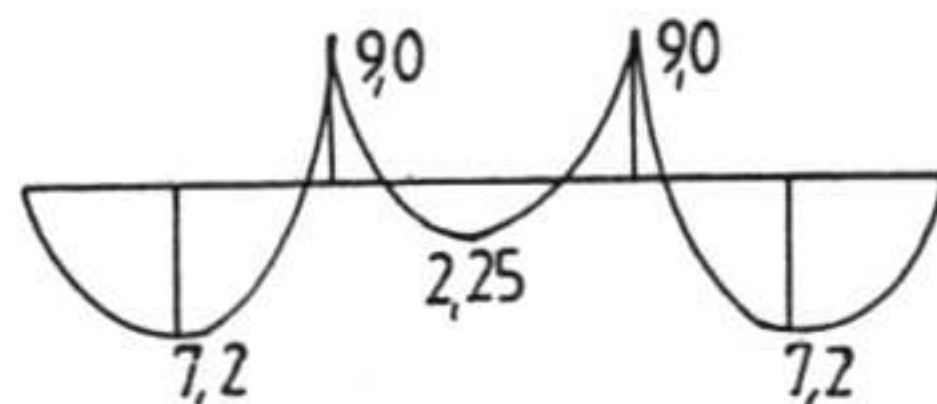
permanent load: $g_k = 1,0$ kN/m
snow load: $s_k = 1,5$ kN/m
wind load (uplift): $w_k = -2,0$ kN/m

3. Relevant load combinations for ultimate limit state verification of purlins

3.1 Relevant load combination for moment distribution (permanent + snow (1,35G + 1,5S))
 $q = 1,35 g + 1,5 s = 1,35 \cdot 1,0 + 1,5 \cdot 1,5 = 3,6$ kN/m

Plastic global analysis: $W_{pl} \geq \frac{q l^2 \gamma_{Mo}}{11,66 f_y} = \frac{3,6 \cdot 5,0^2 \cdot 1,1 \cdot 10^6}{11,66 \cdot 275} = 30,9 \cdot 10^3 \text{ mm}^3$

----- chosen: IPE 140 ($W_{pl} = 88,3 \cdot 10^3 \text{ mm}^3$; $M_{pl,Rd} = 22,1$ kNm)
Moment distribution due to $q = 3,6$ kN/m



3.2 Relevant load combination for uplift (permanent + wind (1,0G - 1,5W))
 $q = 1,0 g + 1,5 w = 1,0 \cdot 1,0 - 1,5 \cdot 2,0 = -2,0$ kN/m

4. Relevant load combinations for serviceability limit state

4.1 Serviceability limit state requirement (example only)
Requirement: $\delta_2 \leq L/200$ (for snow only)
 $q = 1,0 s = 1,5$ kN/m
 $\delta = 18,0 \text{ mm} = L/280 < L/200$

Table 2.1
[2.3.3.1.(1) and (5)]

Purlins are laterally restrained by cladding
→ sufficient resistance to prevent minor axis buckling

No imperfection effects
This load combination leads to the following effects:
- max M_{sd} (strength, lateral torsional buckling)
- max V_{sd} (shear)
- max R_{sd} (reaction)

In view of erection an IPE 140 is chosen.

Table 2.1
[2.3.3.1.(1) and (5)]

This load combination leads to the following effects:
- M_{sd} (lateral torsional buckling)
- R_{sd} (reaction)

Table 2.3;
4.2.1.(1);
Table 4.2;
[2.3.4]

For roofs in general less stringent values may be used [4.2.1.(2)].

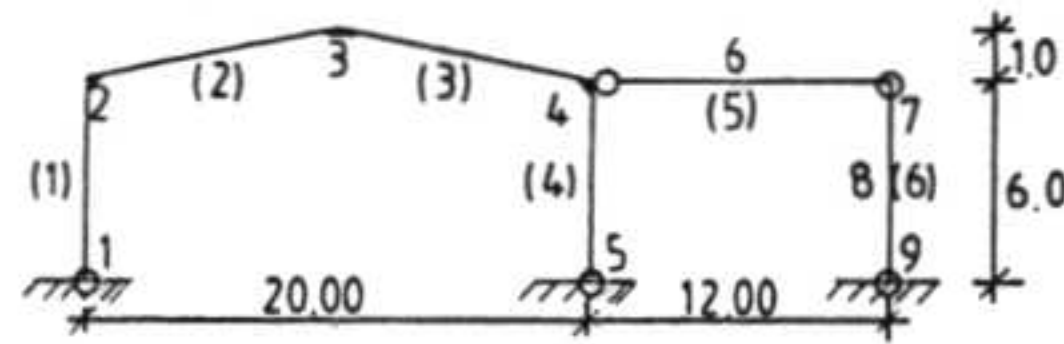
Problem, Sketch, Calculation

Reference

Commentary

Example 1.1.3: Single storey frame

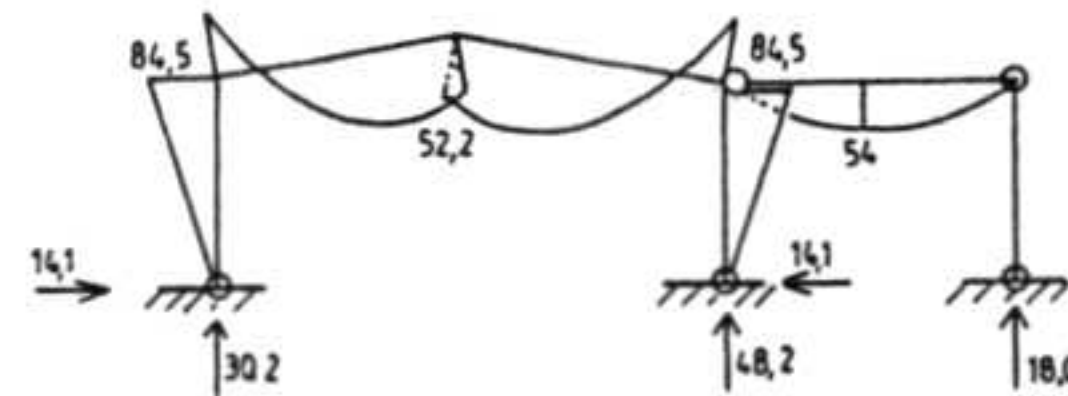
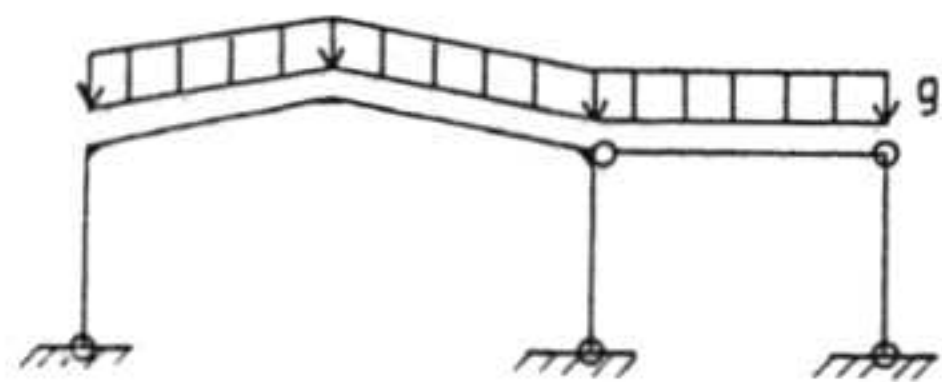
1. System



column (1) and (4): IPE 450
 beam (2) and (3): IPE 400
 column (6): HE 180 A
 beam (5): IPE 400
 Steel grade: Fe 430

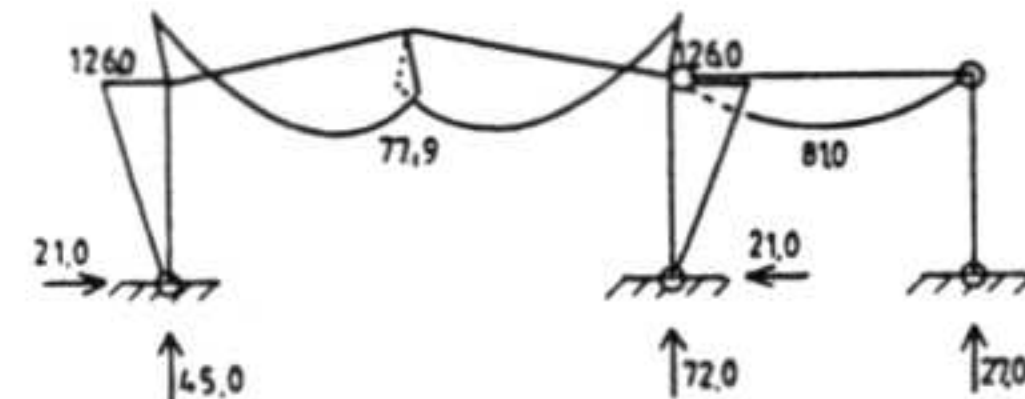
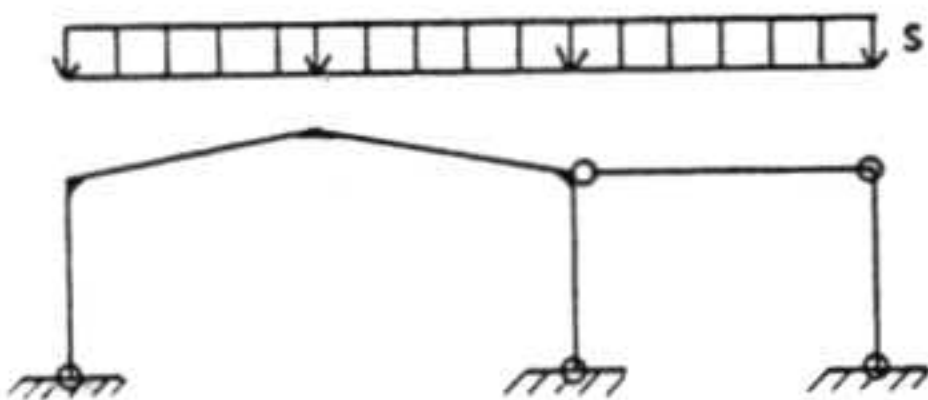
2. Load configuration and distribution of moments permanent:

$g = 3,0 \text{ kN/m}$

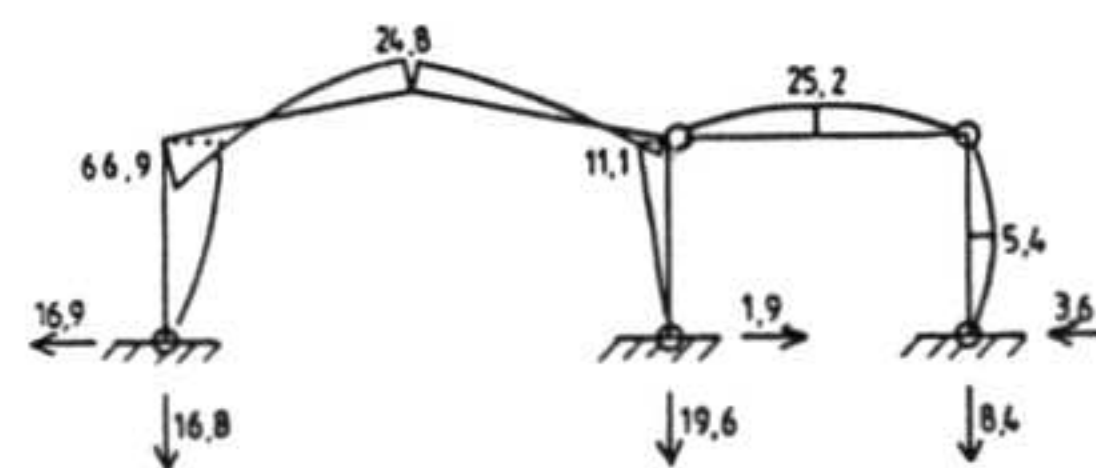
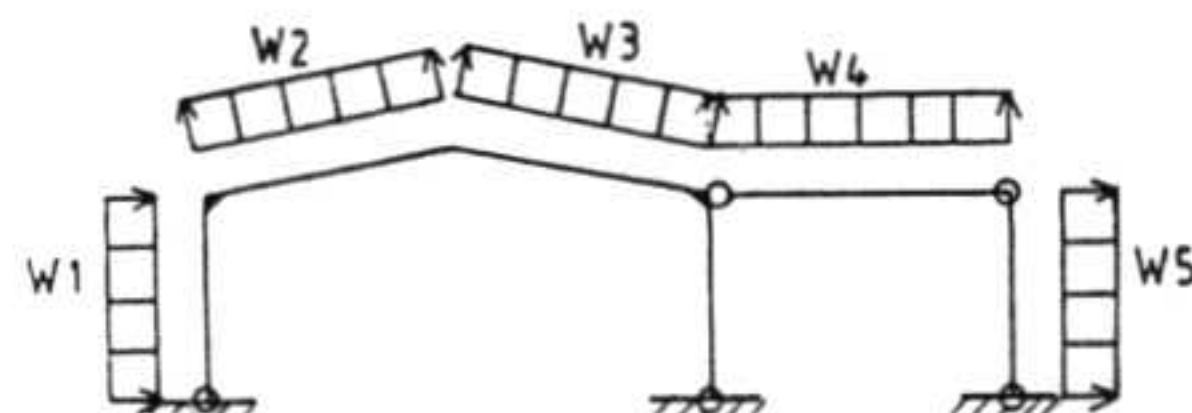


snow:

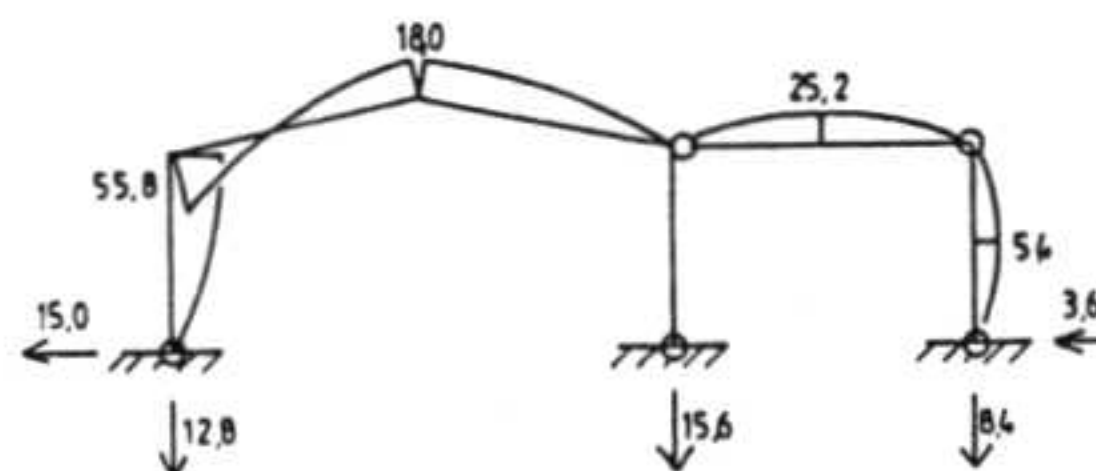
$s = 4,5 \text{ kN/m}$



wind from left side



$w_1 = 1,9 \text{ kN/m}$
 $w_2 = 1,0 \text{ kN/m}$ or $1,4 \text{ kN/m}$
 (whichever gives a larger effect)
 $w_3 = 1,0 \text{ kN/m}$ or $1,4 \text{ kN/m}$
 (whichever gives a larger effect)
 $w_4 = 1,4 \text{ kN/m}$
 $w_5 = 1,2 \text{ kN/m}$

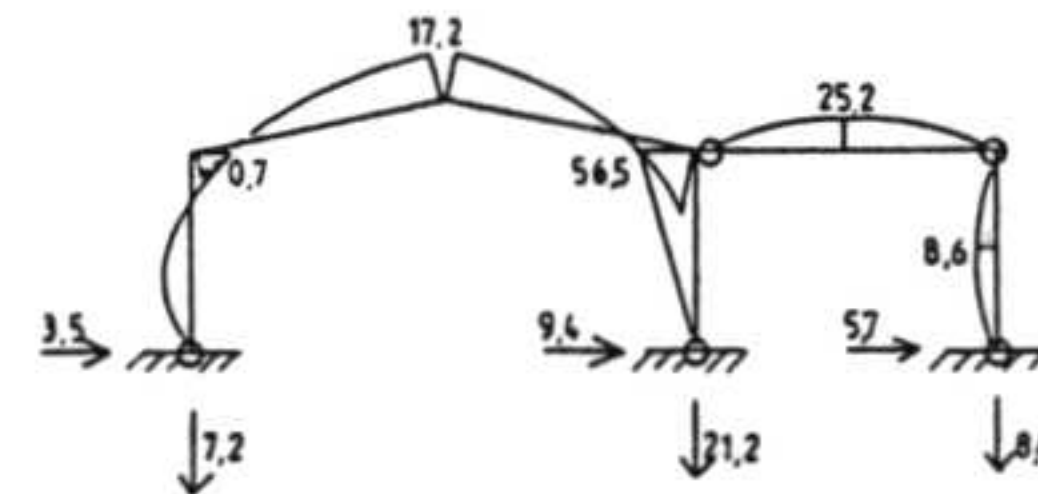
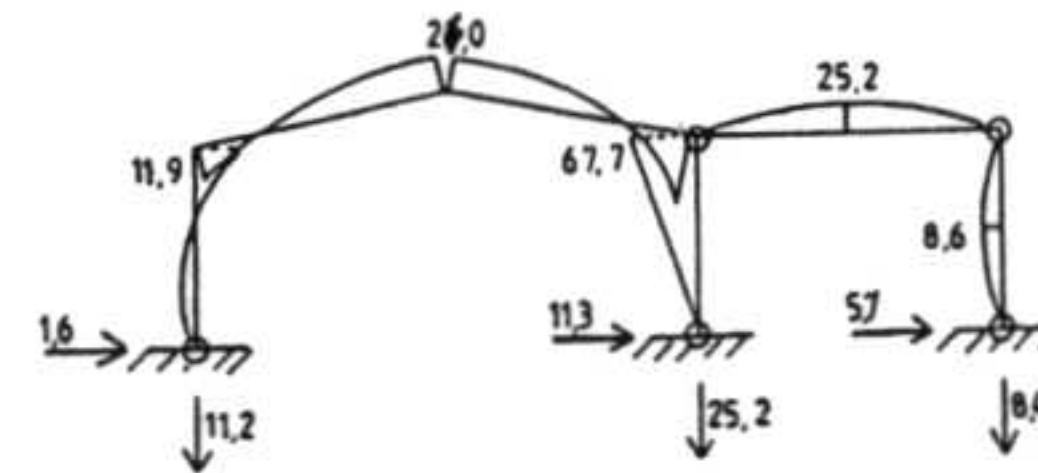
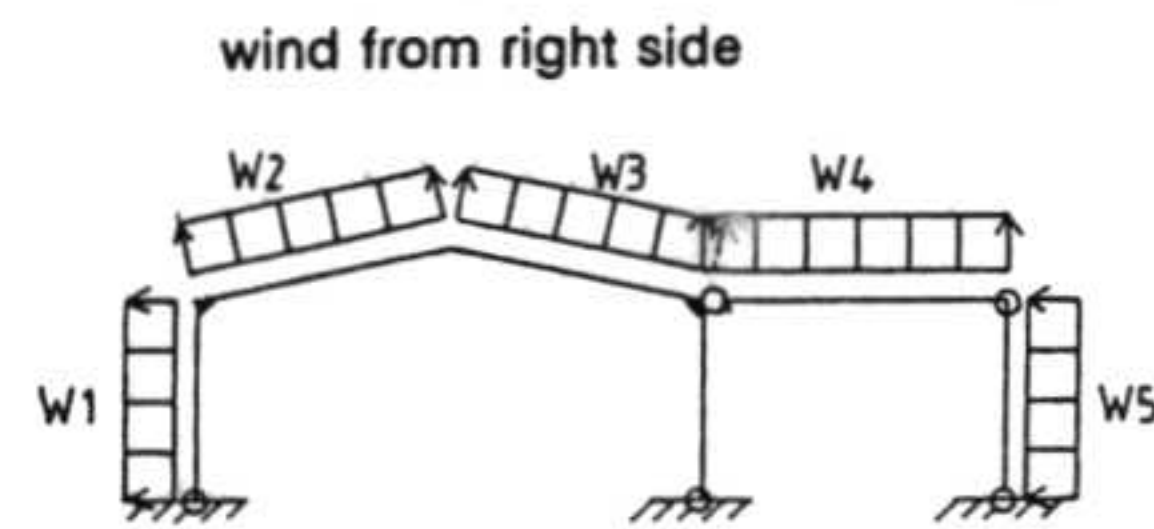


Using first order theory for determining the internal forces and moments with $\gamma_F = 1,0$ for finding out the relevant load combination

Problem, Sketch, Calculation

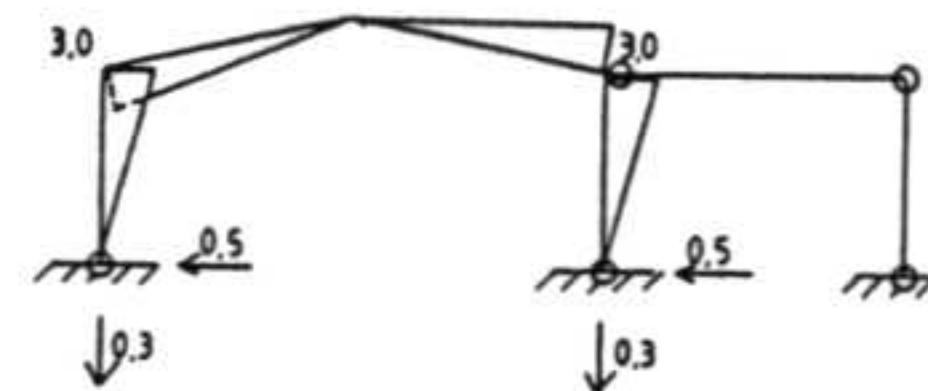
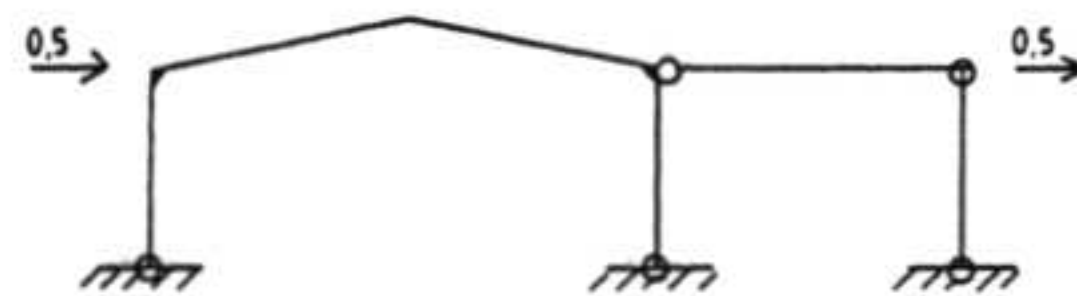
Reference

Commentary



$w_1 = 1,9 \text{ kN/m}$
 $w_2 = 1,0 \text{ kN/m}$ or $1,4 \text{ kN/m}$
 (whichever gives a larger effect)
 $w_3 = 1,0 \text{ kN/m}$ or $1,4 \text{ kN/m}$
 (whichever gives a larger effect)
 $w_4 = 1,4 \text{ kN/m}$
 $w_5 = 1,2 \text{ kN/m}$

Imperfections (equivalent horizontal unit force)



3. Relevant load combinations for ultimate limit state verification
3.1 permanent + snow
 $1,35 \text{ g} + 1,5 \text{ s}$

3.1.1 Determination of the initial sway imperfections
 Column (3) does not contribute to the resistance
 $\rightarrow n_c = 2 ; n_s = 1$
 $\rightarrow \Phi = k_c k_s \varphi_0$ (see table 5.6 in E-EC3)
 $\Phi = 1/200$

Equivalent horizontal forces F_i
 due to g: $\Sigma V_{srf} = 1,35 \cdot [60,6 + 96,6] = 212,2 \text{ kN}$
 due to s: $\Sigma V_{srf} = 1,50 \cdot [45,2 + 72,2] = 176,1 \text{ kN}$
 $F = 212,2 + 176,1 = 388,3 \text{ kN}$
 $F_i = \Phi \cdot F = 388,3/200 = 1,9 \text{ kN}$

Table 2.1;
[2.3.3.1.(5)]

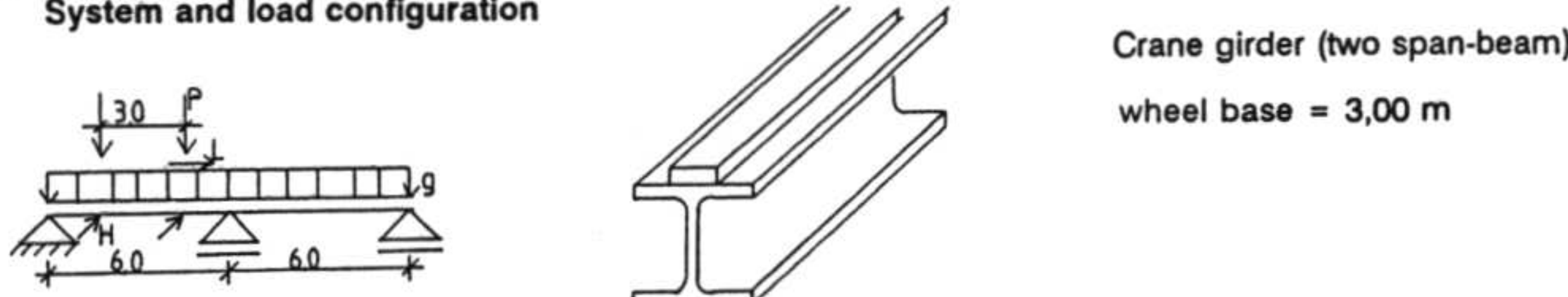
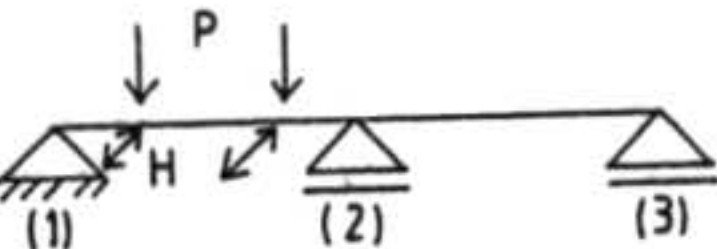
This load combination leads to following effects:
 - max N_{sd} at column (4) and (6)
 - max M_{sd} at 2, 3, 4 and 6
 - max $R_{v, sd}$ at 5 and 9
 $R_{v, sd}$ is needed for base plate design. For foundation design characteristic values are needed.

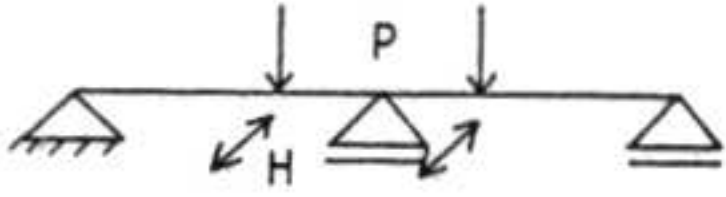
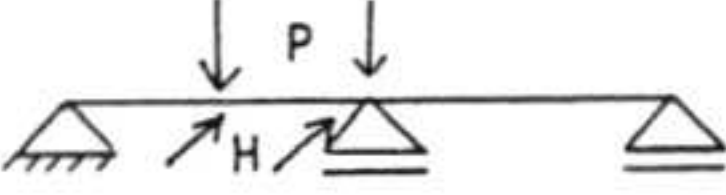
5.2.1.2; [5.2.1.2]

5.2.3.1.2; Table 5.6
[5.2.4.3]

Table 5.5;
[Figure 5.2.4]

Problem, Sketch, Calculation	Reference	Commentary
3.1.2 Maximum effects due to the load combination permanent + snow max N at column (4): $N_{sd} = 1,35 \cdot 48,2 + 1,5 \cdot 72,0 + 0,3 \cdot 1,9 = 173,6 \text{ kN}$ max N at column (6): $N_{sd} = 1,35 \cdot 18,0 + 1,5 \cdot 27,0 = 173,6 \text{ kN}$ max M at 2 and 4: $M_{sd} = 1,35 \cdot 84,9 + 1,5 \cdot 126,0 + 3,0 \cdot 1,9 = 309,3 \text{ kNm}$ max M at 3: $M_{sd} = 1,35 \cdot 52,2 + 1,5 \cdot 77,9 = 187,3 \text{ kNm}$ max M at 6: $M_{sd} = 1,35 \cdot 54,0 + 1,5 \cdot 81,0 = 194,4 \text{ kNm}$		max $R_{v, sd}$ at 5 and 9 are equal N_{sd} at column (4) and (6).
3.2 permanent + snow + wind 1,35 (g + s + w) wind is considered from the right side with w_2 and $w_3 = 1,0 \text{ kN/m}$	Table 2.1 [2.3.3.1.(5)]	This load combination leads to the following effect: - max $R_{h, sd}$ (reaction horizontal). $R_{h, sd}$ is needed for base plate design.
3.2.1 Determination of the initial sway imperfections Column (3) does not contribute to the resistance $\rightarrow \Phi = 1/200$ (as demonstrated in 3.1.1) Equivalent horizontal forces F_i due to g: $\Sigma V_{sd} = 1,35 \cdot (60,6 + 96,6) = 212,2 \text{ kN}$ due to s: $\Sigma V_{sd} = 1,35 \cdot (45,2 + 72,2) = 158,5 \text{ kN}$ due to w: $\Sigma V_{sd} = 1,35 \cdot (-11,2 - 25,2) = -49,1 \text{ kN}$ $F = 212,2 + 158,5 - 49,1 = 321,6 \text{ kN}$ $F_i = \Phi \cdot F = 321,6/200 = 1,6 \text{ kN}$	5.2.1.2; [5.2.1.2] 5.2.3.1.2; Table 5.6 [5.2.4.3] Table 5.5; [Figure 5.2.4]	
3.2.2 Maximum effects due to load combination permanent + snow + wind $R_{h, sd} = 1,35 \cdot (14,1 + 21,0 + 3,5) + 0,5 \cdot 1,6 = 52,9 \text{ kN}$		
3.3 Further commentary Not all possible load combinations are given in this example. It could be that for different elements different load combinations are relevant. E.g. for a beam-column two checks are necessary: (1) max M and associated N (2) max N and associated M Therefore the load combination "permanent + wind" could also be relevant.		

Problem, Sketch, Calculation	Reference	Commentary
Example 1.1.4: Crane girder 1. System and load configuration  Detail: Crane girder (two span-beam) wheel base = 3,00 m 2. Load assumptions permanent: $g = 2,0 \text{ kN/m}$ wheel load: $P = 130 \text{ kN}$ wheel loads must be increased by a dynamic coefficient φ; $\varphi = 1,2$ (purely for calculation) Lateral force: $H = 13 \text{ kN}$ (lateral forces due to crabbing, surging, etc) Braking force: $L = 15 \text{ kN}$ 3. Relevant load combinations for ultimate limit state verification 3.1 Load position at mid-span see sketch  Relevant load combination for max mid-span moment and max reaction force at support (3) $(1,35g + 1,5(P + H))$ $g: 1,35 \cdot 0,0703 \cdot 2,0 \cdot 6,0^2 = 6,8 \text{ kNm}$ $P: 1,50 \cdot 0,23 \cdot 1,2 \cdot 130 \cdot 6,0 = 322,9 \text{ kNm}$ $\rightarrow M_{y,Sd} = 6,8 + 322,9 = 329,7 \text{ kNm}$ $H: M_{z,Sd} = 1,50 \cdot 0,23 \cdot 13 \cdot 6,0 = 26,9 \text{ kNm}$ Relevant load combination for maximum moment at the inner support $(1,35g + 1,5(P + H))$ $g: -1,35 \cdot 0,125 \cdot 2,0 \cdot 6,0^2 = -12,2 \text{ kNm}$ $P: -1,50 \cdot 0,14 \cdot 1,2 \cdot 130 \cdot 6,0 = -196,6 \text{ kNm}$ $\rightarrow M_{y,Sd} = -12,2 - 196,6 = -208,8 \text{ kNm}$ $H: M_{z,Sd} = -1,50 \cdot 0,14 \cdot 13 \cdot 6,0 = -16,4 \text{ kNm}$ Relevant load combination for maximum uplift at the end support $(1,0g + 1,5P)$ $g: 1,0 \cdot 0,375 \cdot 2,0 \cdot 6,0 = 4,5 \text{ kN}$ $P: -1,50 \cdot 0,14 \cdot 1,2 \cdot 130 = -32,8 \text{ kN}$ $\rightarrow R_{z,Sd} = 4,5 - 32,8 = -28,3 \text{ kN}$		No fatigue is included here Lateral forces and wheel loads can act together Braking forces are independent from lateral forces No imperfection effects For the determination of the internal forces and moments influence lines are used

Problem, Sketch, Calculation	Reference	Commentary
3.2 Load position at inner support see sketch Relevant load combination for max reaction force at support (2) $(1,35g + 1,5(P + H))$ g: $1,35 \cdot 1,25 \cdot 2,0 \cdot 6,0 = 20,3 \text{ kN}$ P: $1,50 \cdot 1,82 \cdot 1,2 \cdot 130 = 425,9 \text{ kN}$ $\rightarrow R_{x, Sd} = 20,3 + 425,9 = 446,2 \text{ kN}$ H: $R_{y, Sd} = 1,50 \cdot 1,82 \cdot 13 = 35,5 \text{ kN}$ 		
3.3 Load position at inner support see sketch Relevant load combination for max shear force at support (2) $(1,35g + 1,5(P + H))$ g: $1,35 \cdot 0,625 \cdot 2,0 \cdot 6,0 = 10,1 \text{ kN}$ P: $1,50 \cdot 1,60 \cdot 1,2 \cdot 130 = 374,4 \text{ kN}$ $\rightarrow V_{x, Sd} = 10,1 + 374,4 = 384,5 \text{ kN}$ H: $V_{y, Sd} = 1,50 \cdot 1,6 \cdot 13 = 31,2 \text{ kN}$ 		
3.4 Relevant load combination for determining the horizontal forces which must be applied at the bracing system parallel to crane girder $1,35(g + P + H + L)$ perpendicular to crane girder $1,35g + 1,5 H$		

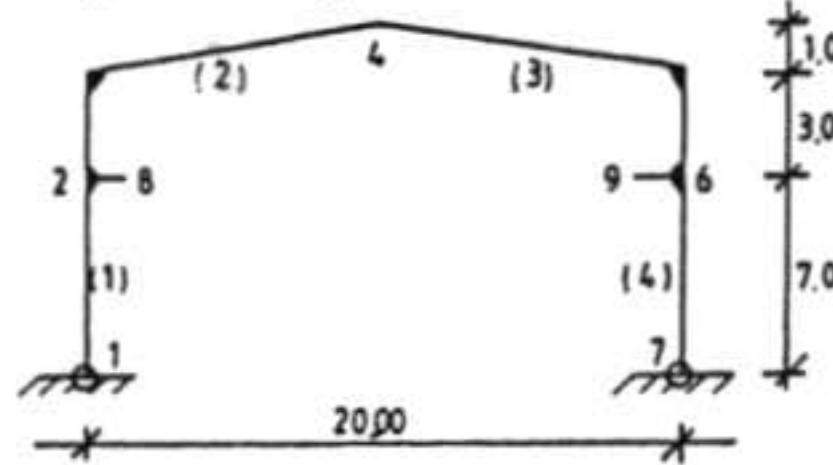
Problem, Sketch, Calculation

Reference

Commentary

Example 1.1.5: Single storey frame including a crane girder

1. System

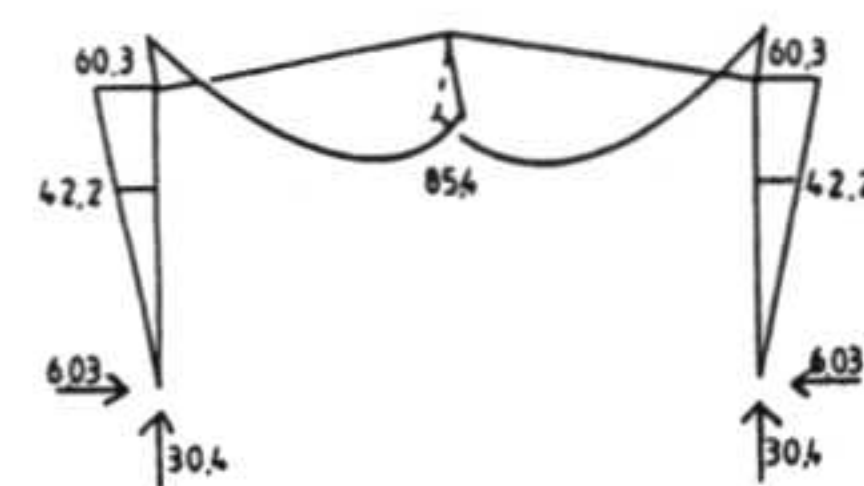
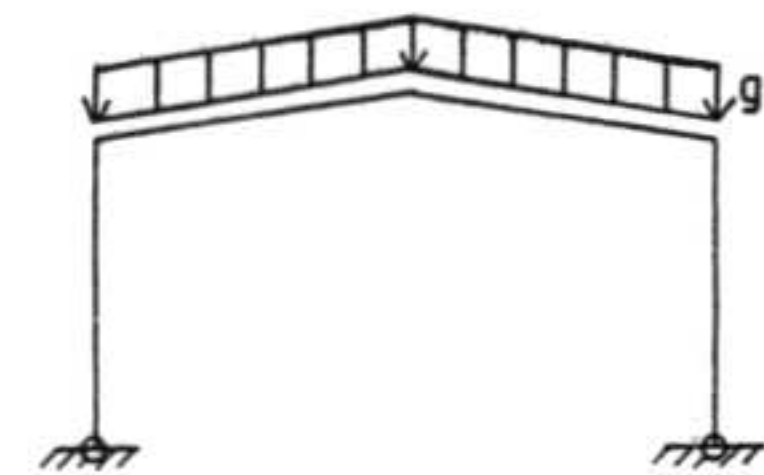


column (1) and (4):
beam (2) and (3):
Steel grade:

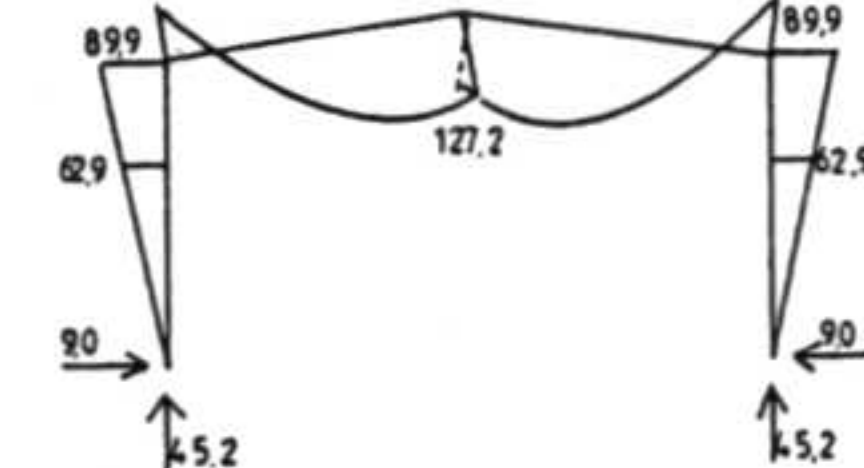
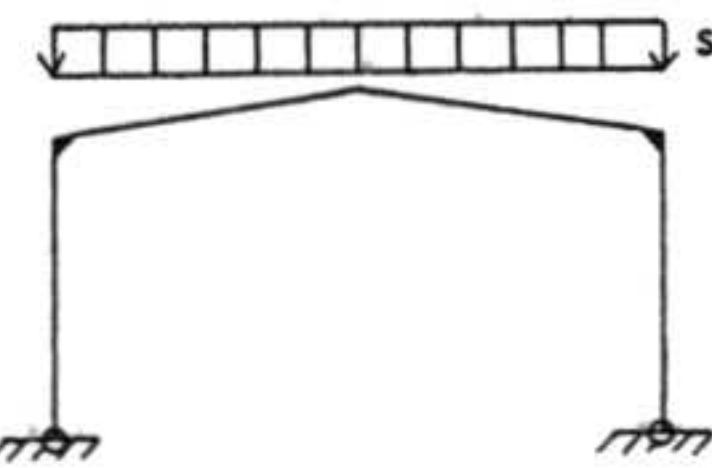
HE 400 A
IPE 600
Fe 430

2. Load configuration and distribution of moments permanent:

$$g = 3,0 \text{ kN/m}$$

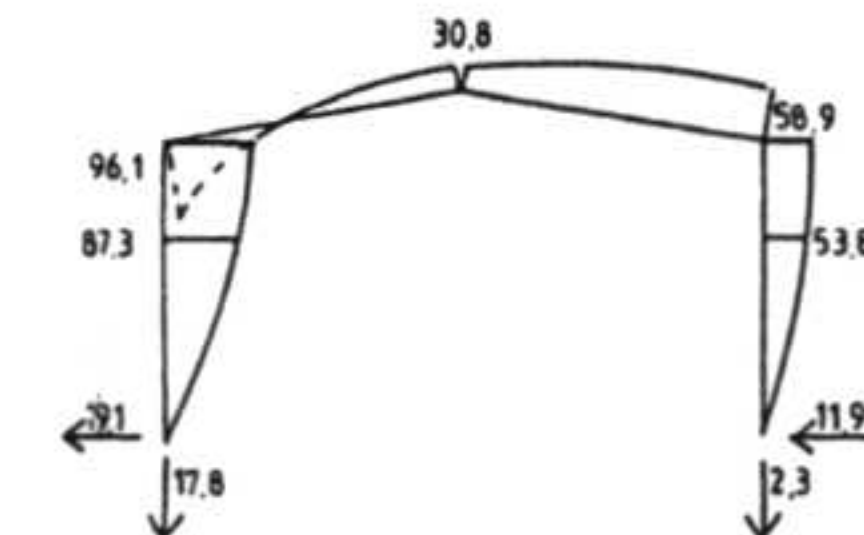
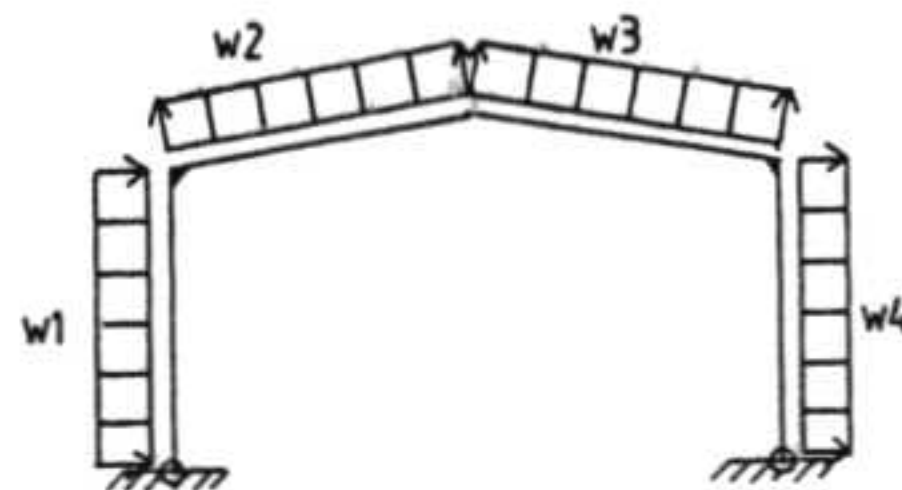


snow:



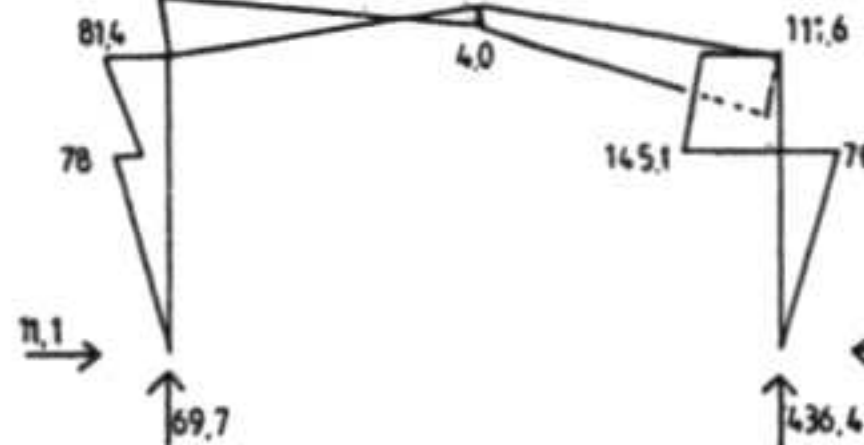
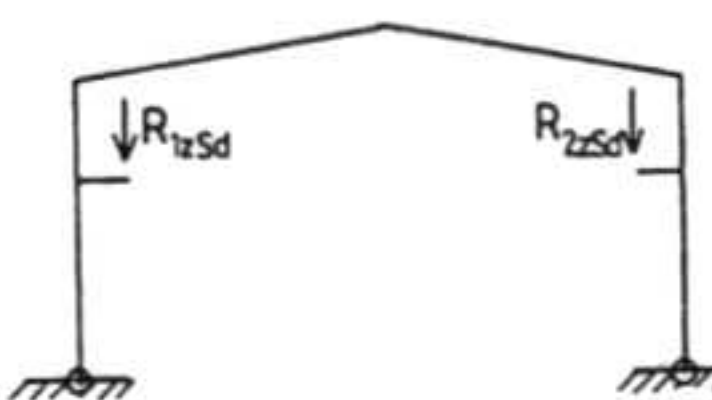
$$s = 4,5 \text{ kN/m}$$

wind from left side



$$\begin{aligned} w_1 &= 1,9 \text{ kN/m} \\ w_2 &= 1,0 \text{ kN/m} \\ w_3 &= 1,0 \text{ kN/m} \\ w_4 &= 1,2 \text{ kN/m} \end{aligned}$$

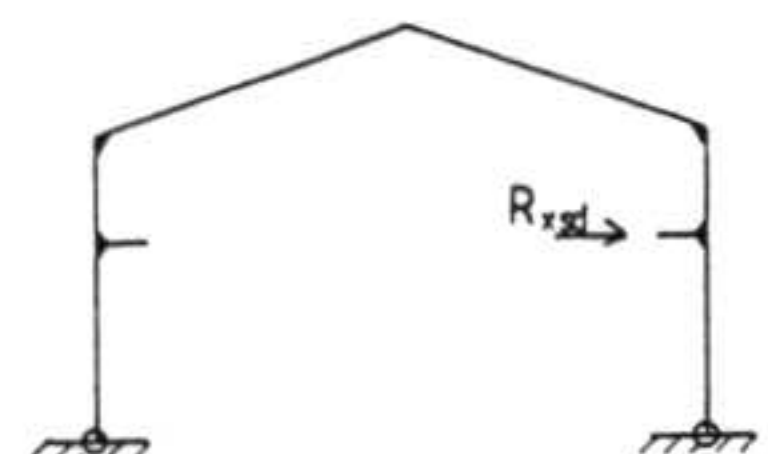
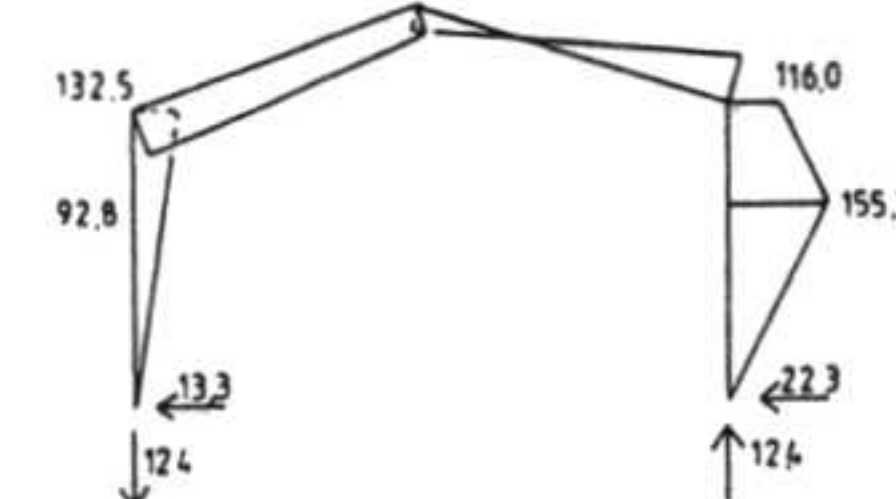
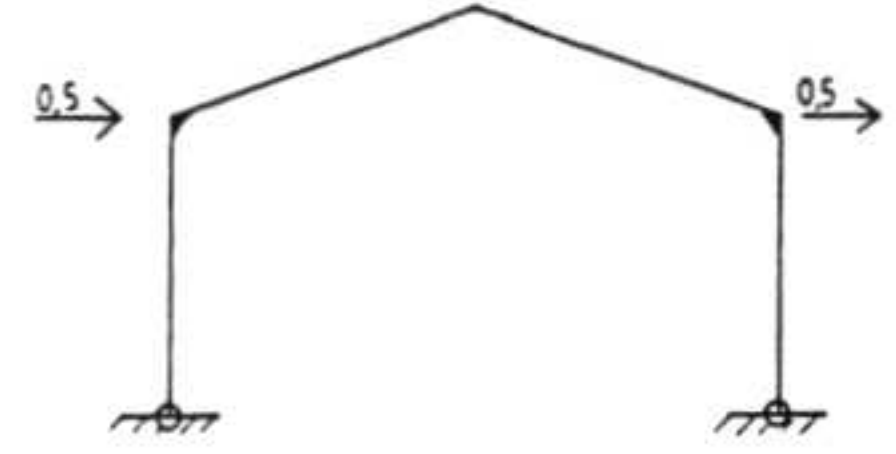
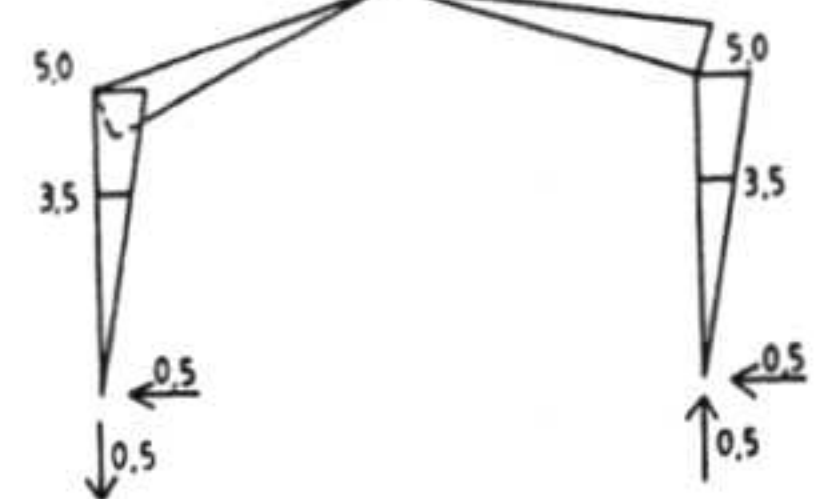
Support load from crane girder (vertical)



$$\begin{aligned} R_{1xSd} &= 60 \text{ kN} \\ R_{2xSd} &= 446 \text{ kN} \end{aligned}$$

Using first order theory for determining the internal forces and moments with $\gamma_F = 1,0$ for finding out the relevant load combination for each element.

Loads of crane girder are already multiplied with partial safety factors.

Problem, Sketch, Calculation	Reference	Commentary
Support load from crane girder (horizontal)   $R_{2x, sd} = 35,5 \text{ kN}$ Imperfections (equivalent horizontal unit force)  		
3. Relevant load combinations for ultimate limit state verification 3.1 permanent + snow including crane loads $1,35 (g + s) + R_{x, sd} + R_{y, sd}$	Table 2.1; [2.3.3.1.(5)]	This load combination leads to the following effects: - max N_{sd} at column (4) - max M_{sd} at 3, 4 and 5 - max $R_{v, sd}$ at 7
3.1.1 Determination of the initial sway imperfections → $n_c = 2$; $n_s = 1$ → $\Phi = k_c k_s \varphi_o$ (see table 5.6 in E-EC3) → $\Phi = 1/200$ Equivalent horizontal forces F_i due to g: $\Sigma V_{sd} = 1,35 \cdot 30,4 \cdot 2 = 82,1 \text{ kN}$ due to s: $\Sigma V_{sd} = 1,50 \cdot 45,2 \cdot 2 = 135,6 \text{ kN}$ $F = 82,1 + 135,6 = 217,7 \text{ kN}$ $F_i = \Phi \cdot F = 217,7/200 = 1,1 \text{ kN}$	5.2.1.2; [5.2.1.2] 5.2.3.1.2; Table 5.6 [5.2.4.3] Table 5.5; [Figure 5.2.4]	For the determination of the initial sway imperfections only self-weight and snow is considered. For the columns it must be checked whether member imperfections may be neglected [5.2.4.2.(4)] with the criterion: $\bar{\lambda} > 0,5 \sqrt{\frac{A f_y}{N_{sd}}}$ If this criterion is fulfilled member imperfections shall be considered.

Problem, Sketch, Calculation		Reference	Commentary
3.1.2	Effects of load combination permanent and snow max N_{sd} in column: $N_{sd} = 1,35 \cdot (30,4 + 45,2) + 436,4 + 12,4 + 1,1 \cdot 0,5 = 551,4 \text{ kN}$ max M_{sd} at 4: $M_{sd} = 1,35 \cdot (85,4 + 127,2) + 4,0 + 8,3 = 299,3 \text{ kNm}$ max M_{sd} at 3 and 5: $M_{sd} = 1,35 \cdot (60,3 + 89,9) + 81,4 + 116,0 + 1,1 \cdot 5,0 = 405,7$		$R_{v, sd}$ is equal to N_{sd} in column.
3.2	permanent + wind $1,35 (g + w) + R_{z, sd} + R_{x, sd}$	Table 2.1 [2.3.3.1.(5)]	This load combination leads to the following effect: - max M_{sd} at 2 and 6
3.2.1	Determination of the initial sway imperfections $\rightarrow n_c = 2 ; n_s = 1$ $\rightarrow \Phi = k_c k_s \varphi_o$ (see table 5.6 in E-EC3) $\Phi = 1/200$ Equivalent horizontal forces F_i due to g: $\Sigma V_{sd} = 1,35 \cdot 30,4 \cdot 2 = 82,1 \text{ kN}$ $F_i = \Phi \cdot \Sigma V_{sd} = 82,1/200 = 0,4 \text{ kN}$	5.2.1.2; [5.2.1.2] 5.2.3.1.2; Table 5.6 [5.2.4.3] Table 5.5; [Figure 5.2.4]	
3.2.2	Effects of load combination permanent and wind max M_{sd} at 2: $M_{sd} = 1,35 \cdot (42,2 + 53,8) + 78,0 + 155,7 + 0,4 \cdot 3,5 = 364,7 \text{ kNm}$ max M_{sd} at 6: $M_{sd} = 1,35 \cdot (42,2 - 87,3) - 145,1 - 92,8 - 0,4 \cdot 3,5 = - 300,2 \text{ kNm}$		
3.3	Further comentary Not all possible load combinations are given in this example. It could be that for different elements different load combinations are relevant, e.g. "permanent and snow without crane loads" ($1,35 g + 1,5 s$).		

1.2 Methods of analysis

These examples demonstrate on continuous beams how the design values of action effects are determined using either plastic global analysis or elastic global analysis and plastic or elastic stress distribution. All of the methods of analysis presented may also be applied on frames. The further steps of design are not treated in this part.

Note: For plastic global analysis special requirements specified in [5.2.7, 5.3.3 and 3.2.2.2] shall be satisfied.

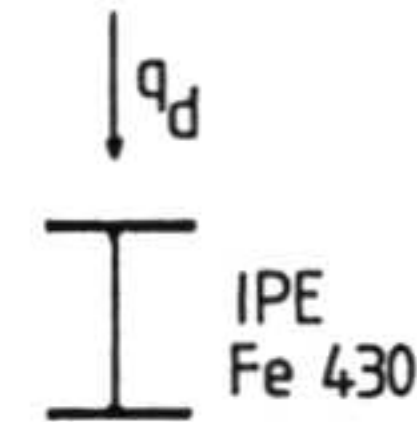
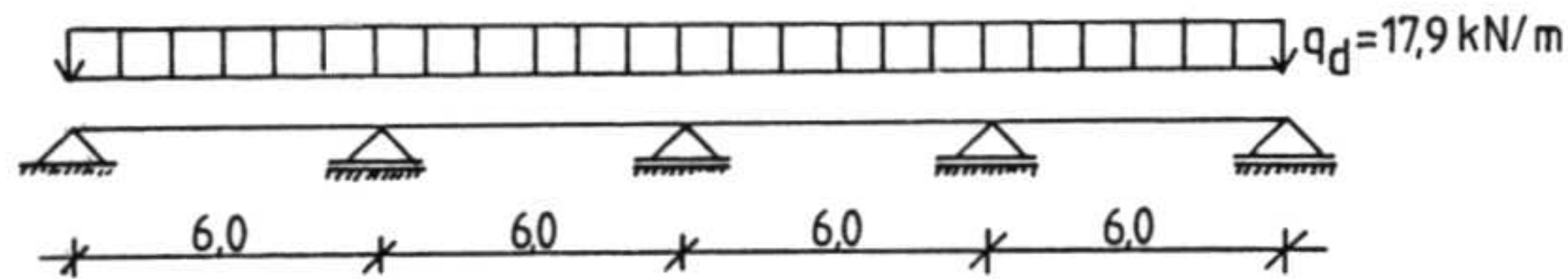
The following examples are included in this chapter:

- | | |
|-----------------------|---|
| Example 1.2.1: | Continuous beam (plastic - plastic) |
| Example 1.2.2: | Continuous beam (elastic - plastic) with limited redistribution |
| Example 1.2.3: | Continuous beam (elastic - elastic) without redistribution |
| Example 1.2.4: | Continuous beam (elastic - elastic) |

Problem, Sketch, Calculation

Example 1.2.1: Continuous beam (plastic - plastic)

1. System



2. Determination of internal forces and moments

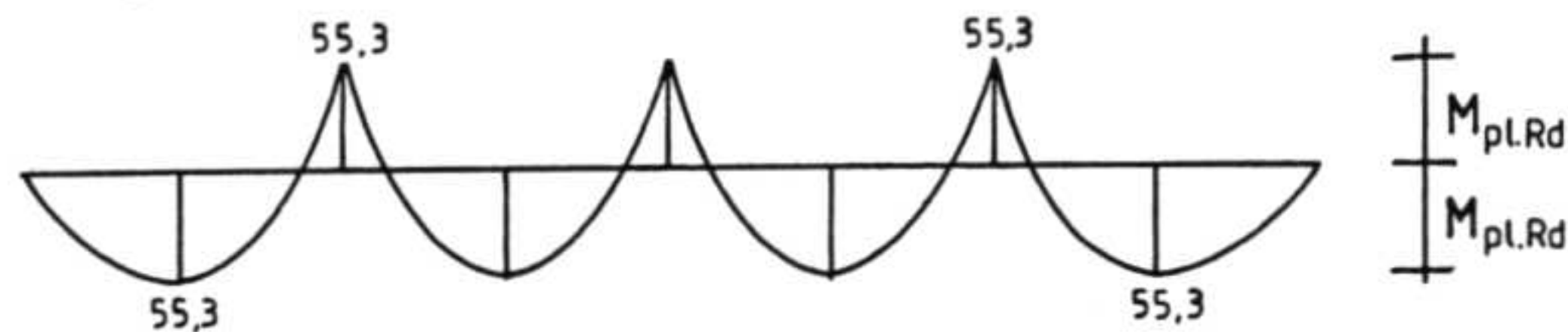
2.1 Plastic global analysis

$$W_{pl} \geq \frac{q_d \cdot l^2 \cdot \gamma_{M0}}{11,666 \cdot f_y} = \frac{17,9 \cdot 6,0^2 \cdot 1,1 \cdot 10^6}{11,666 \cdot 275} = 220,9 \cdot 10^3 \text{ mm}^3$$

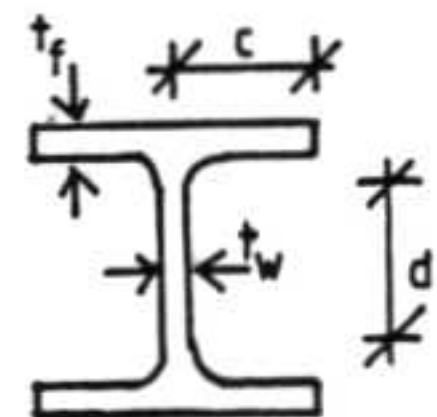
→ chosen: IPE 200 ($W_{pl} = 221 \cdot 10^3 \text{ mm}^3$)

2.2 Plastic bending moment distribution for IPE 200

max $q_d = 17,91 \text{ kN/m}$



2.3 Width-to-thickness ratios



Fe 430 → $\epsilon = 0,92$

Flange:

$$c/t = 50/8,5 = 5,9 < 10\epsilon = 9,2 \quad \rightarrow \text{class 1}$$

Web:

$$d/t = 159/5,6 = 28,4 < 72\epsilon = 66,2 \quad \rightarrow \text{class 1}$$

$$\rightarrow A_{eff} = A; W_{eff} = W_{pl}$$

$$d/t = 28,4 < 69\epsilon = 63,5$$

→ no shear buckling verification is required

Reference

Commentary

The beam is laterally restrained to prevent lateral torsional buckling. Further, lateral restraints have to be provided at all plastic hinge locations, see [5.2.1.4 and 5.3.3].
Fe 430 satisfies the requirements specified in [3.2.2.2]

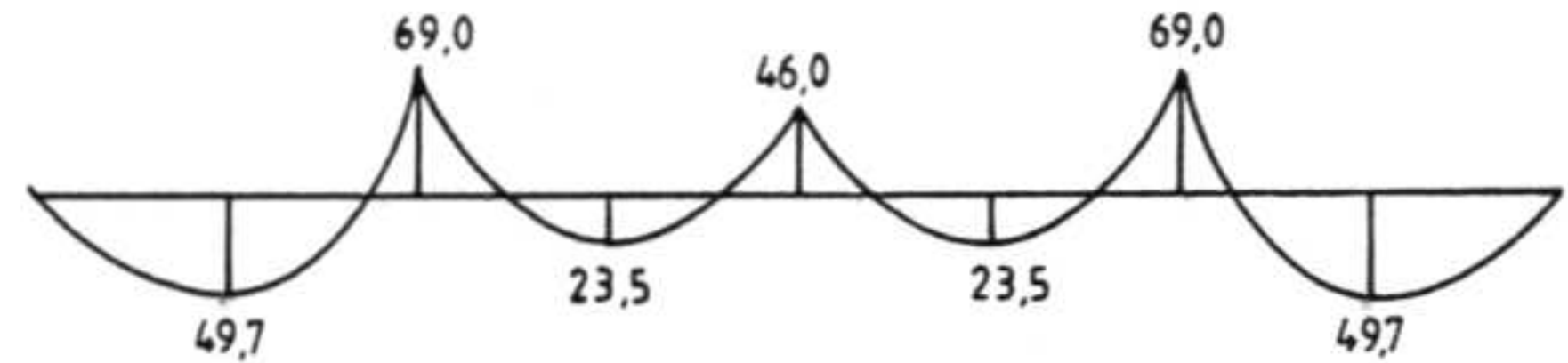
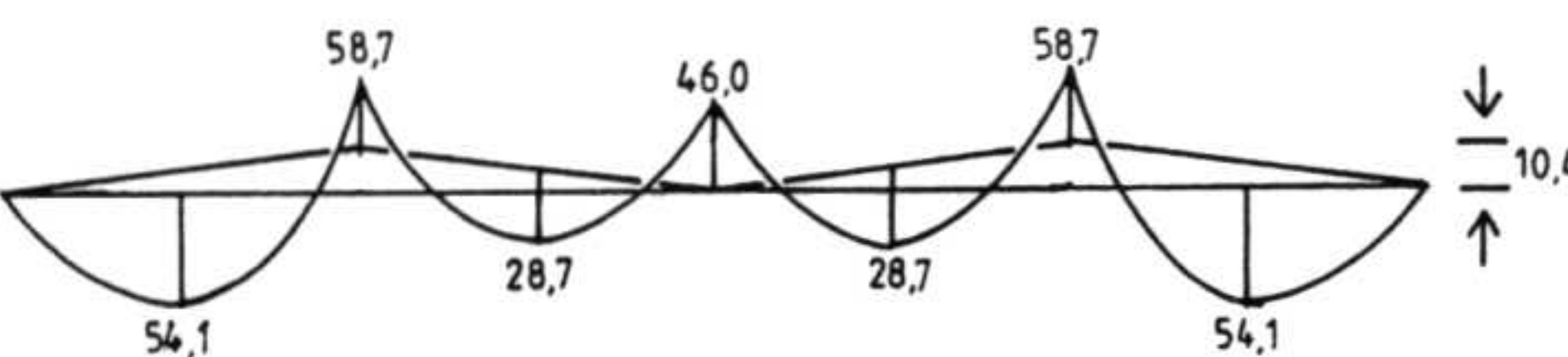
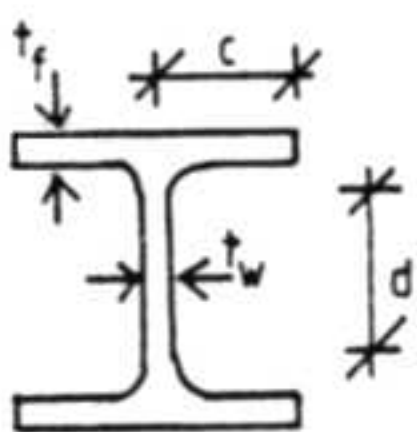
The plastic hinges occur at the first inner support and in the outer span at 2,568 m from the outer support.

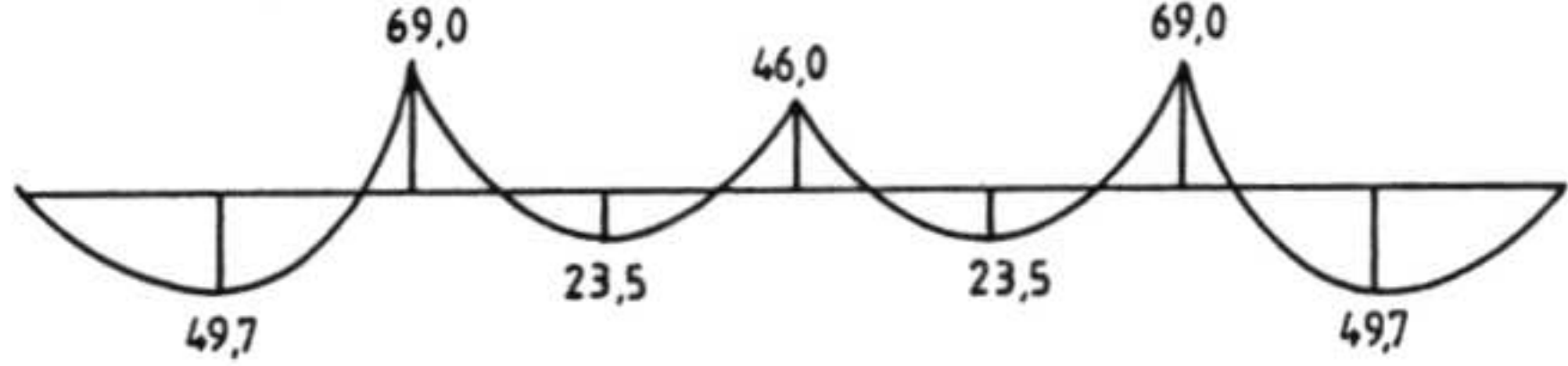
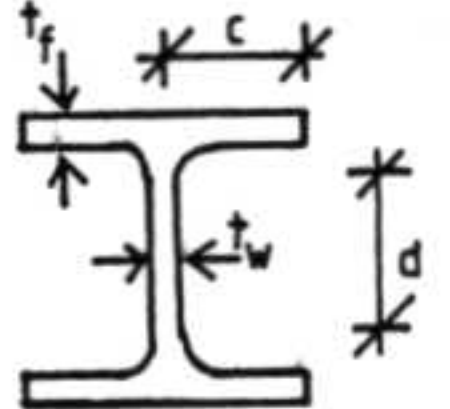
$$M_{pl,Rd} = f_y \cdot W_{pl} / \gamma_{M0} = 275 \cdot 221 \cdot 10^3 \cdot 10^{-6} / 1,1 = 55,3 \text{ kNm}$$

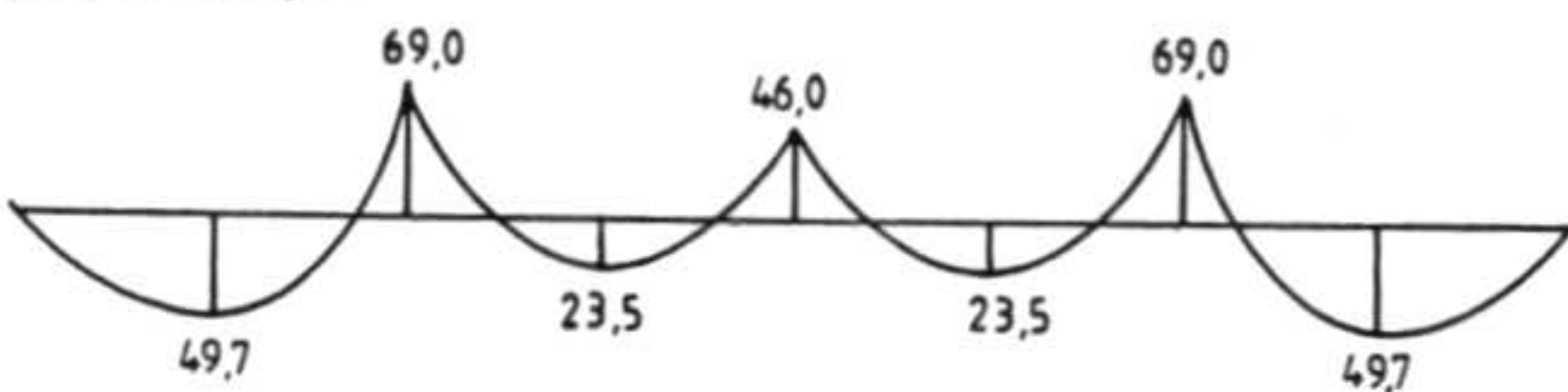
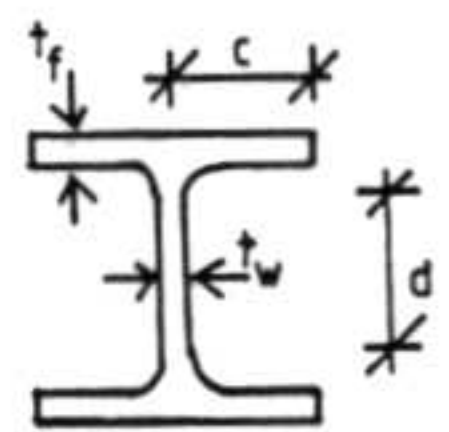
[5.3.3(4);
Table 5.3.1]

5.3.3; [5.4.6]

Plastic global analysis may be used only where the cross-sections satisfy the requirements specified in [5.3.3].

Problem, Sketch, Calculation	Reference	Commentary
Example 1.2.2: Continuous beam (elastic - plastic) with limited redistribution of moments 1. System see example 1 2. Determination of internal forces and moments 2.1 Elastic global analysis 2.1.1 Moment before redistribution  2.1.2 Moment after redistribution  $0,15 \cdot 69,0 = 10,4 \text{ kNm}$ $49,7 + 0,42 \cdot 10,4 = 54,1 \text{ kNm} < 58,7 \text{ kNm}$ $W_{pl} \geq \frac{M_{Sd} \gamma_{Mo}}{f_y} = \frac{58,65 \cdot 1,1 \cdot 10^6}{275} = 234,6 \cdot 10^3 \text{ mm}^3$ → chosen: IPE 220 ($W_{pl} = 285 \cdot 10^3 \text{ mm}^3$) 2.2 Width-to-thickness ratios Fe 430 → $\epsilon = 0,92$ Flange: $c/t = 55/9,2 = 6,0 < 10\epsilon = 9,2$ Web: $d/t = 177/5,9 = 30,0 < 72\epsilon = 66,2$ → $A_{eff} = A$; $W_{eff} = W_{pl}$ $d/t = 30,0 < 69\epsilon = 63,5$ → no shear buckling verification is required 	[5.2.1.3.(3)] Table 5.10; [5.3; Table 5.3.1] 5.3.3; [5.4.6]	The moments calculated using elastic global analysis are modified by redistribution of the peak calculated moment in any member (maximum value equal 15%) and the members have Class 1 or Class 2 cross-sections. Approximative estimation of maximum redistribution: $69 \cdot (1 - x) = 49,7 \cdot (1 + x)$ → $x_{max} \leq 16,3\%$ → redistribution of 15% is allowed

Problem, Sketch, Calculation	Reference	Commentary
Example 1.2.3: Continuous beam (elastic - plastic) without redistribution of moments 1. System see example 1 2. Determination of internal forces and moments 2.1 Elastic global analysis Moment distribution:  $W_{pl} \geq \frac{M_{Sd} \gamma_{Mo}}{f_y} = \frac{69,0 \cdot 1,1 \cdot 10^6}{275} = 276,0 \cdot 10^3 \text{ mm}^3$ → chosen: IPE 220 ($W_{pl} = 285 \cdot 10^3 \text{ mm}^3$) 2.2 Width-to-thickness ratios  Fe 430 → $\epsilon = 0,92$ Flange: $c/t = 55/9,2 = 6,0 < 10\epsilon = 9,2$ Web: $d/t = 177/5,9 = 30 < 72\epsilon = 66,2$ → $A_{eff} = A$; $W_{eff} = W_{pl}$ $d/t = 30 < 69\epsilon = 63,5$ → no shear buckling verification is required	Table 5.10; [5.3; Table 5.3.1] 5.3.3; [5.4.6]	

Problem, Sketch, Calculation	Reference	Commentary
Example 1.2.4: Continuous beam (elastic - elastic) 1. System see example 1 2. Determination of internal forces and moments 2.1 Elastic global analysis  $W_{el} \geq \frac{M_{Sd} \gamma_{Mo}}{f_y} = \frac{69,0 \cdot 1,1 \cdot 10^6}{275} = 276,0 \cdot 10^3 \text{ mm}^3$ → chosen: IPE 240 ($W_{el} = 324 \cdot 10^3 \text{ mm}^3$) 2.2 Width-to-thickness ratios Fe 430 → $\epsilon = 0,92$ Flange: $c/t = 60,0/9,8 = 6,1 < 10\epsilon = 9,2$ Web: $d/t = 190/6,2 = 30,6 < 72\epsilon = 66,2$ → $A_{eff} = A$; $W_{eff} = W_{pl}$ $d/t = 30,6 < 69\epsilon = 63,5$ → no shear buckling verification is required 	Table 5.11; [5.3; Table 5.3.1] 5.3.3; [5.4.6]	

1.3 Frame analysis

The first example demonstrates the application of the criterion "sway - non sway" and how the second order effects in the sway mode are included using first order analysis with amplified sway moments. The second example demonstrate how frame imperfections are determined. The further steps of design are not treated in this part.

The following examples are included in this chapter:

- | | |
|-----------------------|--------------------------------------|
| Example 1.3.1: | Calculation of a sway frame |
| Example 1.3.2: | Determination of frame imperfections |

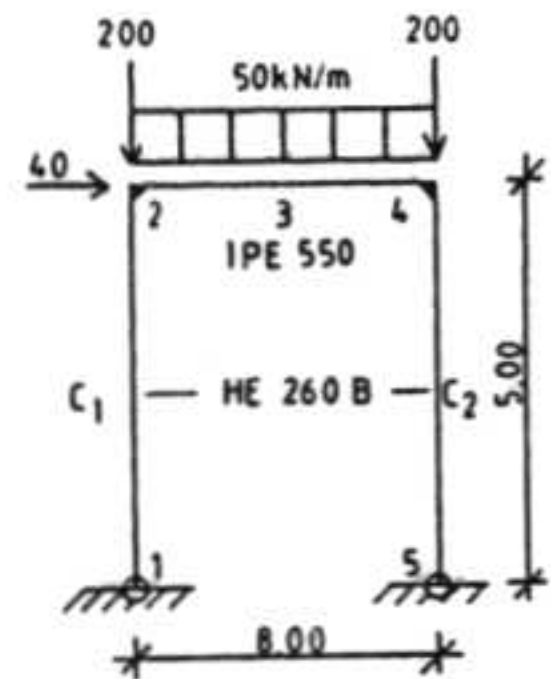
Problem, Sketch, Calculation

Reference

Commentary

Example 1.3.1: Calculation of a sway frame

1. System and loading



2. Frame imperfections

$$F = 2 \cdot 200 + 50 \cdot 8 = 800 \text{ kN}$$

$$\Phi = 1/200$$

$$\rightarrow F_i = 800/200 = 4,0 \text{ kN}$$

3. Internal forces and moments due to 1st order

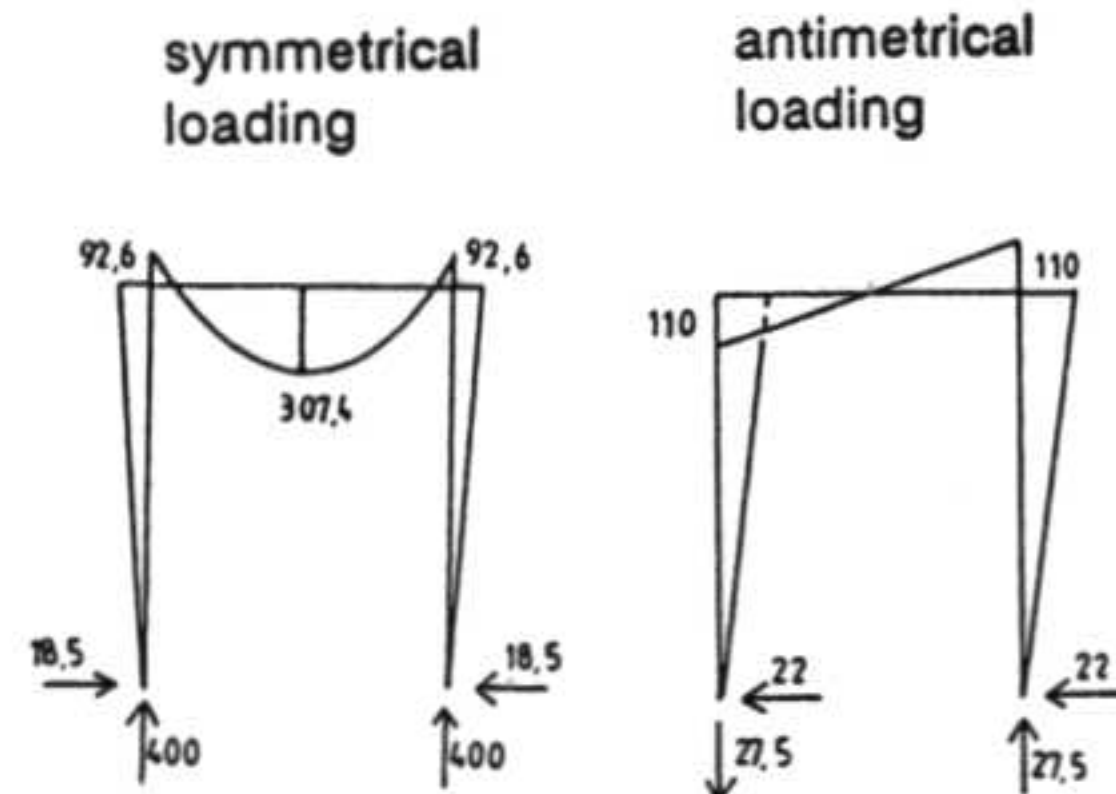
Maximum axial compression in column (C2): $N_{sd} = 427,5 \text{ kN}$

Maximum shear force at point (4) in the beam: $V_{sd} = 227,5 \text{ kN}$

Moment distribution: see sketch

Horizontal deflection of the storey: $\delta = 34,5 \text{ mm}$

Buckling length for non-sway mode of column: $\ell = 3,75 \text{ m}$



4. Criterion sway - non-sway

$$\Sigma V = 800 \text{ kN}$$

$$\Sigma H = 44 \text{ kN (including the equivalent imperfection load)}$$

$$\frac{\delta}{h} \frac{\Sigma V}{\Sigma H} = \frac{34,5 \cdot 10^{-3}}{5,0} \frac{800}{44} = 0,125 > 0,1$$

→ the frame is classified as sway

5.2.3.1.2;
Table 5.6
[5.2.4.3]

Table 5.4
[5.2.5.2.(4)]

The partial safety factor for actions is included in the loading.

The internal forces and moments may generally be determined in the global analysis taking into account the effects of imperfections

constant for frame analysis:

$$k = \frac{I_b}{I_c} \frac{h_c}{\ell_b} = \frac{67120 \cdot 10^4}{14920 \cdot 10^4} \frac{5,0}{8,0} = 2,812$$

In the calculation of the deflection the effects of imperfections are taken into account.

This criterion gives an indication when second order theory should be used for structures as frames, bracing systems, etc. Alternatively the $\Delta M/M$ - method (mentioned in 5.2.5.2(3)) may be used (Normally 2 iteration steps are needed).

$$\frac{\delta}{h} \frac{\Sigma V}{\Sigma H} = \frac{\Delta M}{M}$$

(Requirement: After each iteration step $\Delta M/M = \text{const.}$)

$$M'' = M_0 + M_1 / (1 - M_2/M_1)$$

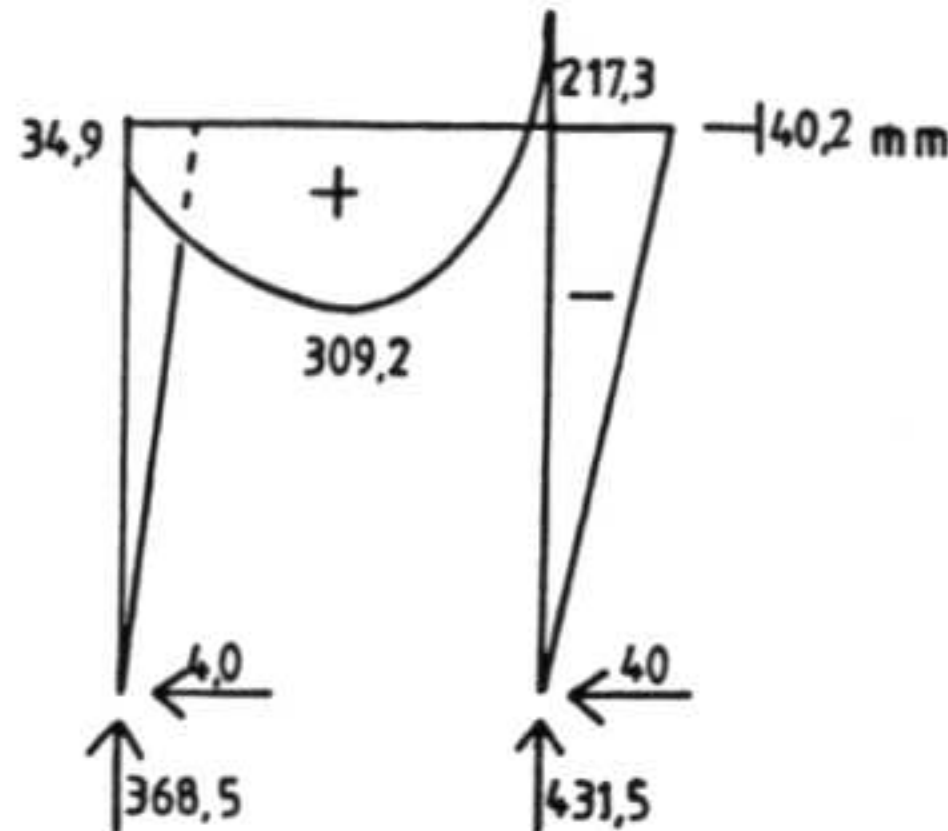
M_0 edge moment due to symmetrical loading

M_1 edge moment due to antimetrical loading

M_2 edge moment after first iteration

$$M_2 = 2,5 \cdot 800 \cdot 34,5 / 5000 = 13,8 \text{ kNm}$$

$$\text{here: } M_2/M_1 = 13,8/110 = 0,125$$

Problem, Sketch, Calculation	Reference	Commentary
5. Verification using 1st order theory with amplified sway moments 5.1 Internal forces and moments $\alpha = \frac{1}{1 - \frac{\delta \sum V}{h \sum H}} = \frac{1}{1 - 0,125} = 1,14$ For comparison: Moment distribution (calculation by computer)  $M_{4.Sd}^{II} = 92,6 + 1,14 \cdot 110 = 218 \text{ kNm}$ $N_{c.Sd}^{II} = 400,0 + 1,14 \cdot 27,5 = 431,4 \text{ kN}$ $V_{4b.Sd}^{II} = 200,0 + 1,14 \cdot 27,5 = 231,4 \text{ kN}$ $V_{4c.Sd}^{II} = 18,5 + 1,14 \cdot 22 = 43,6 \text{ kN}$ 5.2 Moment resistance $M_{pl.Rd} = W_{pl} \frac{f_y}{\gamma_{Mo}} = 1283 \cdot 10^3 \cdot \frac{235}{1,1} \cdot 10^{-6} = 274,1 \text{ kNm}$ 5.3 Shear resistance of the cross-section Beam: $V_{Rd} = \frac{A_v f_y}{\sqrt{3} \gamma_{Mo}} = \frac{61,0 \cdot 10^2 \cdot 235}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 752,4 \text{ kN}$ $0,5V_{Rd} = 376,2 \text{ kN} > 200 + 1,14 \cdot 27,5 = 231,4 \text{ kN}$ $\rightarrow$ no interaction bending - shear is required Column: $V_{Rd} = \frac{A_v f_y}{\sqrt{3} \gamma_{Mo}} = \frac{26,0 \cdot 10^2 \cdot 235}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 320,7 \text{ kN}$ $0,5V_{Rd} = 160,3 \text{ kN} > 18,5 + 1,14 \cdot 22,0 = 43,6 \text{ kN}$ $\rightarrow$ no interaction bending - shear is required 5.4 Axial compression Column: $N_{pl.Rd} = \frac{A \cdot f_y}{\gamma_{Mo}} = \frac{118 \cdot 10^2 \cdot 235}{1,1} \cdot 10^{-3} = 2520,9 \text{ kN}$ $0,25N_{Rd} = 630,2 \text{ kN}$ $dt f_y / 2 \gamma_{Mo} = 177 \cdot 10 \cdot 235 / (2 \cdot 1,1) \cdot 10^{-3} = 189,1 \text{ kN} < 400 + 1,14 \cdot 27,5 = 431,4 \text{ kN}$ $\rightarrow$ interaction bending - axial compression is required	5.2.3.1.3; [5.2.6.2] 5.3.2; [5.4.5] 5.3.3 [5.4.6] 5.3.1 [5.4.4]	In the amplified sway moment method the sway internal forces and moments found by first order elastic analysis should be increased by multiplying them by the following factor: $\frac{1}{1 - \frac{\delta \sum V}{h \sum H}} = \frac{1}{1 - \frac{\Delta M}{M}}$

Problem, Sketch, Calculation	Reference	Commentary
5.5 Interaction bending - axial compression $M_{N,Rd} = 1,11 \cdot \left(1 - \frac{N_{Sd}}{N_{pl,Rd}}\right) \cdot M_{pl,Rd} = 1,11 \cdot \left(1 - \frac{427,5}{2520,9}\right) \cdot 274,1 = 252,7 \text{ kNm} > 92,6 + 110 \cdot 1,14 = 218,0 \text{ kNm}$ 5.6 Member resistance of the column Buckling length of column: $\ell = 3,98 \text{ m}$ From the moment distribution $\rightarrow \beta_M = 1,8$ $\bar{\lambda} = \frac{\ell}{i} \cdot \frac{1}{\lambda_1} = \frac{3,98 \cdot 10^3}{112} \cdot \frac{1}{93,9} = 0,38$ using buckling curve b $\rightarrow \chi_b = 0,9339$ $N_{b,Rd} = \chi_b \cdot \frac{A \cdot f_y}{\gamma_{M1}} = 0,9339 \cdot \frac{118 \cdot 10^2 \cdot 235}{1,1} \cdot 10^{-3} = 2354,3 \text{ kN} > 431,0 \text{ kN}$ $n = N_{Sd}/N_{b,Rd} = 431,0/2354,3 = 0,183$ $\alpha = W_{pl}/W_{el} - 1 = 1283 \cdot 10^3 / 1150 \cdot 10^3 - 1 = 0,116$ $\mu = \lambda_2(2\beta_M - 4) + \alpha = 0,36(2 \cdot 1,8 - 4) + 0,116 = -0,028 < 0,9$ $1 - \mu n / \gamma_{M1} = 1 + 0,028 \cdot 0,183 / 1,1 = 1,005 < 1,5 \rightarrow k = 1,005$ $n + \frac{k_y M_{Sd}}{W_{y,pl} \frac{f_y}{\gamma_{M0}}} = 0,183 + \frac{1,005 \cdot 218,0}{1283 \cdot 10^3 \cdot \frac{235}{1,1}} \cdot 10^6 = 0,98 < 1 \quad \text{this check is fulfilled}$	5.3.4; [5.4.9] 5.4.2; [5.5.1; 5.2.6.2.(2)]	When second order elastic global analysis is used, in-plane buckling lengths for the non-sway mode may be used for member design.

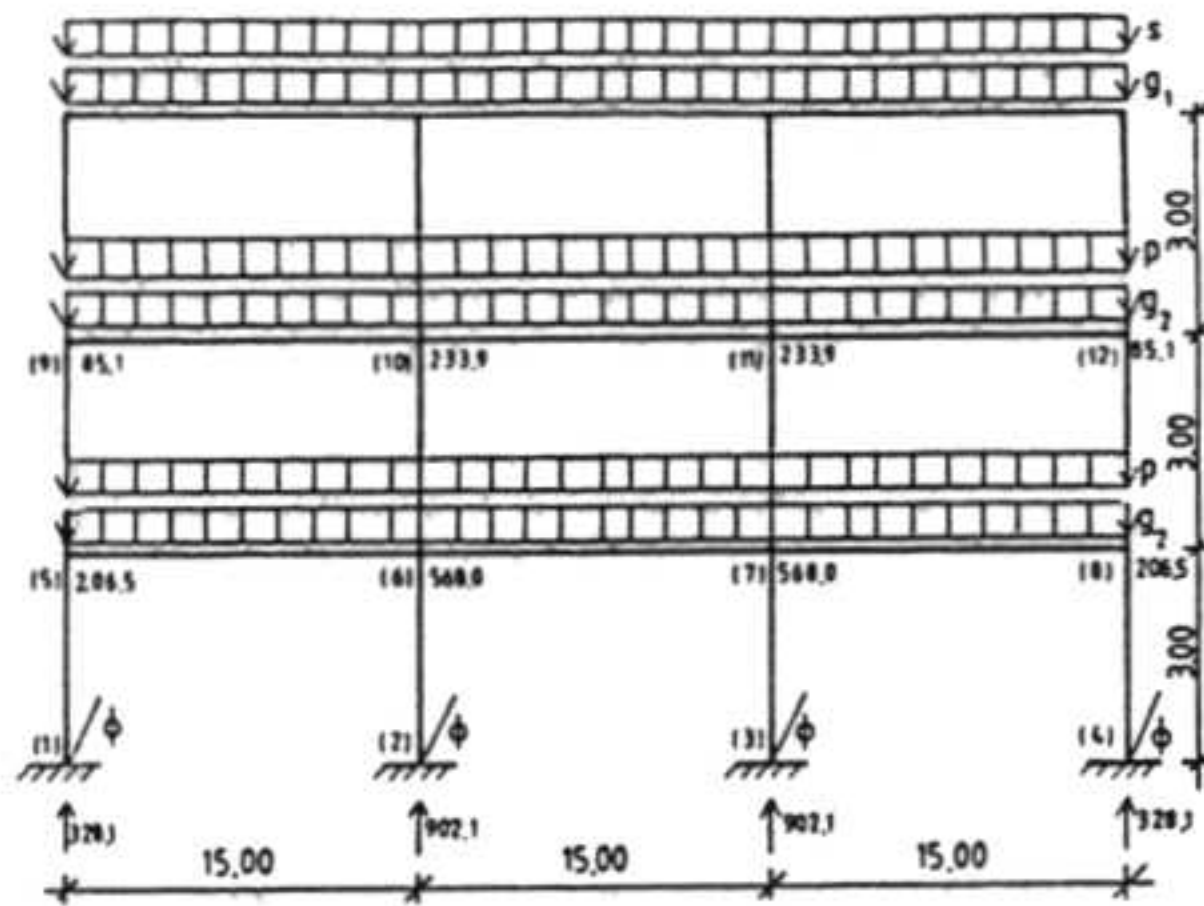
Problem, Sketch, Calculation

Reference

Commentary

Example 1.3.2: Determination of frame imperfections

1. System and loading



$$\begin{aligned} g_1 &= 6,0 \text{ kN/m} \\ g_2 &= 10,0 \text{ kN/m} \\ p &= 5,0 \text{ kN/m} \\ s &= 4,5 \text{ kN/m} \end{aligned}$$

2. Determination of the imperfections for the load case 1,35 (g + p + s)

2.1 Total vertical reaction per storey

2.1.1 Ground floor

$$\Sigma V_{sd} = 1,35 \cdot [6,0 + 4,5 + 2 \cdot (10,0 + 5)] \cdot 15,0 \cdot 3,0 = 2460,4 \text{ kN}$$

Half of the "mean reaction force" per column:

$$0,5 \cdot \bar{V}_{sd} = 0,5 \cdot \Sigma V_{sd} / 4 = 0,5 \cdot 2460,4 / 4 = 307,6 \text{ kN}$$

The axial force of each column at this floor is higher than 307,6 kN, therefore all columns are included in n_c .

2.1.2 1st floor

$$\Sigma V_{sd} = 1,35 \cdot [6,0 + 4,5 + 10,0 + 5] \cdot 15,0 \cdot 3,0 = 1549,1 \text{ kN}$$

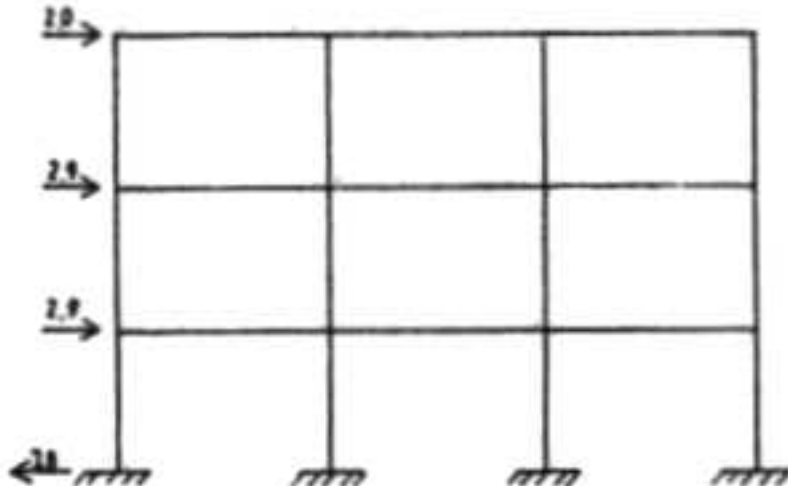
Half of the "mean reaction force" per column:

$$0,5 \cdot \bar{V}_{sd} = 0,5 \cdot \Sigma V_{sd} / 4 = 0,5 \cdot 1549,1 / 4 = 193,6 \text{ kN}$$

The axial force of each column at this floor is higher than 193,6 kN, therefore all columns are included in n_c .

5.2.3.1.2;
[5.2.4.3]

5.2.3.1.2;
[5.2.4.3]

Problem, Sketch, Calculation	Reference	Commentary
2.1.3 2nd floor $\Sigma V_{sd} = 1,35 \cdot [6,0 + 4,5] \cdot 15,0 \cdot 3,0 = 637,9 \text{ kN}$ Half of the "mean reaction force" per column: $0,5 \cdot \bar{V}_{sd} = 0,5 \cdot \Sigma V_{sd} / 4 = 0,5 \cdot 637,9 / 4 = 79,7 \text{ kN}$ The axial force of each column at this floor is higher than 79,7 kN, therefore all columns are included in n_c.	5.2.3.1.2; [5.2.4.3]	
2.2 Determination of the initial sway imperfection All columns extend through all storeys and therefore are included in n_s. → $n_c = 4$; $n_s = 3$ → $\Phi = k_c k_s \varphi_0$ (see table 5.6 in E-EC3) $\Phi = 1/315$  $F_{L,2nd} = 637,9 / 315 = 2,0 \text{ kN}$ $F_{L,1st} = 911,3 / 315 = 2,9 \text{ kN}$ $F_{L,Gf} = 911,3 / 315 = 2,9 \text{ kN}$	5.2.3.1.2; [5.2.4.3]	

1.4 Bracing system analysis

The first three examples demonstrate the application of the criterion "braced - unbraced". The fourth example demonstrates how bracing imperfections are determined. The further steps of design are not treated in this part.

The following examples are included in this chapter:

- | | |
|-----------------------|---|
| Example 1.4.1: | A frame is braced by a bracing system |
| Example 1.4.2: | A frame is braced by a frame |
| Example 1.4.3: | A frame is braced by a bracing system |
| Example 1.4.4: | Bracing system imperfections for a wind bracing |

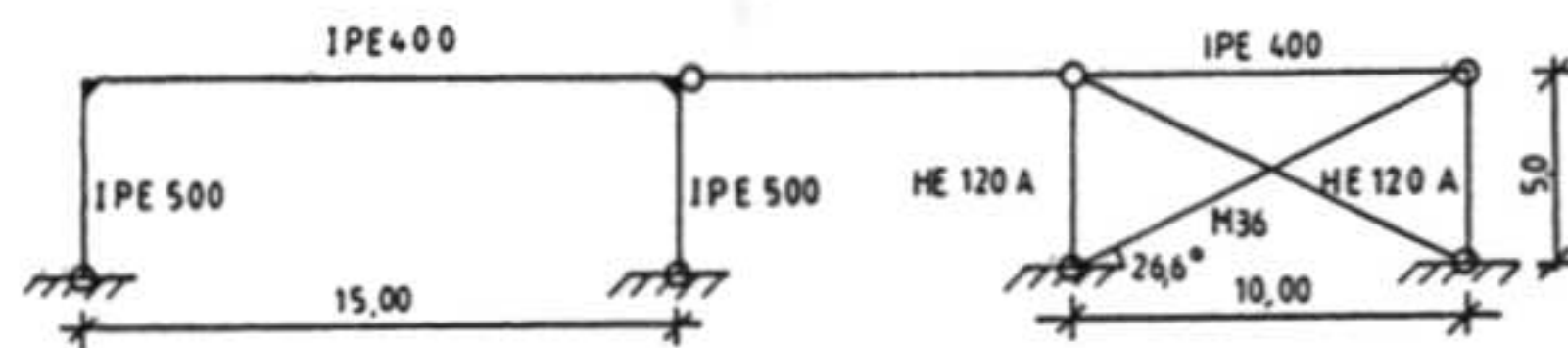
Problem, Sketch, Calculation

Reference

Commentary

Example 1.4.1: A frame is braced by a bracing system

1. System



Frame:

column: IPE 500
beam: IPE 400
column 1: HE 120 A
column 2: HE 120 A
beam: IPE 400
diagonals: ϕ 36

Bracing system:

2. Stiffness of the frame

Assumption: $EA \approx \infty$

$$\delta = \int \frac{M M'}{EI} dx$$

$$\delta = \frac{2,5^2 \cdot 5,0}{3 \cdot 210000 \cdot 48200 \cdot 10^4} \cdot 2 \cdot 10^9 + \frac{2,5^2 \cdot 7,5}{3 \cdot 210000 \cdot 23130 \cdot 10^4} \cdot 2 \cdot 10^9 = 0,00085 \text{ m/kN}$$

$$S_{\text{frame}} = \frac{1}{\delta} = 1177,6 \text{ kN/m}$$

3. Stiffness of the bracing system

$$\delta = \int \frac{N N'}{EA} dx$$

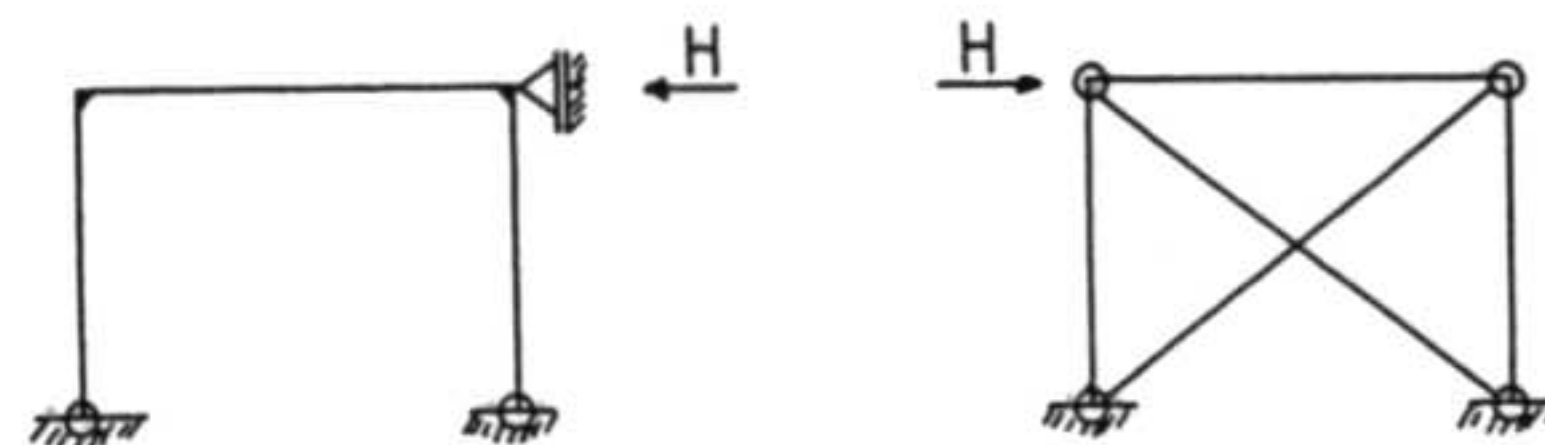
$$\delta = \frac{1,0}{210000 \cdot 10,2 \cdot 10^2 \cdot \cos^2 26,6^\circ} \cdot \frac{10,0}{\cos 26,6^\circ} \cdot 10^3 = 0,000065 \text{ m/kN}$$

$$S_{\text{brace}} = \frac{1}{\delta} = 15336 \text{ kN/m}$$

4. Criterion

$$S_{\text{brace}}/S_{\text{frame}} = 15336/1177,6 = 13,02 > 5$$

→ the continued system can be calculated as a unique system or separated into sub-systems where the frame may be considered as braced and the bracing system is subjected with the horizontal forces



The criterion braced - unbraced gives an indication to which part of a structure or sub-structure the horizontal loads are transferred (i.e. any horizontal loads plus the initial sway imperfections applied on a braced frame are treated as affecting only the bracing system). However it is possible that the bracing system itself could be classified as sway.

A braced frame may be treated as fully supported laterally.

Only the bracing diagonal loaded in tension is used to calculate the system stiffness (since the compression diagonal has insignificant resistance).

5.2.3; Table 5.3
[5.2.5.3]

A frame may be classified as braced if the stiffness of the bracing system is at least 5 times greater than the stiffness of the frame.

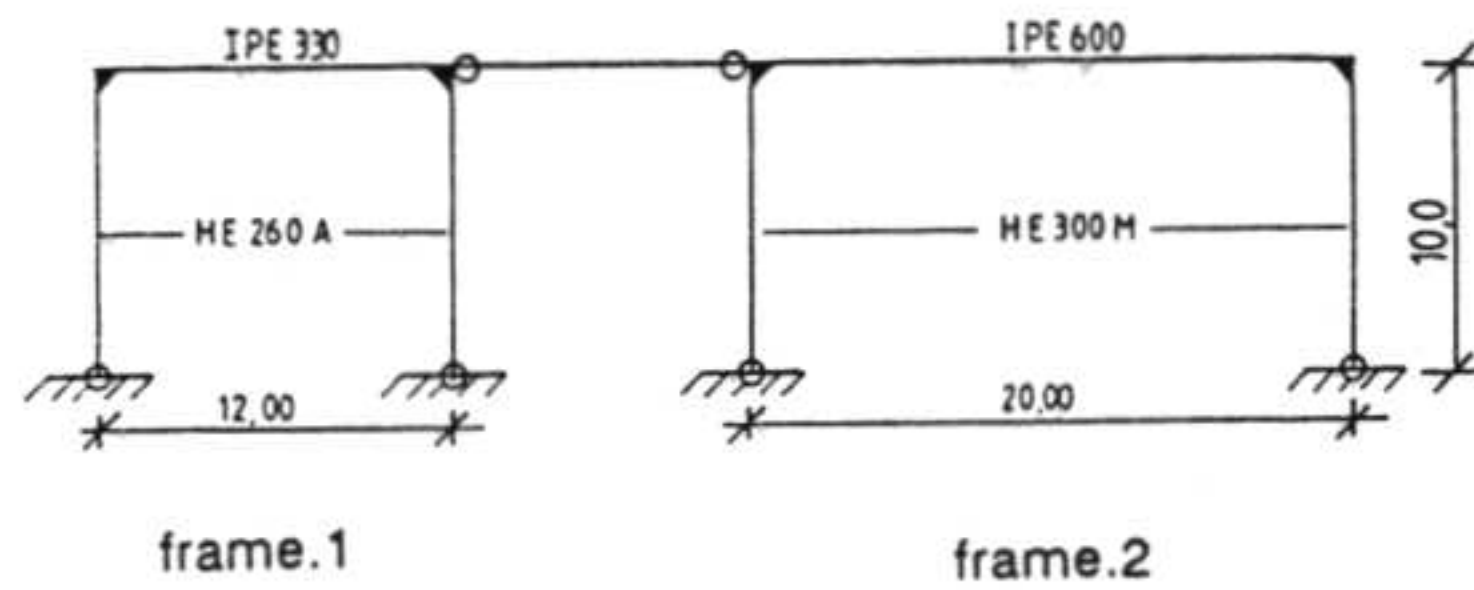
Problem, Sketch, Calculation

Reference

Commentary

Example 1.4.2: A frame is braced by a frame

1. System



Frame 1: column: HE 260 A
beam: IPE 330
Frame 2: column: HE 300 M
beam: IPE 600

2. Stiffness of the frame.1

$$\delta = \int \frac{M M'}{EI} dx \quad \text{Assumption: } EA \approx \infty$$

$$\delta = \frac{5,0^2 \cdot 10,0}{3 \cdot 210000 \cdot 10450 \cdot 10^4} \cdot 2 \cdot 10^9 + \frac{5,0^2 \cdot 6,0}{3 \cdot 210000 \cdot 11770 \cdot 10^4} \cdot 2 \cdot 10^9 = 0,0116 \text{ m/kN}$$

$$S_{\text{frame.1}} = \frac{1}{\delta} = 85,9 \text{ kN/m}$$

3. Stiffness of the frame.2

$$\delta = \int \frac{M M'}{EI} dx \quad \text{Assumption: } EA \approx \infty$$

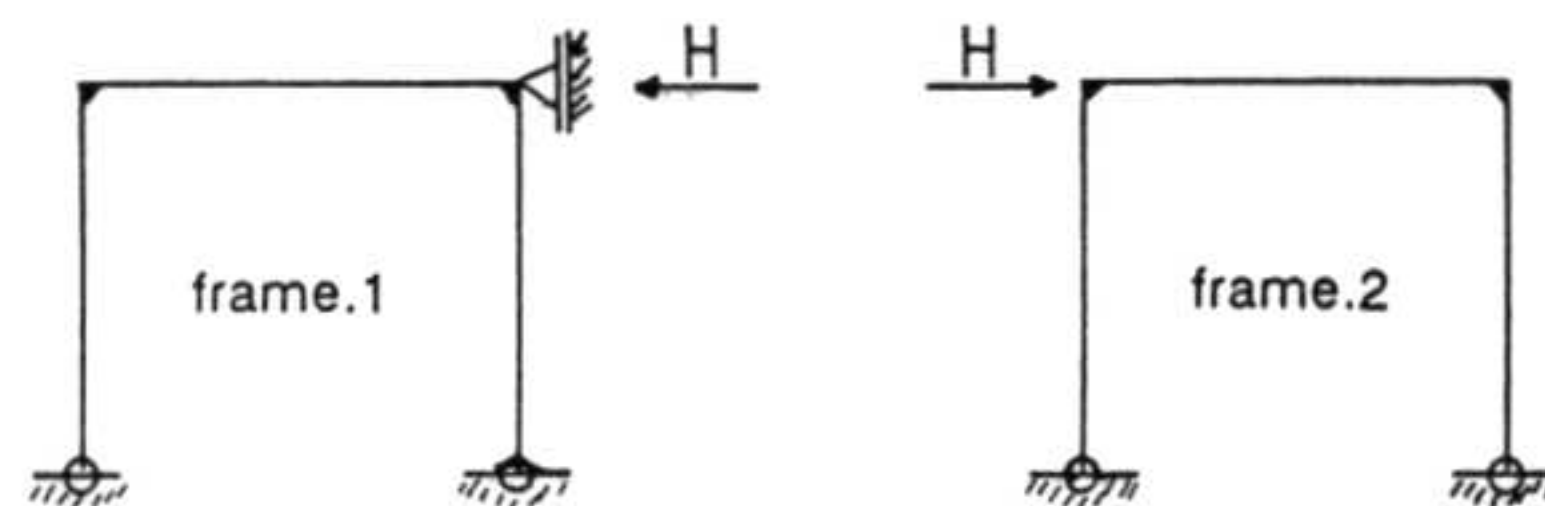
$$\delta = \frac{5,0^2 \cdot 10,0}{3 \cdot 210000 \cdot 59200 \cdot 10^4} \cdot 2 \cdot 10^9 + \frac{5,0^2 \cdot 10,0}{3 \cdot 210000 \cdot 92080 \cdot 10^4} \cdot 2 \cdot 10^9 = 0,0022 \text{ m/kN}$$

$$S_{\text{frame.2}} = \frac{1}{\delta} = 454,0 \text{ kN/m}$$

4. Criterion

$$S_{\text{frame.2}}/S_{\text{frame.1}} = 454/85,9 = 5,29 > 5 \cdot S_{\text{frame.1}}$$

→ frame.1 may be considered as braced



5.2.3; Table 5.3
[5.2.5.3]

Frame.1 may be classified as braced if the stiffness of frame.2 is 5 times greater than the stiffness of the frame.1.
The continued system may be separated into subframes where the frame.1 may be considered as braced and the frame.2 which is designed to resist the horizontal forces.

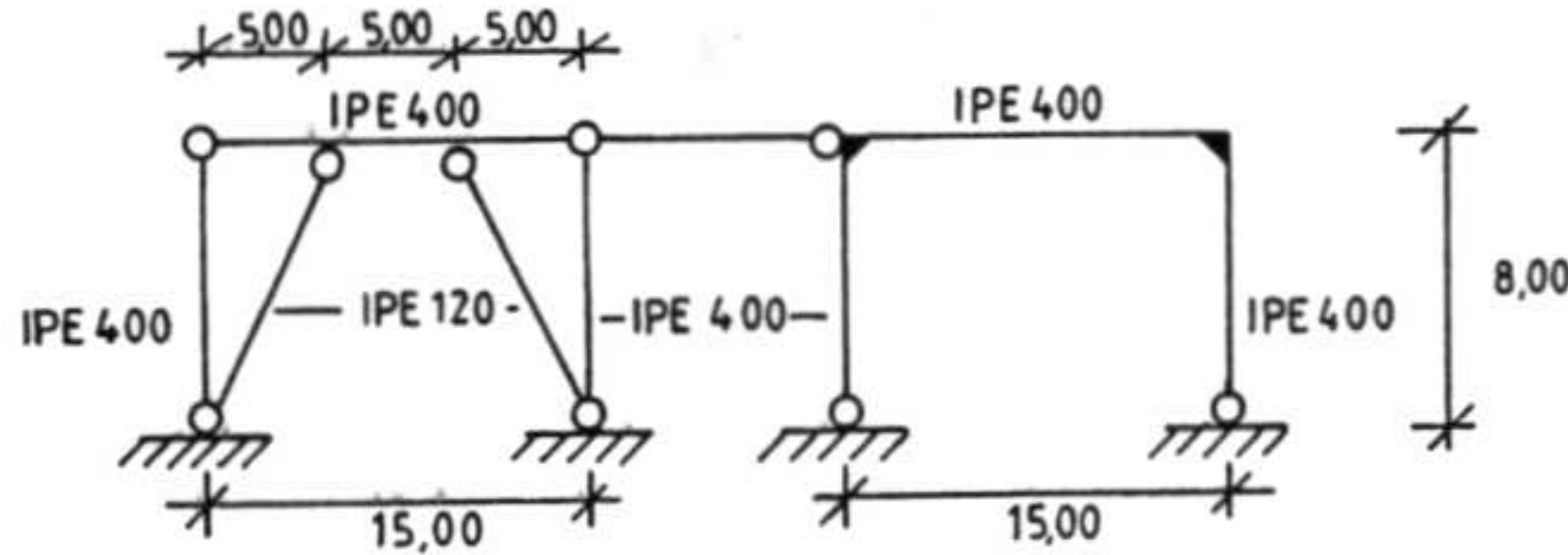
Problem, Sketch, Calculation

Reference

Commentary

Example 1.4.3: A frame is braced by a bracing system

1. System



Frame: column: IPE 400
beam: IPE 400
Bracing system: column: IPE 400
beam: IPE 400
Diagonals: IPE 120

2. Stiffness of the frame

Assumption: $EA \approx \infty$

$$\delta = \int \frac{N N'}{EA} dx + \int \frac{M M'}{EI} dx$$

$$\delta = \frac{4,0^2 \cdot 8,0}{3 \cdot 210000 \cdot 23130 \cdot 10^4} \cdot 2 \cdot 10^9 + \frac{4,0^2 \cdot 7,5}{3 \cdot 210000 \cdot 23130 \cdot 10^4} \cdot 2 \cdot 10^9 = 0,0034 \text{ m/kN}$$

$$S_{\text{frame},1} = \frac{1}{\delta} = 293,8 \text{ kN/m}$$

3. Stiffness of the bracing system

$$\delta = \int \frac{N N'}{EA} dx + \int \frac{M M'}{EI} dx$$

$$\delta = \frac{0,27^2 \cdot 8,0 \cdot 2 \cdot 10^3}{210000 \cdot 84,5 \cdot 10^2} + \frac{0,94^2 \cdot 9,434 \cdot 2 \cdot 10^3}{210000 \cdot 13,2 \cdot 10^2} + \frac{5,0 \cdot 10^3}{210000 \cdot 84,5 \cdot 10^2} + \frac{0,5^2 \cdot 5,0 \cdot 10^3}{210000 \cdot 84,5 \cdot 10^2} +$$

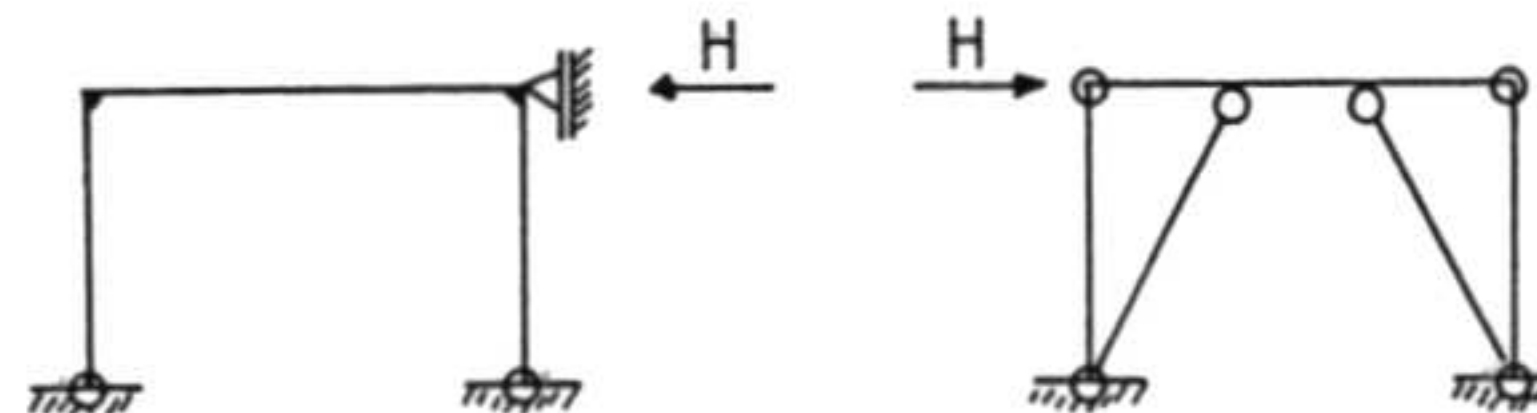
$$\frac{1,33^2 \cdot 5,0 \cdot 2 \cdot 10^9}{3 \cdot 210000 \cdot 23130 \cdot 10^4} + \frac{1,33^2 \cdot 2,5 \cdot 2 \cdot 10^9}{3 \cdot 210000 \cdot 23130 \cdot 10^4} = 0,00024641 \text{ m/kN}$$

$$S_{\text{frame},2} = \frac{1}{\delta} = 4058,3 \text{ kN/m}$$

4. Criterion

$$S_{\text{brace}}/S_{\text{frame}} = 4058,3/293,8 = 13,8 > 5$$

→ the frame could be considered as braced



5.2.3; Table 5.3
[5.2.5.3]

For the calculation of the system - stiffness both diagonals are considered (The connections of the diagonals must be formed such that the compression forces can be transferred).

The frame may be classified as braced if the stiffness of the bracing system is at least 5 times greater than the stiffness of the frame. The frame may be considered as braced and the bracing system is designed to resist the horizontal forces.

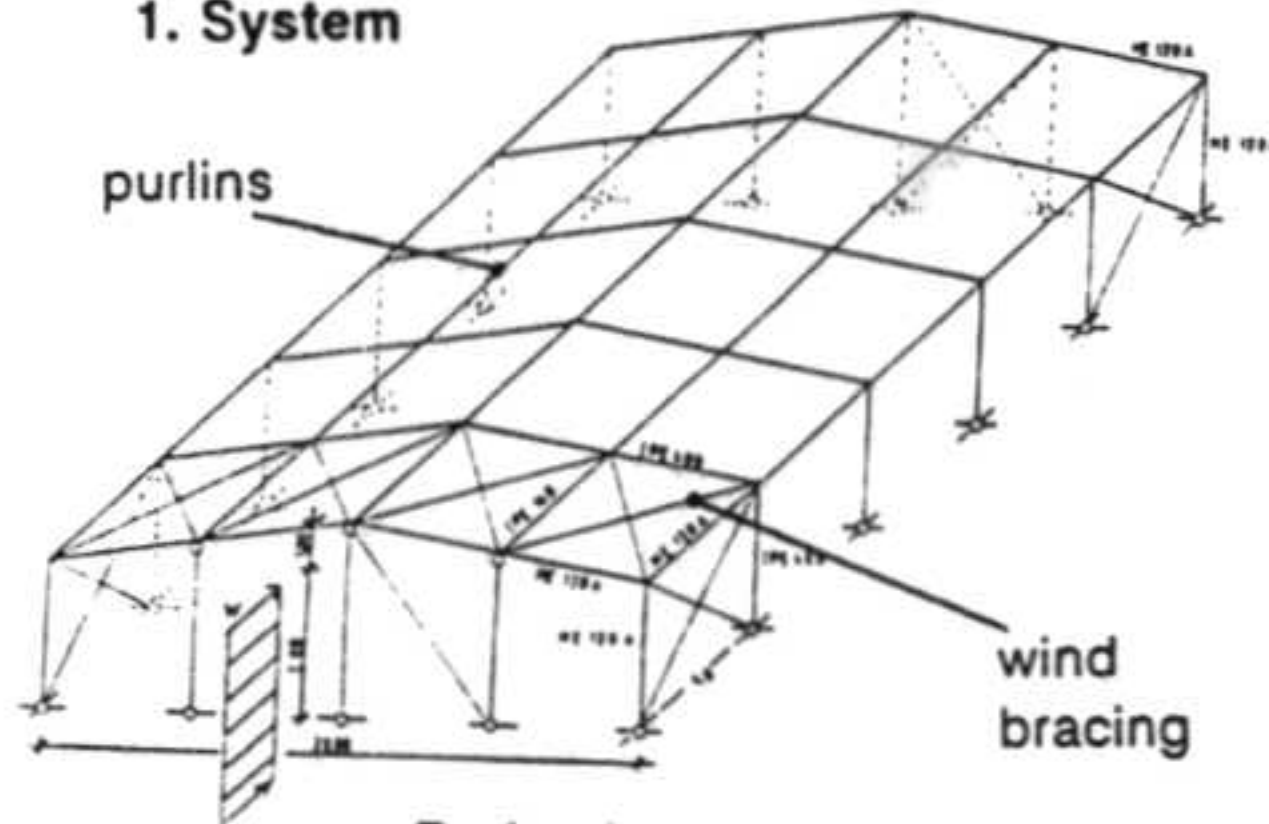
Problem, Sketch, Calculation

Reference

Commentary

Example 1.4.4: Bracing system imperfections for a wind bracing

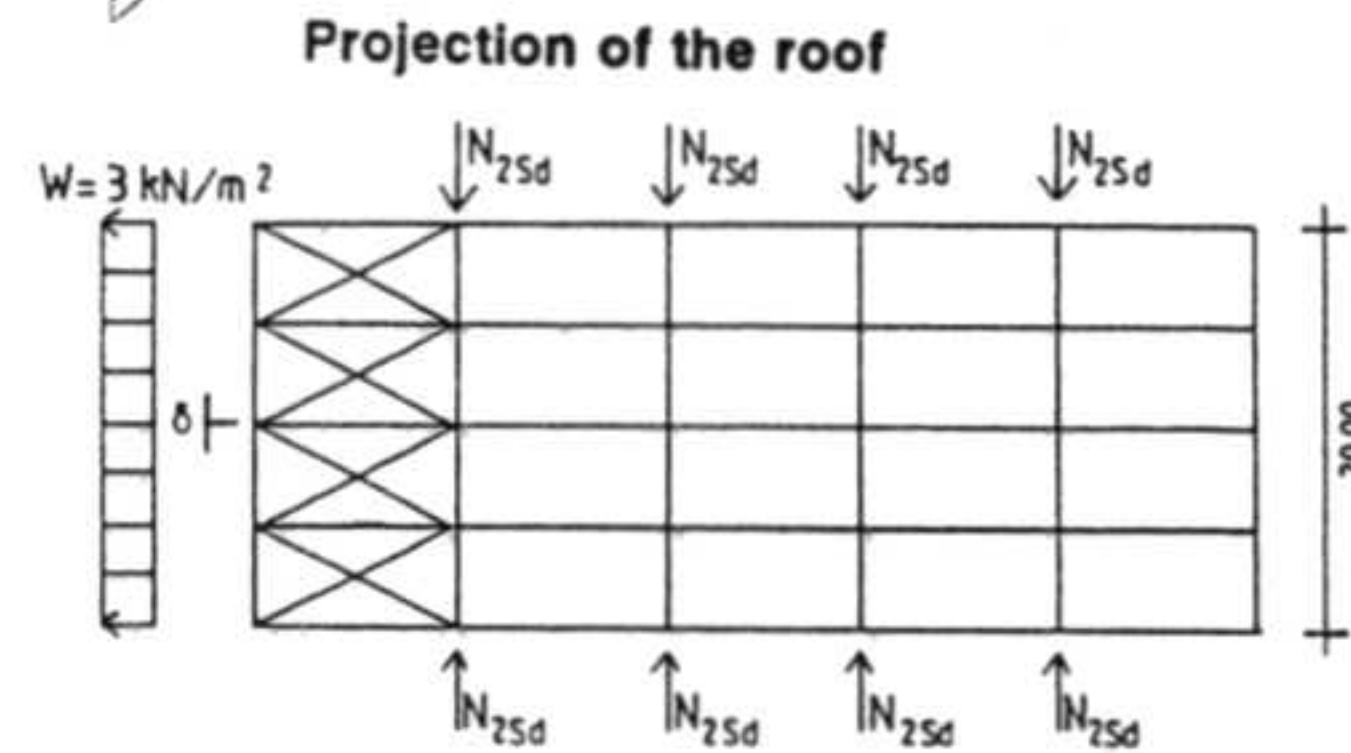
1. System



Frames: column: IPE 500
 beams: IPE 400
 Bracing: Diagonals: L 60x40x5
 Chord 1: HE 120 A
 Chord 2: IPE 400

2. Loading

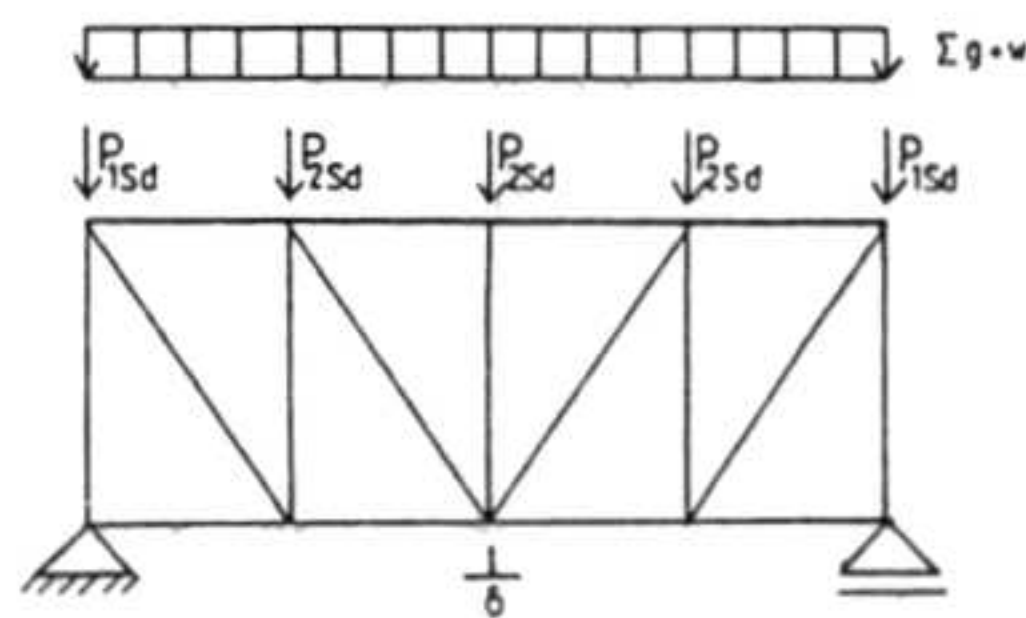
$w = 1,5 \cdot 2,0 = 3,0 \text{ kN/m}$
 Mid-span moment of beam:
 $M_{Sd} = 310 \text{ kNm}$
 Axial force applied on the bracing system:
 $N_{Sd} = M_{Sd}/h = 310/0,4 = 775 \text{ kN}$



3. Determination of the bracing system imperfections

Assumption: the total deflection δ is lower than $L/2000$

with $n_i = 4$ (number of members to be restrained)
 $n_p = 4$ (number of panels) $\rightarrow \zeta = 1,0$
 $\Sigma N_{Sd} = 4 \cdot 775 = 3100 \text{ kN}$



Equivalent stabilizing force Σq

$$\Sigma q = \frac{1,0}{67,9} \frac{3100,0}{20,0} = 2,28 \text{ kN/m}$$

4. Determination of the deformation δ of the bracing system

The total loads applied on the wind bracing are $\Sigma q + w$

Equivalent panel loads from $\Sigma q + w$

$$P_{2Sd} = (2,28 + 3,0) \cdot 20/4 = 26,4 \text{ kN}$$

$$P_{1Sd} = P_{2Sd}/2 = 13,2 \text{ kN}$$

$\delta = 9,5 \text{ mm} < 10 \text{ mm} = L/2000$

$$\gamma_F = 1,5$$

h - depth of the beam

Table 5.8
 [5.2.4.4.(3)]

5.2.3.2.1;
 Table 5.7;
 Table 5.8
 [5.2.4.4]

Table 5.8;
 [Figure 5.2.5]

Table 5.8
 [Figure 5.2.5]

The bracing system has to be designed to resist [5.2.5.3.(6)]:

- any loads applied directly on the bracing system
- any horizontal loads applied to the frames which it braces
- the effects of the initial imperfections

Part 2

Members in compression
Members in bending
Combined loading - Bending and compression
Local stresses

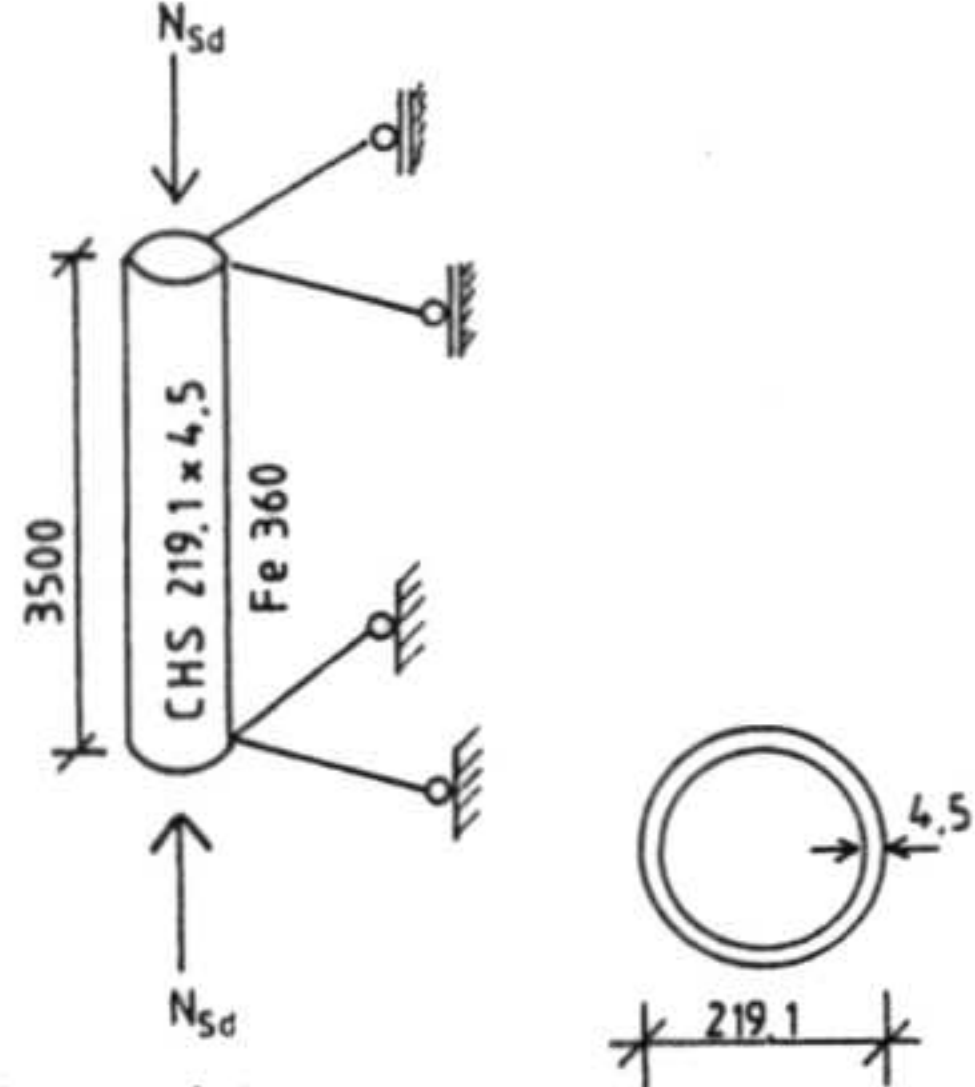
2.1 Members in compression

These examples demonstrate the verification of members in compression assuming design values of action effects (N_{sd}) which have been calculated by an analysis of the structure and these values already include ψ , γ_F etc.

The results presented in the examples are rounded values. For the purpose of easy re-calculation each formula and each check is calculated with the rounded values.

The following examples are included in this chapter:

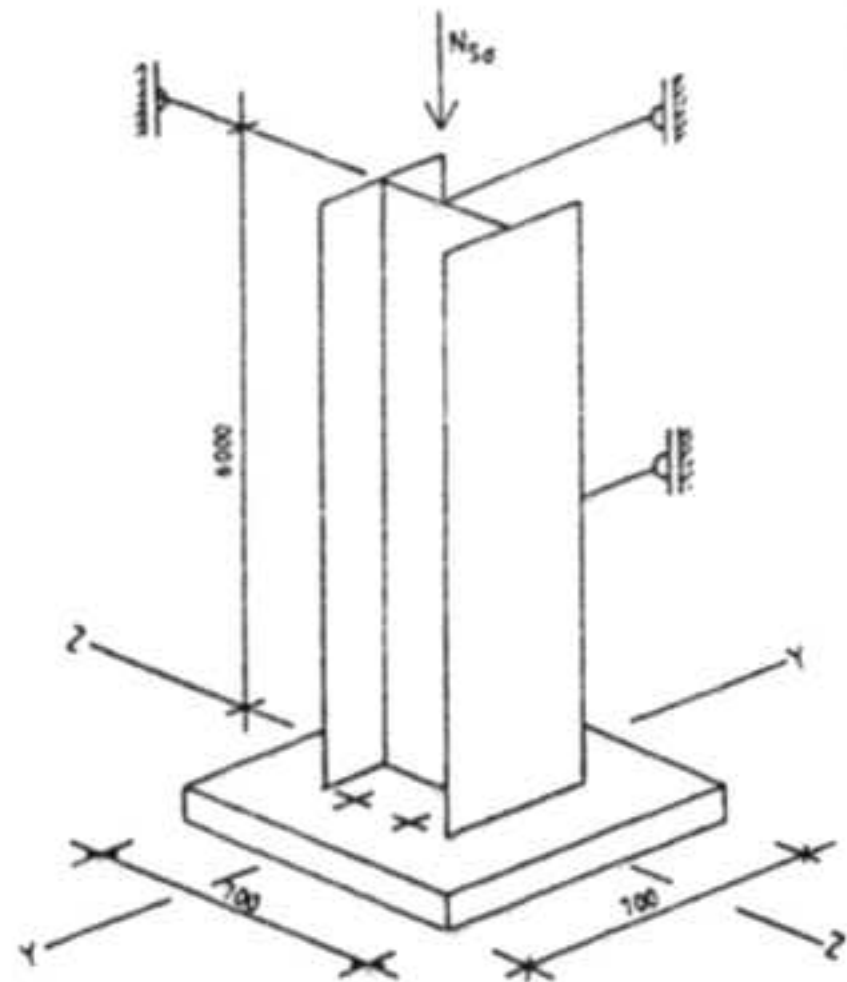
- | | |
|-----------------------|--|
| Example 2.1.1: | Circular hollow section as a column |
| Example 2.1.2: | HEB profile as column |
| Example 2.1.3: | Cold formed RHS as a strut of a lattice girder |
| Example 2.1.4: | Angle as a strut of a lattice girder |
| Example 2.1.5: | Cold formed RHS (class 4 cross-section) |

Problem, Sketch, Calculation	Reference	Commentary
Example 2.1.1: Circular hollow section as a column 1. System  Column: CHS 219,1 x 4,5 hot finished Steel grade: Fe 360 System length = Buckling length = $\ell = 3,50$ m 2. Loading Compression force: $N_{sd} = 600$ kN 3. Determination of the resistance 3.1 Diameter-to-thickness ratio Fe 360 $\rightarrow \epsilon = 1,0$ $d/t = 219,1/4,5 = 48,7 < 90\epsilon^2 = 90$ $\rightarrow A_{eff} = A$ 3.2 Member resistance Buckling length: $\ell = 3,50$ m $\bar{\lambda} = \frac{\ell}{i} \frac{1}{\lambda_1} = \frac{3,50 \cdot 10^3}{75,9} \cdot \frac{1}{93,9} = 0,49$ using buckling curve a $\rightarrow \chi_s = 0,9273$ $N_{b,Rd} = \chi_s \cdot \frac{A \cdot f_y}{\gamma_{M1}} = 0,9273 \cdot \frac{30,3 \cdot 10^2 \cdot 235}{1,1} \cdot 10^{-3} = 600,3 \text{ kN} > 600 \text{ kN}$	5.3.1; Table 5.9; [5.3; Table 5.3.1] 5.4.2; Table 5.21 [5.5.1; Table 5.5.3]	The wording in Eurocode 3 used for "hot finished" is "hot rolled" The whole cross-section is effective to resist the loadings. Determination of χ, see tables in Appendix A

Problem, Sketch, Calculation

Example 2.1.2: HEB profile as column

1. System



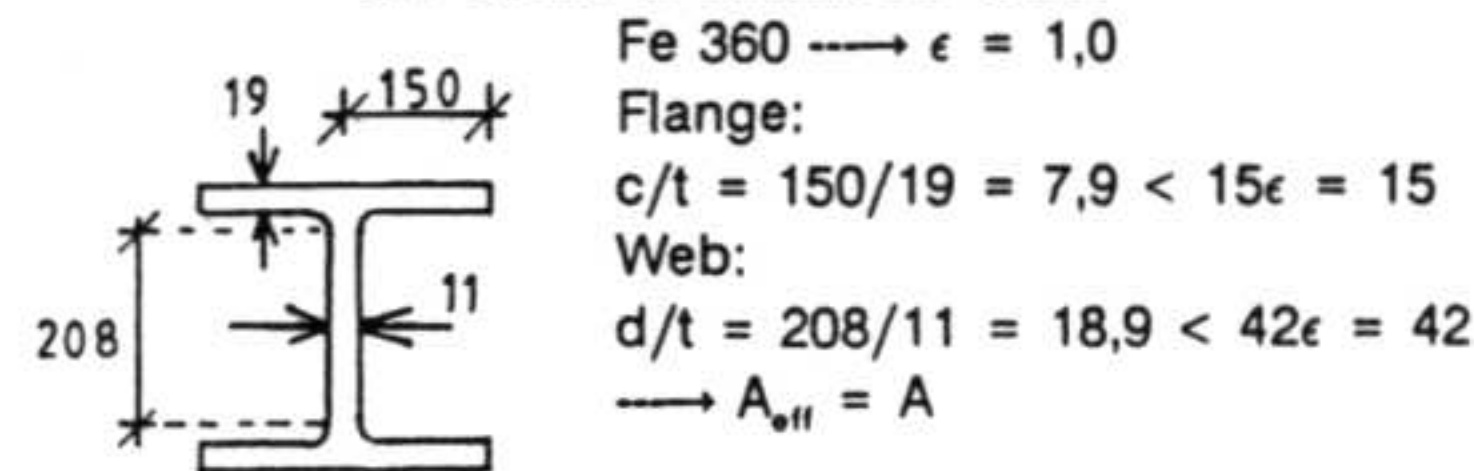
Column: HE 300 B
 Steel grade: Fe 360
 System length: $L = 6,0 \text{ m}$
 Buckling length: $\ell_y = 0,7 \cdot 6,0 \text{ m} = 4,20 \text{ m}$
 $\ell_z = 0,88 \cdot 3,0 \text{ m} = 2,64 \text{ m}$

2. Loading

Compression force: $N_{s,d} = 2900 \text{ kN}$

3. Determination of the resistance

3.1 Width-to-thickness ratio



Fe 360 $\rightarrow \epsilon = 1,0$
 Flange:
 $c/t = 150/19 = 7,9 < 15\epsilon = 15$
 Web:
 $d/t = 208/11 = 18,9 < 42\epsilon = 42$
 $\rightarrow A_{eff} = A$

3.2 Member resistance

3.2.1 Buckling about the strong axis

Buckling length: $\ell_y = 4,20 \text{ m}$

$$\bar{\lambda}_y = \frac{\ell_y}{i_y} \cdot \frac{1}{\lambda_1} = \frac{4,20 \cdot 10^3}{130} \cdot \frac{1}{93,9} = 0,344 \approx 0,35$$

using buckling curve b $\rightarrow \chi_b = 0,9455$

$$N_{b,Rd} = \chi_b \cdot \frac{A \cdot f_y}{\gamma_{M1}} = 0,9455 \cdot \frac{149 \cdot 10^2 \cdot 235}{1,1} \cdot 10^{-3} = 3009,6 \text{ kN} > 2900 \text{ kN}$$

3.2 Member resistance

3.2.2 Buckling about the weak axis

Buckling length: $\ell_z = 2,64 \text{ m}$

$$\bar{\lambda}_z = \frac{\ell_z}{i_z} \cdot \frac{1}{\lambda_1} = \frac{2,64 \cdot 10^3}{75,8} \cdot \frac{1}{93,9} = 0,37$$

using buckling curve c $\rightarrow \chi_c = 0,9131$

$$N_{b,Rd} = \chi_c \cdot \frac{A \cdot f_y}{\gamma_{M1}} = 0,9131 \cdot \frac{149 \cdot 10^2 \cdot 235}{1,1} \cdot 10^{-3} = 2906,6 \text{ kN} > 2900 \text{ kN}$$

Reference

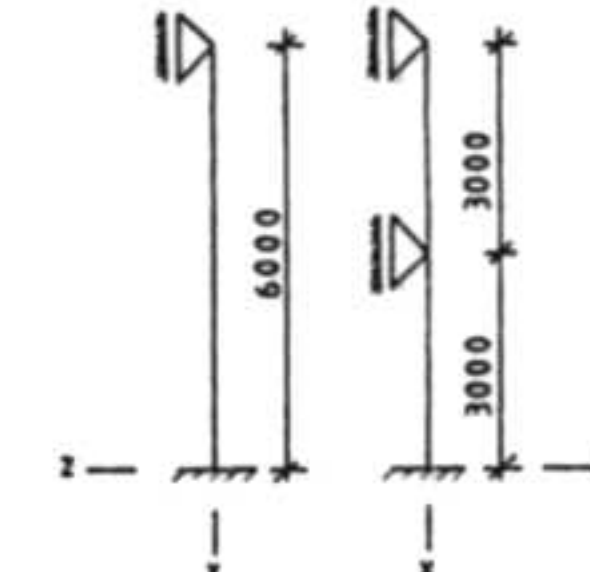
5.3.1; Table 5.9;
 [5.3;
 Table 5.3.1]

5.4.2; Table 5.21
 [5.5.1;
 Table 5.5.3]

5.4.2; Table 5.21
 [5.5.1;
 Table 5.5.3]

Commentary

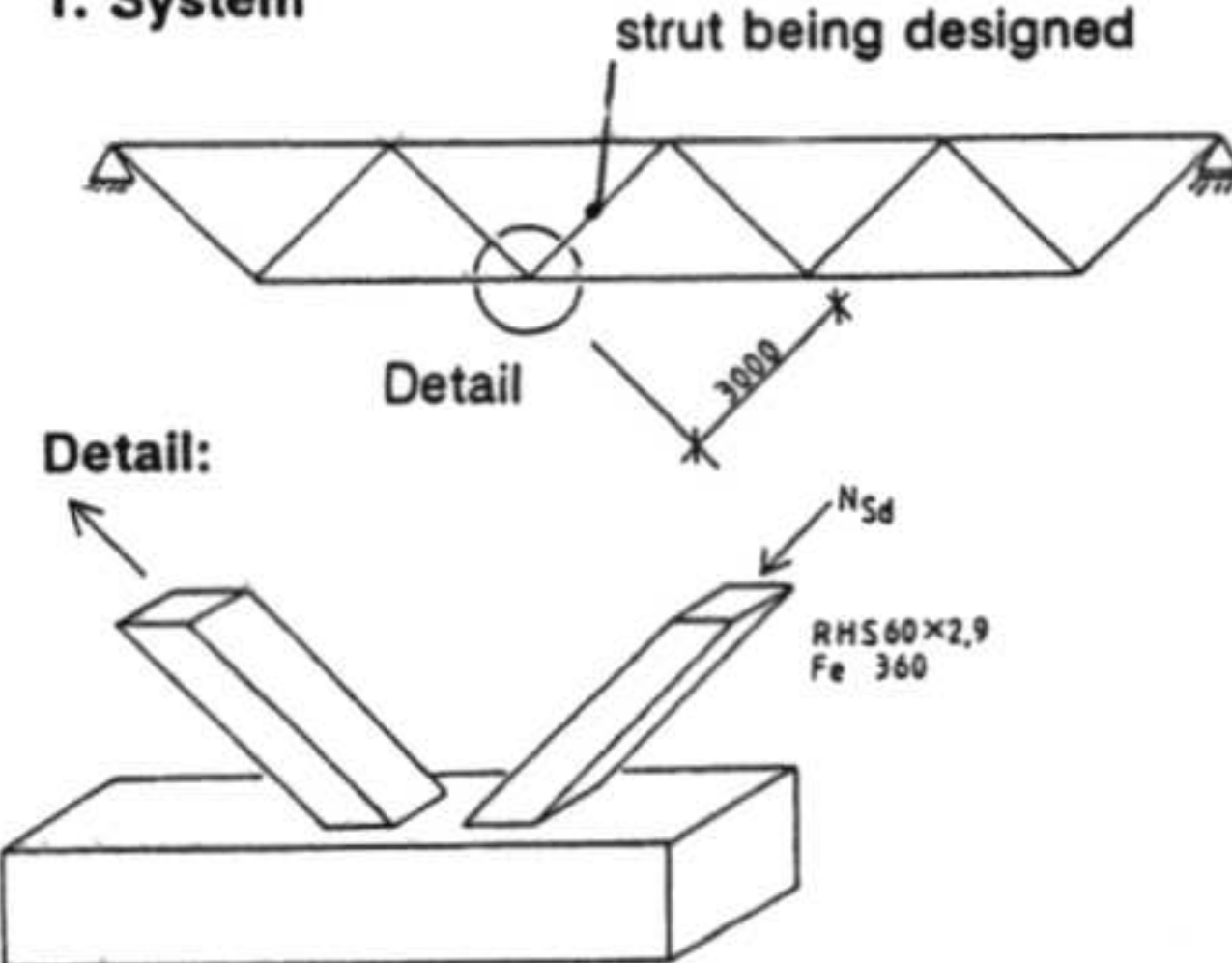
The determination of the buckling length is based on a simple assumption [Annex E.2.1]:

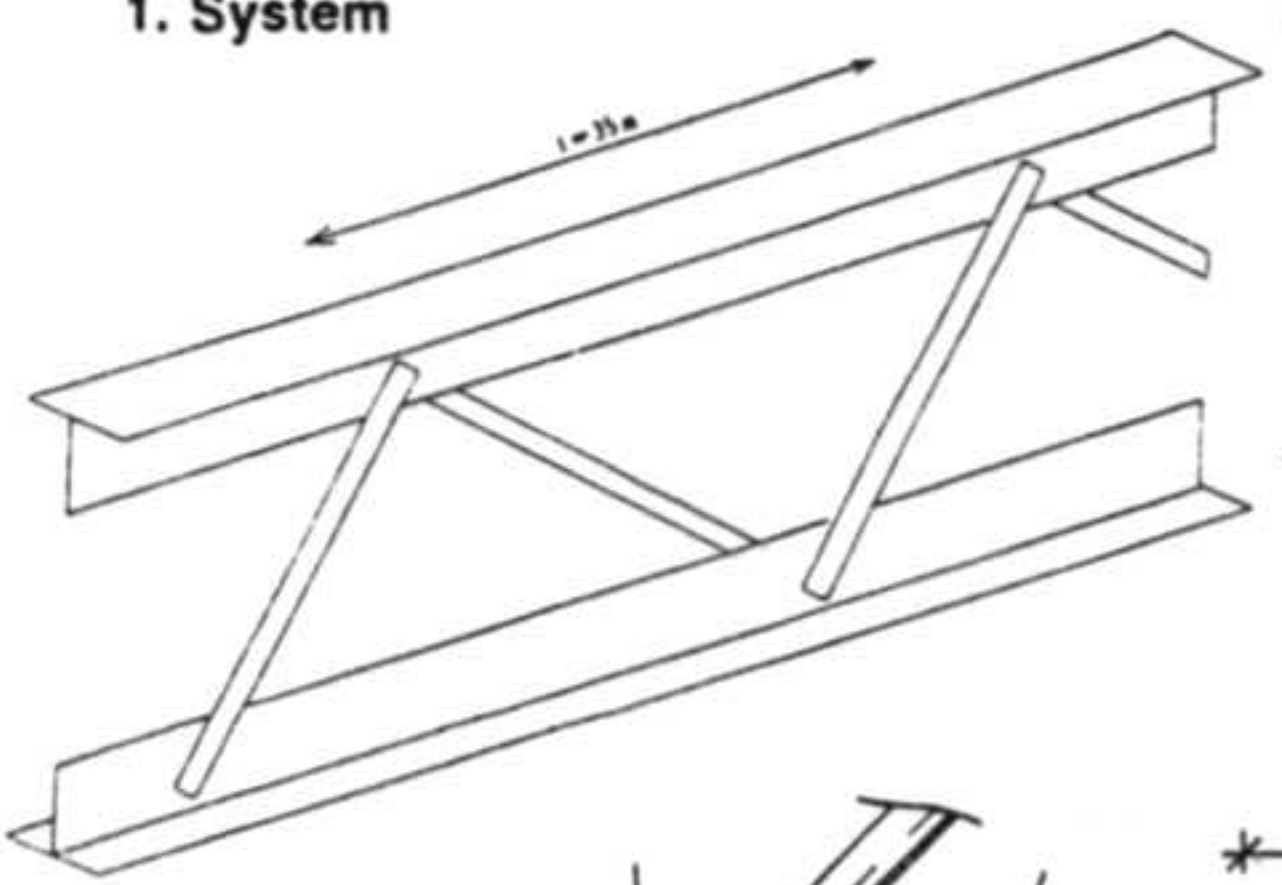
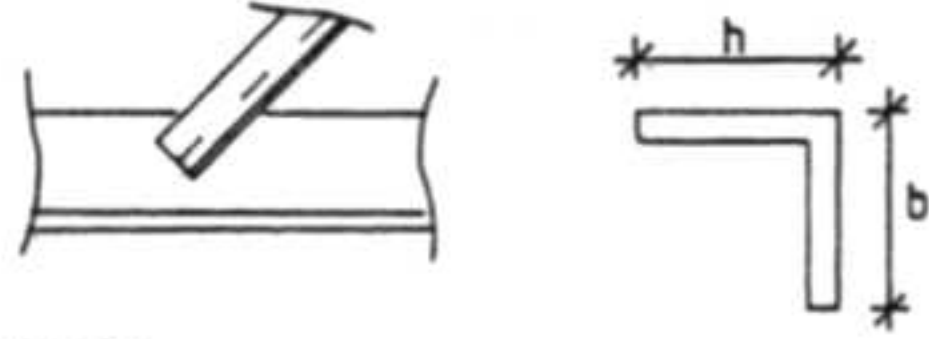


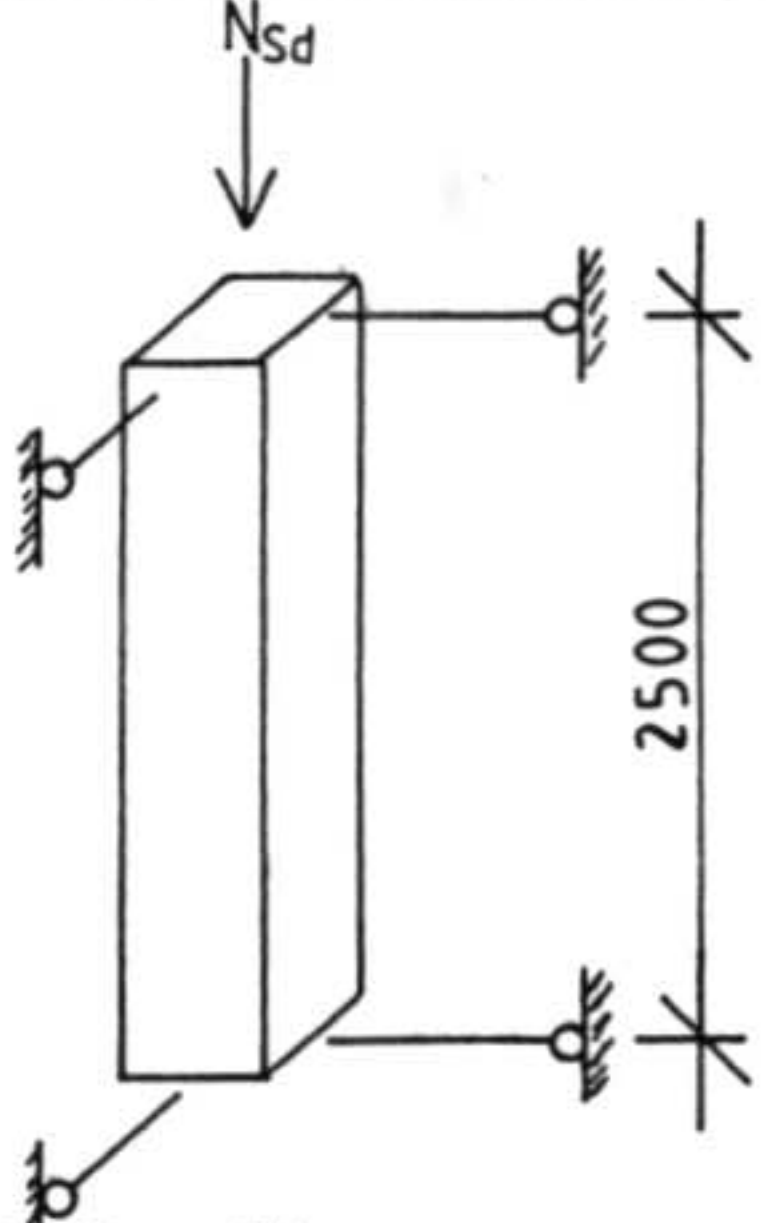
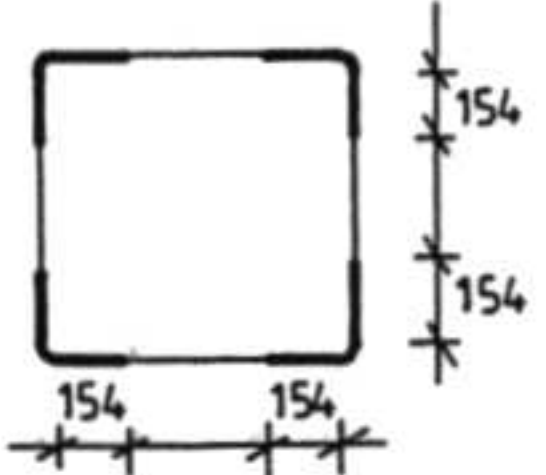
The whole cross-section is effective to resist the loadings.

The stability of compression members needs to be checked according to the two principal axes of the section with appropriate effective lengths. Determination of χ , see tables in Appendix A

Determination of χ , see tables in Appendix A

Problem, Sketch, Calculation	Reference	Commentary
Example 2.1.3: Cold formed RHS as a strut of a lattice girder 1. System  Strut: RHS 60 x 2,9 cold formed Steel grade: Fe 360 System length: L = 3,0 m Buckling length: $\ell = 0,75 \cdot 3,0 \text{ m} = 2,25 \text{ m}$ 2. Loading Compression force: $N_{sd} = 70 \text{ kN}$ 3. Determination of the resistance 3.1 Width-to-thickness ratio Fe 360 $\rightarrow \epsilon = 1,0$ $d = b - 3t = 60 - 3 \cdot 2,9 = 51,3 \text{ mm}$ $d/t = 51,3/2,9 = 17,7 < 38\epsilon = 38$ $\rightarrow A_{eff} = A$ 3.2 Member resistance Buckling length: $\ell = 2,25 \text{ m}$ $\bar{\lambda} = \frac{\ell}{i} \frac{1}{\lambda_1} = \frac{2,25 \cdot 10^3}{23,1} \cdot \frac{1}{93,9} = 1,04$ using buckling curve b $\rightarrow \chi_b = 0,5719$ $N_{b,Rd} = \chi_b \cdot \frac{A \cdot f_y}{\gamma_{M1}} = 0,5719 \cdot \frac{6,41 \cdot 10^2 \cdot 235}{1,1} \cdot 10^{-3} = 78,3 \text{ kN} > 70 \text{ kN}$ Alternative verification by using the average yield strength f_{ya} $f_{ya} = f_{yb} + \frac{k n t^2}{A_0} (f_{ub} - f_{yb}) = 235 + \frac{7 \cdot 4 \cdot 2,9^2}{641} \cdot (360 - 235) = 280,9 \text{ N/mm}^2$ $< f_{ub} = 360 \text{ N/mm}^2$ $< 1,2 \cdot f_{yb} = 282 \text{ N/mm}^2$ $\lambda_1 = \pi \sqrt{\frac{E}{f_{ya}}} = \pi \sqrt{\frac{210000}{280,9}} = 85,9 \rightarrow \bar{\lambda} = \frac{\ell}{i} \frac{1}{\lambda_1} = \frac{2,25 \cdot 10^3}{23,1} \cdot \frac{1}{85,9} = 1,13$ using buckling curve c $\rightarrow \chi_c = 0,4685$ $N_{b,Rd} = \chi_c \cdot \frac{A \cdot f_{ya}}{\gamma_{M1}} = 0,4685 \cdot \frac{6,41 \cdot 10^2 \cdot 280,9}{1,1} \cdot 10^{-3} = 76,7 \text{ kN} > 70 \text{ kN}$	5.3.1; Table 5.9; [5.3; Table 5.3.1] 5.4.2; Table 5.21 [5.5.1; Table 5.5.3] 5.5.1.4; Fig. 5.5.2; Table 5.5.3]	The buckling length of hollow section bracings welded around the perimeter without cropping and flattening can be taken as $0,75 \cdot L$ for in and out of plane buckling. [Annex KK (in preparation)] The choice of the profile should be made considering the practical and cost aspects of the connection design. The whole cross-section is effective to resist the loadings. Class 2 is necessary because of redistribution of secondary effects. Determination of χ, see tables in Appendix A f_{ya} = average yield strength f_{yb} = yield strength of basic material $k = 7$ for cold rolling $n = 4$ bends of 90° Determination of χ, see tables in Appendix A

Problem, Sketch, Calculation	Reference	Commentary
Example 2.1.4: Angle as a strut of a lattice girder 1. System  Strut: L 100 x 10 Steel grade: Fe 430 System length: L = 3,50 m Buckling length: $\ell = 3,50$ m 2. Loading Compression force: $N_{sd} = 100$ kN 3. Determination of the resistance 3.1 Width-to-thickness ratio Fe 430 $\rightarrow \epsilon = 0,92$ $(h+b)/2t = 200/20 = 10 < 11,5\epsilon = 10,6$ $\rightarrow A_{eff} = A$  3.2 Member resistance Buckling length: $\ell = 3,50$ m $\bar{\lambda}_v = \frac{\ell}{i_v} \cdot \frac{1}{\lambda_1} = \frac{3,50 \cdot 10^3}{19,5} \cdot \frac{1}{86,8} = 2,07 \quad \rightarrow \bar{\lambda}_{eff,v} = 0,35 + 0,7\bar{\lambda}_v = 0,35 + 0,7 \cdot 2,07 = 1,80$ using buckling curve c $\rightarrow \chi_c = 0,2345$ $N_{b,Rd} = \chi_c \cdot \frac{A \cdot f_y}{\gamma_{M1}} = 0,2345 \cdot \frac{19,2 \cdot 10^2 \cdot 275}{1,1} \cdot 10^{-3} = 112,6 \text{ kN} > 100 \text{ kN}$	[5.8.3] 5.3.1; Table 5.9; [5.3; Table 5.3.1] 5.4.2; Table 5.21 [5.5.1; Table 5.5.3]	If the chords supply appropriate end restraint and the end connections supply appropriate fixity (e.g. at least 2 bolts or welded) the effective length of a web member as an angle is taken as 1,0L. The eccentricities of the connection may be neglected and an effective slenderness ratio used in such cases. $h/t = 100/10 = 10 < 15\epsilon = 13,8$ $\rightarrow$ criterion only decisive, when $b < 0,53 \cdot h$ The whole cross-section is effective to resist the loadings. Buckling about the weak axis v is critical. Determination of χ, see tables in Appendix A.

Problem, Sketch, Calculation	Reference	Commentary
Example 2.1.5: Cold formed RHS (class 4 cross-section) 1. System  compression member: RHS 400 x 8,0 , cold formed Steel grade: Fe 510 System length: L = 2,50 m Buckling length: $\ell = 2,50$ m 2. Loading Compression force: $N_{Sd} = 3200$ kN 3. Determination of the resistance 3.1 Width-to-thickness ratio Fe 510 $\rightarrow \epsilon = 0,81$ $d = b - 3t = 400 - 3 \cdot 8,0 = 376$ mm $d/t = 376/8,0 = 47 > 42\epsilon = 34,0$ 3.2 Determination of A_{eff}  Fe 510 $\rightarrow 1/\epsilon = 1,23$ $\bar{b} = b - 3t = 376$ mm ; $\bar{\lambda}_p = \frac{(b - 3t)/t}{28,4 \epsilon \sqrt{k_\sigma}} = \frac{b - 3t}{t} \cdot \frac{1}{56,8 \epsilon} = 47 \cdot \frac{1,23}{56,8} = 1,018 > 0,673$ $\rho = \frac{\bar{\lambda}_p - 0,22}{\bar{\lambda}_p^2} = \frac{1,018 - 0,22}{1,018^2} = 0,77$ $d_{eff} = \rho \cdot b = 0,77 \cdot 400 = 308$ mm $A_{eff} = A - 4t(b - d_{eff}) = 123 \cdot 10^2 - 4 \cdot 8,0 \cdot (376 - 308) = 101,24 \cdot 10^2$ mm² 3.3 Buckling resistance Buckling length: $\ell = 2,50$ m $\bar{\lambda} = \frac{\ell}{i} \cdot \frac{1}{\lambda_1} \sqrt{\frac{A_{eff}}{A}} = \frac{2,50 \cdot 10^3}{159} \cdot \frac{1}{76,4} \cdot \sqrt{\frac{101,24 \cdot 10^2}{123,00 \cdot 10^2}} = 0,187 < 0,2 \rightarrow \chi = 1$ $N_{c,Rd} = \frac{A_{eff} \cdot f_y}{\gamma_{M1}} = \frac{101,24 \cdot 10^2 \cdot 355}{1,1} \cdot 10^{-3} = 3267,3 \text{ kN} > 3200,0 \text{ kN}$	5.3.1; Table 5.9; [5.3; Table 5.3.1] 5.3.5; Table 5.18 [5.3.5; Table 5.3.2] 5.4.2; [5.5.1]	The whole cross-section is not effective to resist the loadings. Therefore the effective cross-section properties have to be determined. The effective cross-section properties are calculated on the basis of f_y for compression elements. As there is no shift of the centroidal axis when the effective cross section is subject to uniform compression, then no additional moment needs to be considered. The radius of gyration of the gross section is used to determine the non-dimensional slenderness, but the ratio of A_{eff} to A has to be taken into account. Because of $\bar{\lambda} < 0,2$ no allowance for buckling is necessary, i.e. only cross-section verification is required $\rightarrow$ use of γ_{M1} because of class 4 section.

2.2 Members in bending

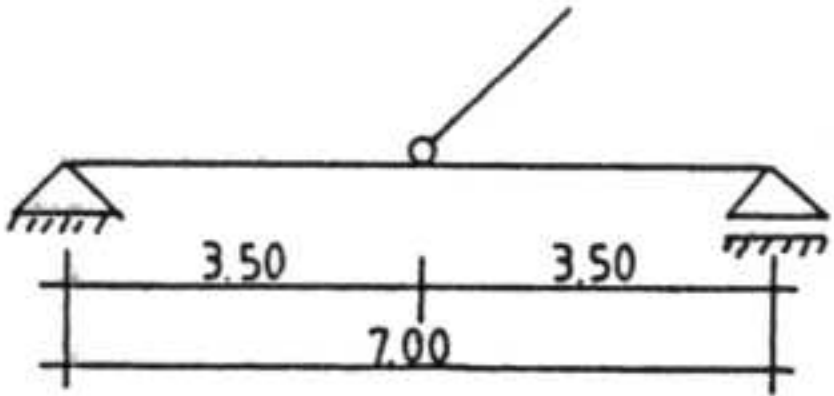
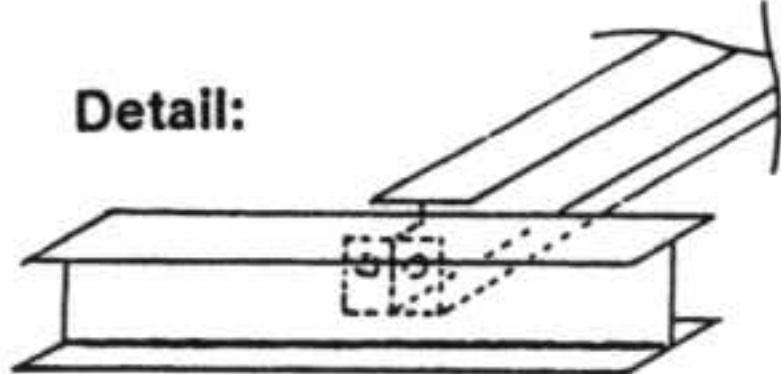
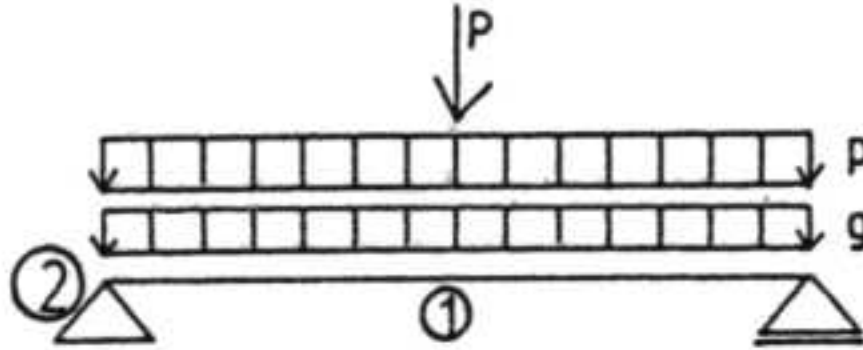
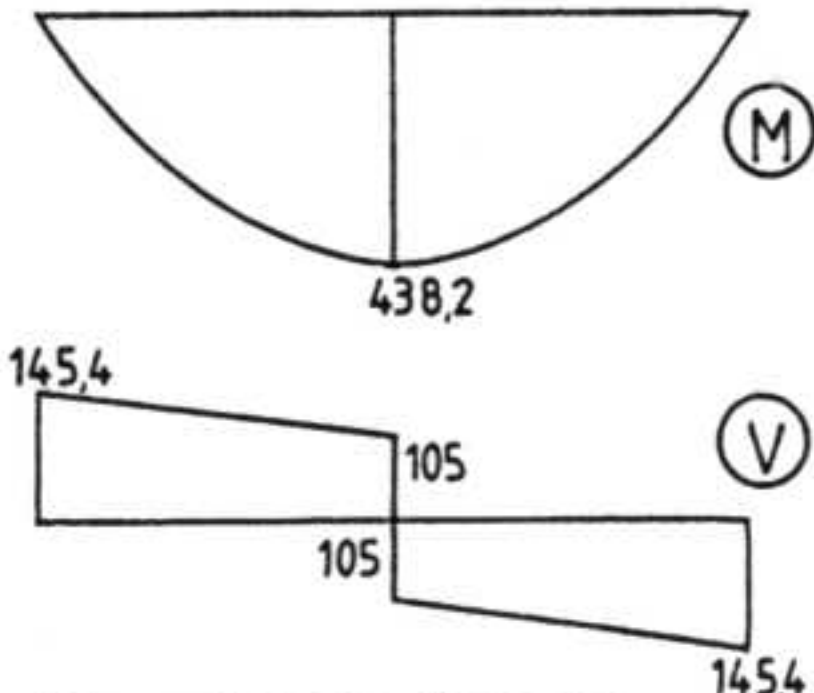
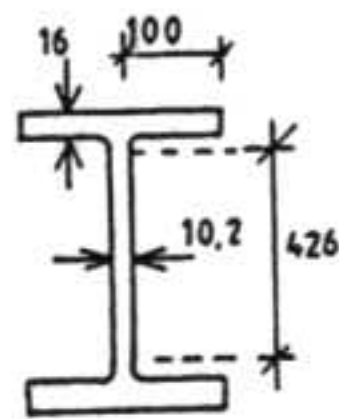
Examples 1, 2 and 3 demonstrate the verification of members in bending assuming design values of action effects (V_{sd} , M_{sd} , etc) which are calculated by an analysis of the sub-structure.

Example 4 demonstrates the verification of members in bending assuming design values of action effects (V_{sd} , M_{sd} , etc) which have been calculated by an analysis of the structure and these values already include ψ , γ_F etc.

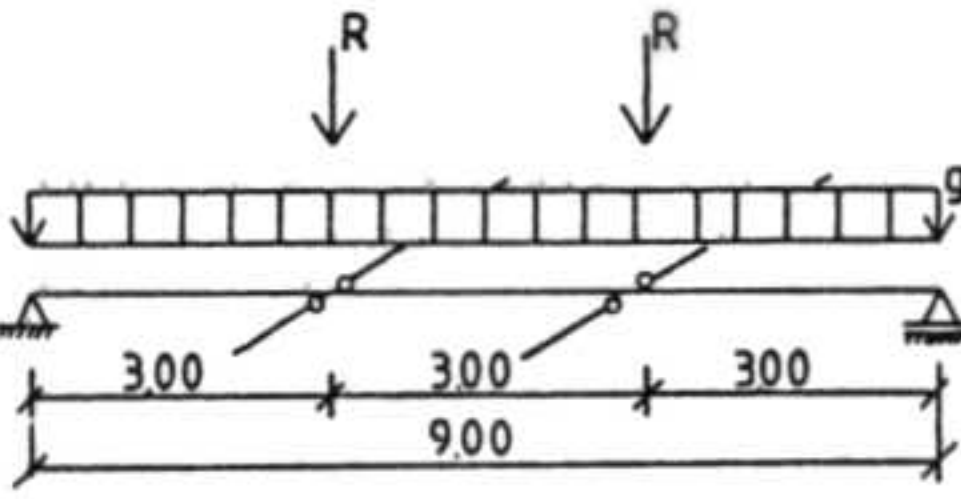
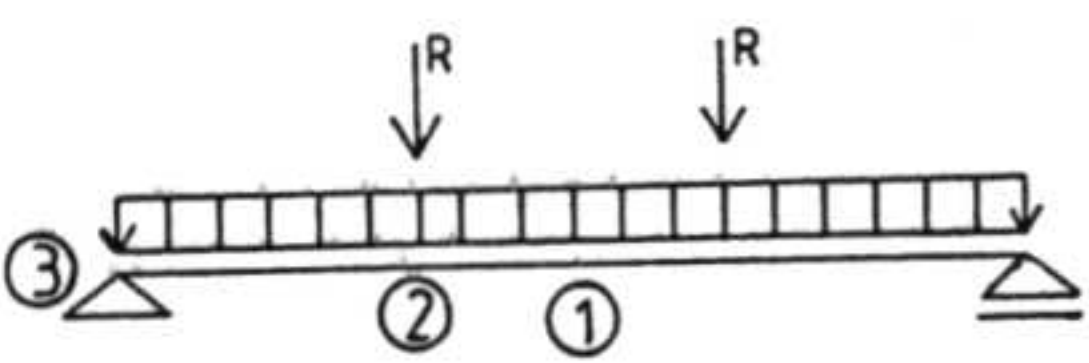
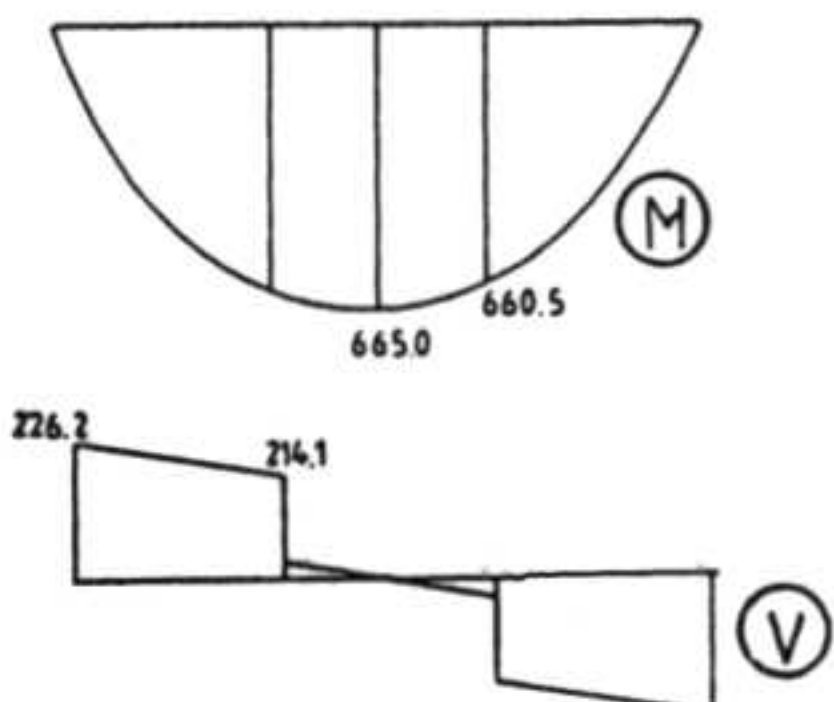
The results presented in the examples are rounded values. For the purpose of easy re-calculation each formula and each check is calculated with the rounded values.

The following examples are included in this chapter:

Example 2.2.1:	Single span beam
Example 2.2.2:	Single span beam with lateral torsional buckling
Example 2.2.3:	Single span beam with shear buckling verification
Example 2.2.4:	Class 4 cross-section loaded in bending

Problem, Sketch, Calculation	Reference	Commentary
Example 2.2.1: Single span beam 1. System  Detail:  Main girder: IPE 500 Steel grade: Fe 430 System length: $l = 7,00 \text{ m}$ 2. Loading  Dead load: $g = 3,0 \text{ kN/m}$ Imposed load: $p = 5,0 \text{ kN/m}$; $P = 140 \text{ kN}$ Relevant load combination of this load case: $1,35 \cdot g + 1,5 \cdot (p + P)$ 3. Internal forces and moments  $M_{Sd} = (1,35 \cdot 3,0 + 1,5 \cdot 5,0) \cdot 7^2 / 8 + 1,5 \cdot 140 \cdot 7 / 4 = 438,2 \text{ kNm}$ $V_{1.Sd} = 1,5 \cdot 140 / 2 = 105 \text{ kN}$ $V_{2.Sd} = R_{Sd} = 1,5 \cdot 140 / 2 + (1,35 \cdot 3,0 + 1,5 \cdot 5,0) \cdot 7 / 2 = 145,4 \text{ kN}$ 4. Determination of the resistance 4.1 Width-to-thickness ratios  Fe 430 $\rightarrow \epsilon = 0,92$ Flange: $c/t = 100/16 = 6,3 < 11\epsilon = 10,1$ Web: $d/t = 426/10,2 = 41,8 < 83\epsilon = 76,4$ $\rightarrow A_{eff} = A$; $W_{eff} = W_{pl}$ $d/t = 41,8 < 69\epsilon = 63,5$ $\rightarrow$ no shear buckling verification is required	2.3.2.2; Table 2.1 [2.3.3.1.(5)] 5.3.1; Table 5.10 [5.3; Table 5.3.1]	At mid span a side girder is connected by a nominally pinned connection. Lateral restraints are assumed at the supports. For determining the cross-section properties I_w, I_t, I, W, A, A_w, i etc. dimension tables are used. Dead load includes the weight of floor, etc. Since the imposed loads p and P are not independent from each other they are considered as one variable action. The classification of cross-sections could be made by appropriate dimension tables. For rolled sections the shear buckling verification need not normally be considered.

Problem, Sketch, Calculation	Reference	Commentary
4.2 Shear resistance verification 4.2.1 At support $V_{Rd} = \frac{A_v f_y}{\sqrt{3} \gamma_{Mo}} = \frac{51,0 \cdot 10^2 \cdot 275}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 736,1 \text{ kN} > 145,4 \text{ kN}$ 4.2.2 At mid-span $0,5 V_{Rd} = 0,5 \cdot \frac{A_v f_y}{\sqrt{3} \gamma_{Mo}} = 0,5 \cdot \frac{51,0 \cdot 10^2 \cdot 275}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 368,1 \text{ kN} > 105,0 \text{ kN} \rightarrow \text{no interaction bending - shear is required}$ 4.3 Member resistance verification 4.3.1 Determination of the critical bending moment M_{cr} Buckling length $\ell_{LT} = 3,50 \text{ m}$ $k = k_w = 1,0$; $\psi = 0 \rightarrow C_1 = 1,88$ $M_{cr} = C_1 \frac{\pi^2 E I_z}{\ell_{LT}^2} \sqrt{\left(\frac{k}{k_w}\right)^2 \frac{I_w}{I_z} + 0,039 \ell_{LT}^2 \frac{I_t}{I_z}} = 1,88 \cdot \frac{\pi^2 \cdot 210000 \cdot 2140 \cdot 10^4}{3,50^2 \cdot 10^{12}} \sqrt{\frac{1249000 \cdot 10^8 + 0,039 \cdot 3,50^2 \cdot 10^8 \cdot 89,7 \cdot 10^4}{2140 \cdot 10^4}} = 1905,8 \text{ kNm}$ 4.3.2 Lateral torsional buckling verification $\bar{\lambda}_{LT} = \sqrt{\frac{W_{pl} f_y}{M_{cr}}} = \sqrt{\frac{2194 \cdot 10^3 \cdot 275}{1905,8} \cdot 10^{-6}} = 0,56 > 0,4$ using buckling curve a $\rightarrow \chi_{LT,a} = 0,9045$ $M_{b,Rd} = \chi_{LT,a} \frac{W_{pl} f_y}{\gamma_{M1}} = 0,9045 \cdot \frac{2194 \cdot 10^3 \cdot 275}{1,1} \cdot 10^{-6} = 496,1 \text{ kNm} > 438,2 \text{ kNm}$ 5. Serviceability verification Relevant load combination: 1,0 g + 1,0 p + 1,0 P Deflection due to dead loads: $\frac{5}{384} \frac{g \cdot \ell^4}{E I} = \frac{5}{384} \frac{3,0 \cdot 7,0^4}{210000 \cdot 48200 \cdot 10^4} \cdot 10^{12} = 0,93 \text{ mm}$ Deflection due to imposed loads: $\delta_2 = \frac{5}{384} \frac{g \cdot \ell^4}{E I} + \frac{1}{48} \frac{P \ell^3}{E I} = \frac{5}{384} \frac{3,0 \cdot 7,0^4}{210000 \cdot 48200 \cdot 10^4} \cdot 10^{12} + \frac{1}{48} \frac{140 \cdot 7,0^3}{210000 \cdot 48200 \cdot 10^4} \cdot 10^{12}$ $= 11,4 \text{ mm} = \ell/614 < \ell/300$ Maximum deflection: $\delta_{max} = 11,4 + 0,93 = 12,33 \text{ mm} = \ell/570 < \ell/250$ $\rightarrow$ EC 3 recommendations are fulfilled	5.3.3; [5.4.6] 5.3.4; [5.4.7] 5.4.3; [Annex F] 5.4.3; [5.5.2] 4.2.2; Table 4.1 [4.4.2; Table 4.1]	$A_v = h \cdot t_w = 500 \cdot 10,2 = 5100 \text{ mm}^2$ It is allowed to consider larger values by using the formulae given in EC 3 [5.4.6] Due to the side girder which provides a lateral support the buckling length is half the span.

Problem, Sketch, Calculation	Reference	Commentary
Example 2.2.2: Single span beam with lateral torsional buckling 1. System  Main girder: HE 450 A Steel grade: Fe 360 System length: $l = 9,00 \text{ m}$ 2. Loading  Dead load: $g = 3,0 \text{ kN/m}$ Reaction force: $R = 208 \text{ kN}$ Relevant load combination of this load case: $1,35 \cdot g + R$ 3. Internal forces and moments  $M_{1.Sd} = 1,35 \cdot 3,0 \cdot 9^2 / 8 + 208 \cdot 3,0 = 665,0 \text{ kNm}$ $M_{2.Sd} = 1,35 \cdot 3,0 \cdot 3^2 + 208 \cdot 3,0 = 660,5 \text{ kNm}$ $V_{3.Sd} = R_{3.Sd} = 1,35 \cdot 3,0 \cdot 9,0 / 2 + 208 = 226,2 \text{ kN}$ $V_{2.Sd} = -1,35 \cdot 3,0 \cdot 3,0 / 6 + 226,2 = 214,1 \text{ kN}$ 4. Determination of the resistance 4.1 Width-to-thickness ratios Fe 360 $\rightarrow \epsilon = 1,0$ Flange: $c/t = 150/21 = 7,1 < 11\epsilon = 11$ Web: $d/t = 344/11,5 = 29,9 < 83\epsilon = 83$ $\rightarrow A_{eff} = A ; W_{eff} = W_{pl}$ $d/t = 29,9 < 69\epsilon = 69$ $\rightarrow$ no shear buckling verification is required	2.3.2.2; Table 2.1 [2.3.3.1.(5)] 5.3.1; Table 5.10 [5.3; Table 5.3.1]	At the third points of the main girder side girders are connected by a nominally pinned connections. Lateral restraints are assumed at the supports. For determining the cross-section properties $I_w, I_t, I, W, A, A_v, i$ etc. dimension tables are used. Dead load includes the weight of floor, etc. R is the largest reaction force from the design of the side girders. The reaction force R includes the load factors. The classification of cross-sections could be made by appropriate dimension tables. For rolled sections the shear buckling verification need not normally be considered.

Problem, Sketch, Calculation	Reference	Commentary
4.2 Shear resistance verification 4.2.1 At support $V_{Rd} = \frac{A_v f_y}{\sqrt{3} \gamma_{M0}} = \frac{50,6 \cdot 10^2 \cdot 235}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 624,1 \text{ kN} > 226,2 \text{ kN}$	5.3.3; [5.4.6]	$A_v = h \cdot t_w = 440 \cdot 11,5 = 5060 \text{ mm}^2$ It is allowed to consider larger values by using the formulae given in EC 3 [5.4.6].
4.2.2 At mid-span $0,5 V_{Rd} = 0,5 \cdot \frac{A_v f_y}{\sqrt{3} \gamma_{M0}} = 0,5 \cdot \frac{50,6 \cdot 10^2 \cdot 235}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 312,1 \text{ kN} > 214,1 \text{ kN} \rightarrow \text{no interaction bending - shear is required}$	5.3.4; [5.4.7]	
4.3 Member resistance verification 4.3.1 Determination of the critical bending moment M_{cr} Buckling length $\ell_{LT} = 3,0 \text{ m}$ $k = k_w = 1,0$; $\psi \approx 1,0 \rightarrow C_1 = 1,0$ $M_{cr} = C_1 \frac{\pi^2 E I_z}{\ell_{LT}^2} \sqrt{\left(\frac{k}{k_w}\right)^2 \frac{I_w}{I_z} + 0,039 \ell_{LT}^2 \frac{I_1}{I_z}} = 1,0 \cdot \frac{\pi^2 \cdot 210000 \cdot 9470 \cdot 10^4}{3,0^2 \cdot 10^{12}} \sqrt{\frac{4146000 \cdot 10^8 + 0,039 \cdot 3,0^2 \cdot 10^8 \cdot 245 \cdot 10^4}{9470 \cdot 10^4}} = 5014,1 \text{ kNm}$	5.4.3; [Annex F]	Because of lateral restraints the buckling length is limited to 3,0 m
4.3.2 Lateral torsional buckling verification $\bar{\lambda}_{LT} = \sqrt{\frac{W_{pl} f_y}{M_{cr}}} = \sqrt{\frac{3216 \cdot 10^3 \cdot 235}{5014,1} \cdot 10^{-6}} = 0,39 < 0,4$ $M_{b,Rd} = \frac{W_{pl} f_y}{\gamma_{M0}} = \frac{3216 \cdot 10^3 \cdot 235}{1,1} \cdot 10^{-6} = 687,1 \text{ kNm} > 665,0 \text{ kNm}$	5.4.3; [5.5.2]	No lateral torsional buckling to be considered. Only cross-section verification is necessary, i.e. γ_{M0} is used

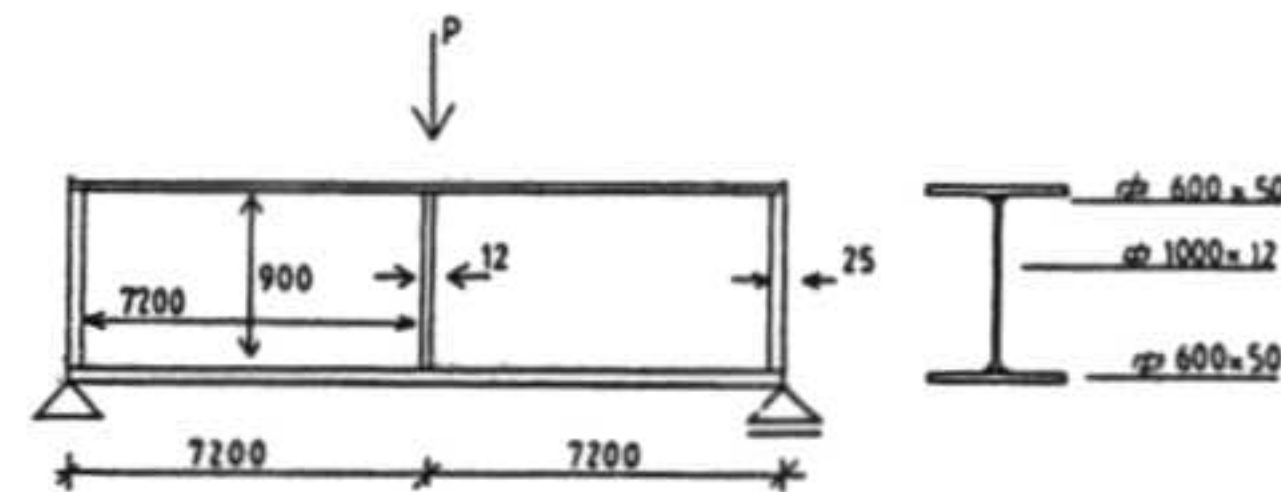
Problem, Sketch, Calculation

Reference

Commentary

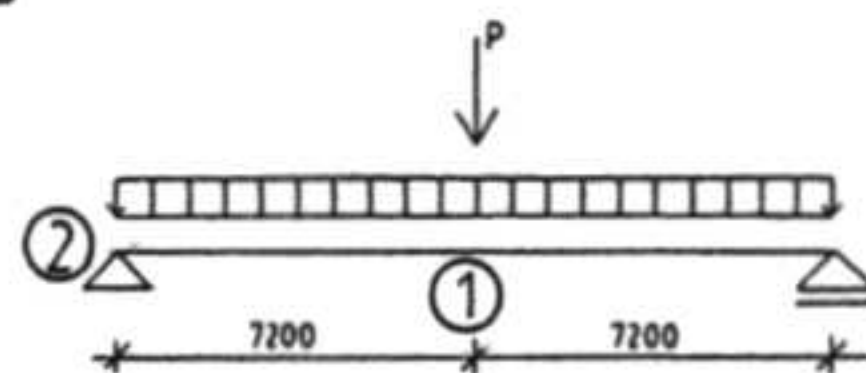
Example 2.2.3: Single span girder with shear buckling verification

1. System



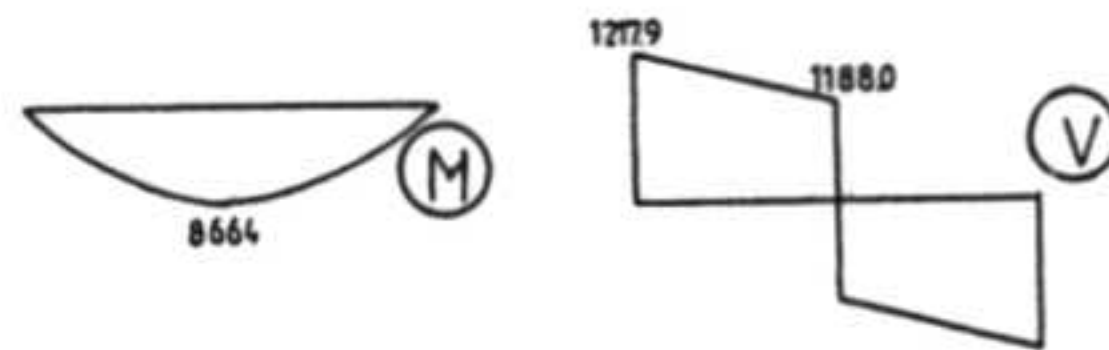
Beam: Welded section with stiffeners at the supports and at mid span
Dimensions see sketch
Steel grade: Fe 510
System length: $l = 14,40$ m

2. Loading



Dead load: $g = 3,0$ kN/m
Reaction force: $P = 1585,0$ kN
Relevant load combination of this load case:
 $1,35 \cdot g + 1,5 \cdot P$

3. Internal forces and moments



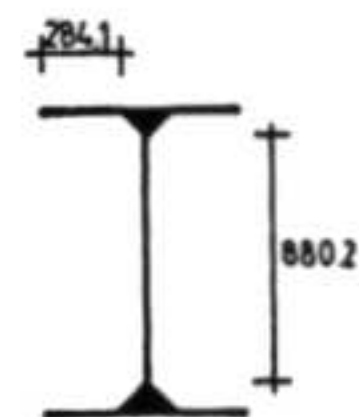
$$M_{1, sd} = 1,35 \cdot 3,0 \cdot 14,4^2 / 8 + 1,5 \cdot 1585,0 \cdot 14,4 / 4 = 8664,0 \text{ kNm}$$

$$V_{1, sd} = 0,5 \cdot 1,5 \cdot 1585,0 = 1188,8 \text{ kN}$$

$$V_{2, sd} = R_{2, sd} = 1,35 \cdot 3,0 \cdot 14,4 / 2 + 1,5 \cdot 1585,0 / 2 = 1217,9 \text{ kN}$$

4. Determination of the resistance

4.1 Width-to-thickness ratios



Flange: $t_f = 50 \text{ mm} > 40 \text{ mm} \rightarrow f_y = 335 \text{ N/mm}^2$

Fe 510 $\rightarrow \epsilon = \sqrt{235/335} = 0,84$

$c = 300 \cdot 12/2 \cdot \sqrt{2} \cdot 7 = 284,1 \text{ mm}$

$c/t = 284,1/50 = 5,7 < 14\epsilon = 11,8$

Web:

Fe 510 $\rightarrow \epsilon = 0,81$

$d/t = 900/12 = 75 < 124\epsilon = 100,4$

$\rightarrow A_{eff} = A; W_{eff} = W_{el}$

$$a/d = 7200/900 = 8,0 \rightarrow k_t = 5,34 + \frac{4}{(a/d)^2} = 5,34 + \frac{4}{8^2} = 5,4$$

$$d/t = 75 > 30\epsilon\sqrt{k_t} = 56,5$$

$\rightarrow$ shear buckling verification is required

2.3.2.2;
Table 2.1
[2.3.3.1.(5)]

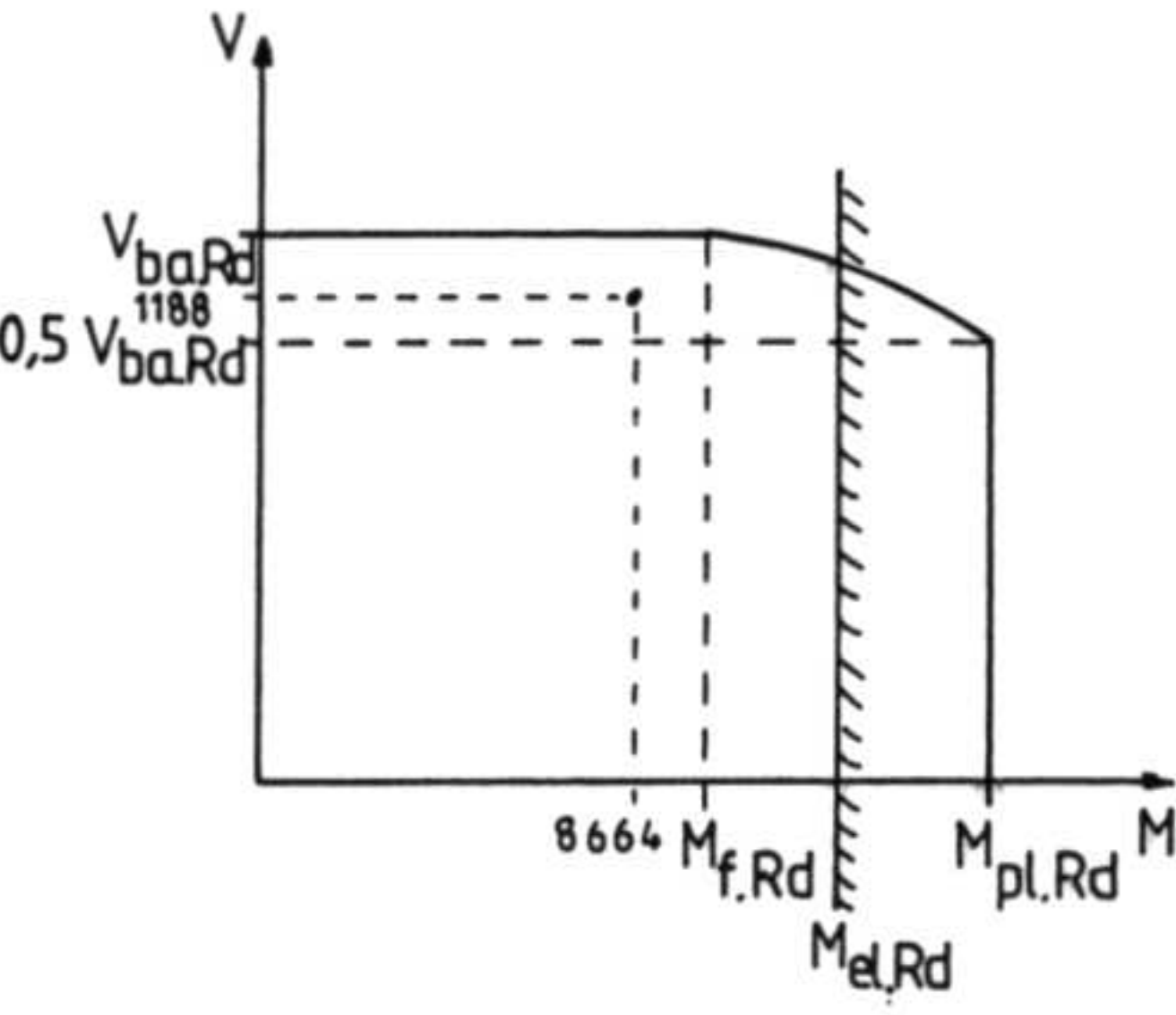
- ;
[5.3;
Table 5.3.1]

Table 5.13;
[5.4.6.(7)]

The beam is continuously laterally restrained.
Cross-section properties
 $A = 708 \cdot 10^2 \text{ mm}^2$
 $I = 1427900 \cdot 10^4 \text{ mm}^4$
throat thickness between flange and web:
 $a = 7 \text{ mm}$

Dead load includes the weight of floor, etc.

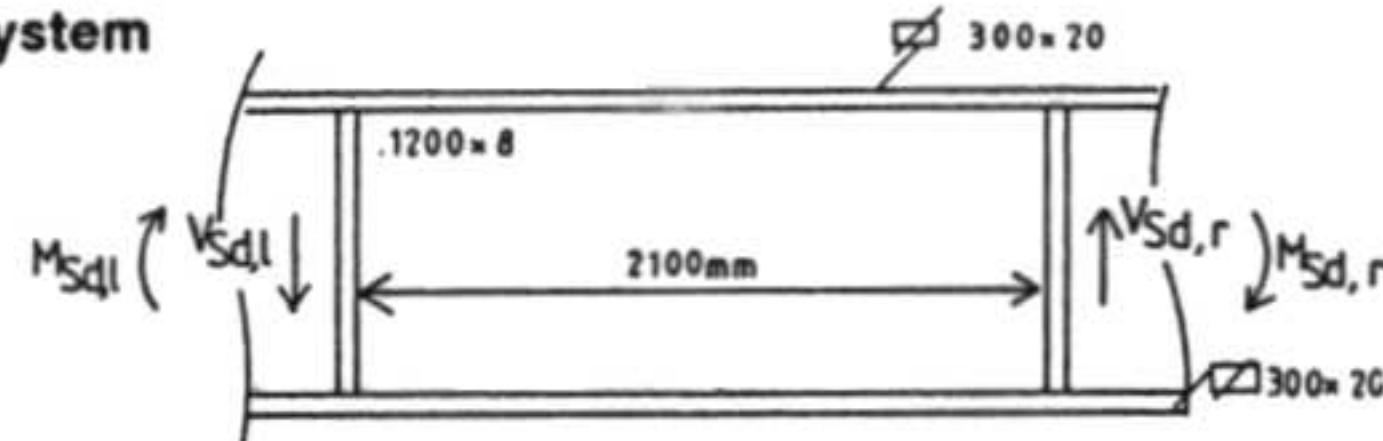
Note that where the plate thickness is greater than 40 mm the yield strength has to be reduced. In the classification of welded cross-sections the welds are taken into account.

Problem, Sketch, Calculation	Reference	Commentary
4.2 Flange induced buckling	5.3.5.(6); [5.7.7]	k = 0,55 for class 3 and class 4 section
$\frac{d}{t_w} = 73,4 < k \frac{E}{f_{yf}} \sqrt{\frac{A_w}{A_{fc}}} = 0,55 \cdot \frac{210000}{335} \sqrt{\frac{900 \cdot 12}{600 \cdot 50}} = 206,9 > 75$		
4.3 Shear buckling verification of web	5.4.5; [5.6.3]	The cross-section may be assumed to be satisfactory, without investigating the effect of shear force on the design resistance moment when both checks are satisfied ($V_{sd} \leq V_{Rd}$; $M_{sd} \leq M_{f,Rd}$).
$\bar{\lambda}_w = \frac{d/t_w}{37,4 \epsilon_w \sqrt{k_t}} = \frac{75}{37,4 \cdot 0,81 \cdot \sqrt{5,4}} = 1,065$ $\tau_{ba} = f_{vw} / \sqrt{3} (1,5 - 0,625 \bar{\lambda}_w) = 35,5 / \sqrt{3} (1,5 - 0,625 \cdot 1,065) \cdot 10 = 171 \text{ N/mm}^2$ $V_{ba,Rd} = \frac{d t_w \tau_{ba}}{\gamma_{M1}} = \frac{900 \cdot 12 \cdot 171}{1,1} \cdot 10^{-3} = 1678,9 \text{ kN} > 1217,9 \text{ kN}$ 1188,0 > 0,5V_{ba,Rd} = 839,5 kN → Interaction bending - shear is required	[5.6.7.2.(1)]	The simple post critical method is used for determining the shear buckling resistance. M _{f,Rd} The design moment resistance of a cross section consisting of the flanges only.
4.4 Determination of the design moment resistance consisting the flanges only $M_{f,Rd} = A_t (h - t_f) f_{yt} / \gamma_{M0} = 600 \cdot 50 \cdot (1000 - 50) \cdot 335 \cdot 10^{-6} / 1,1 = 8679,5 \text{ kNm} > 8664,0 \text{ kNm}$		
4.5 Bending moment resistance $W_{el} = 28558 \cdot 10^3 \text{ mm}^3$ $M_{c,Rd} = W_{el} f_y / \gamma_{M0} = 28558 \cdot 10^3 \cdot 335 \cdot 10^{-6} / 1,1 = 8697,2 \text{ kN} > 8664,0 \text{ kNm}$	5.3.2; [5.4.5]	M _{w,Rd} the design moment resistance of a cross section consisting of the web only
4.6 Determination of the design plastic moment $M_{w,Rd} = d^2 t_w f_{yw} / 4 \cdot 1,1 = 900^2 \cdot 12 / 4 \cdot 355 \cdot 10^{-6} / 1,1 = 784,2 \text{ kNm}$ $M_{pl,Rd} = (M_{f,Rd} + M_{w,Rd}) = 8679,5 + 784,2 = 9463,7 \text{ kNm}$	5.4.5.3 [5.6.7.2]	
4.7 Interaction bending - shear buckling verification → not necessary because of M _{sd} < M _{f,Rd}		
		

Problem, Sketch, Calculation

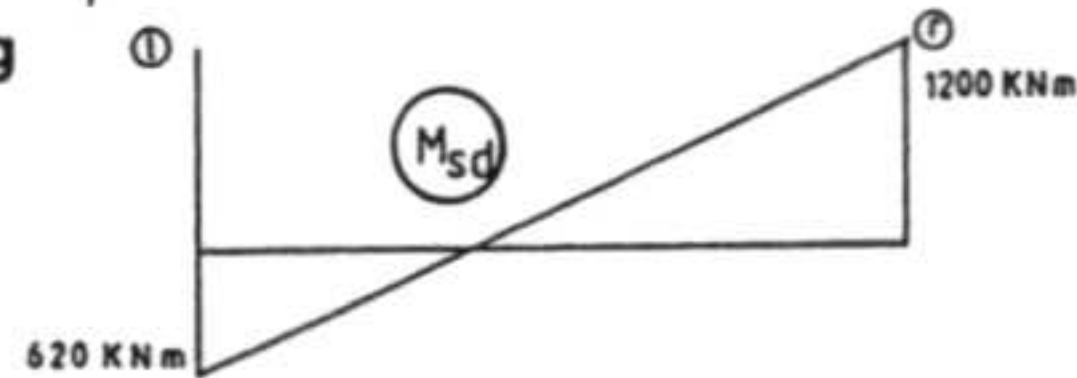
Example 2.2.4: Class 4 cross-section loaded in bending

1. System



Part of a Beam: Welded section
Dimensions see sketch
Steel grade: Fe 360

2. Loading



Bending moment distribution: see sketch
Shear force: $V_{Sd} = 700,0 \text{ kN}$

3. Determination of the resistance

3.1 Width-to-thickness ratios

Fe 360 $\rightarrow \epsilon = 1,0$

Flange:

$$c = 150 - 8/2 - \sqrt{2} \cdot 5 = 138,9 \text{ mm}$$

$$c/t = 138,9/20 = 6,9 < 14\epsilon = 14,0$$

$\rightarrow$ Flange is fully effective

Web:

$$d/t = 1200/8 = 150,0 > 124\epsilon = 124,0$$

$\rightarrow$ Determination of W_{eff}

$$a/d = 2100/1200 = 1,75 > 1,0 \rightarrow k_t = 5,34 + \frac{4}{(a/d)^2} = 5,34 + \frac{4}{1,75^2} = 6,646$$

$$d/t = 150,0 > 30\epsilon\sqrt{k_t} = 77,3 \rightarrow \text{shear buckling verification is required}$$

3.2 Determination of W_{eff} of the web

Fe 360 $\rightarrow 1/\epsilon = 1,0$

$$\bar{b} = d = 1200 \text{ mm}; \quad \bar{\lambda}_p = \frac{d/t}{28,4 \epsilon \sqrt{k_t}} = \frac{d}{t} \frac{1}{138,8 \epsilon} = 150,0 \cdot \frac{1,0}{138,8} = 1,081 > 0,673$$

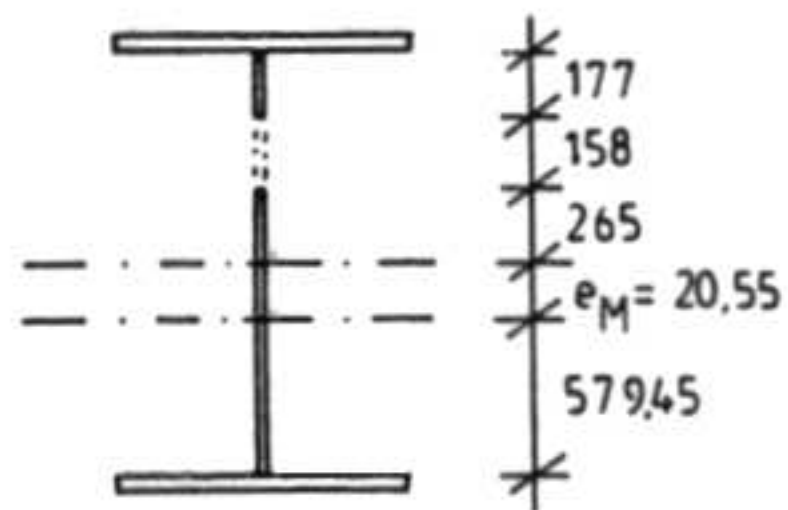
$$\rho = \frac{\bar{\lambda}_p - 0,22}{\bar{\lambda}_p^2} = \frac{1,081 - 0,22}{1,081^2} = 0,737$$

$$d_{eff,1} = 0,4 \cdot \rho \cdot d/2 = 0,4 \cdot 0,737 \cdot 1200/2 = 177 \text{ mm}$$

$$d_{eff,2} = 0,6 \cdot \rho \cdot d/2 = 0,6 \cdot 0,737 \cdot 1200/2 = 265 \text{ mm}$$

$$A_{eff} = 203,9 \cdot 10^2 \text{ mm}^2; e_M = 20,55 \text{ mm}$$

$$I_{eff} = 547148,5 \cdot 10^4 \text{ mm}^4; W_{eff} = 8542 \cdot 10^3 \text{ mm}^3$$



Reference

Commentary

Cross-section properties

$$A = 216 \cdot 10^2 \text{ mm}^2$$

$$I = 561760 \cdot 10^4 \text{ mm}^4$$

throat thickness between flange and web:

$$a = 5 \text{ mm}$$

- ;
[5.3;
Table 5.3.1]

By the classification of welded cross-sections the welds are taken into account.

5.4.5.2; [5.6.3]

5.3.3; [5.4.6]

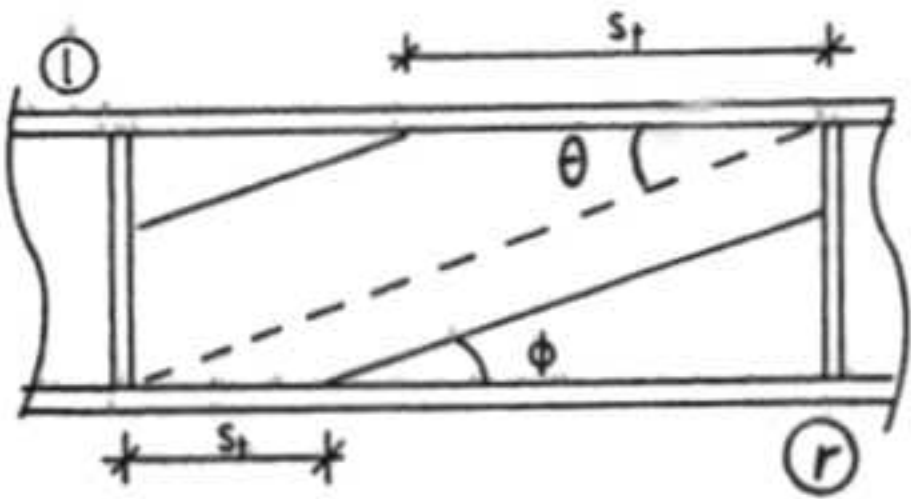
5.3.5; [5.3.5]

Flange is fully effective

$$A_{eff} = (265 + 177 + 593 + 2\sqrt{2} \cdot 5) \cdot 8 + 300 \cdot 20 \cdot 2 = 203,9 \cdot 10^2 \text{ mm}^2$$

The effective width of the web is calculated on the basis of f_y and $\Psi = -1$.

Problem, Sketch, Calculation	Reference	Commentary
3.3 Shear buckling verification of web using "simple post-critical method" $\bar{\lambda}_w = \frac{d/t_w}{37,4 \cdot \epsilon_w \sqrt{k_\tau}} = \frac{150}{37,4 \cdot \sqrt{6,646}} = 1,556 > 1,2$ $\tau_{bs} = f_{yw} / \sqrt{3} \cdot 0,9 / \bar{\lambda}_w = 235 / \sqrt{3} \cdot 0,9 / 1,556 = 78,5 \text{ N/mm}^2$ $V_{bs,Rd} = \frac{d \cdot t_w \cdot \tau_{bs}}{\gamma_{M1}} = \frac{1200 \cdot 8 \cdot 78,5}{1,1} \cdot 10^{-3} = 685,1 \text{ kN} < 700,0 \text{ kN}$ → Criterion is not satisfied	5.4.5; [5.6]	Criterion is not fulfilled, therefore → use a thicker plate → or decrease the aspect ratio → or use a more precise verification
3.4 Shear buckling verification using tension field method Requirement: $1 \leq a/d = 1,75 \leq 3$	- ; [5.6.4] - ; [5.6.2]	A more precise verification is presented
3.4.1 Initial shear buckling strength $\lambda_w = 1,556 > 1,25 \rightarrow \tau_{bb} = f_{yw} / \sqrt{3} \cdot 1 / \lambda_w^2 = 235 / \sqrt{3} \cdot 1 / 1,556^2 = 56,0 \text{ N/mm}^2$	- ; [5.6.4.1.(2)]	
3.4.2 Determination of the reduced plastic moment resistance $M_{r,Rk}$: On the right: $N_{r,Rk} = M_{sd} / (h - t_f) = 1200 / (1240 - 20) \cdot 10^3 = 983,6 \text{ kN}$ On the left: $N_{l,Rk} = M_{sd} / (h - t_f) = 620 / (1240 - 20) \cdot 10^3 = 508,2 \text{ kN}$ $M_{r,Rk} = 0,25 b t_f^2 f_{yf} \left[1 - \left(\frac{N_{r,Rk}}{b t_f f_{yf} / \gamma_{M0}} \right)^2 \right] = 0,25 \cdot 300 \cdot 20^2 \cdot 235 \left[1 - \left(\frac{983,6 \cdot 10^3}{300 \cdot 20 \cdot 235 / 1,1} \right)^2 \right] \cdot 10^{-6} = 2,90 \text{ kNm}$ $M_{l,Rk} = 0,25 b t_f^2 f_{yf} \left[1 - \left(\frac{N_{l,Rk}}{b t_f f_{yf} / \gamma_{M0}} \right)^2 \right] = 0,25 \cdot 300 \cdot 20^2 \cdot 235 \left[1 - \left(\frac{508,2 \cdot 10^3}{300 \cdot 20 \cdot 235 / 1,1} \right)^2 \right] \cdot 10^{-6} = 5,94 \text{ kNm}$	- ; [5.6.4.1.(4)]	Only the flanges carry the moments whilst the web carries the shear force. In this example both flange hinges occur in the tension flange
3.4.3 Determination of the strength of the tension field $\Theta = \arctan d/a = \arctan 1200/2100 = 29,7^\circ$ conservative assumption: $\Phi \approx \Theta / 1,5 = 19,8^\circ$ $\psi = 1,5 \tau_{bb} \sin(2\Phi) = 1,5 \cdot 56,0 \cdot \sin(2 \cdot 19,8) = 53,5 \text{ N/mm}^2$ $\sigma_{bb} = \sqrt{f_{yw}^2 - 3 \tau_{bb}^2 + \psi^2} - \psi = \sqrt{235^2 - 3 \cdot 56,0^2 + 53,5^2} - 53,5 = 167,1 \text{ N/mm}^2$	- ; [5.6.4.2] - ; [5.6.4.1.(1)]	

Problem, Sketch, Calculation	Reference	Commentary
3.4.4 Anchorage lengths of the tension field  $s_{t,t} = \frac{2}{\sin \Phi} \sqrt{\frac{M_{f,c,Rk}}{t_w \sigma_{bb}}} = \frac{2}{\sin 19,8} \sqrt{\frac{2,90 \cdot 10^6}{8 \cdot 167,1}} = 275 \text{ mm}$ $s_{t,l} = \frac{2}{\sin \Phi} \sqrt{\frac{M_{l,c,Rk}}{t_w \sigma_{bb}}} = \frac{2}{\sin 19,8} \sqrt{\frac{5,94 \cdot 10^6}{8 \cdot 167,1}} = 393,6 \text{ mm}$	- ; [5.6.4.1.(3)]	
3.4.5 Width of the tension field $g = d \cdot \cos \Phi - (a - s_{t,t} - s_{t,l}) \cdot \sin \Phi = 1200 \cdot \cos 19,8 - (2100 - 275 - 393,6) \cdot \sin 19,8 = 644,2 \text{ mm}$	- ; [5.6.4.1.(3)]	
3.4.6 Shear buckling resistance $V_{bb,Rd} = \frac{d t_w \tau_{bb} + 0,9 g t_w \sigma_{bb} \sin \Phi}{\gamma_{M1}} = \frac{1200 \cdot 8 \cdot 56,0 + 0,9 \cdot 644,2 \cdot 8 \cdot 167,1 \cdot \sin 19,8}{1,1} \cdot 10^{-3} = 727,4 \text{ kN} > 700 \text{ kN}$ $700 > 0,5 V_{bb,Rd} = 359,8 \text{ kN}$ ----> interaction bending - shear is required	- ; [5.6.4.1.(1)]	
3.4.7 Determination of the design resistance moment considering the flanges only $M_{f,Rd} = A_t (d + t_f) f_{yt} / \gamma_{Mo} = 300 \cdot 20 \cdot (1200 + 20) \cdot 235 \cdot 10^{-6} / 1,1 = 1563,8 \text{ kNm} > 1200,0 \text{ kNm}$	- ; [5.6.7.3.(1)]	
3.4.8 Interaction bending - shear buckling ----> not necessary because of $M_{Sd} < M_{f,Rd}$	- ; [5.6.7.3.(1)]	

2.3 Combined loading - Bending and compression

These examples demonstrate the verification of members loaded by the combination of bending and compression assuming design values of action effects (N_{sd} , V_{sd} , M_{sd} , etc) which have been calculated by an analysis of the structure and these values already include ψ , γ_F etc. The second order effects are considered by using first order elastic analysis with sway-mode buckling lengths. The results presented in the examples are rounded values. For the purpose of easy re-calculation each formula and each check is calculated with the rounded values.

The following examples are included in this chapter:

- | | |
|-----------------------|---|
| Example 2.3.1: | RHS column loaded in bending and axial compression |
| Example 2.3.2: | HEA profile loaded in bending and axial compression |
| Example 2.3.3: | Column of a frame |
| Example 2.3.4: | Rafter of a frame |

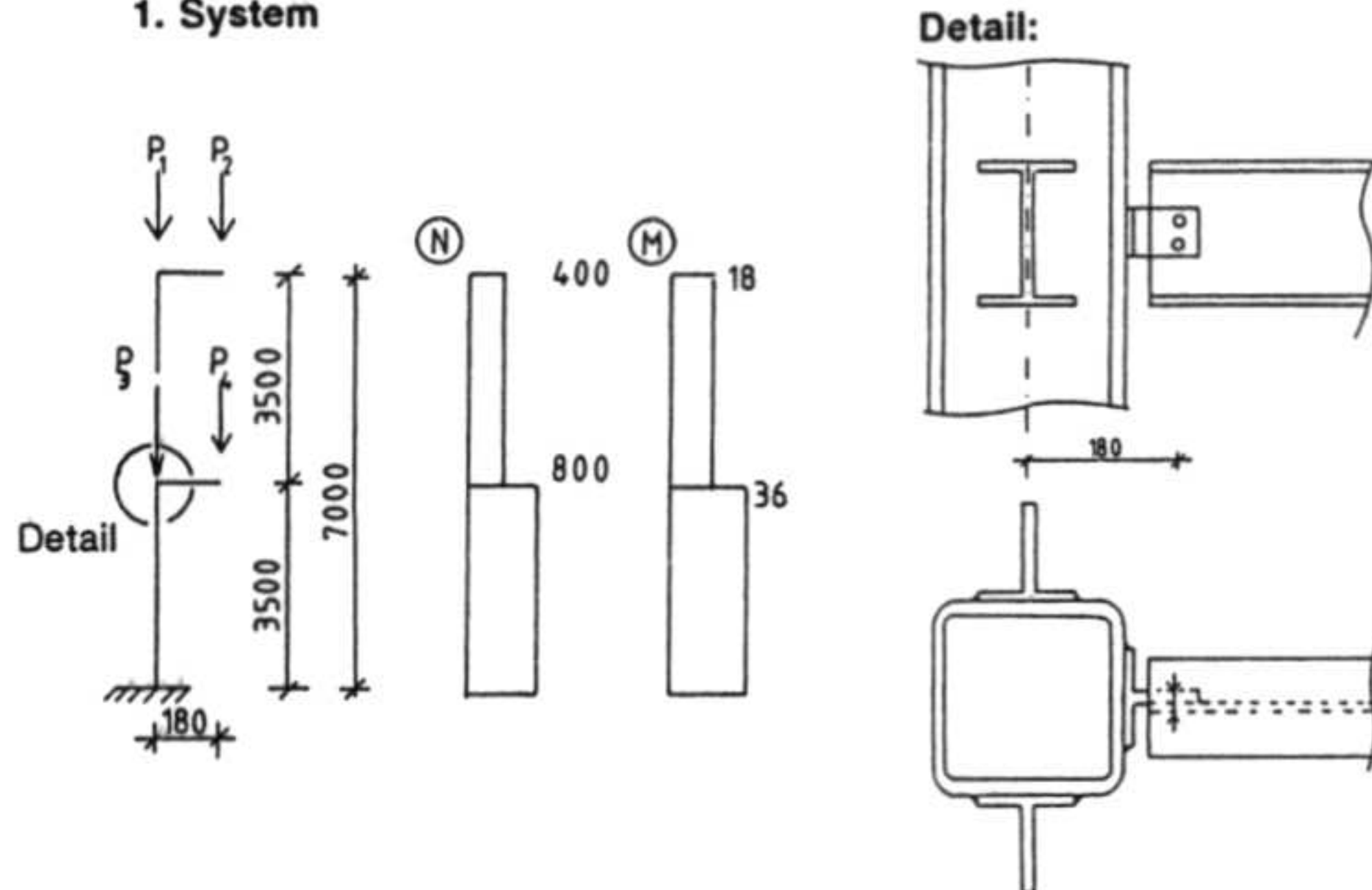
Problem, Sketch, Calculation

Reference

Commentary

Example 2.3.1: RHS profile loaded in bending and axial compression

1. System



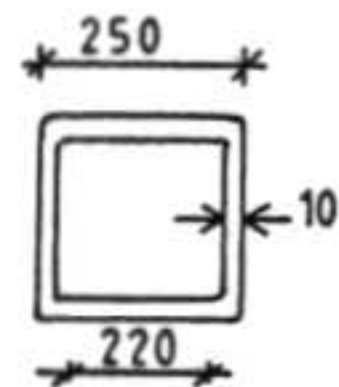
Column: RHS 250 x 10,0, hot rolled
Steel grade: Fe 430
System length: $L_1 = L_2 = 3,50$ m
Buckling lengths: $\ell_1 = 10,85$ m
 $\ell_2 = 15,34$ m

2. Loading

Axial compression and moment distribution see sketch
 $P_1 = P_3 = 300$ kN; $P_2 = P_4 = 100$ kN

3. Determination of the resistance

3.1 Width-to-thickness ratios



Fe 430 $\rightarrow \epsilon = 0,92$

Flange:

$$d = b - 3t = 250 - 3 \cdot 10 = 220 \text{ mm}$$

$$d/t = 220/10 = 22 < 38\epsilon = 35$$

Web:

$$d/t = 220/10 = 22 < 83\epsilon = 76,4$$

$$\rightarrow A_{eff} = A; W_{eff} = W_{pl}$$

$$d/t = 22 < 69\epsilon = 63,5$$

$\rightarrow$ no shear buckling verification is required

3.2 Member resistance verification

3.2.1 Axial compression resistance

Part 1:

Buckling length: $\ell = 10,85$ m

$$\bar{\lambda} = \frac{\ell}{i} \frac{1}{\lambda_1} = \frac{10,85 \cdot 10^3}{96,7} \cdot \frac{1}{86,8} = 1,29$$

using buckling curve a $\rightarrow \chi_a = 0,4760$

$$N_{b,Rd} = \chi_a \cdot \frac{A \cdot f_y}{\gamma_{M1}} = 0,4760 \cdot \frac{91,7 \cdot 10^2 \cdot 275}{1,1} \cdot 10^{-3} = 1091,2 \text{ kN} > 800,0 \text{ kN}$$

No restraints at the load introduction points.
For determining the cross-section properties $I_w, I_y, I_z, W, A, A_w, i$ etc. dimension tables are used.
The buckling lengths are determined by a separate calculation using the literature

Load factors are included. Global imperfections are included in the global analysis to determine the applied loads.

5.3.1; Table 5.10
[5.3;
Table 5.3.1]

The classification of cross-sections could be made by appropriate dimension tables.

Firstly a stress distribution of pure bending in the web is considered. When the real stress distribution is known a final check has to be made. This final check is not made.

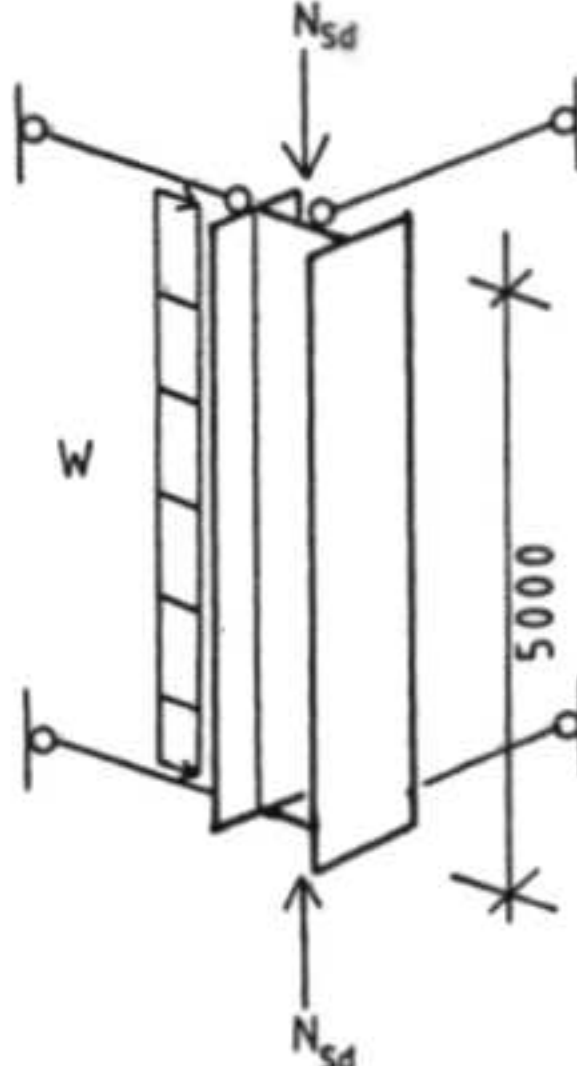
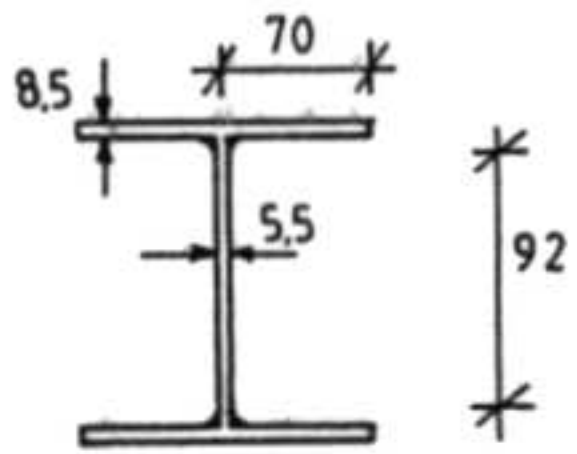
5.3.3; [5.4.6]

For rolled sections the shear buckling verification need normally not to be considered.

5.4.2; [5.5.1]

The buckling check is made for the two individual segments (Part 1 and Part 2) by using first order analysis with sway-mode buckling lengths.
The global column buckling mode is included in the check of the first segment (Part 1).

Problem, Sketch, Calculation		Reference	Commentary
Part 2:	Buckling length: $\ell = 15,34 \text{ m}$ $\bar{\lambda} = \frac{\ell}{i} \frac{1}{\lambda_1} = \frac{15,34 \cdot 10^3}{96,7} \cdot \frac{1}{86,8} = 1,83$ using buckling curve a $\rightarrow \chi_s = 0,2623$ $N_{b,Rd} = \chi_s \cdot \frac{A \cdot f_y}{\gamma_{M1}} = 0,2623 \cdot \frac{91,7 \cdot 10^2 \cdot 275}{1,1} \cdot 10^{-3} = 601,3 \text{ kN} > 400,0 \text{ kN}$	5.4.2; [5.5.1]	
3.2.2 Bending moment resistance	$M_{y,Rd} = \frac{W_{pl} f_y}{\gamma_{M0}} = \frac{832,8 \cdot 10^3 \cdot 275}{1,1} \cdot 10^{-6} = 208,2 \text{ kNm} > 40,0 \text{ kNm}$	5.3.2; [5.4.5]	RHS $\rightarrow$ No lateral torsional buckling verification is required.
3.2.3 Interaction Bending - axial compression without lateral torsional buckling phenomena		5.4.4; Table 5.26 [5.5.4]	
Part 1:	$n = N_{Sd}/N_{b,Rd} = 800,0/1091,2 = 0,73$ $\Psi = 1,0 \rightarrow \beta_M = 1,1$ $\alpha = W_{pl}/W_{el} - 1 = 832,8 \cdot 10^3 / 707,3 \cdot 10^3 - 1 = 0,177$ $\mu = \lambda_1 (2\beta_M - 4) + \alpha = 1,21 (2 \cdot 1,1 - 4) + 0,177 = -2,001 < 0,9$ $1 - \mu n / \gamma_{M1} = 1 + 2,001 \cdot 0,73 / 1,1 = 2,33 > 1,5 \rightarrow k = 1,5$ $n + \frac{1,5 M_{Sd}}{W_{pl} \frac{f_y}{\gamma_{M1}}} = 0,73 + \frac{1,50 \cdot 36,0}{832,8 \cdot 10^3 \cdot \frac{275}{1,1} \cdot 10^{-6}} = 0,99 < 1$		
Part 2:	$n = N_{Sd}/N_{b,Rd} = 400,0/601,3 = 0,67$ $\Psi = 1,0 \rightarrow \beta_M = 1,1$ $\alpha = W_{pl}/W_{el} - 1 = 832,8 \cdot 10^3 / 707,3 \cdot 10^3 - 1 = 0,177$ $\mu = \lambda_2 (2\beta_M - 4) + \alpha = 1,92 (2 \cdot 1,1 - 4) + 0,177 = -3,279 < 0,9$ $1 - \mu n / \gamma_{M1} = 1 + 3,279 \cdot 0,67 / 1,1 = 3,0 > 1,5 \rightarrow k = 1,5$ $n + \frac{1,5 M_{Sd}}{W_{pl} \frac{f_y}{\gamma_{M1}}} = 0,67 + \frac{1,50 \cdot 18,0}{832,8 \cdot 10^3 \cdot \frac{275}{1,1} \cdot 10^{-6}} = 0,80 < 1$	5.4.4; Table 5.26 [5.5.4]	

Problem, Sketch, Calculation	Reference	Commentary
Example 2.3.2: HEA profile loaded in bending and axial compression 1. System  Column: HE 140 A Steel grade: Fe 360 System length: L = 5,00 m Buckling length: $\ell_y = 5,00$ m $\ell_z = 5,00$ m 2. Loading Axial compression: $N_{Sd} = 120$ kN Uniform distributed load: $w = 2,67$ kN/m Moment at mid-span: $M_{Sd} = 1,5 \cdot 2,67 \cdot 5^2 / 8 = 12,5$ kNm		For determining the cross-section properties I_w, I_y, I_z, W, A, A_w, i etc. dimension tables are used.
3. Determination of the resistance 3.1 Width-to-thickness ratios Fe 360 $\rightarrow \epsilon = 1,00$ Flange: $c/t = 70/8,5 = 8,2 < 11\epsilon = 11$ Web: $d/t = 92/5,5 = 16,7 < 83\epsilon = 83$ $\rightarrow A_{eff} = A$; $W_{eff} = W_{pl}$ $d/t = 16,7 < 69\epsilon = 69$ $\rightarrow$ no shear buckling verification is required 	5.3.1; Table 5.10 [5.3; Table 5.3.1]	The classification of cross-sections could be made by appropriate dimension tables. Firstly a stress distribution on pure bending in the web is considered. When the real stress distribution is known a final check has to be made. The final check is not made.
3.2 Member resistance verification 3.2.1 Axial compression resistance Strong axis: Buckling length: $\ell_y = 5,00$ m $\bar{\lambda}_y = \frac{\ell_y}{i_y} \cdot \frac{1}{\lambda_1} = \frac{5,00 \cdot 10^3}{57,3} \cdot \frac{1}{93,9} = 0,93$ using buckling curve b $\rightarrow \chi_b = 0,6419$ $N_{y,b,Rd} = \chi_b \cdot \frac{A \cdot f_y}{\gamma_{M1}} = 0,6419 \cdot \frac{31,4 \cdot 10^2 \cdot 235}{1,1} \cdot 10^{-3} = 430,6 \text{ kN} > 120,0 \text{ kN}$	5.3.3; [5.4.6] 5.4.2; [5.5.1]	For rolled sections the shear buckling verification need normally not to be considered.

Problem, Sketch, Calculation	Reference	Commentary
Weak axis: Buckling length: $\ell_z = 5,00 \text{ m}$ $\bar{\lambda}_z = \frac{\ell_z}{i_z} \frac{1}{\lambda_1} = \frac{5,00 \cdot 10^3}{35,2} \cdot \frac{1}{93,9} = 1,51$ using buckling curve c $\rightarrow \chi_c = 0,3113$ $N_{z,b,Rd} = \chi_c \cdot \frac{A \cdot f_y}{\gamma_{M1}} = 0,3113 \cdot \frac{31,4 \cdot 10^2 \cdot 235}{1,1} \cdot 10^{-3} = 208,8 \text{ kN} > 120,0 \text{ kN}$	5.4.2; [5.5.1]	
3.2.2 Bending moment resistance 3.2.2.1 Determination of the critical bending moment M_{cr} Buckling length $\ell_{LT} = 5,0 \text{ m}$ $k = k_w = 1,0$; Moment distribution  $\rightarrow C_1 = 1,132$ $M_{cr} = C_1 \frac{\pi^2 E I_z}{\ell_{LT}^2} \sqrt{\left(\frac{k}{k_w}\right)^2 \frac{I_w}{I_z} + 0,039 \ell_{LT}^2 \frac{I_t}{I_z}} = 1,132 \cdot \frac{\pi^2 \cdot 210000 \cdot 389 \cdot 10^4}{5,0^2 \cdot 10^{12}} \sqrt{\frac{15060 \cdot 10^8 + 0,039 \cdot 5,0^2 \cdot 10^8 \cdot 8,16 \cdot 10^4}{389 \cdot 10^4}} = 56,9 \text{ kNm}$ $\bar{\lambda}_{LT} = \sqrt{\frac{W_{pl} f_y}{M_{cr}}} = \sqrt{\frac{173 \cdot 10^3 \cdot 235}{56,9 \cdot 10^6}} = 0,85 > 0,4$ using buckling curve a $\rightarrow \chi_{LT,a} = 0,7659$ 3.2.2.2 Bending moment resistance $M_{b,Rd} = \chi_{LT} \frac{W_{pl} f_y}{\gamma_{M1}} = 0,7659 \cdot \frac{173,0 \cdot 10^3 \cdot 235}{1,1} \cdot 10^{-6} = 28,3 \text{ kNm} > 12,5 \text{ kNm}$	5.4.3; [Annex F; 5.5.2]	
3.2.3 Interaction Bending - axial compression without lateral torsional buckling phenomena $n_y = N_{Sd}/N_{y,b,Rd} = 120,0/430,6 = 0,28$ $\beta_M = 1,3$ $\alpha = W_{pl}/W_{el} - 1 = 173,0 \cdot 10^3 / 155,0 \cdot 10^3 - 1 = 0,116$ $\mu_y = \lambda_y (2\beta_M - 4) + \alpha = 0,93 (2 \cdot 1,3 - 4) + 0,116 = -1,186 < 0,9$ $1 - \mu_y n_y / \gamma_{M1} = 1 + 1,186 \cdot 0,28 / 1,1 = 1,30 < 1,5 \rightarrow k_y = 1,30$ $n_y + \frac{k_y M_{y,Sd}}{W_{pl,y} \frac{f_y}{\gamma_{M1}}} = 0,28 + \frac{1,30 \cdot 12,5 \cdot 10^6}{173 \cdot 10^3 \cdot \frac{235}{1,1}} = 0,72 < 1$	5.4.4; Table 5.26 [5.5.4]	For mono-axial bending one should take the relevant buckling curve for buckling in the moment plane.

Problem, Sketch, Calculation	Reference	Commentary
3.2.4 Interaction Bending - axial compression with lateral torsional buckling phenomena $n_z = N_{sd}/N_{z,b,Rd} = 120,0/208,8 = 0,575$ $\beta_{M,LT} = 1,3$ $\mu_{LT} = 0,15\lambda_z\beta_{M,LT} - 0,15 = 0,15 \cdot 1,51 \cdot 1,3 - 0,15 = 0,144 < 0,9$ $1 - \mu_{LT}n_z/\gamma_{M1} = 1 - 0,144 \cdot 0,575/1,1 = 0,925 < 1,0 \longrightarrow k_{LT} = 0,925$ $n_z + \frac{k_{LT} M_{y,sd}}{M_{b,Rd}} = 0,575 + \frac{0,925 \cdot 12,5}{28,3} = 0,98 < 1$	5.4.4; Table 5.26 [5.5.4]	

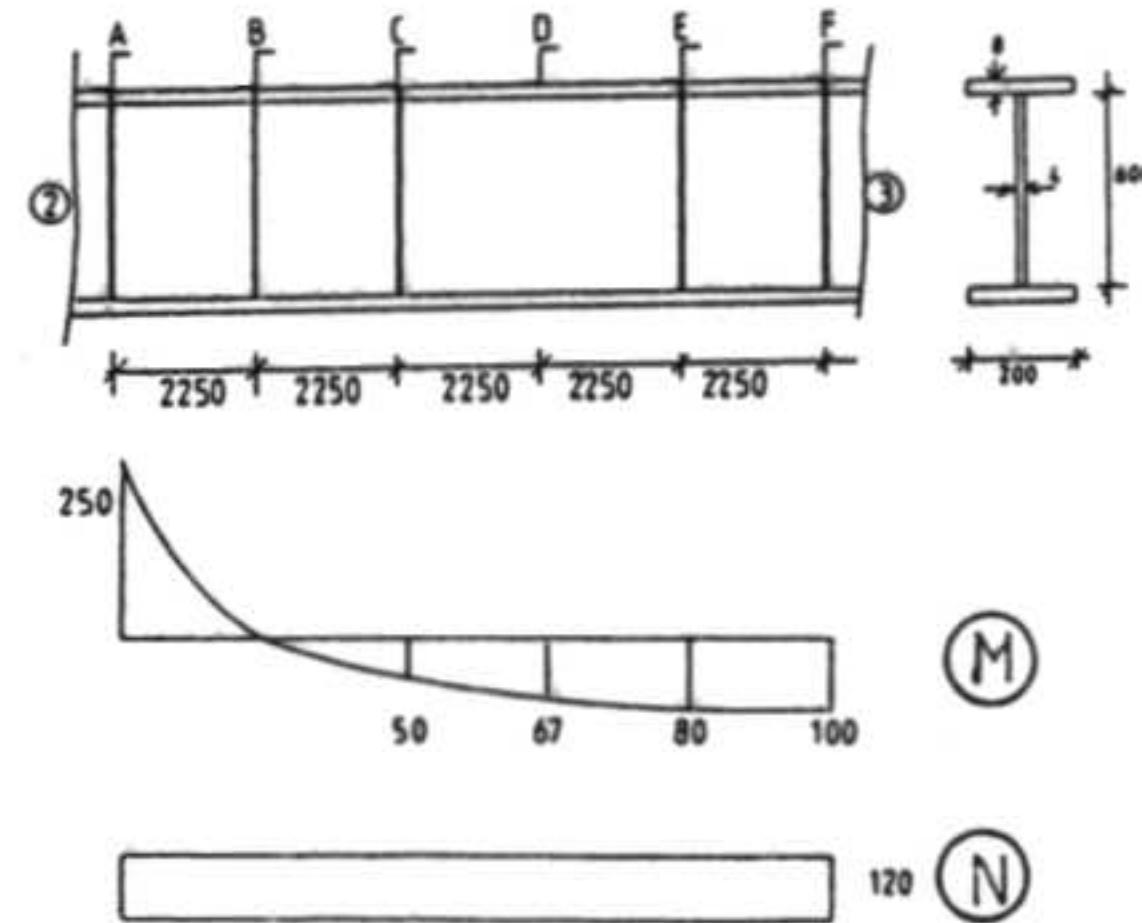
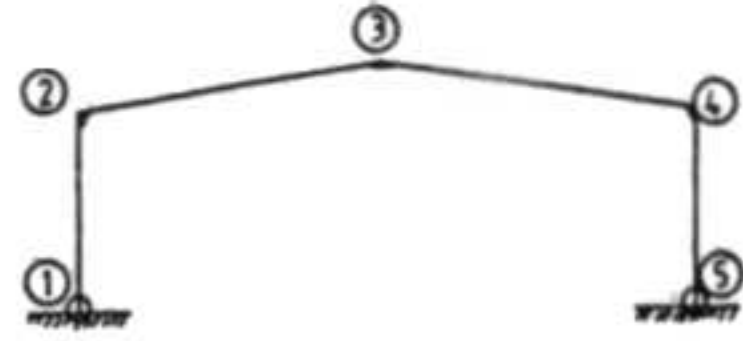
Problem, Sketch, Calculation	Reference	Commentary
Weak axis: Section 1: Buckling length: $\ell_{1,z} = 3,00 \text{ m}$ $\bar{\lambda}_{1,z} = \frac{\ell_{1,z}}{i_z} \frac{1}{\lambda_1} = \frac{3,00 \cdot 10^3}{38,4} \cdot \frac{1}{76,4} = 1,02$ using buckling curve b $\rightarrow \chi_b = 0,5844$ $N_{1,z,b,Rd} = \chi_b \cdot \frac{A \cdot f_y}{\gamma_{M1}} = 0,5844 \cdot \frac{64,0 \cdot 10^2 \cdot 355}{1,1} \cdot 10^{-3} = 1207,1 \text{ kN} > 100,0 \text{ kN}$ Section 2: Buckling length: $\ell_{2,z} = 1,50 \text{ m}$ $\bar{\lambda}_{2,z} = \frac{\ell_{2,z}}{i_z} \frac{1}{\lambda_1} = \frac{1,50 \cdot 10^3}{38,4} \cdot \frac{1}{76,4} = 0,51$ using buckling curve b $\rightarrow \chi_b = 0,8798$ $N_{2,z,b,Rd} = \chi_b \cdot \frac{A \cdot f_y}{\gamma_{M1}} = 0,8798 \cdot \frac{64,0 \cdot 10^2 \cdot 355}{1,1} \cdot 10^{-3} = 1817,2 \text{ kN} > 100,0 \text{ kN}$	5.4.2; [5.5.1]	
3.2.2 Bending moment resistance 3.2.2.1 Section 1 Buckling length $\ell_{LT} = 3,0 \text{ m}$ $k = k_w = 1,0$; $\Psi = 0 \rightarrow C_1 = 1,879$ $M_{\alpha} = C_1 \frac{\pi^2 E I_z}{\ell_{LT}^2} \sqrt{\left(\frac{k}{k_w}\right)^2 \frac{I_w}{I_z} + 0,039 \ell_{LT}^2 \frac{I_t}{I_z}} = 1,879 \cdot \frac{\pi^2 \cdot 210000 \cdot 944 \cdot 10^4}{3,0^2 \cdot 10^{12}} \sqrt{\frac{282000 \cdot 10^8 + 0,039 \cdot 3,0^2 \cdot 10^8 \cdot 26,5 \cdot 10^4}{944 \cdot 10^4}} = 814,2 \text{ kNm}$ $\bar{\lambda}_{LT} = \sqrt{\frac{W_{pl} f_y}{M_{\alpha}}} = \sqrt{\frac{907 \cdot 10^3 \cdot 355}{814,2 \cdot 10^6}} = 0,63 > 0,4$ using buckling curve a $\rightarrow \chi_{LT,a} = 0,8783$ $M_{1,b,Rd} = \chi_{LT,a} \frac{W_{pl} f_y}{\gamma_{M1}} = 0,8783 \cdot \frac{907,0 \cdot 10^3 \cdot 355}{1,1} \cdot 10^{-6} = 257,1 \text{ kNm} > 133,3 \text{ kNm}$	5.4.3; [5.5.2]	

Problem, Sketch, Calculation	Reference	Commentary
3.2.2.2 Section 2 Buckling length $\ell_{LT} = 1,5 \text{ m}$ $k = k_w = 1,0$; $\Psi = 133/200 \rightarrow C_1 = 1,199$ $M_{\alpha} = C_1 \frac{\pi^2 E I_z}{\ell_{LT}^2} \sqrt{\left(\frac{k}{k_w}\right)^2 \frac{I_w}{I_z} + 0,039 \ell_{LT}^2 \frac{I_t}{I_z}} = 1,199 \cdot \frac{\pi^2 \cdot 210000 \cdot 944 \cdot 10^4}{1,5^2 \cdot 10^{12}} \sqrt{\frac{282000 \cdot 10^8 + 0,039 \cdot 1,5^2 \cdot 10^8 \cdot 26,5 \cdot 10^4}{944 \cdot 10^4}} = 1874,9 \text{ kNm}$ $\bar{\lambda}_{LT} = \sqrt{\frac{W_{pl,y} f_y}{M_{\alpha}}} = \sqrt{\frac{907 \cdot 10^3 \cdot 355}{1874,9 \cdot 10^6}} = 0,41 > 0,4 \text{ using buckling curve a} \rightarrow \chi_{LT,a} = 0,9501$ $M_{2b,Rd} = \chi_{LT,a} \frac{W_{pl,y} f_y}{\gamma_{M1}} = 0,9501 \cdot \frac{907 \cdot 10^3 \cdot 355}{1,1} \cdot 10^{-6} = 278,1 \text{ kNm} > 200 \text{ kNm}$	5.4.3; [5.5.2]	
3.2.3 Interaction Bending - axial compression without lateral torsional buckling phenomena $n_y = N_{Sd}/N_{y,Rd} = 100,0/719,2 = 0,14$ $\Psi = 0 \rightarrow \beta_{M,y} = 1,8$ $\alpha = W_{pl,y}/W_{el,y} - 1 = 907 \cdot 10^3 / 812 \cdot 10^3 - 1 = 0,12$ $\mu_y = \lambda_y (2\beta_{M,y} - 4) + \alpha = 1,56 (2 \cdot 1,8 - 4) + 0,12 = -0,5 < 0,9$ $1 - \mu_y n_y / \gamma_{M1} = 1 + 0,5 \cdot 0,14 / 1,1 = 1,06 < 1,5 \rightarrow k_y = 1,06$ $n_y + \frac{k_y M_{y,Sd}}{W_{pl,y} \frac{f_y}{\gamma_{M1}}} = 0,14 + \frac{1,06 \cdot 200,0 \cdot 10^6}{907 \cdot 10^3 \cdot \frac{355}{1,1}} = 0,86 < 1$	5.4.4; Table 5.26 [5.5.4]	
3.2.4 Interaction Bending - axial compression with lateral torsional buckling phenomena 3.2.4.1 Section 1: $n_z = N_{Sd}/N_{z,Rd} = 100,0/1207,1 = 0,08$ $\Psi = 0 \rightarrow \beta_{M,z} = 1,8$ $\mu_{LT} = 0,15 \lambda_{1,z} \beta_{M,LT} - 0,15 = 0,15 \cdot 1,02 \cdot 1,8 - 0,15 = 0,125 < 0,9$ $1 - \mu_{LT} n_z / \gamma_{M1} = 1 - 0,125 \cdot 0,08 / 1,1 = 0,991 < 1,0 \rightarrow k_{LT} = 0,991$ $n_z + \frac{k_{LT} M_{z,Sd}}{M_{1b,Rd}} = 0,08 + \frac{0,991 \cdot 133,3}{257,1} = 0,59 < 1$	5.4.4; Table 5.26 [5.5.4]	
3.2.4.2 Section 2: $n_z = N_{Sd}/N_{z,Rd} = 100,0/1817,2 = 0,06$ $\Psi = 133,3/200 = 0,67 \rightarrow \beta_{M,LT} = 1,8 - 0,7 \cdot 0,67 = 1,331$ $\mu_{LT} = 0,15 \lambda_{2,z} \beta_{M,LT} - 0,15 = 0,15 \cdot 0,51 \cdot 1,331 - 0,15 = -0,05 < 0,9$ $1 - \mu_{LT} n_z / \gamma_{M1} = 1 + 0,05 \cdot 0,06 / 1,1 = 1,003 > 1,0 \rightarrow k_{LT} = 1,0$ $n_z + \frac{k_{LT} M_{z,Sd}}{M_{2b,Rd}} = 0,06 + \frac{1,0 \cdot 200,0}{278,1} = 0,78 < 1$	5.4.4; Table 5.26 [5.5.4]	

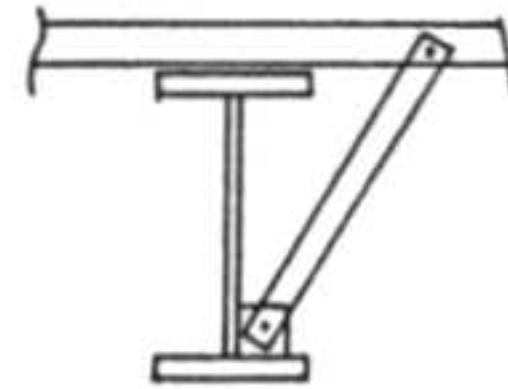
Problem, Sketch, Calculation

Example 2.3.4: Rafter of a frame

1. System



Detail of lateral restraints



Welded section Steel grade Fe 510
Dimensions: see sketch
Buckling length: $\ell_y = 14,2875 \text{ m}$
Rafter laterally restraint at A, B, C, E, F

2. Loading

see sketch

3. Determination of the resistance

3.1 Width-to-thickness ratios

Fe 510 $\rightarrow \epsilon = 0,81$

Flange: $c = 100 - 2 \cdot \sqrt{2} \cdot 3 = 94 \text{ mm}$

$c/t = 94/8 = 11,8 > 14\epsilon = 11,3$

Web:

$d/t = 600/4 = 150 > 124\epsilon = 100,4$

$\rightarrow$ Determination of A_{eff} and W_{eff}

$d/t = 150 > 69\epsilon = 55,9$

$\rightarrow$ shear buckling verification is required

3.2 Determination of effective properties of the cross-section

3.2.1 A_{eff} for compression

Fe 510 $\rightarrow 1/\epsilon = 1,23$

Flange:

$$\bar{b} = c = 94 \text{ mm}; \quad \bar{\lambda}_p = \frac{c/t}{28,4 \epsilon \sqrt{k_\sigma}} = \frac{c}{t} \frac{1}{18,6 \epsilon} = 11,8 \cdot \frac{1,23}{18,6} = 0,78 > 0,673$$

$$\rho = \frac{\bar{\lambda}_p - 0,22}{\bar{\lambda}_p^2} = \frac{0,78 - 0,22}{0,78^2} = 0,92$$

$$c_{eff} = \rho \cdot c = 0,92 \cdot 94 = 86,5 \text{ mm}$$

Reference

Commentary

Weld between flange and web: $a = 3 \text{ mm}$
 $A = 56 \cdot 10^2 \text{ mm}^2$
 $I_y = 36775 \cdot 10^4 \text{ mm}^4$ $i_y = 256,3 \text{ mm}$
 $I_z = 1067 \cdot 10^4 \text{ mm}^4$ $i_z = 43,6 \text{ mm}$
The buckling length is calculated separately.

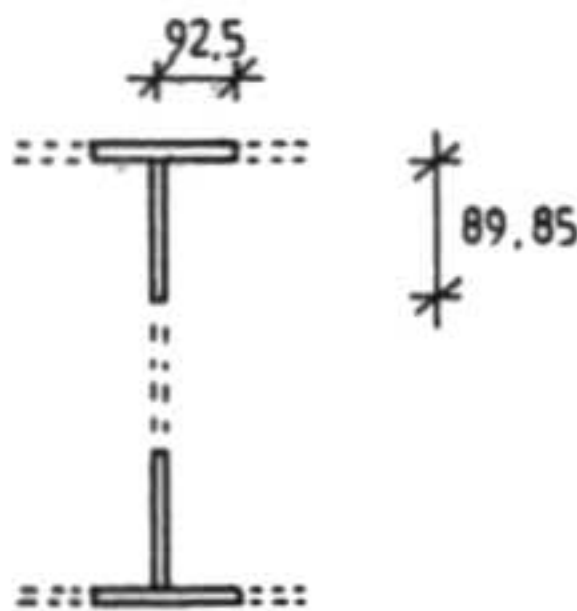
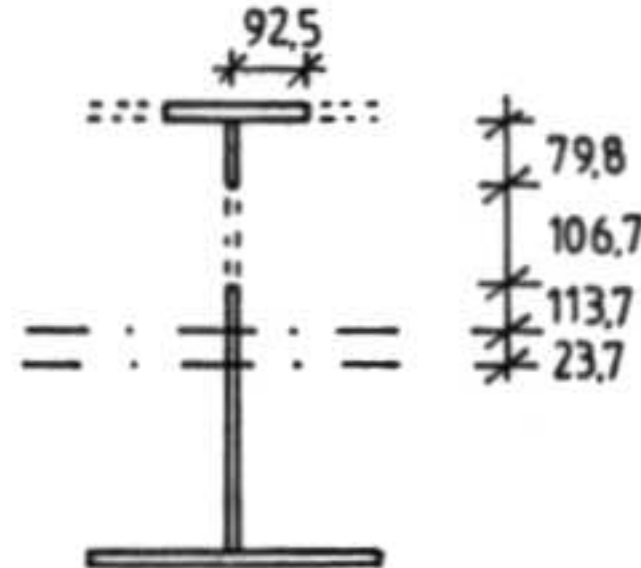
5.3.1; Table 5.10
[5.3;
Table 5.3.1]

The classification with the exact stress distribution leads also to a class 4 section.

The verification of shear buckling is not contained in this example.

5.3.3 [5.4.6]

5.3.5; Table 5.18
[5.3.5]

Problem, Sketch, Calculation	Reference	Commentary
Web: $\bar{b} = d = 592 \text{ mm}; \quad \bar{\lambda}_p = \frac{d/t}{28,4 \sqrt{k_e}} = \frac{d}{t} \frac{1}{56,8 \sqrt{e}} = 148 \cdot \frac{1,23}{56,8} = 3,20 > 0,673$ $\rho = \frac{\bar{\lambda}_p - 0,22}{\bar{\lambda}_p^2} = \frac{3,20 - 0,22}{3,20^2} = 0,29$ $d_{eff} = \rho \cdot d = 0,29 \cdot 592 = 171,7 \text{ mm}$ $A_{eff} = 2 \cdot (2 \cdot 86,5 + 12) \cdot 8 + (171,7 + 8) \cdot 4 = 36,8 \cdot 10^2 \text{ mm}^2$ 3.2.2 W_{eff} for bending Flange: The compressed flange is reduced as above $c_{eff} = \rho \cdot c = 0,92 \cdot 94 = 86,5 \text{ mm}$ Web: $\bar{b} = d = 592 \text{ mm}; \quad \bar{\lambda}_p = \frac{d/t}{28,4 \sqrt{k_e}} = \frac{d}{t} \frac{1}{138,8 \sqrt{e}} = 148 \cdot \frac{1,23}{138,8} = 1,31 > 0,673$ $\rho = \frac{\bar{\lambda}_p - 0,22}{\bar{\lambda}_p^2} = \frac{1,31 - 0,22}{1,31^2} = 0,64$ $d_{eff,1} = 0,4 \cdot \rho \cdot d_c = 0,4 \cdot 0,64 \cdot 0,5 \cdot 592 = 75,8 \text{ mm}$ $d_{eff,2} = 0,6 \cdot \rho \cdot d_c = 0,6 \cdot 0,64 \cdot 0,5 \cdot 592 = 113,7 \text{ mm}$ $A_{eff} = (2 \cdot 86,5 + 12 + 200) \cdot 8 + (75,8 + 113,7 + 0,5 \cdot 592 + 8) \cdot 4 = 50,5 \cdot 10^2 \text{ mm}^2$ $e = (608 \cdot 185 \cdot 8 + 80 \cdot 4 \cdot 564 + 413,9 \cdot 4 \cdot 211) / 505 = 280,3 \text{ mm}$ $\Delta e_M = 304 - 280,3 = 23,70 \text{ mm}$ $I_{eff,y} = 4 \cdot (80^3 + 413,9^3) / 12 + 2 \cdot 8^3 \cdot (179 + 200) / 12 + 280,3^2 \cdot 200 \cdot 8 + 327,2^2 \cdot 179 \cdot 8 + 285,8^2 \cdot 80 \cdot 4 + 413,9 \cdot 4 \cdot 69,45^2$ $= 33698 \cdot 10^4 \text{ mm}^4$ $W_{eff,y} = 33698 \cdot 10^4 / 331,7 = 1016 \cdot 10^3 \text{ mm}^3$	5.3.5; Table 5.18 [5.3.5]   5.4.2; [5.5.1]	
3.3 Member resistance verification 3.3.1 Axial compression resistance Strong axis: Buckling length: $\ell_y = 14,2875 \text{ m}$ $\bar{\lambda}_y = \frac{\ell_y}{i_y} \frac{1}{\lambda_1} \sqrt{\frac{A_{eff}}{A}} = \frac{14,2875 \cdot 10^3}{256,3} \cdot \frac{1}{76,4} \cdot \sqrt{\frac{36,8 \cdot 10^2}{56 \cdot 10^2}} = 0,59$ using buckling curve b $\rightarrow \chi_b = 0,8420$ $N_{y,b,Rd} = \chi_b \cdot \frac{A_{eff} \cdot f_y}{\gamma_{M1}} = 0,8420 \cdot \frac{36,8 \cdot 10^2 \cdot 355}{1,1} \cdot 10^{-3} = 1000,0 \text{ kN} > 120,0 \text{ kN}$		For determining the slenderness λ of a class 4 section the radius of gyration about the relevant axis from the gross cross-section should always be used.

Problem, Sketch, Calculation	Reference	Commentary
Weak axis: between A-B, B-C and E-F: Buckling length: $\ell_z = 2,25 \text{ m}$ $\bar{\lambda}_z = \frac{\ell_z}{i_z} \frac{1}{\lambda_1} \sqrt{\frac{A_{eff}}{A}} = \frac{2,25 \cdot 10^3}{43,6} \cdot \frac{1}{76,4} \cdot \sqrt{\frac{36,8 \cdot 10^2}{56 \cdot 10^2}} = 0,55$ using buckling curve c $\rightarrow \chi_c = 0,8146$ $N_{z.b.Rd} = \chi_c \cdot \frac{A_{eff} \cdot f_y}{\gamma_{M1}} = 0,8146 \cdot \frac{36,8 \cdot 10^2 \cdot 355}{1,1} \cdot 10^{-3} = 967,4 \text{ kN} > 120,0 \text{ kN}$ Weak axis: between C - E: Buckling length: $\ell_z = 4,50 \text{ m}$ $\bar{\lambda}_z = \frac{\ell_z}{i_z} \frac{1}{\lambda_1} \sqrt{\frac{A_{eff}}{A}} = \frac{4,5 \cdot 10^3}{43,6} \cdot \frac{1}{76,4} \cdot \sqrt{\frac{36,8 \cdot 10^2}{56 \cdot 10^2}} = 1,10$ using buckling curve c $\rightarrow \chi_c = 0,4842$ $N_{z.b.Rd} = \chi_c \cdot \frac{A_{eff} \cdot f_y}{\gamma_{M1}} = 0,4842 \cdot \frac{36,8 \cdot 10^2 \cdot 355}{1,1} \cdot 10^{-3} = 575,1 \text{ kN} > 120,0 \text{ kN}$		
3.3.2 Bending moment resistance Part A - B: buckling length $\ell_{LT} = 2,25 \text{ m}$ $k = k_w = 1,0$; $\Psi \approx 0 \rightarrow C_1 = 1,879$ $M_{\alpha} = C_1 \frac{\pi^2 E I_z}{\ell_{LT}^2} \sqrt{\left(\frac{k}{k_w}\right)^2 \frac{I_w}{I_z} + 0,039 \ell_{LT}^2 \frac{I_1}{I_z}} = 1,879 \cdot \frac{\pi^2 \cdot 210000 \cdot 1067 \cdot 10^4}{2,25^2 \cdot 10^{12}} \sqrt{\frac{986000 \cdot 10^8 + 0,039 \cdot 2,25^2 \cdot 10^8 \cdot 8,1 \cdot 10^4}{1067 \cdot 10^4}} = 2515,3 \text{ kNm}$ $\bar{\lambda}_{LT} = \sqrt{\frac{W_{eff} f_y}{M_{\alpha}}} = \sqrt{\frac{1016 \cdot 10^3 \cdot 355}{2515,3 \cdot 10^6}} = 0,38 < 0,4$ $\rightarrow$ no lateral torsional buckling verification is required	5.4.3; [5.5.2]	For determining M_{cr} the properties of the gross cross-section should always be used.

Problem, Sketch, Calculation	Reference	Commentary
Part D - E: buckling length $\ell_{LT} = 2,25 \text{ m}$ $k = k_w = 1,0$; $\Psi = 67/80 = 0,838 \rightarrow C_1 = 1,091$ $M_{\alpha} = C_1 \frac{\pi^2 E I_z}{\ell_{LT}^2} \sqrt{\left(\frac{k}{k_w}\right)^2 \frac{I_w}{I_z} + 0,039 \ell_{LT}^2 \frac{I_t}{I_z}} = 1,091 \frac{\pi^2 \cdot 210000 \cdot 1067 \cdot 10^4}{2,25^2 \cdot 10^{12}} \sqrt{\frac{986000 \cdot 10^8 + 0,039 \cdot 2,25^2 \cdot 10^8 \cdot 8,1 \cdot 10^4}{1067 \cdot 10^4}} = 1460,5 \text{ kNm}$ $\bar{\lambda}_{LT} = \sqrt{\frac{W_{eff} f_y}{M_{\alpha}}} = \sqrt{\frac{1016 \cdot 10^3 \cdot 355}{1460,5 \cdot 10^6}} = 0,50 > 0,4$ $\rightarrow$ using buckling curve c $\rightarrow \chi_{LT,c} = 0,8430$ $M_{b,Rd} = \chi_{LT,c} \frac{W_{eff} f_y}{\gamma_{M1}} = 0,8430 \cdot \frac{1016 \cdot 10^3 \cdot 355}{1,1} \cdot 10^{-6} = 276,4 \text{ kNm} > 80,0 \text{ kNm}$ Part E - F: buckling length $\ell_{LT} = 2,25 \text{ m}$ $k = k_w = 1,0$; $\Psi = 80/100 = 0,8 \rightarrow C_1 = 1,113$ $M_{\alpha} = C_1 \frac{\pi^2 E I_z}{\ell_{LT}^2} \sqrt{\left(\frac{k}{k_w}\right)^2 \frac{I_w}{I_z} + 0,039 \ell_{LT}^2 \frac{I_t}{I_z}} = 1,113 \frac{\pi^2 \cdot 210000 \cdot 1067 \cdot 10^4}{2,25^2 \cdot 10^{12}} \sqrt{\frac{986000 \cdot 10^8 + 0,039 \cdot 2,25^2 \cdot 10^8 \cdot 8,1 \cdot 10^4}{1067 \cdot 10^4}} = 1489,9 \text{ kNm}$ $\bar{\lambda}_{LT} = \sqrt{\frac{W_{eff} f_y}{M_{\alpha}}} = \sqrt{\frac{1016 \cdot 10^3 \cdot 355}{1489,9 \cdot 10^6}} = 0,49 > 0,4$ $\rightarrow$ using buckling curve c $\rightarrow \chi_{LT,c} = 0,8486$ $M_{b,Rd} = \chi_{LT,c} \frac{W_{eff} f_y}{\gamma_{M1}} = 0,8486 \cdot \frac{1016 \cdot 10^3 \cdot 355}{1,1} \cdot 10^{-6} = 278,2 \text{ kNm} > 100,0 \text{ kNm}$		
3.3.3 Interaction Bending - Axial compression without lateral torsional buckling Part A-B: $n_y = N_{sd}/N_{y,b,Rd} = 120,0/1000,0 = 0,12$ $\Psi = -100/250 = -0,4$ $\rightarrow \beta_{M,\psi} \approx 1,8 + 0,7 \cdot 0,4 = 2,08$ $\beta_{M,y} = \beta_{M,\psi} + M_0/\Delta M \cdot (\beta_{M,0} - \beta_{M,\psi}) = 2,08 + 160/350 \cdot (1,3 - 2,08) = 1,72$ $\mu_y = \lambda_y (2\beta_{M,y} - 4) = 0,58 (2 \cdot 1,72 - 4) = -0,325 < 0,9$ $1 - \mu_y n_y / \gamma_{M1} = 1 + 0,325 \cdot 0,12 / 1,1 = 1,04 < 1,5 \rightarrow k_y = 1,04$ $n_y + \frac{k_y M_{y,Ed}}{W_{eff,y} \frac{f_y}{1,1}} = 0,12 + \frac{1,04 \cdot 250,0 \cdot 10^6}{1016 \cdot 10^3 \cdot \frac{355}{1,1}} = 0,91 < 1$	5.4.4; Table 5.26 [5.5.4]	$M_0 = 160 \text{ kNm}$; $\Delta M = 350 \text{ kNm}$ 

Problem, Sketch, Calculation	Reference	Commentary
3.3.4 Interaction Bending - Axial compression with lateral torsional buckling Part C - E: $n_z = N_{sd}/N_{z,b,Rd} = 120,0/575,1 = 0,21$ $\Psi = 67/80 = 0,84 \longrightarrow \beta_{M,LT} = 1,8 - 0,7 \cdot 0,84 = 1,212$ $\mu_{LT} = 0,15\lambda_{1,z}\beta_{M,LT} - 0,15 = 0,15 \cdot 1,08 \cdot 1,212 - 0,15 = 0,05 < 0,9$ $1 - \mu_{LT}n_z/\gamma_{M1} = 1 - 0,05 \cdot 0,21/1,1 = 0,99 < 1,0 \longrightarrow k_{LT} = 0,99$ $n_z + \frac{k_{LT} M_{y,Ed}}{M_{1,b,Rd}} = 0,21 + \frac{0,99 \cdot 80}{276,4} = 0,50 < 1$ Part E - F: $n_z = N_{sd}/N_{z,b,Rd} = 120,0/967,4 = 0,12$ $\Psi = 80/100 = 0,8 \longrightarrow \beta_{M,LT} = 1,8 - 0,7 \cdot 0,8 = 1,24$ $\mu_{LT} = 0,15\lambda_{1,z}\beta_{M,LT} - 0,15 = 0,15 \cdot 0,54 \cdot 1,24 - 0,15 = -0,05 < 0,9$ $1 - \mu_{LT}n_z/\gamma_{M1} = 1 + 0,05 \cdot 0,12/1,1 = 1,01 > 1,0 \longrightarrow k_{LT} = 1,0$ $n_z + \frac{k_{LT} M_{y,Ed}}{M_{1,b,Rd}} = 0,13 + \frac{1,0 \cdot 100}{276,4} = 0,49 < 1$	5.4.4; Table 5.26 [5.5.4]	Lateral torsional buckling is considered between D - E whereas flexural buckling is considered between C - E.

2.4 Local stresses

These examples demonstrate the verification of load introduction problems assuming design values of action effects (N_{sd} , V_{sd} , M_{sd} , etc) which have been calculated by an analysis of the structure and these values already include ψ , γ_F etc.

The results presented in the examples are rounded values. For the purpose of easy re-calculation each formula and each check is calculated with the rounded values.

The following examples are included in this chapter:

- | | |
|-----------------------|--|
| Example 2.4.1: | Design of transverse stiffeners (continuation of example 3 of chapter "Members in Bending") |
| Example 2.4.2: | Design of intermediate transverse stiffeners (continuation of example 4 of chapter "Members in Bending") |
| Example 2.4.3: | Axially loaded column supported by a beam |
| Example 2.4.4: | Load introduction of wheel loads from cranes |
| Example 2.4.5: | Beam supported by a beam |

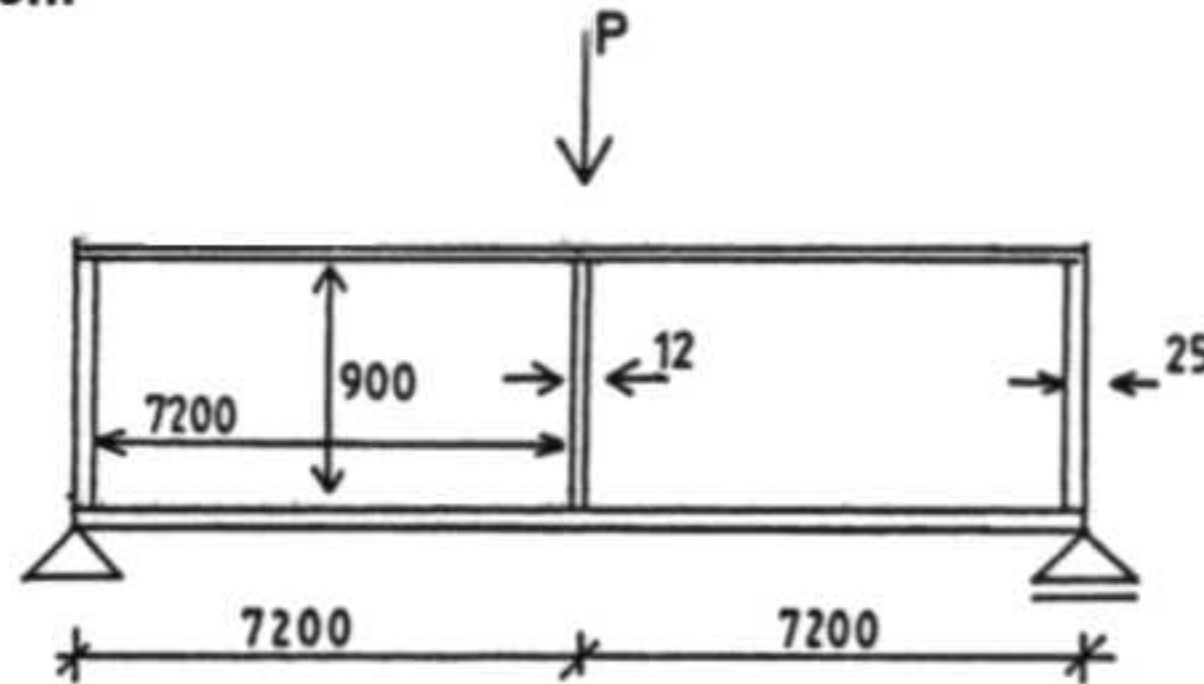
Problem, Sketch, Calculation

Reference

Commentary

Example 2.4.1: Design of transverse stiffeners (continuation of example 2.2.3 of chapter "Members in Bending")

1. System



Beam: Welded section with stiffeners at the supports and at mid-span
 $\phi 600 \times 50$
 $\phi 1000 \times 12$
 $\phi 600 \times 50$
 Dimensions see sketch
 Steel grade: Fe 510
 System length: $L = 14,40$ m

2. Loading

At mid-span:
 Internal forces and moments at mid span:
 $P_{sd} = 2377,5$ kN
 $M_{sd} = 8664,0$ kNm
 $V_{sd} = 1188,8$ kN

3. Determination of the resistance of the transverse stiffener at mid span

3.1 Verification of the stiffness effect of the stiffener

$$a/d = 8,0 > \sqrt{2}$$

$$\rightarrow I_s = 600^3 \cdot 12 / 12 = 21600 \cdot 10^4 \text{ mm}^4 > 0,75 d t_w^3 = 0,75 \cdot 880,2 \cdot 12^3 = 114,1 \cdot 10^4 \text{ mm}^4$$

3.2 Buckling resistance verification (perpendicular to the web-plane)

3.2.1 Effective cross-section of the stiffener

$$\text{Fe 510} \rightarrow \epsilon = 0,81$$

$$b/t = 294/12 = 24,5 > 15\epsilon = 12,2 \rightarrow \text{calculation of effective cross-section of stiffener}$$

$$\bar{\lambda}_p = \frac{b/t}{28,4 \epsilon \sqrt{k_\sigma}} = \frac{24,5}{28,4 \cdot 0,81 \cdot \sqrt{0,43}} = 1,624 > 0,673 \quad \rho = \frac{\bar{\lambda}_p - 0,22}{\bar{\lambda}_p^2} = \frac{1,624 - 0,22}{1,624^2} = 0,532$$

$$15\epsilon t_w = 15 \cdot 0,81 \cdot 12 = 146 \text{ mm}$$

$$A_{eff} = 0,532 \cdot 2 \cdot 294 \cdot 12 + (2 \cdot 146 + 12) \cdot 12 = 37,5 \cdot 10^2 + 36,5 \cdot 10^2 = 74 \cdot 10^2 \text{ mm}^2$$

$$A = 146 \cdot 12 \cdot 2 + 12 \cdot 600 = 107,0 \cdot 10^2 \text{ mm}^2$$

$$I_s = 600^3 \cdot 12 / 12 = 21600 \cdot 10^4 \text{ mm}^4$$

$$i_s = \sqrt{I_s/A} = \sqrt{21600 \cdot 10^4 / 107 \cdot 10^2} = 142 \text{ mm}$$

3.2.2 Buckling length

$$\text{Assumption: } \ell = 0,9 \text{ m} > 0,675 \text{ m} = 0,75d$$

5.4.5.4.(6);
[5.6.5.(3)]

5.4.5.4.(1);
[5.7.6.(1)]

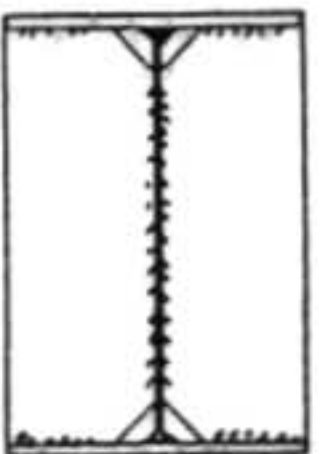
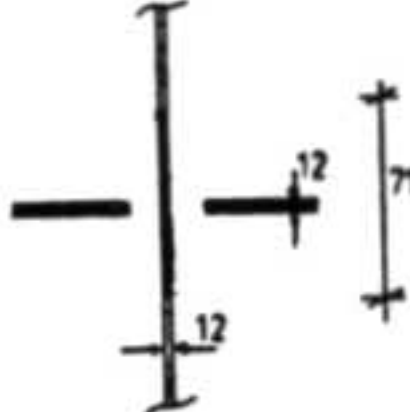
5.3.5(3)

5.4.5.4.(2);
[5.7.6.(2)]

The beam is continuously laterally restrained.
 Throat thickness between flange and web:
 $a = 7$ mm

In determining the effective cross-section the cut off for the welds of the stiffeners need not to be considered.

0,75d as a buckling length could be taken if appropriate restraint is provided

Problem, Sketch, Calculation	Reference	Commentary
3.2.4 Buckling verification $\bar{\lambda} = \frac{l}{i_z} \frac{1}{\lambda_1} \sqrt{\frac{A_{eff}}{A}} = \frac{0,9 \cdot 10^3}{142} \frac{1}{76,4} \sqrt{\frac{74 \cdot 10^2}{107 \cdot 10^2}} = 0,07 < 0,2$ → no reduction due to buckling is required $\frac{N_{sd}}{A_{eff} \frac{f_y}{\gamma_{M1}}} = \frac{2377,5 \cdot 10^3}{74 \cdot 10^2 \cdot \frac{355}{1,1}} = 0,996 < 1$	5.4.2; [5.5.1]	When $\bar{\lambda} < 0,2$ no buckling mode is govern. Only cross-section verification is necessary. γ_{M1} is used because of class 4 cross-section.
3.3 Crushing resistance verification of the stiffened web and the stiffener $\sigma_{t,Ed} = M \cdot z / I = 8664,0 \cdot 100 \cdot 500 / 1427900 = 303 \text{ N/mm}^2$ $b_t = 600 \text{ mm} < 25t_t = 25 \cdot 50 = 1250 \text{ mm}$   $s_y = 2 t_t \sqrt{\frac{b_t}{t_w}} \sqrt{\frac{f_{yt}}{f_{yw}}} \sqrt{1 - \left(\frac{\gamma_{M0} \sigma_{t,Ed}}{f_{yt}} \right)^2} = 2 \cdot 50 \sqrt{\frac{600}{12}} \sqrt{\frac{1,1 \cdot 303}{335}} = 71 \text{ mm}$ $< 2 \cdot 15 \varepsilon t_w + t_s = 2 \cdot 146 + 12 = 304 \text{ mm}$ $f_{yw} = f_{y, \text{stiffener}} \rightarrow A = s_y t_w + t_s (b_t - t_w - 2c) = 71 \cdot 12 + 12 \cdot (600 - 12 - 2 \cdot 20) = 74,3 \cdot 10^2 \text{ mm}^2$ $R_{y,Rd} = A \frac{f_y}{\gamma_{M1}} = 74,3 \cdot 10^2 \cdot \frac{355}{1,1} \cdot 10^{-3} = 2397,9 \text{ kN} > 2377,5 \text{ kN}$	5.6.3 [5.7.6.(6); 5.7.3]	Allowance needs to be made for the openings cut in the stiffener to clear the web-flange welds. This check is only necessary for a load bearing stiffener.

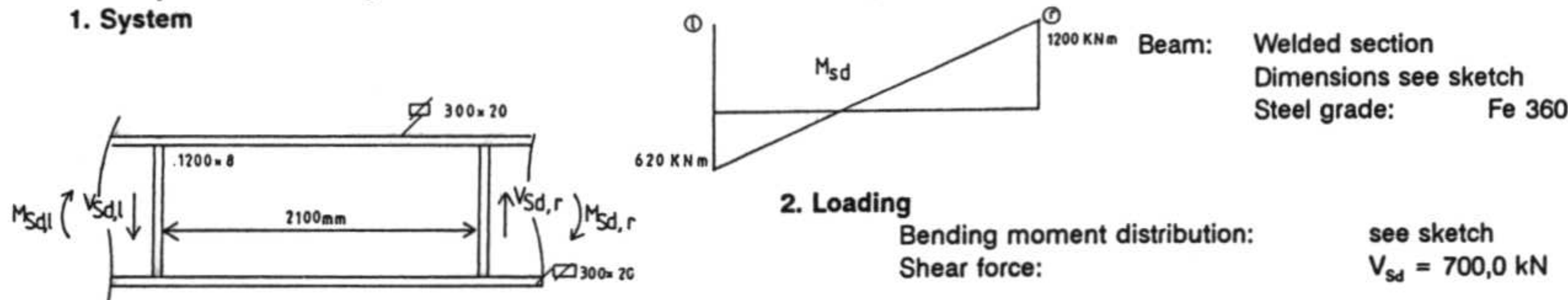
Problem, Sketch, Calculation

Reference

Commentary

Example 2.4.2: Design of intermediate transverse stiffeners (continuation of example 2.2.4 of chapter "Members in Bending")

1. System



2. Loading

Bending moment distribution: see sketch
Shear force: $V_{sd} = 700,0 \text{ kN}$

3. Determination of the resistance of the intermediate transverse stiffeners

3.1 Verification of the stiffness effect of the stiffener

$$a/d = 1,75 > \sqrt{2}$$

$$\rightarrow I_s = 300^3 \cdot 6/12 = 1350 \cdot 10^4 \text{ mm}^4 > 0,75 d t_w^3 = 0,75 \cdot 1482,5 \cdot 8^3 = 56,9 \cdot 10^4 \text{ mm}^4$$

3.2 Buckling resistance verification (perpendicular to the web-plane)

3.2.1 Determination of the compression force in the stiffener

$$V_{sd} = 700,0 \text{ kN}$$

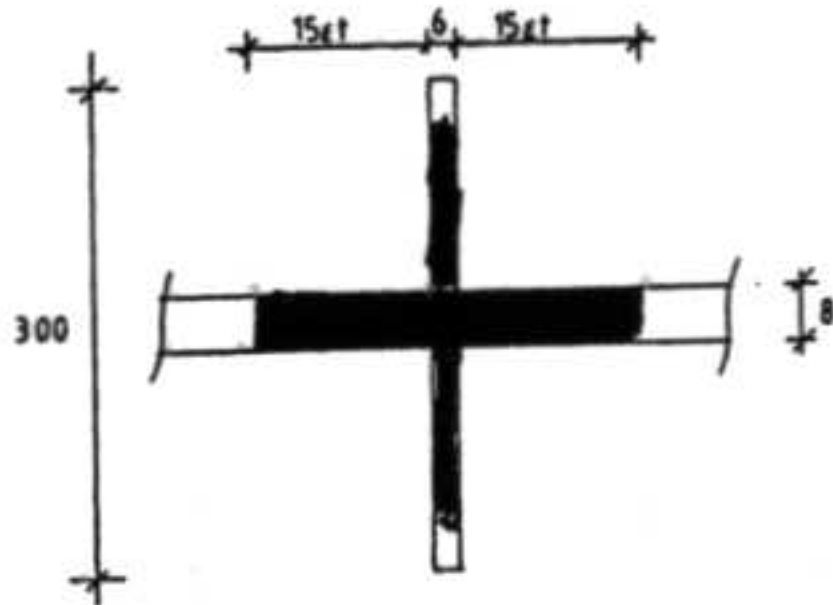
$$\lambda_w = 1,556 \rightarrow \tau_{bb} = (1/\lambda_w^2) f_{yw}/\sqrt{3} = 1/1,556^2 \cdot 235/\sqrt{3} = 56,0 \text{ N/mm}^2$$

$$N_{sd} = V_{sd} \cdot d t_w \tau_{bb} / \gamma_{M1} = 700,0 \cdot 1200 \cdot 8 \cdot 56,0 \cdot 10^{-3} / 1,1 = 211,3 \text{ kN}$$

3.2.2 Effective cross-section of the stiffener

$$\text{Fe 360} \rightarrow \epsilon = 1,0 ; k_s = 0,43$$

$$b/t = 146/6 = 24,3 > 15\epsilon = 15 \rightarrow \text{calculation of effective cross-section of stiffener}$$



$$\bar{\lambda}_p = \frac{b/t}{28,4 \epsilon \sqrt{k_s}} = \frac{24,3}{28,4 \cdot 1,0 \cdot \sqrt{0,43}} = 1,305 > 0,673$$

$$\rho = \frac{\bar{\lambda}_p - 0,22}{\bar{\lambda}_p^2} = \frac{1,305 - 0,22}{1,305^2} = 0,637$$

$$15\epsilon t_w = 15 \cdot 1,0 \cdot 8 = 120 \text{ mm}$$

$$A_{eff} = 0,637 \cdot 2 \cdot 146 \cdot 6 + (2 \cdot 120 + 6) \cdot 8 = 11,2 \cdot 10^2 + 19,7 \cdot 10^2 = 30,9 \cdot 10^2 \text{ mm}^2$$

$$A = 120 \cdot 8 \cdot 2 + 6 \cdot 300 = 37,2 \cdot 10^2 \text{ mm}^2$$

$$I_s = 300^3 \cdot 6/12 = 1350 \cdot 10^4 \text{ mm}^4$$

$$i_z = \sqrt{I_s/A} = \sqrt{1350 \cdot 10^4 / 37,2 \cdot 10^2} = 60,2 \text{ mm}$$

3.2.3 Buckling length

$$\text{Assumption: } \ell = 1,20 \text{ m} > 0,9 \text{ m} = 0,75d$$

3.2.4 Buckling verification

$$\bar{\lambda} = \frac{\ell}{i_z} \frac{1}{\lambda_1} \sqrt{\frac{A_{eff}}{A}} = \frac{1,20 \cdot 10^3}{60,2} \frac{1}{93,9} \sqrt{\frac{30,9}{37,2}} = 0,19$$

$$N_{b,Rd} = A_{eff} \frac{f_y}{\gamma_{M1}} = 30,9 \cdot 10^2 \cdot \frac{235}{1,1} \cdot 10^{-3} = 660,1 \text{ kN} > 211,3 \text{ kN}$$

5.4.5.4.(6);
[5.6.5.(3)]

5.4.5.4.(1);
[5.6.5.(1)]

5.4.5.4.(1);
[5.7.6.(1)]

5.4.5.4.(2);
[5.7.6.(2)]

5.4.2; [5.5.1]

The beam is continuously laterally restrained.
Throat thickness between flange and web:
 $a = 7 \text{ mm}$

No change of sign in the shear force line

In determining the effective cross-section the cut off, for the welds, of the stiffeners need not to be considered.

0,75d as a buckling length could be taken if appropriate restraint is provided

When $\bar{\lambda} < 0,2$ no buckling mode is govern. Only cross-section verification is necessary. γ_{M1} is used because of class 4 cross-section.
For intermediate transverse stiffeners it is only necessary to check the buckling resistance provided that they are not subject to external loads

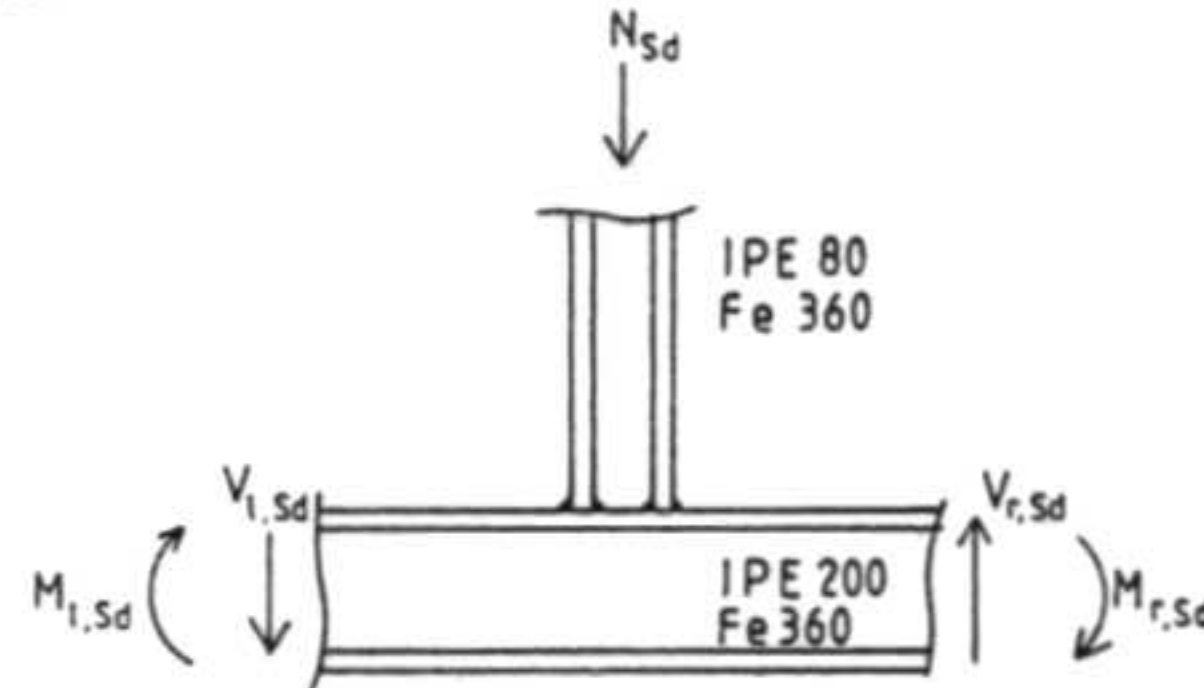
Problem, Sketch, Calculation

Reference

Commentary

Example 2.4.3: Axial loaded column supported by a beam

1. System



Beam:	IPE 200	Steel grade:	Fe 360
Bracing:	IPE 80	Steel grade:	Fe 360

2. Loading

Compression force:	$N_{Sd} = 100,0 \text{ kN}$
Bending moment:	$M_{r,Sd} = 40,0 \text{ kNm}$ $M_{l,Sd} = 40,0 \text{ kNm}$
Shear force:	$V_{r,Sd} = 50,0 \text{ kN}$ $V_{l,Sd} = 50,0 \text{ kN}$

3. Determination of the resistances

3.1 Determination of the reduced effective breadth resistance

$$b_{eff} = \min \left\{ \begin{array}{l} t_w + 2r + 7t_f \\ t_w + 2r + 7 \frac{t_f^2}{t_p} \frac{f_y}{f_{yp}} \end{array} \right.$$

$$= \min \left\{ \begin{array}{l} 5,6 + 2 \cdot 12 + 7 \cdot 8,5 = 89,1 \text{ mm} \\ 5,6 + 2 \cdot 12 + 7 \cdot \frac{8,5^2}{5,2} = 126,9 \text{ mm} \end{array} \right.$$

$$\rightarrow b_{eff} = 89,1 \text{ mm} > b = 80 \text{ mm}$$

→ the flanges of the bracing are fully effective and no stiffening of the joint is required

3.2 Crushing resistance

3.2.1 Determination of the length of stiff bearing s_z

$$s_z = h_{bracing} = 80 \text{ mm}$$

3.2.2 Determination of s_y

$$\sigma_{l,Ed} = M_{Sd}/W_{el} = 40 \cdot 10^6 / 194 \cdot 10^3 = 206 \text{ N/mm}^2$$

$$b_f = 100 \text{ mm} < 212,5 \text{ mm} = 25t_f$$

$$s_y = 2t_f \sqrt{\frac{b_f}{t_w} \frac{f_{yf}}{f_{yw}} \left[1 - \left(\frac{\sigma_{l,Ed}}{f_{yf}} \gamma_{M0} \right)^2 \right]} = 2 \cdot 8,5 \sqrt{\frac{100}{5,6} \left[1 - \left(\frac{206 \cdot 1,1}{235} \right)^2 \right]} = 19,0 \text{ mm}$$

3.2.3 Verification

$$R_{y,Rd} = (s_z + s_y) t_w \frac{f_{yw}}{\gamma_{M1}} = (80 + 19) \cdot 5,6 \cdot \frac{235}{1,1} \cdot 10^{-3} = 118,4 \text{ kN} > 100 \text{ kN}$$

6.3.6; [6.6.8]

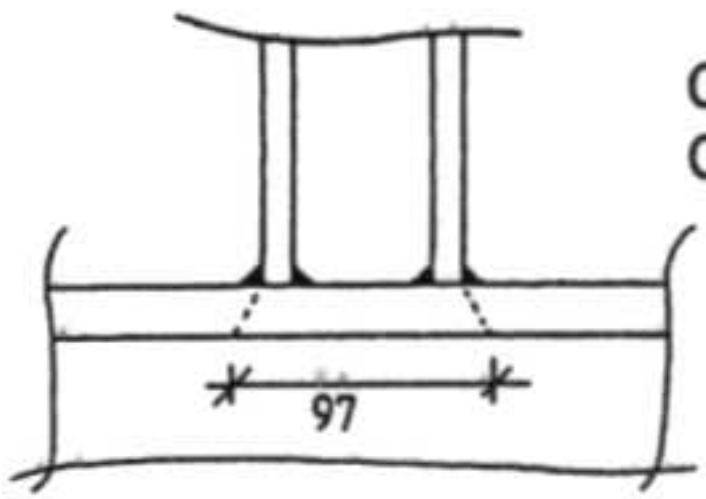
5.6.2; [5.7.2]

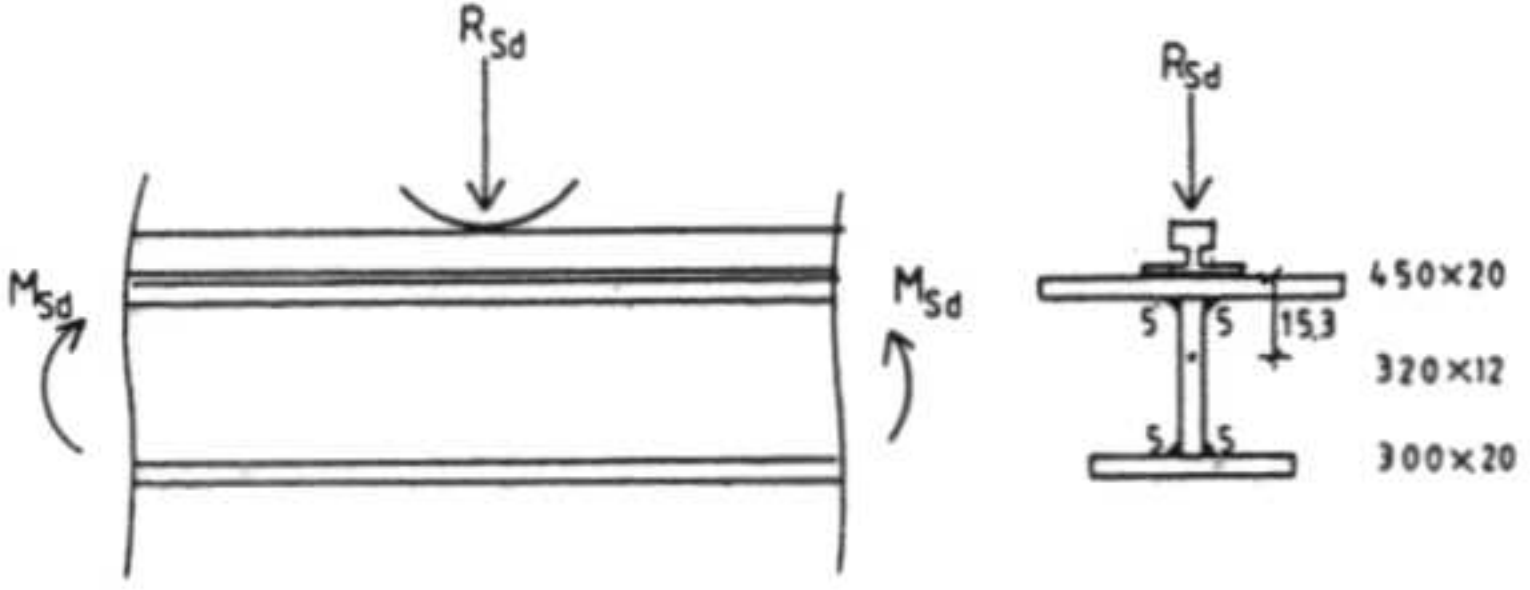
5.6.3; [5.7.3]

5.6.3; [5.7.3]

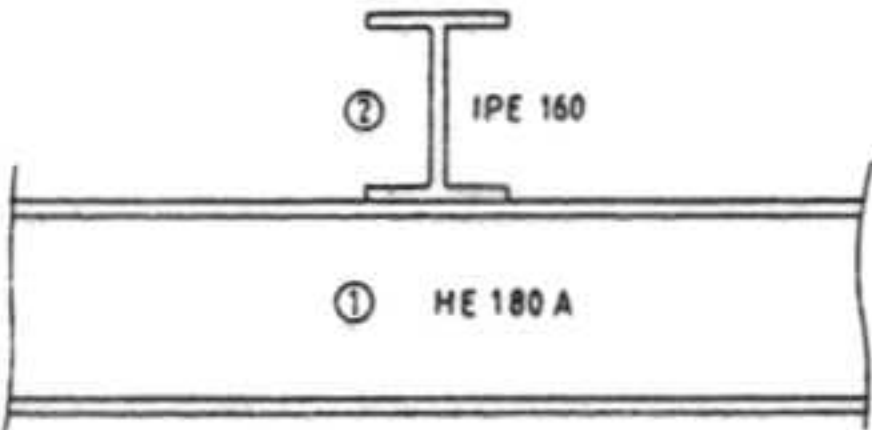
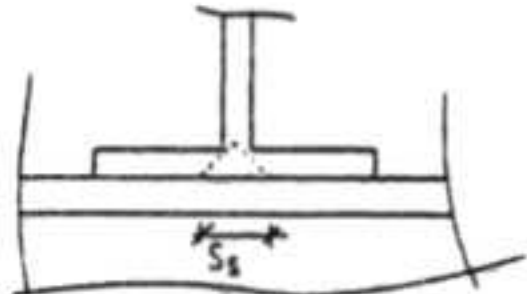
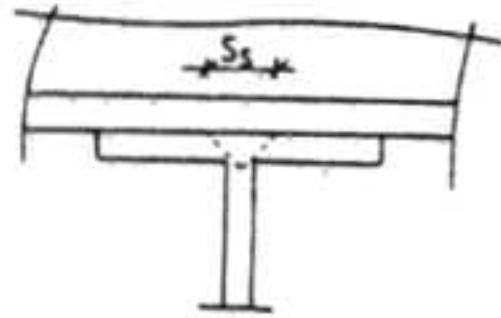
Throat thickness of the weld connecting the chord member and the bracing member: $a = 3 \text{ mm}$

The length of stiff bearing on the flange is the distance over which the applied force is effectively distributed

Problem, Sketch, Calculation	Reference	Commentary
3.3 Crippling resistance $s_w/d = 80/159 = 0,5 > 0,2 \rightarrow s_w/d = 0,2$ $R_{a,Rd} = 0,5 t_w^2 \sqrt{E f_{yw}} \left[\sqrt{\frac{t_f}{t_w}} + 3 \frac{t_w}{t_f} \frac{s_s}{d} \right] / \gamma_{M1} = 0,5 \cdot 5,6^2 \sqrt{210000 \cdot 235} \left[\sqrt{\frac{8,5}{5,6}} + 3 \frac{5,6}{8,5} \cdot 0,2 \right] \cdot 10^{-3} / 1,1 = 163,0 \text{ kN} > 100 \text{ kN}$ $M_{c,Rd} = W_{pl} \cdot f_y / \gamma_{M0} = 221 \cdot 10^3 \cdot 235 \cdot 10^{-6} / 1,1 = 47,2 \text{ kNm} > 40 \text{ kNm}$	5.6.4.(1); [5.7.4.(1)]	
3.3.1 Interaction $\frac{F_{Sd}}{R_{a,Rd}} + \frac{M_{Sd}}{M_{c,Rd}} = \frac{100,0}{163,0} + \frac{40,0}{47,2} = 1,46 < 1,5$	5.6.4.(2); [5.7.4.(2)]	Where the member is subject to bending moments, the interaction criterion has to be fulfilled
3.4 Buckling resistance 3.4.1 Determination of the effective breadth of the web $b_{eff} = \sqrt{(h^2 + s_w^2)} = \sqrt{(200^2 + 80^2)} = 215,4 \text{ mm}$	5.6.5; [5.7.5]	Where forces are applied through one flange and resisted by shear forces in the web, the buckling resistance of the web to transverse forces need not be considered.
3.4.2 Buckling resistance verification Assumption: buckling length $\ell = 0,159 \text{ m}$ $A = 215,4 \cdot 5,6 = 12,1 \cdot 10^2 \text{ mm}^2$; $I = 215,4 \cdot 5,6^3 / 12 = 0,315 \cdot 10^4 \text{ mm}^4$ $i = \sqrt{0,315 \cdot 10^4 / 12,1 \cdot 10^2} = 1,6 \text{ mm}$ $\bar{\lambda} = \frac{\ell}{i} \frac{1}{\lambda_1} = \frac{0,159 \cdot 10^3}{1,6} \frac{1}{93,9} = 1,06$ using buckling curve c $\rightarrow \chi_c = 0,5059$ $R_{b,Rd} = \chi_c A \frac{f_{yw}}{\gamma_{M1}} = 0,5059 \cdot 12,1 \cdot 10^2 \cdot \frac{235}{1,1} \cdot 10^{-3} = 130,8 \text{ kN} > 100 \text{ kN}$	5.6.5; [5.7.5]	When the detail is such that there is a doubt as to whether buckling mode governs, it should be considered.
3.5 Cross-section resistance verification $V_{pl,Rd} = A_v f_v / \sqrt{3} / \gamma_{M0} = 11,2 \cdot 10^2 \cdot 235 \cdot 10^{-3} / \sqrt{3} / 1,1 = 138,1 \text{ kN} > 50 \text{ kN}$ $0,5 V_{pl,Rd} = 69,1 \text{ kN} > 50 \text{ kN} \rightarrow$ no interaction bending shear is required Compression: $\sigma_{x,Ed} = M_{Sd} / W_{el} = 40 \cdot 10^6 / 194 \cdot 10^3 \cdot 91,5 / 100 = 189 \text{ N/mm}^2 < 214 \text{ N/mm}^2 = 235 / 1,1 = f_y / \gamma_{M0}$ Compression: $\sigma_{z,Ed} = N_{Sd} / (s_s t_w) = 100 \cdot 10^3 / (97 \cdot 5,6) = 184 \text{ N/mm}^2 < 214 \text{ N/mm}^2$  $\left(\frac{\sigma_{x,Ed}}{f_{yd}} \right)^2 + \left(\frac{\sigma_{z,Ed}}{f_{yd}} \right)^2 - \frac{\sigma_{x,Ed}}{f_{yd}} \frac{\sigma_{z,Ed}}{f_{yd}} = \left(\frac{189}{214} \right)^2 + \left(\frac{184}{214} \right)^2 - \frac{189}{214} \frac{184}{214} = 0,78 + 0,74 - 0,88 \cdot 0,86 = 0,76 < 1$	5.4.2; [5.5.1]	
	-; [5.4.10]	$A_v = 11,2 \cdot 10^2 \text{ mm}^2$ Higher values may be taken by using the formula given in [5.4.6(2)] The effect of the transverse force on the bending moment resistance of the member should also be considered.

Problem, Sketch, Calculation	Reference	Commentary
Example 2.4.4: Load introduction of wheel loads from cranes 1. System  Beam: Welded section (see sketch) Steel grade: Fe 360 Crane rail: A 45 2. Loading Wheel load: $R_{Sd} = 200,0 \text{ kN}$ Bending moment: $M_{r,Sd} = 366,0 \text{ kNm}$ $M_{l,Sd} = 366,0 \text{ kNm}$ 3. Determination of the resistances 3.1 Crushing resistance 3.1.1 Determination of s_y $\sigma_{l,Ed} = M_{Sd}/W_{el} = 366 \cdot 10^6 / 153 / 45296 \cdot 10^4 = 124 \text{ N/mm}^2$ $I_R = 91 \cdot 10^4 \text{ mm}^4$ $I_l = 450 \cdot 20^3 / 12 = 30 \cdot 10^4 \text{ mm}^4$ $s_y = k_r \left(\frac{I_l + I_R}{t_w} \right)^{1/3} \sqrt{1 - \left(\frac{\sigma_{l,Ed} \gamma_{M0}}{f_{yf}} \right)^2} = 3,25 \left(\frac{30 \cdot 10^4 + 91 \cdot 10^4}{12} \right)^{1/3} \sqrt{1 - \left(\frac{124 \cdot 1,1}{235} \right)^2} = 123,2 \text{ mm}$ 3.1.2 Verification $R_{y,Rd} = s_y t_w \frac{f_{yw}}{\gamma_{M1}} = 123,2 \cdot 12 \cdot \frac{235}{1,1} \cdot 10^{-3} = 315,8 \text{ kN} > 200 \text{ kN}$ 3.2 Crippling resistance $s_s = 0$ $R_{a,Rd} = 0,5 t_w^2 \sqrt{E f_{yw}} \left[\sqrt{\frac{t_l}{t_w}} + 3 \frac{t_w}{t_l} \frac{s_s}{d} \right] / \gamma_{M1} = 0,5 \cdot 12^2 \sqrt{210000 \cdot 235} \left[\sqrt{\frac{20}{12}} \right] \cdot 10^{-3} / 1,1 = 593,6 \text{ kN} > 200 \text{ kN}$ $M_{c,Rd} = W_{el} \cdot f_y / \gamma_{M0} = 2188 \cdot 10^3 \cdot 235 \cdot 10^{-6} / 1,1 = 467,4 \text{ kNm} > 366,0 \text{ kNm}$ 3.2.1 Interaction $\frac{F_{Sd}}{R_{a,Rd}} + \frac{M_{Sd}}{M_{c,Rd}} = \frac{200,0}{593,6} + \frac{366,0}{467,4} = 1,12 < 1,5$	; 5.7.3.(4) 5.6.3; [5.7.3] 5.6.4.(1); [5.7.4.(1)] 5.6.4.(2); [5.7.4.(2)]	$I_x = 45296 \cdot 10^4 \text{ mm}^4$ Throat thickness of the weld connecting the flange and the web: $a = 5 \text{ mm}$ The crane rail is not welded on the flange. Where the details are such that there is doubt over which mode (crushing, crippling or buckling) governs, all three modes should be considered. k_r is a constant taken as 3,25 for crane rails mounted directly on the flange Where the member is subject to bending moments, the interaction criterion has to be fulfilled

A diagram of a beam with a central point load. A horizontal line represents the beam, with a vertical line passing through its center. A dashed line is drawn horizontally through the center of the beam, representing the neutral axis. A small vertical line segment is drawn at the top center of the beam, representing the point load.

Problem, Sketch, Calculation	Reference	Commentary
Example 2.4.5: Beam supported by a beam 1. System  2. Loading Reaction force: $R_{ed} = 60 \text{ kN}$ Bending moment: Beam "1": 55 kNm Beam "2": 16 kNm 3. Determination of the resistance 3.1 Crushing resistance 3.1.1 Beam "1" (HE 180 A) 3.1.1.1 Bearing stiff length $s_s = 5,0 + 2 \cdot 7,4 + 4 \cdot 9 \cdot (1 - \sqrt{2}/2) = 30,3 \text{ mm}$  3.1.1.2 Determination of s_y $\sigma_{f,Ed} = M_{Sd}/W_{el} = 55 \cdot 10^6 / 294 \cdot 10^3 = 187 \text{ N/mm}^2$ $b_f = 180 \text{ mm} < 237,5 \text{ mm} = 25t_f$ $s_y = 2t_f \sqrt{\frac{b_f}{t_w} \frac{f_{yf}}{f_{yw}} \left[1 - \left(\frac{\sigma_{f,Ed} \gamma_{Mo}}{f_{yf}} \right)^2 \right]} = 2 \cdot 9,5 \sqrt{\frac{180}{6} \left[1 - \left(\frac{187 \cdot 1,1}{235} \right)^2 \right]} = 50,3 \text{ mm}$ 3.1.1.3 Verification $R_{y,Rd} = (s_s + s_y) t_w \frac{f_{yw}}{\gamma_{M1}} = (30,3 + 50,3) \cdot 6 \cdot \frac{235}{1,1} \cdot 10^{-3} = 103,3 \text{ kN} > 60 \text{ kN}$ 3.1.2 Beam "2" (IPE 160) 3.1.2.1 Bearing stiff length $s_s = 6,0 + 2 \cdot 9,5 + 4 \cdot 15 \cdot (1 - \sqrt{2}/2) = 42,6 \text{ mm}$  3.1.2.2 Determination of s_y $\sigma_{f,Ed} = M_{Sd}/W_{el} = 16 \cdot 10^6 / 109 \cdot 10^3 = 147 \text{ N/mm}^2$ $b_f = 82 \text{ mm} < 185 \text{ mm} = 25t_f$ $s_y = 2t_f \sqrt{\frac{b_f}{t_w} \frac{f_{yf}}{f_{yw}} \left[1 - \left(\frac{\sigma_{f,Ed} \gamma_{Mo}}{f_{yf}} \right)^2 \right]} = 2 \cdot 7,4 \sqrt{\frac{82}{5} \left[1 - \left(\frac{147 \cdot 1,1}{235} \right)^2 \right]} = 43,5 \text{ mm}$ 3.1.2.3 Verification $R_{y,Rd} = (s_s + s_y) t_w \frac{f_{yw}}{\gamma_{M1}} = (42,6 + 43,5) \cdot 5 \cdot \frac{235}{1,1} \cdot 10^{-3} = 92,0 \text{ kN} > 60 \text{ kN}$	5.6.2; [5.7.2.(3)] 5.6.3; [5.7.3.(1)] 5.6.3; [5.7.3.(1)] 5.6.2; [5.7.2.(3)] 5.6.3; [5.7.3.(1)] 5.6.3; [5.7.3.(1)]	For rolled sections: $s_s = t_w + 2t_f + 4r(1 - \sqrt{2}/2)$ For rolled sections: $s_s = t_w + 2t_f + 4r(1 - \sqrt{2}/2)$

Problem, Sketch, Calculation	Reference	Commentary
3.2 Crippling resistance 3.2.1 Beam "1" (HE 180 A) $s_w/d = 30,3/122 = 0,25 > 0,2 \longrightarrow s_w/d = 0,2$ $R_{b,Rd} = 0,5 t_w^2 \sqrt{E f_{yw}} \left[\sqrt{\frac{t_f}{t_w}} + 3 \frac{t_w}{t_f} \frac{s_w}{d} \right] / \gamma_{M1} = 0,5 \cdot 6^2 \sqrt{210000 \cdot 235} \left[\sqrt{\frac{9,5}{6}} + 3 \frac{6}{9,5} 0,2 \right] \cdot 10^{-3} / 1,1 = 188,2 \text{ kN} > 60 \text{ kN}$ $M_{c,Rd} = W_{pl} \cdot f_y / \gamma_{M0} = 325 \cdot 10^3 \cdot 235 \cdot 10^{-6} / 1,1 = 69,4 \text{ kNm} > 55 \text{ kNm}$	5.6.4.(1) [5.7.4.(1)]	
3.2.1.1 Interaction $\frac{F_{Sd}}{R_{b,Rd}} + \frac{M_{Sd}}{M_{c,Rd}} = \frac{60,0}{188,2} + \frac{55,0}{69,4} = 1,11 < 1,5$	5.6.4.(2) [5.7.4.(2)]	
3.2.2 Beam "2" (IPE 160) $s_w/d = 42,6/127 = 0,34 > 0,2 \longrightarrow s_w/d = 0,2$ $R_{b,Rd} = 0,5 t_w^2 \sqrt{E f_{yw}} \left[\sqrt{\frac{t_f}{t_w}} + 3 \frac{t_w}{t_f} \frac{s_w}{d} \right] / \gamma_{M1} = 0,5 \cdot 5^2 \sqrt{210000 \cdot 235} \left[\sqrt{\frac{7,4}{5}} + 3 \frac{5}{7,4} 0,2 \right] \cdot 10^{-3} / 1,1 = 129,5 \text{ kN} > 60 \text{ kN}$ $M_{c,Rd} = W_{pl} \cdot f_y / \gamma_{M0} = 124 \cdot 10^3 \cdot 235 \cdot 10^{-6} / 1,1 = 26,5 \text{ kNm} > 16 \text{ kNm}$	5.6.4.(1) [5.7.4.(1)]	
3.2.2.1 Interaction $\frac{F_{Sd}}{R_{b,Rd}} + \frac{M_{Sd}}{M_{c,Rd}} = \frac{60,0}{129,5} + \frac{16,0}{26,5} = 1,07 < 1,5$	5.6.4.(2) [5.7.4.(2)]	Here: Local buckling does not need to be checked
3.3 Cross-section verification 3.3.1 Beam "1" (HE 180 A) Assumption: no interaction bending shear is required Compression: $\sigma_x = M_{Sd}/W_{el} = 55 \cdot 10^6 / 294 \cdot 10^3 \cdot 76/85,5 = 166 \text{ N/mm}^2 < 214 \text{ N/mm}^2 = 235/1,1 = f_y/\gamma_{M0}$ Compression: $\sigma_z = N_{Sd}/(s_w t_w) = 60 \cdot 10^3 / ((30,3 + 2 \cdot 9,5) \cdot 6) = 202,8 \text{ N/mm}^2 < 214 \text{ N/mm}^2$ $\left(\frac{\sigma_{x,Ed}}{f_{yd}} \right)^2 + \left(\frac{\sigma_{z,Ed}}{f_{yd}} \right)^2 - \frac{\sigma_{x,Ed}}{f_{yd}} \frac{\sigma_{z,Ed}}{f_{yd}} = \left(\frac{166}{214} \right)^2 + \left(\frac{203}{214} \right)^2 - \frac{166}{214} \frac{203}{214} = 0,60 + 0,90 - 0,78 \cdot 0,95 = 0,76 < 1$	;- [5.4.10]	
3.3.2 Beam "1" (IPE 160) Assumption: no interaction bending shear is required Compression: $\sigma_x = M_{Sd}/W_{el} = 16 \cdot 10^6 / 109 \cdot 10^3 \cdot 72,6/80 = 133 \text{ N/mm}^2 < 214 \text{ N/mm}^2 = 235/1,1 = f_y/\gamma_{M0}$ Compression: $\sigma_z = N_{Sd}/(s_w t_w) = 60 \cdot 10^3 / ((42,6 + 2 \cdot 7,4) \cdot 5) = 209 \text{ N/mm}^2 < 214 \text{ N/mm}^2$ $\left(\frac{\sigma_{x,Ed}}{f_{yd}} \right)^2 + \left(\frac{\sigma_{z,Ed}}{f_{yd}} \right)^2 - \frac{\sigma_{x,Ed}}{f_{yd}} \frac{\sigma_{z,Ed}}{f_{yd}} = \left(\frac{133}{214} \right)^2 + \left(\frac{209}{214} \right)^2 - \frac{133}{214} \frac{209}{214} = 0,39 + 0,95 - 0,62 \cdot 0,98 = 0,73 < 1$	;- [5.4.10]	

Part 3

Bolted connections
Welded connections
Pin connections

3.1 Bolted connections

These examples demonstrate the verification of bolted connections assuming design values of action effects (N_{sd} , V_{sd} , M_{sd} , etc) which have been calculated by an analysis of the structure and these values already include ψ , γ_F etc.

The results presented in the examples are rounded values. For the purpose of easy re-calculation each formula and each check is calculated with the rounded values.

The following examples are included in this chapter:

Example 3.1.1:	Bolted connection of a tension member to a gusset plate
Example 3.1.2:	Erection splice at mid span of a lattice girder
Example 3.1.3:	Angle connected to a gusset plate
Example 3.1.4:	Fin plate connection to H section column
Example 3.1.5:	Fin plate connection to RHS column
Example 3.1.6:	Flexible end plate connection
Example 3.1.7:	Beam to beam connection with cleats
Example 3.1.8:	Splice of an unsymmetrical I-section
Example 3.1.9:	Bolted end plate connection

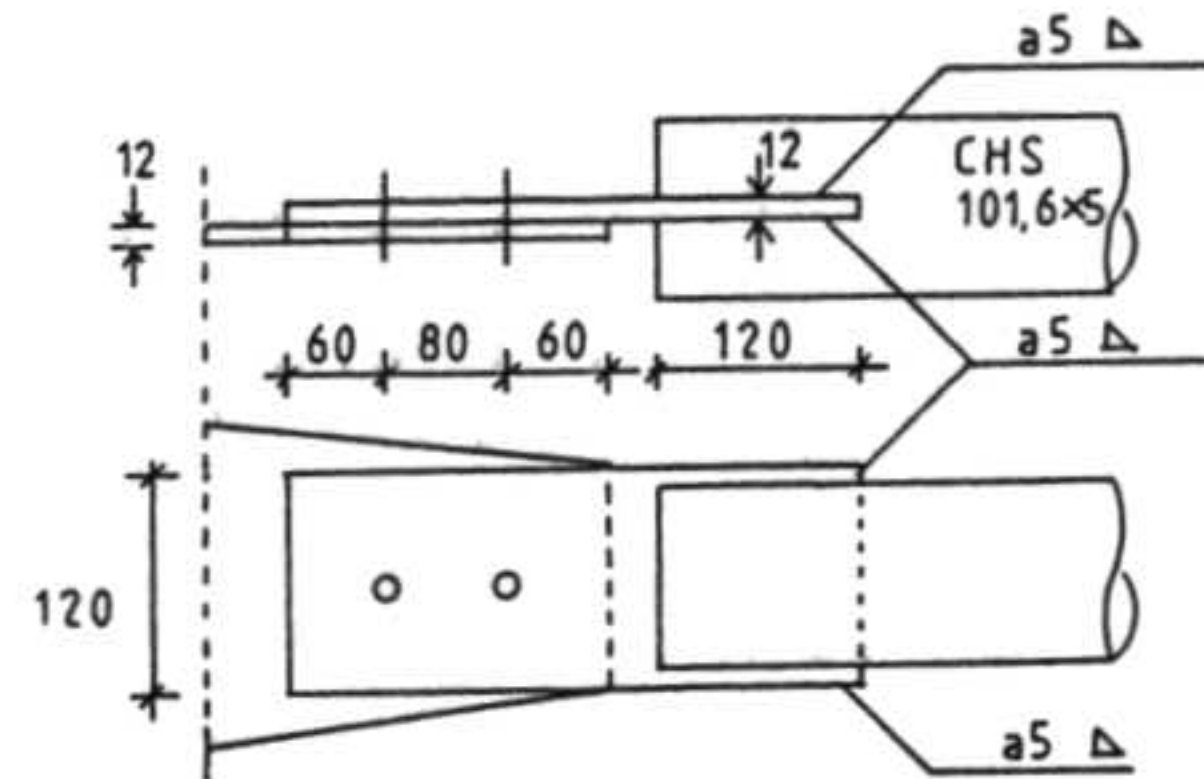
Problem, Sketch, Calculation

Reference

Commentary

Example 3.1.1: Bolted connection of a tension bar on a gusset plate

1. System



Member:	CHS 101,6 x 5,0	Steel grade:	Fe 360
Plate:	120 x 12	Steel grade:	Fe 360
Gusset plate:	120 x 12	Steel grade:	Fe 360
Bolts:	M 24, 8.8		
	$d_o = 26 \text{ mm}$		
weld length:	$\ell = 120 \text{ mm}$		
throat thickness:	$a = 5 \text{ mm}$		

2. Loading

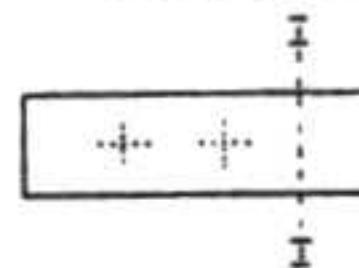
Tension force: $N_{sd} = 260 \text{ kN}$

3. Determination of the resistance of the members

3.1 Gross section (Circular hollow section)

$$N_{pl,Rd} = \frac{A \cdot f_y}{\gamma_{Mo}} = \frac{15,17 \cdot 10^2 \cdot 235}{1,1} \cdot 10^{-3} = 324,1 \text{ kN} > 260 \text{ kN}$$

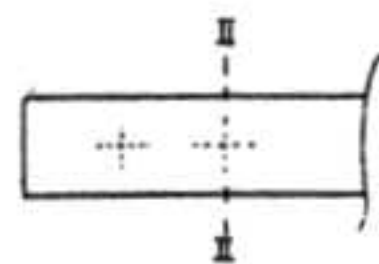
3.2 Gross section (Plate and gusset plate)



$$N_{pl,Rd} = \frac{A \cdot f_y}{\gamma_{Mo}} = \frac{120 \cdot 12 \cdot 235}{1,1} \cdot 10^{-3} = 307,6 \text{ kN} > 260 \text{ kN}$$

3.3 Net section (Plate and gusset plate)

$$A_{net} = (120 - 26) \cdot 12 = 11,3 \cdot 10^2 \text{ mm}^2$$



$$N_{u,Rd} = 0,9 \frac{A_{net} \cdot f_u}{\gamma_{M2}} = 0,9 \frac{11,3 \cdot 10^2 \cdot 360}{1,25} \cdot 10^{-3} = 292,9 \text{ kN} > 260 \text{ kN}$$

4. Determination of the connection resistance

4.1 Bolted connection

Shear resistance of bolts:

$$F_{v,Rd} = \frac{0,6 f_{ub} A_s}{\gamma_{Mb}} = \frac{0,6 \cdot 800 \cdot 3,53 \cdot 10^2}{1,25} \cdot 10^{-3} = 135,6 \text{ kN} > \frac{260}{2} = 130,0 \text{ kN}$$

$$F_{v,Rd} = \frac{169,4}{1,25} = 135,5 \text{ kN} > \frac{260}{2} = 130,0 \text{ kN}$$

Table 6.7; [7.5.2]

5.5.1; [5.4.3]

5.5.1; [5.4.3]

5.5.1; [5.4.3]

[6.5.5;
Table 6.5.3]

6.2.3.2;
Table 6.7

Assumption: Shear in thread

The eccentricity Δe is covered by the design resistances for bolts and need not to be taken into account

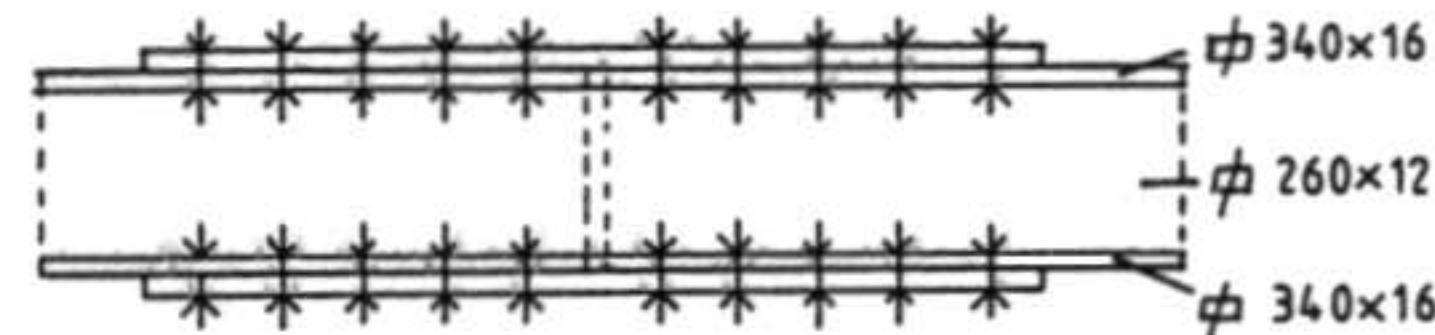
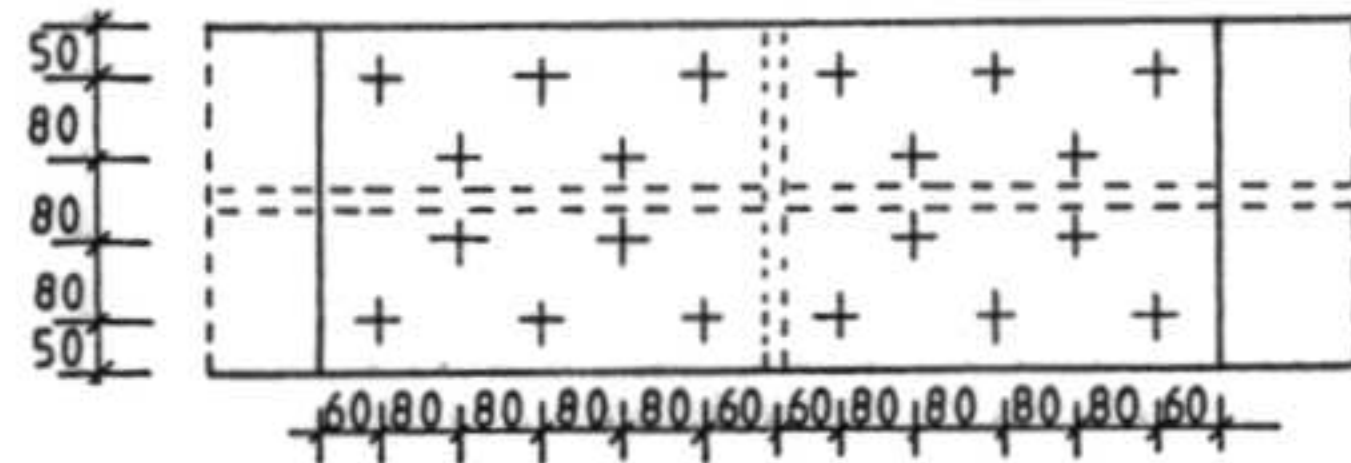
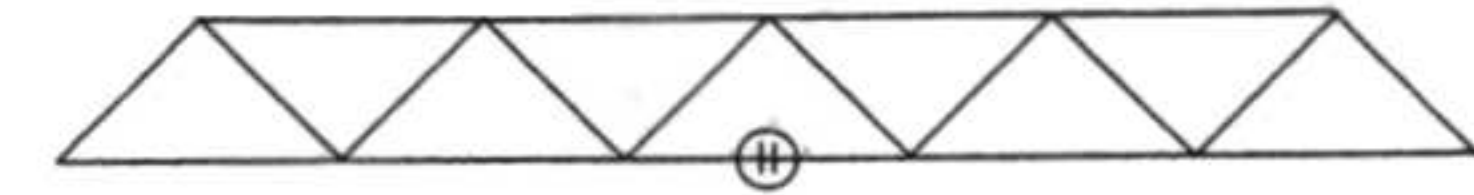
Problem, Sketch, Calculation	Reference	Commentary
Bearing resistance: $e_1/3d_o = 60/(3 \cdot 26) = 0,77$; $p_1/3d_o - 1/4 = 80/(3 \cdot 26) - 1/4 = 0,77$; $\rightarrow \alpha = 0,77$ $F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,77 \cdot 360 \cdot 24 \cdot 12}{1,25} \cdot 10^{-3} = 159,7 \text{ kN} > \frac{260}{2} = 130,0 \text{ kN}$ $F_{b,Rd} = \frac{166,2}{1,25} \cdot \frac{12}{10} = 159,6 \text{ kN} > \frac{260}{2} = 130,0 \text{ kN}$	[6.5.5; Table 6.5.3] 6.2.3.1; Table 6.6	The shear resistance is governing ($F_{v,Rd} < F_{b,Rd}$). If high deformation capacity is needed, the shear resistance must be increased.
4.2 Welded connection (Plate - CHS) Resistance of the fillet weld $F_{w,Rd} = 4 \frac{f_u}{\sqrt{3} \beta_w} \frac{a L}{\gamma_{Mw}} = 4 \cdot \frac{360}{\sqrt{3} \cdot 0,8} \cdot \frac{5 \cdot 120}{1,25} \cdot 10^{-3} = 498,8 \text{ kN} > 260 \text{ kN}$ $F_{w,Rd} = 4 \cdot \frac{129,9}{1,25} \cdot \frac{120}{100} = 498,8 \text{ kN} > 260 \text{ kN}$	[6.6.5.3] 6.3.4.3; Table 6.15	
Local shear resistance for CHS $V_{pl,Rd} = 4 t L \frac{f_y}{\sqrt{3} \gamma_{Mo}} = 4 \cdot 5 \cdot 120 \cdot \frac{235}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 296,0 \text{ kN} > 260,0 \text{ kN}$	5.3.3; [5.4.6]	CHS is critical for local shear failure. The verification for the plate may be neglected.

Problem, Sketch, Calculation

Reference

Commentary

Example 3.1.2: Erection splice at mid span of a lattice girder



1. System

Member: Welded section
Steel grade: Fe 510
see sketch
Splice plates: 880 x 340 x 16 mm thick
Steel grade: Fe 510
Bolts: M 24, 8.8
→ $d_o = 26$ mm

2. Loading

Tension force: $N_{sd} = 3000$ kN

3. Determination of the resistance of the members

3.1 Gross section I - I

Welded section:

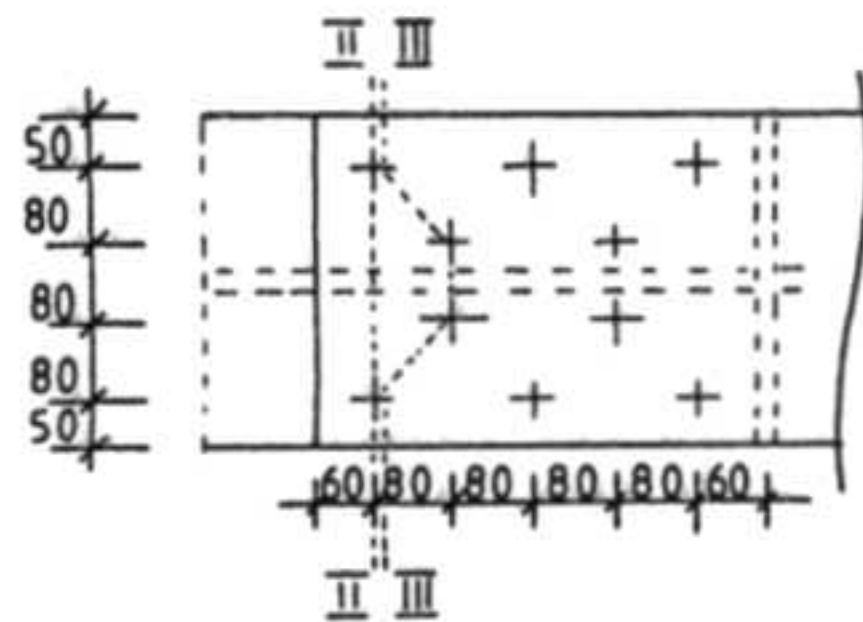
$$A = 340 \cdot 16 \cdot 2 + 260 \cdot 12 = 140,0 \cdot 10^2 \text{ mm}^2$$

Splice plates:

$$A = 2 \cdot 340 \cdot 16 = 108,8 \cdot 10^2 \text{ mm}^2$$

$$N_{pl,Rd} = \frac{A \cdot f_y}{\gamma_{M0}} = \frac{108,8 \cdot 10^2 \cdot 355}{1,1} \cdot 10^{-3} = 3511,3 \text{ kN} > 3000 \text{ kN}$$

3.2 Net section



$$\Delta A_{II} = 4 \cdot 26 \cdot 16 = 16,64 \cdot 10^2 \text{ mm}^2$$

$$\Delta A_{III} = \sum \Delta A_i - \sum \frac{s_i^2 t}{4 p_i} = \left(26 \cdot 16 \cdot 4 - 2 \cdot \frac{80^2 \cdot 16}{4 \cdot 80} \right) \cdot 2 = 20,48 \cdot 10^2 \text{ mm}^2$$

$$A_{net} = A - \max(\Delta A_{II}; \Delta A_{III}) = 108,8 \cdot 10^2 - 20,48 \cdot 10^2 = 88,32 \cdot 10^2 \text{ mm}^2$$

$$N_{u,Rd} = 0,9 \frac{A_{net} \cdot f_u}{\gamma_{M2}} = 0,9 \frac{88,32 \cdot 10^2 \cdot 510}{1,25} \cdot 10^{-3} = 3243,1 \text{ kN} > 3000 \text{ kN}$$

Table 6.7;
[7.5.2]

5.5.1; [5.4.3]

- ; [5.4.2.2.(4)]

5.5.1; [5.4.3]

If $A_{net} \leq 1,11 \frac{f_y \gamma_{M2}}{f_u \gamma_{M0}} A$ then only net section verification is necessary.

Problem, Sketch, Calculation	Reference	Commentary
4. Determination of the connection resistance 4.1 Bolted connection Shear resistance of bolts: $F_{v,Rd} = \frac{0,6 f_{ub} A}{\gamma_{Mb}} = \frac{0,6 \cdot 800 \cdot 4,52 \cdot 10^2}{1,25} \cdot 10^{-3} = 173,6 \text{ kN} > \frac{3000}{2 \cdot 10} = 150,0 \text{ kN}$ Bearing resistance: $e_1/3d_o = 60/(3 \cdot 26) = 0,77$; $p_1/3d_o - 1/4 = 160/(3 \cdot 26) - 1/4 = 1,8$; $\rightarrow \alpha = 0,77$ $F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,77 \cdot 510 \cdot 24 \cdot 16}{1,25} \cdot 10^{-3} = 301,6 \text{ kN} > 150,0 \text{ kN}$ $F_{b,Rd} = \frac{235,4}{1,25} \cdot \frac{16}{10} = 301,3 \text{ kN} > 150,0 \text{ kN}$	6.2.3.2 [6.5.5; Table 6.5.3] [6.5.5; Table 6.5.3] 6.2.3.1; Table 6.6	Assumption: Shear in shank $\rightarrow A = \pi d^2/4 = 4,52 \cdot 10^2 \text{ mm}^2$

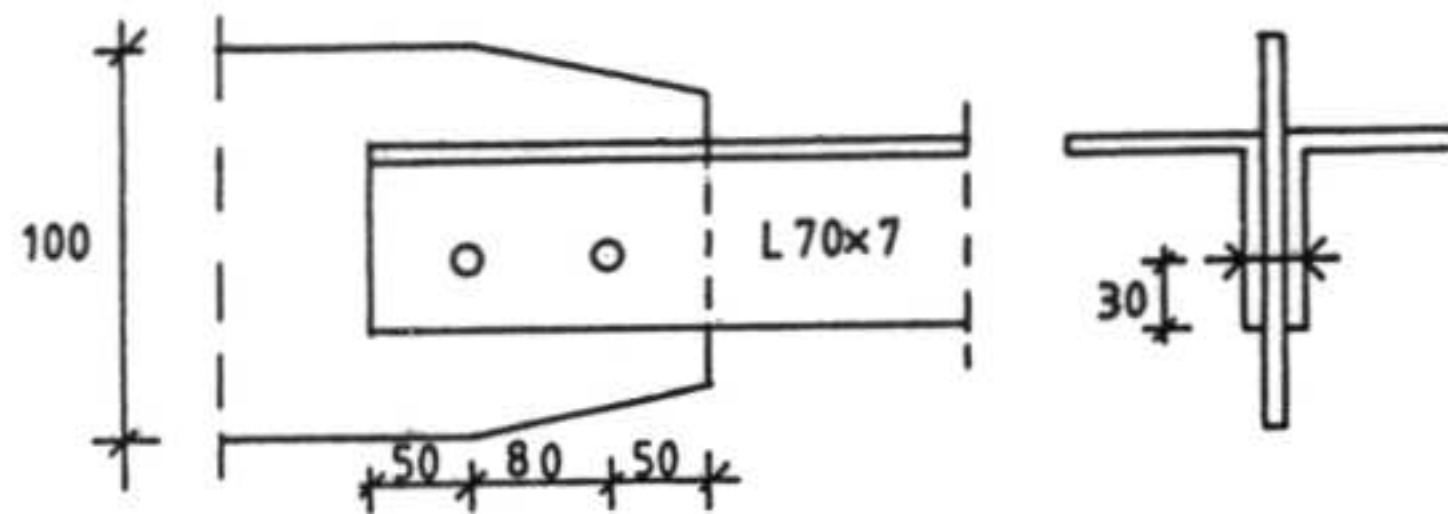
Problem, Sketch, Calculation

Reference

Commentary

Example 3.1.3: Angle connected on a gusset plate

1. System



Member: 2 L 70 x 7 Steel grade: Fe 430
 Plate: 100 x 15 mm Steel grade: Fe 430
 Bolts: M 20, 8.8
 $\rightarrow d_o = 22 \text{ mm}$

2. Loading

Tension force: $N_{sd} = 285 \text{ kN}$

3. Determination of the resistance

3.1 Gross section (angle)

$$N_{pl,Rd} = \frac{A \cdot f_y}{\gamma_{Mo}} = 2 \cdot \frac{9,4 \cdot 10^2 \cdot 275}{1,1} \cdot 10^{-3} = 470,0 \text{ kN} > 285 \text{ kN}$$

3.2 Gross section (plate)

$$N_{pl,Rd} = \frac{A \cdot f_y}{\gamma_{Mo}} = \frac{100 \cdot 15 \cdot 275}{1,1} \cdot 10^{-3} = 375,0 \text{ kN} > 285 \text{ kN}$$

3.3 Net section (Angle)

$$A_{net} = 9,4 \cdot 10^2 - 22 \cdot 7 = 7,86 \cdot 10^2 \text{ mm}^2$$

Determination of β_2 by linear interpolation:

$$\beta_2 = 0,4 + \frac{0,3}{5 \cdot 22 - 2,5 \cdot 22} (80 - 2,5 \cdot 22) = 0,536$$

$$N_{u,Rd} = \beta_2 \frac{A_{net} \cdot f_u}{\gamma_{M2}} = 2 \cdot 0,536 \frac{7,86 \cdot 10^2 \cdot 430}{1,25} \cdot 10^{-3} = 289,9 \text{ kN} > 285,0 \text{ kN}$$

Table 6.7;
[7.5.2]

5.5.1; [5.4.3]

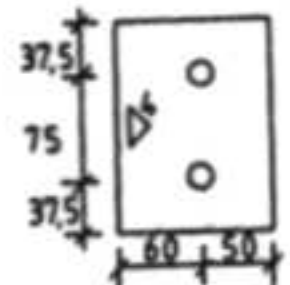
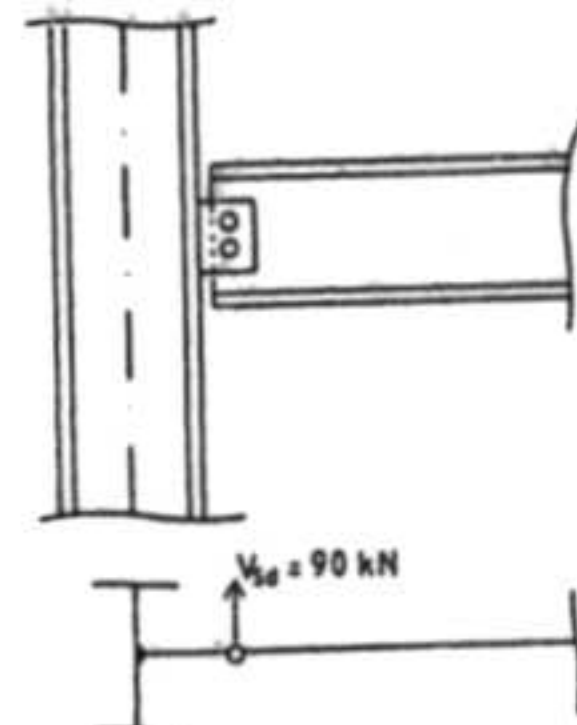
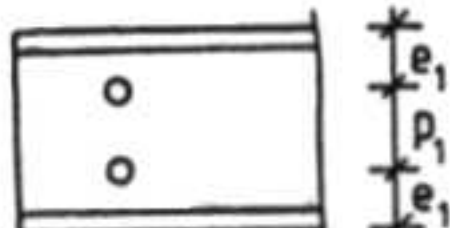
5.5.1; [5.4.3]

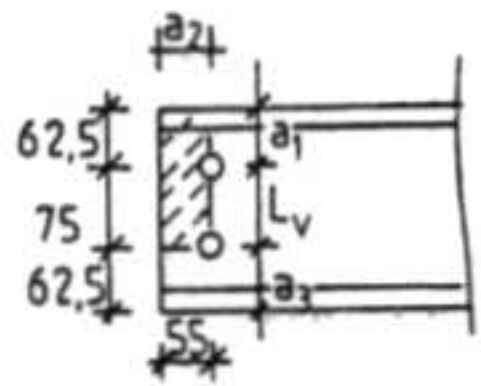
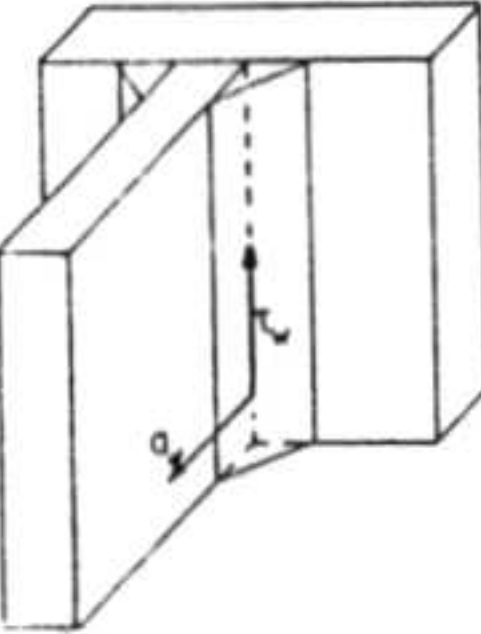
5.5.1;
Table 5.33
[5.4.3; 6.5.2.3]

In the case of joints with angles connected by at least two bolts, the setting out lines for the bolts in the angles may be substituted for the centroidal axes for the purpose of intersection at the joint.

Angles connected by a single row of bolts in one leg may be treated as concentrically loaded if the design ultimate resistance is determined as 5.5.1.(2); [6.5.2.3], i.e. the effect of eccentricities is included in the joint design.

Problem, Sketch, Calculation	Reference	Commentary
4. Determination of the connection resistance 4.1 Bolted connection		Here: Two shear planes Assumption: Shear in thread
Shear resistance of bolts: $F_{v,Rd} = 2 \frac{0,6 f_{ub} A_s}{\gamma_{Mb}} = 2 \frac{0,6 \cdot 800 \cdot 2,45 \cdot 10^2}{1,25} \cdot 10^{-3} = 188,2 \text{ kN} > \frac{285}{2} = 142,5 \text{ kN}$ or $F_{v,Rd} = 2 \frac{117,6}{1,25} = 188,2 \text{ kN} > \frac{285}{2} = 142,5 \text{ kN}$	[6.5.5; Table 6.5.3] 6.2.3.2; Table 6.7	
Bearing resistance: Plate: $e_1/3d_o = 50/3 \cdot 22 = 0,76 ; p_1/3d_o - 1/4 = 80/3 \cdot 22 - 1/4 = 0,96 ; \longrightarrow \alpha = 0,76$ $F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,76 \cdot 430 \cdot 20 \cdot 15}{1,25} \cdot 10^{-3} = 196,1 \text{ kN} > \frac{285}{2} = 142,5 \text{ kN}$	[6.5.5; Table 6.5.3] 6.2.3.1; Table 6.6	If the end distance is equal then only one calculation is necessary.
Angles: $e_1/3d_o = 50/3 \cdot 22 = 0,76 ; p_1/3d_o - 1/4 = 80/3 \cdot 22 - 1/4 = 0,96 ; \longrightarrow \alpha = 0,76$ $e_2 = 30 \text{ mm} = 1,36d_o < 1,5d_o$ $\longrightarrow \text{Reduction of the bearing resistance required}$ Bearing resistance without reduction: $F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,76 \cdot 430 \cdot 20 \cdot 7}{1,25} \cdot 10^{-3} = 91,5 \text{ kN}$ or $F_{b,Rd} = \frac{162,9}{1,25} \cdot \frac{7}{10} = 91,2 \text{ kN}$	[6.5.5; Table 6.5.3] 6.2.3.1; Table 6.6	The bearing resistance is calculated per member (here: per angle).
Reduktion of the bearing resistance: $F_{b,Rd} = 91,5 \cdot \left(\frac{2}{3} + \frac{1,36 - 1,20}{1,50 - 1,20} \cdot \frac{1}{3} \right) = 77,3 \text{ kN} > \frac{285}{2 \cdot 2} = 71,3 \text{ kN}$	[6.5.5(10)]	

Problem, Sketch, Calculation	Reference	Commentary																				
Example 3.1.4: Fin plate connection with supporting member as an I-section  1. System: <table><tr><td>Column member:</td><td>HE 200 B</td><td>Steel grade:</td><td>Fe 430</td></tr><tr><td>Beam member:</td><td>IPE 200</td><td>Steel grade:</td><td>Fe 510</td></tr><tr><td>Fin plate:</td><td>150 x 110 x 10</td><td>Steel grade:</td><td>Fe 510</td></tr><tr><td>Bolts:</td><td>M 20, 8.8</td><td></td><td></td></tr><tr><td></td><td>→ do = 22 mm</td><td></td><td></td></tr></table> 2. Loading Shear force: Vsd = 90 kN 3. Determination of the resistance of the connection: 3.1 Design resistance of the connection (fin plate - beam) Shear resistance of bolts: $F_{v,Rd} = \frac{0,6 f_{ub} A_s}{\gamma_{Mb}} = \frac{0,6 \cdot 800 \cdot 2,45 \cdot 10^2}{1,25} \cdot 10^{-3} = 94,1 \text{ kN} > \frac{90}{2} = 45 \text{ kN}$ or $F_{v,Rd} = \frac{117,6}{1,25} = 94,1 \text{ kN} > \frac{90}{2} = 45 \text{ kN}$ Bearing resistance: Fin plate: $e_1/3d_o = 37,5/3 \cdot 22 = 0,57$; $p_1/3d_o - 1/4 = 75/3 \cdot 22 - 1/4 = 0,89$; → $\alpha = 0,57$ $F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,57 \cdot 510 \cdot 20 \cdot 10}{1,25} \cdot 10^{-3} = 116,3 \text{ kN} > 45 \text{ kN}$ Beam web: $p_1/3d_o - 1/4 = 75/3 \cdot 22 - 1/4 = 0,89$; → $\alpha = 0,89$  $F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,89 \cdot 510 \cdot 20 \cdot 5,6}{1,25} \cdot 10^{-3} = 101,7 \text{ kN} > 45 \text{ kN}$ 3.2 Shear resistance of the beam $A_{v,net} = 11,2 \cdot 10^2 - 2 \cdot 22 \cdot 5,6 = 8,7 \cdot 10^2 \text{ mm}^2 \geq A_v \cdot f_v / f_u = 11,2 \cdot 10^2 \cdot 355 / 510 = 7,8 \cdot 10^2 \text{ mm}^2$ → fastener holes need not be allowed for in shear verification $V_{pl,Rd} = A_v \frac{f_y}{\sqrt{3} \gamma_{Mo}} = 11,2 \cdot 10^2 \cdot \frac{355}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 208,7 \text{ kN} > 90,0 \text{ kN}$	Column member:	HE 200 B	Steel grade:	Fe 430	Beam member:	IPE 200	Steel grade:	Fe 510	Fin plate:	150 x 110 x 10	Steel grade:	Fe 510	Bolts:	M 20, 8.8				→ do = 22 mm			[6.5.5; Table 6.5.3] 6.2.3.2; Table 6.7 6.2.3.1; Table 6.6 [6.5.5; Table 6.5.3] 5.3.3; [5.4.6]	Fin plate connections are assumed as hinged connections. The location of the hinge is controlled by the stiffness of the supporting member. Here: The column web is a rigid support of the fin plate. Therefore it may be assumed that the hinge occurs at the bolts. Assumption: Shear in thread The edge distance e_1 should be taken as the minimum value of both sides when the acting force effects not parallel to one of both edge distances $A_v = h \cdot t_w = 11,2 \cdot 10^2 \text{ mm}^2$ It is allowed to consider larger values by using the formulae given in EC 3 [5.4.6]
Column member:	HE 200 B	Steel grade:	Fe 430																			
Beam member:	IPE 200	Steel grade:	Fe 510																			
Fin plate:	150 x 110 x 10	Steel grade:	Fe 510																			
Bolts:	M 20, 8.8																					
	→ do = 22 mm																					

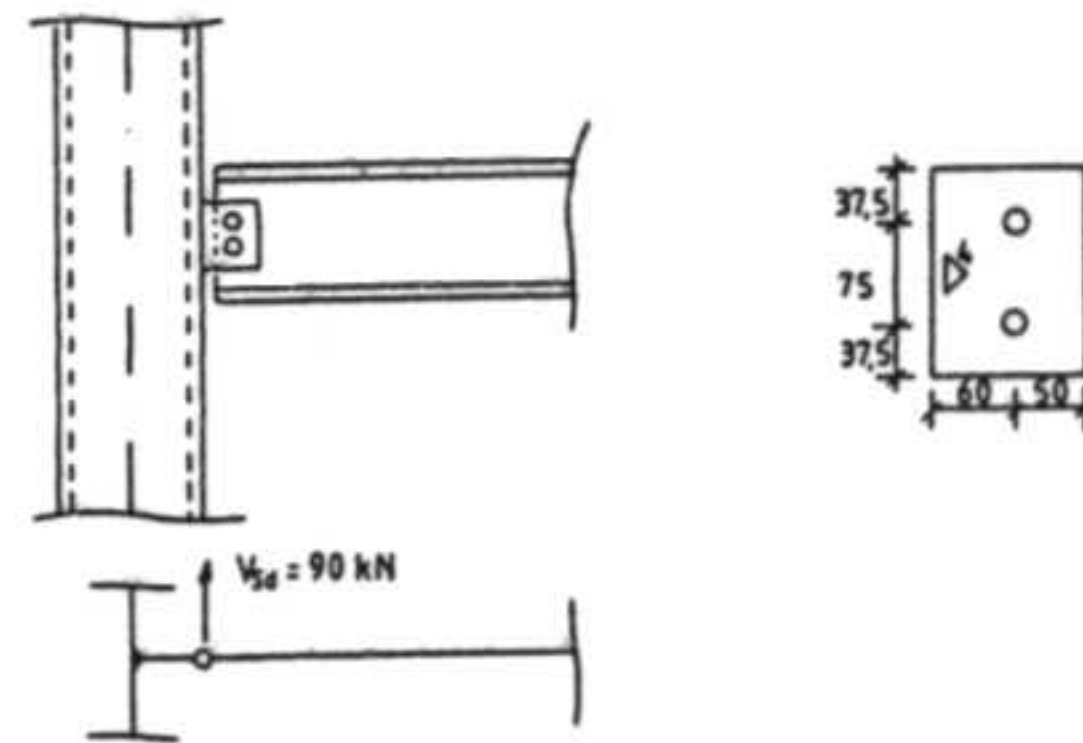
Problem, Sketch, Calculation	Reference	Commentary
3.3 Design resistance of fin plate $A_{v,net} = 150 \cdot 10 - 2 \cdot 22 \cdot 5,6 = 12,5 \cdot 10^2 \text{ mm}^2 \geq A_v \cdot f_y / f_u = 15,0 \cdot 10^2 \cdot 355 / 510 = 10,4 \cdot 10^2 \text{ mm}^2$ → fastener holes need not be allowed for in shear verification $V_{pl,Rd} = A_v \frac{f_y}{\sqrt{3} \gamma_{M0}} = 15,0 \cdot 10^2 \cdot \frac{355}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 279,5 \text{ kN} > 90,0 \text{ kN}$ $V_{Sd} < 0,5 V_{pl,Rd}$ → no interaction bending - shear is required $W_{el} = 150^2 \cdot 10 / 6 = 37,5 \cdot 10^3 \text{ mm}^3$ $M_{Rd} = W_{el} f_y / \gamma_{M0} = 37,5 \cdot 10^3 \cdot 355 \cdot 10^{-6} / 1,1 = 12,1 \text{ kNm} > 5,4 \text{ kNm}$	5.3.3; [5.4.6] 5.3.2; [5.4.5]	
3.4 Block shear resistance  $L_1 = a_1 = 62,5 \text{ mm} < 5d = 5 \cdot 20 = 100 \rightarrow L_1 = 62,5 \text{ mm}$ $L_2 = (a_2 - k d_{0,1}) (f_u / f_y) = (50 - 0,5 \cdot 22) 510 / 355 = 56 \text{ mm}$ $L_3 = L_v + a_1 + a_3 = 75 + 62,5 + 62,5 = 200 \text{ mm}$ $< (L_v + a_1 + a_3 - n d_{0,v}) (f_u / f_y) = (200 - 2 \cdot 22) 510 / 355 = 224,1 \text{ mm}$ $L_{v,eff} = L_v + L_1 + L_2 = 75 + 62,5 + 56 = 193,5 \text{ mm} < L_3 = 200 \text{ mm} \rightarrow L_{v,eff} = 193,5 \text{ mm}$ $V_{eff,Rd} = L_{v,eff} t \frac{f_y}{\sqrt{3} \gamma_{M0}} = 193,5 \cdot 5,6 \cdot \frac{355}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 201,9 \text{ kN} > 90 \text{ kN}$	5.5.2; [6.5.2.2]	The formulae given in table 5.34 of E-EC3 are not used.
3.5 Resistance of the fillet weld Action effects at fillet weld: Shear force: $V_{Sd} = 90 \text{ kN}$ Bending moment: $M_{Sd} = 90 \cdot 60 \cdot 10^{-3} = 5,4 \text{ kNm}$  Geometrical data of the weld: $a = 4 \text{ mm}$ $A_w = 2 \cdot 150 \cdot 4 = 12 \cdot 10^2 \text{ mm}^2$ $W_w = A_w \cdot h / 6 = 12 \cdot 10^2 \cdot 150 / 6 = 30 \cdot 10^3 \text{ mm}^3$ Weld stresses: $\tau_w = V_{Sd} / A_w = 90 \cdot 10^3 / 12 \cdot 10^2 = 75 \text{ N/mm}^2$ $\sigma_w = M_{Sd} / W_w = 5,4 \cdot 10^6 / 30 \cdot 10^3 = 180 \text{ N/mm}^2$ $\sqrt{(\tau_w^2 + \sigma_w^2)} = \sqrt{(75^2 + 180^2)} = 195 \text{ N/mm}^2$ $f_{w,d} = \frac{f_u}{\sqrt{3} \beta_w \gamma_{Mw}} = \frac{430}{\sqrt{3} \cdot 0,85 \cdot 1,25} = 234 \text{ N/mm}^2 > 195 \text{ N/mm}^2$	6.3.4.3; [6.6.5.3]	elastic stress distribution is assumed The alternative method given in Annex M of Eurocode 3 may lead to a smaller weld size, however the calculation is more complex.

Problem, Sketch, Calculation

Reference

Commentary

Example 3.1.5: Fin plate connection with a supporting member as a RHS



1. System:

Column member:	RHS 200 x 8,0	Steel grade:	Fe 430
Beam member:	IPE 200	Steel grade:	Fe 510
Fin plate:	150 x 110 x 10	Steel grade:	Fe 510
Bolts:	M 20, 8.8	→ d _o = 22 mm	

2. Loading

Shear force:	V _{sd} = 90 kN
--------------	-------------------------

3. Determination of the resistance of the connection:

3.1 Design resistance of the connection (fin plate - beam)

Bending moment:	M _{sd} = 90 · 60 · 10 ⁻³ = 5,4 kNm
Horizontal shear force per bolt:	M _{sd} /75 · 10 ⁻³ = 5,4/75 · 10 ⁻³ = 72 kN
Vertical shear force per bolt:	V _{sd} = 90/2 = 45 kN
Resultant shear force per bolt:	F _{sd} = √(45 ² + 72 ²) = 84,9 kN

Shear resistance of bolts:
$$F_{v,Rd} = \frac{0,6 f_{ub} A_s}{\gamma_{Mb}} = \frac{0,6 \cdot 800 \cdot 2,45 \cdot 10^2}{1,25} \cdot 10^{-3} = 94,1 \text{ kN} > 84,9 \text{ kN}$$

or
$$F_{v,Rd} = \frac{117,6}{1,25} = 94,1 \text{ kN} > 84,9 \text{ kN}$$

Bearing resistance:

Fin plate:

$$e_1/3d_o = 37,5/3 \cdot 22 = 0,57 ; p_1/3d_o - 1/4 = 75/3 \cdot 22 - 1/4 = 0,89 ; \rightarrow \alpha = 0,57$$

$$F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,57 \cdot 510 \cdot 20 \cdot 10}{1,25} \cdot 10^{-3} = 116,3 \text{ kN} > 84,9 \text{ kN}$$

Beam web:

$$e_1/3d_o = 50/3 \cdot 22 = 0,76 ; p_1/3d_o - 1/4 = 75/3 \cdot 22 - 1/4 = 0,89 ; \rightarrow \alpha = 0,76$$

$$F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,76 \cdot 510 \cdot 20 \cdot 5,6}{1,25} \cdot 10^{-3} = 86,8 \text{ kN} > 84,9 \text{ kN}$$

3.2 Shear resistance of the beam

$$A_{v,net} = 11,2 \cdot 10^2 - 2 \cdot 22 \cdot 5,6 = 8,7 \cdot 10^2 \text{ mm}^2 \geq A_v \cdot f_y / f_u = 11,2 \cdot 10^2 \cdot 355 / 510 = 7,8 \cdot 10^2 \text{ mm}^2$$

→ fastener holes need not be allowed for in shear verification

$$V_{pl,Rd} = A_v \frac{f_y}{\sqrt{3} \gamma_{Mo}} = 11,2 \cdot 10^2 \cdot \frac{355}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 208,7 \text{ kN} > 90,0 \text{ kN}$$

Fin plate connections are assumed as hinged connections. The location of the hinge is controlled by the stiffness of the supporting member.

Here: The column flange is a flexible support of the fin plate. Therefore it could be assumed that the hinge occurs at the welds.

However; in practice, most supports are neither fully rigid nor entirely flexible. Therefore it is safer to design both the bolts and the welds for shear force and moment. This principle may also be taken where bolted connections are used instead of weldings in fin plate connections.

Assumption: Shear in thread

[6.5.5;
Table 6.5.3]

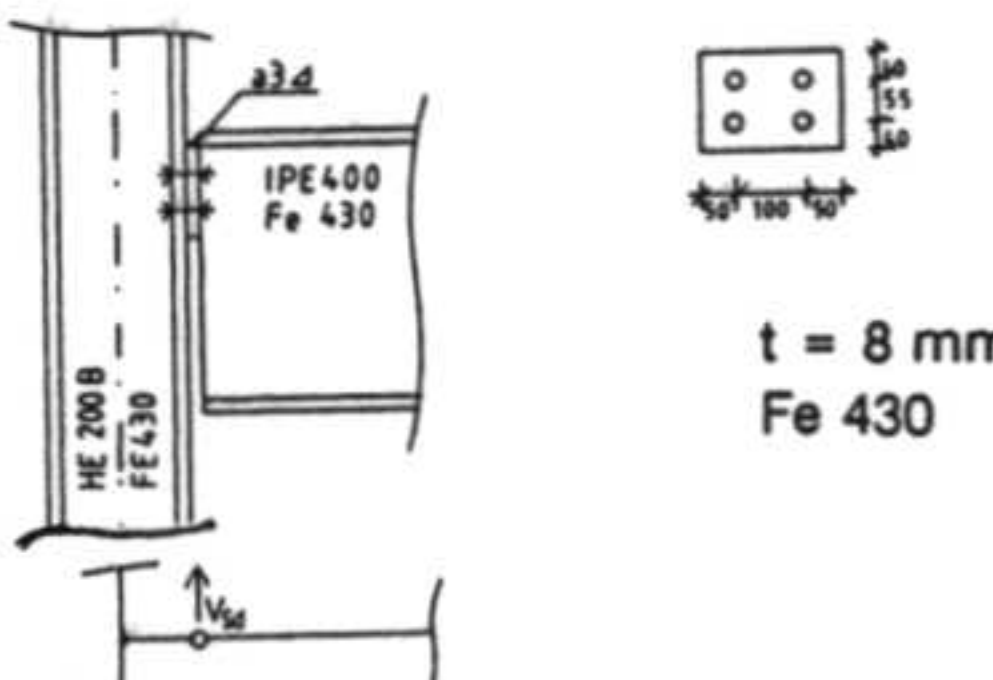
6.2.3.2;
Table 6.7

6.2.3.1;
Table 6.6
[6.5.5;
Table 6.5.3]

The edge distance e₁ should be taken as the minimum value of both sides when the acting force effects not parallel to one of both edge distances

5.3.3; [5.4.6]

Problem, Sketch, Calculation	Reference	Commentary
3.3 Design resistance of fin plate $A_{v,net} = 150 \cdot 10 - 2 \cdot 22 \cdot 5,6 = 12,5 \cdot 10^2 \text{ mm}^2 \geq A_v \cdot f_v / f_u = 15,0 \cdot 10^2 \cdot 355 / 510 = 10,4 \cdot 10^2 \text{ mm}^2$ → fastener holes need not be allowed for in shear verification $V_{pl,Rd} = A_v \frac{f_y}{\sqrt{3} \gamma_{Mo}} = 15,0 \cdot 10^2 \cdot \frac{355}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 279,5 \text{ kN} > 90,0 \text{ kN}$ $V_{Sd} < 0,5 V_{pl,Rd}$ → no interaction bending - shear is required $W_{el} = 150^2 \cdot 10 / 6 = 37,5 \cdot 10^3 \text{ mm}^3$ $M_{Rd} = W_{el} f_y / \gamma_{Mo} = 37,5 \cdot 10^3 \cdot 355 \cdot 10^{-6} / 1,1 = 12,1 \text{ kNm} > 5,4 \text{ kNm}$ 3.4 Block shear resistance $L_1 = a_1 = 62,5 \text{ mm} < 5d = 5 \cdot 20 = 100 \rightarrow L_1 = 62,5 \text{ mm}$ $L_2 = (a_2 - k d_{o,v}) (f_u / f_y) = (55 - 0,5 \cdot 22) 510 / 355 = 63,2 \text{ mm}$ $L_3 = L_v + a_1 + a_3 = 75 + 62,5 + 62,5 = 200 \text{ mm}$ $< (L_v + a_1 + a_3 - n d_{o,v}) (f_u / f_y) = (200 - 2 \cdot 22) 510 / 355 = 224,1 \text{ mm}$ $L_{v,eff} = L_v + L_1 + L_2 = 75 + 62,5 + 63,2 = 200,7 \text{ mm} > L_3 = 200 \text{ mm} \rightarrow L_{v,eff} = 200 \text{ mm}$ $V_{eff,Rd} = L_{v,eff} t \frac{f_y}{\sqrt{3} \gamma_{Mo}} = 200 \cdot 5,6 \cdot \frac{355}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 208,7 \text{ kN} > 90 \text{ kN}$ 3.5 Resistance of the fillet weld Action effects at fillet weld: Shear force: $V_{Sd} = 90 \text{ kN}$ Bending moment: $M_{Sd} = 90 \cdot 60 \cdot 10^{-3} = 5,4 \text{ kNm}$ Weld stresses: $\tau_f = V_{Sd} / A_w = 90 \cdot 10^3 / 12 \cdot 10^2 = 75 \text{ N/mm}^2$ $\sigma_\perp = M_{Sd} / W_w = 5,4 \cdot 10^6 / 30 \cdot 10^3 = 180 \text{ N/mm}^2$ $\sqrt{(\tau_f)^2 + (\sigma_\perp)^2} = \sqrt{75^2 + 180^2} = 195 \text{ N/mm}^2$ $f_{vw,d} = \frac{f_u}{\sqrt{3} \beta_w \gamma_{Mw}} = \frac{430}{\sqrt{3} \cdot 0,85 \cdot 1,25} = 234 \text{ N/mm}^2 > 195 \text{ N/mm}^2$ 3.6 Joint resistance of fin plate welded to RHS - chord $M_{p,Rd} = \frac{f_{yo} t_o^2}{1 - \frac{t_l}{b_o}} \left(2 \frac{h_l}{b_o} + 4 \cdot \sqrt{1 - \frac{t_l}{b_o}} \right) \frac{h_l}{2} k_n$ $= \frac{275 \cdot 8,0^2}{1 - \frac{10}{200}} \left(2 \frac{150}{200} + 4 \cdot \sqrt{1 - \frac{10}{200}} \right) \frac{150}{2} = 7,5 \text{ kNm} > 5,4 \text{ kNm}$	5.3.3; [5.4.6] 5.3.2; [5.4.5] 5.5.2; [6.5.2.2] 6.3.4.3; [6.6.5.3] [Annex K, revised version]	Elastic stress distribution is assumed for the welds. a = 4 mm $A_w = 2 \cdot 150 \cdot 4 = 12 \cdot 10^2 \text{ mm}^2$ $W_w = A_w \cdot h / 6 = 12 \cdot 10^2 \cdot 150 / 6 = 30 \cdot 10^3 \text{ mm}^3$ For compression: $k_n = 1,3 + 0,4 / (t_l / b_o) \cdot n$ but $k_n \leq 1,0$ where $n = \sigma_o / f_{yo}$ Here: Assumption: $k_n = 1,0$ For tension: $k_n = 1,0$

Problem, Sketch, Calculation	Reference	Commentary																								
Example 3.1.6: Flexible end plate connection  1. System <table><tr><td>Column member:</td><td>HE 200 B</td><td>Steel grade:</td><td>Fe 430</td></tr><tr><td>Beam member:</td><td>IPE 400</td><td>Steel grade:</td><td>Fe 430</td></tr><tr><td>End plate:</td><td>135x115x8</td><td>Steel grade:</td><td>Fe 430</td></tr><tr><td>Bolts:</td><td>M 16, 8.8</td><td></td><td></td></tr><tr><td></td><td>$\rightarrow d_o = 18 \text{ mm}$</td><td></td><td></td></tr></table> 2. Loading <table><tr><td>Shear force:</td><td>$V_{sd} = 150 \text{ kN}$</td></tr><tr><td>per bolt:</td><td>$V_{sd}/4 = 37,5 \text{ kN}$</td></tr></table> 3. Determination of the resistance of the connection: 3.1 Weld resistance throat thickness chosen to be $a = 3 \text{ mm}$ $F_{w,Rd} = \frac{f_u}{\sqrt{3} \beta_w \gamma_{Mw}} aL = 2 \cdot \frac{430}{\sqrt{3} \cdot 0,85 \cdot 1,25} \cdot 3 \cdot 135 \cdot 10^{-3} = 189,3 \text{ kN} > 150,0 \text{ kN}$ 3.2 Shear resistance of bolts $F_{v,Rd} = \frac{0,6 A_s f_{ub}}{\gamma_{Mb}} = \frac{0,6 \cdot 157 \cdot 800}{1,25} \cdot 10^{-3} = 60,3 \text{ kN} > 37,5 \text{ kN}$ or $F_{v,Rd} = \frac{0,6 A_s f_{ub}}{\gamma_{Mb}} = \frac{75,4}{1,25} = 60,3 \text{ kN} > 37,5 \text{ kN}$ 3.3 Bearing resistance of the endplate $e_1/3d_o = 40/3 \cdot 18 = 0,74$ $p_1/3d_o - 1/4 = 55/3 \cdot 18 - 1/4 = 0,77 \rightarrow \alpha = 0,74$ $F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,74 \cdot 430 \cdot 16 \cdot 8}{1,25} \cdot 10^{-3} = 81,5 \text{ kN} > 37,5 \text{ kN}$ or $F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{127,4}{1,25} \cdot \frac{8}{10} = 81,5 \text{ kN} > 37,5 \text{ kN}$ 3.4 Bearing resistance of the column flange not relevant because of $t_f > t_p$	Column member:	HE 200 B	Steel grade:	Fe 430	Beam member:	IPE 400	Steel grade:	Fe 430	End plate:	135x115x8	Steel grade:	Fe 430	Bolts:	M 16, 8.8				$\rightarrow d_o = 18 \text{ mm}$			Shear force:	$V_{sd} = 150 \text{ kN}$	per bolt:	$V_{sd}/4 = 37,5 \text{ kN}$	Table 6.7 [7.5.2] 6.3.4.3; Table 6.15 [6.6.5.3] 6.2.3.2 Table 6.7 [6.5.5; Table 6.5.3] 6.2.3.1; Table 6.6 [6.5.5; Table 6.5.3]	From practical evidence flexible end plate connections are assumed as hinged connections. Assumption: Shear in thread The choice of $e_1 = 40 \text{ mm}$ is conservative in view of the presence of the transverse weld between flange and flexible endplate. f_{ub}/f_u is normally greater than 1,0; however α is equal or lesser than 1,0.
Column member:	HE 200 B	Steel grade:	Fe 430																							
Beam member:	IPE 400	Steel grade:	Fe 430																							
End plate:	135x115x8	Steel grade:	Fe 430																							
Bolts:	M 16, 8.8																									
	$\rightarrow d_o = 18 \text{ mm}$																									
Shear force:	$V_{sd} = 150 \text{ kN}$																									
per bolt:	$V_{sd}/4 = 37,5 \text{ kN}$																									

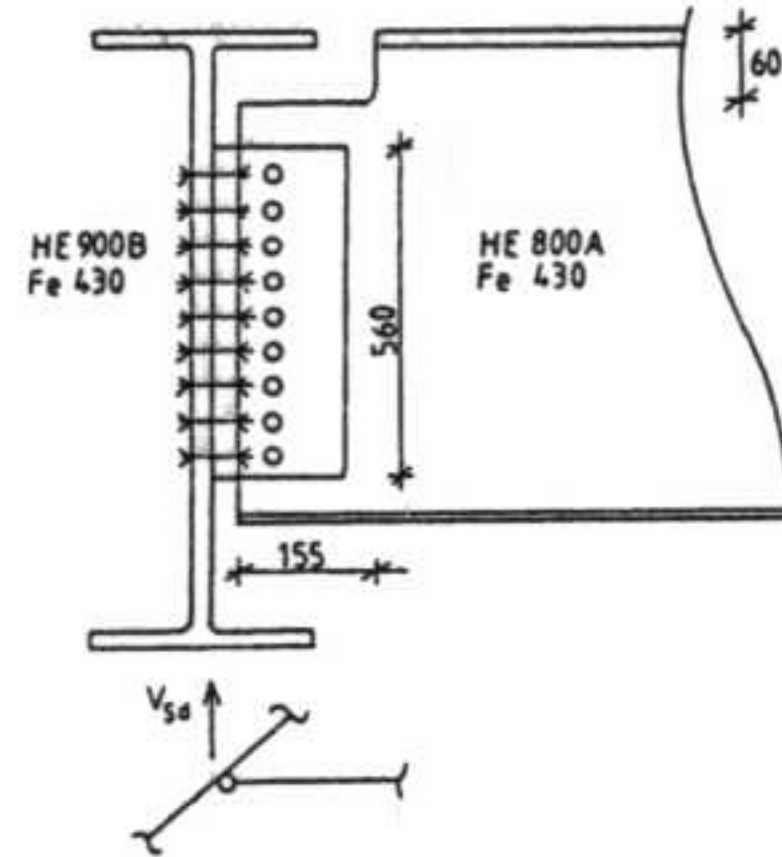
Problem, Sketch, Calculation	Reference	Commentary
3.5 Shear resistance of the endplate $A_{v,net} = 2 \cdot (135 - 2 \cdot 18) \cdot 8 = 15,8 \cdot 10^2 \text{ mm}^2 > A_v f_y / f_u = (135 \cdot 8 \cdot 2) \cdot 275 / 430 = 13,8 \cdot 10^2 \text{ mm}^2$ $V_{pl,Rd} = A_v \frac{f_y}{\sqrt{3} \gamma_{Mo}} = 21,6 \cdot 10^2 \cdot \frac{275}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 311,8 \text{ kN} > 150,0 \text{ kN}$ 3.6 Local shear resistance of the beam web $A_v = 8,6 \cdot 135 = 11,61 \cdot 10^2 \text{ mm}^2$ $V_{pl,Rd} = A_v \frac{f_y}{\sqrt{3} \gamma_{Mo}} = 11,61 \cdot 10^2 \cdot \frac{275}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 167,6 \text{ kN} > 150,0 \text{ kN}$	5.3.3 [5.4.6]	

Problem, Sketch, Calculation

Reference

Commentary

Example 3.1.7: Beam to beam connection with cleats



1. System

Main girder:	HE 900 B	Steel grade:	Fe 430
Side girder:	HE 800 A	Steel grade:	Fe 430
Angles:	2 L 90 x 9	Steel grade:	Fe 430
Bolts:	M 22, 8.8		
	→ $d_o = 24$ mm		

2. Loading

Shear force: $V_{sd} = 890$ kN

3. Determination of the resistance of the connection

3.1 Design resistance of the connection (angle - main girder)

Shear resistance of bolts: $F_{v,Rd} = \frac{0,6 f_{ub} A_s}{\gamma_{Mb}} = \frac{0,6 \cdot 800 \cdot 303}{1,25} \cdot 10^{-3} = 116,4 \text{ kN} > \frac{890}{18} = 49,4 \text{ kN}$

or $F_{v,Rd} = \frac{145,4}{1,25} = 116,3 \text{ kN} > \frac{890}{18} = 49,4 \text{ kN}$

Bearing resistance:

Angle: $e_1/3d_o = 40/(3 \cdot 24) = 0,56$; $p_1/3d_o - 1/4 = 60/(3 \cdot 24) - 1/4 = 0,58$; → $\alpha = 0,56$

$F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,56 \cdot 430 \cdot 22 \cdot 9}{1,25} \cdot 10^{-3} = 95,4 \text{ kN} > \frac{890}{2 \cdot 9} = 49,4 \text{ kN}$

Beam web: $p_1/3d_o - 1/4 = 60/(3 \cdot 24) - 1/4 = 0,58$; → $\alpha = 0,58$

$F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,58 \cdot 430 \cdot 22 \cdot 18,5}{1,25} \cdot 10^{-3} = 203,0 \text{ kN} > \frac{890}{9} = 98,9 \text{ kN}$

3.2 Determination of the moment due to eccentricity of the bolt group (angle - side girder)

$M_{sd} = 59,25 \cdot 10^{-3} \cdot 890 = 52,7 \text{ kNm}$

3.3 Design resistance of the connection (angle - side girder)

$I_p = 2160 \cdot 10^4 \text{ mm}^2$

Horizontal shear force: $M_{sd} \cdot z / I_p = 52,7 \cdot 10^3 \cdot 240 / 2160 \cdot 10^4 = 58,6 \text{ kN}$

Vertical shear force per bolt: $V_{sd} / 9 = 98,9 \text{ kN}$

Resultant shear force: $F_{sd} = \sqrt{(98,9^2 + 58,6^2)} = 115,0 \text{ kN}$

Table 6.7 [7.5.2]

[6.5.5;
Table 6.5.3]

6.2.3.2;
Table 6.7

6.2.3.1;
Table 6.6
[6.5.5;
Table 6.5.3]

Cleated connections on web only are assumed as hinged connections.
Torsional effects may be neglected because of the large length of the cleats.

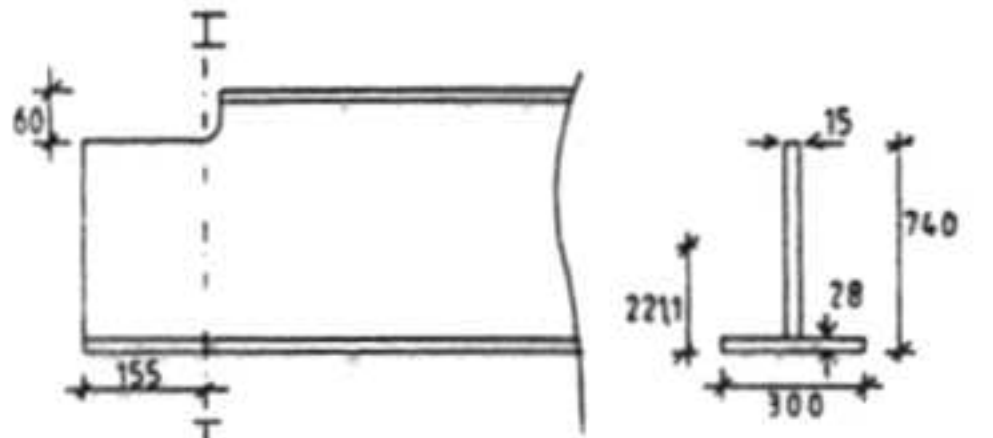
Assumption: Shear in thread

The bearing resistance is calculated per member.

The reaction force V_{sd} is applied to the connection at the notched main girder - angle cleat position

The distribution of internal forces between fasteners needs to be proportional to the distance from the centre of rotation in slip-resistant connections
→ polar moment of inertia of the bolt configuration:

$$I_p = \sum y^2 + \sum z^2 = 2 \cdot (60^2 + 120^2 + 180^2 + 240^2) = 2160 \cdot 10^4 \text{ mm}^2$$

Problem, Sketch, Calculation	Reference	Commentary
Shear resistance of bolts: $F_{v,Rd} = m_s \frac{0,6 f_{ub} A_s}{\gamma_{Mb}} = 2 \frac{0,6 \cdot 800 \cdot 303}{1,25} \cdot 10^{-3} = 232,7 \text{ kN} > 115,0 \text{ kN}$ $F_{v,Rd} = 2 \frac{145,4}{1,25} = 232,6 \text{ kN} > 115,0 \text{ kN}$ Bearing resistance: Angle: $e_1/3d_o = 40/(3 \cdot 24) = 0,56 ; p_1/3d_o - 1/4 = 60/(3 \cdot 24) - 1/4 = 0,58 ; \longrightarrow \alpha = 0,56$ $F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,56 \cdot 430 \cdot 22 \cdot 9}{1,25} \cdot 10^{-3} = 95,4 \text{ kN} > \frac{115,0}{2} = 57,5 \text{ kN}$ Beam web: $e_1/3d_o = 36/(3 \cdot 24) = 0,5 ; p_1/3d_o - 1/4 = 60/3 \cdot 24 - 1/4 = 0,58 ; \longrightarrow \alpha = 0,50$ $F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,50 \cdot 430 \cdot 22 \cdot 15}{1,25} \cdot 10^{-3} = 141,9 \text{ kN} > 115,0 \text{ kN}$	6.2.3.2; Table 6.7 [6.5.5; Table 6.5.3] 6.2.3.1; Table 6.6 [6.5.5; Table 6.5.3] 6.2.2; [6.5.4]	Assumptions: m_s = number of members; here: two shear planes - Shear in thread - elastic distribution of internal forces The bearing resistance is calculated per member. The edge distance e_1 should be taken as the minimum value of both sides when the acting force effects not parallel to one of both edge distances.
3.4 Shear resistance of the angle $A_{v,net} = 560 \cdot 9 - 9 \cdot 24 \cdot 9 = 31,0 \cdot 10^2 \text{ mm}^2 \leq A_v \cdot f_u / f_u = 560 \cdot 9 \cdot 275 / 430 = 32,2 \cdot 10^2 \text{ mm}^2$ → fastener holes have to be allowed for in shear verification $A_{v,eff} = A_{v,net} f_u / f_u = 31,0 \cdot 10^2 \cdot 430 / 275 = 48,5 \cdot 10^2 \text{ mm}^2 < 50,4 \cdot 10^2 \text{ mm}^2 = A_v$ $V_{pl,Rd} = A_v \frac{f_y}{\sqrt{3} \gamma_{Mo}} = 48,5 \cdot 10^2 \cdot \frac{275}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 700,0 \text{ kN} > \frac{890,0}{2} = 445,0 \text{ kN}$	5.3.3; [5.4.6]	Two angle cleats connect the side girder to the main girder. The shear resistance is calculated per member.
3.5 Design resistance of the notched beam at cut I - I $V_{Sd} = 890 \text{ kN} \longrightarrow M_{I,Sd} = 890 \cdot 155 \cdot 10^{-3} = 138 \text{ kNm}$ Cross-sectional dimensions at cut I - I $A_t = 300 \cdot 28 + 712 \cdot 15 = 190,8 \cdot 10^2 \text{ mm}^2$ $z_t = 712 \cdot 15 \frac{712/2 + 14}{190,8 \cdot 10^2} + 14 = 221,1 \text{ mm}$ $I_t = 712^3 \cdot 15 / 12 + 300 \cdot 28 \cdot 207,1^2 + 712 \cdot 15 \cdot 162,9^2 = 109486,8 \cdot 10^4 \text{ mm}^4$ $W_{t,el} = 109486,8 \cdot 10^4 / 518,9 = 2110 \cdot 10^3 \text{ mm}^3$ $A_{t,v} = 712 \cdot 15 = 106,8 \cdot 10^2 \text{ mm}^2$ 		

Problem, Sketch, Calculation	Reference	Commentary
$M_{ed,Rd} = \frac{W_{el} f_y}{\gamma_{Mo}} = \frac{2110 \cdot 10^3 \cdot 275}{1,1} \cdot 10^{-6} = 527,5 \text{ kNm} > 138,0 \text{ kNm}$ $0,5 V_{Rd} = 0,5 \frac{A_v f_y}{\sqrt{3} \gamma_{Mo}} = 0,5 \cdot \frac{106,8 \cdot 10^2 \cdot 275}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 770,8 \text{ kN} < 890,0 \text{ kNm}$ → Interaction Bending - shear is required $M_{ed,V,Rd} = M_{ed,Rd} (1 - \rho)$ $\rho = \left(2 \frac{V_{Sd}}{V_{Rd}} - 1 \right)^2 = \left(\frac{2 \cdot 890}{2 \cdot 770,8} - 1 \right)^2 = 0,024$ $M_{ed,V,Rd} = 527,5 (1 - 0,024) = 514,8 \text{ kNm} > 138 \text{ kNm}$	5.3.2; [5.4.5] 5.3.3; [5.4.6] Table 5.16	More accurate method $M_{ed,V,Rd} = [(51,89^2 \cdot 1,5 \cdot 27,5/3 + 19,31^2 \cdot 1,5 \cdot 10,23/3) \cdot (1 - 0,024) + 2,8 \cdot 30 \cdot (19,31 + 2,8/2) \cdot (11,72 + 10,23)/2] / (100 \cdot 1,1) = 519,0 \text{ kNm}$
3.6 Block shear resistance $L_1 = a_1 = 130 \text{ mm} > 5d = 5 \cdot 22 = 110 \rightarrow L_1 = 110 \text{ mm}$ $L_2 = (a_2 - kd_{o,v}) (f_u/f_y) = (36 - 0,5 \cdot 24) 430/275 = 37,5 \text{ mm}$ $L_3 = L_v + a_1 + a_3 = 480 + 130 + 130 = 740 \text{ mm}$ $< (L_v + a_1 + a_3 - nd_{o,v}) (f_u/f_y) = (740 - 9 \cdot 24) 430/275 = 819,3 \text{ mm}$ $L_{v,eff} = L_v + L_1 + L_2 = 480 + 110 + 37,5 = 627,5 \text{ mm} < L_3 = 740 \text{ mm}$ $V_{eff,Rd} = L_{v,eff} t \frac{f_y}{\sqrt{3} \gamma_{Mo}} = 627,5 \cdot 15 \cdot \frac{275}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 1358,6 \text{ kN} > 890 \text{ kN}$	5.5.2; [6.5.2.2]	
3.7 Design shear resistance of side girder $A_{v,net} = (740 - 28) \cdot 15 - 9 \cdot 24 \cdot 15 = 74,4 \cdot 10^2 \text{ mm}^2 \geq A_v \cdot f_y/f_u = (740 - 28) \cdot 15 \cdot 275/430 = 68,3 \cdot 10^2 \text{ mm}^2$ → fastener holes need not be allowed for in shear verification $V_{pl,Rd} = A_v \frac{f_y}{\sqrt{3} \gamma_{Mo}} = 106,8 \cdot 10^2 \cdot \frac{275}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 1541,5 \text{ kN} > 890,0 \text{ kN}$	5.3.3; [5.4.6]	Though it looks like a "long joint", it is not because the rules for long joints refer to load introduction problems only. The progressive introduction of the load rather than simultaneous introduction as is the case here.

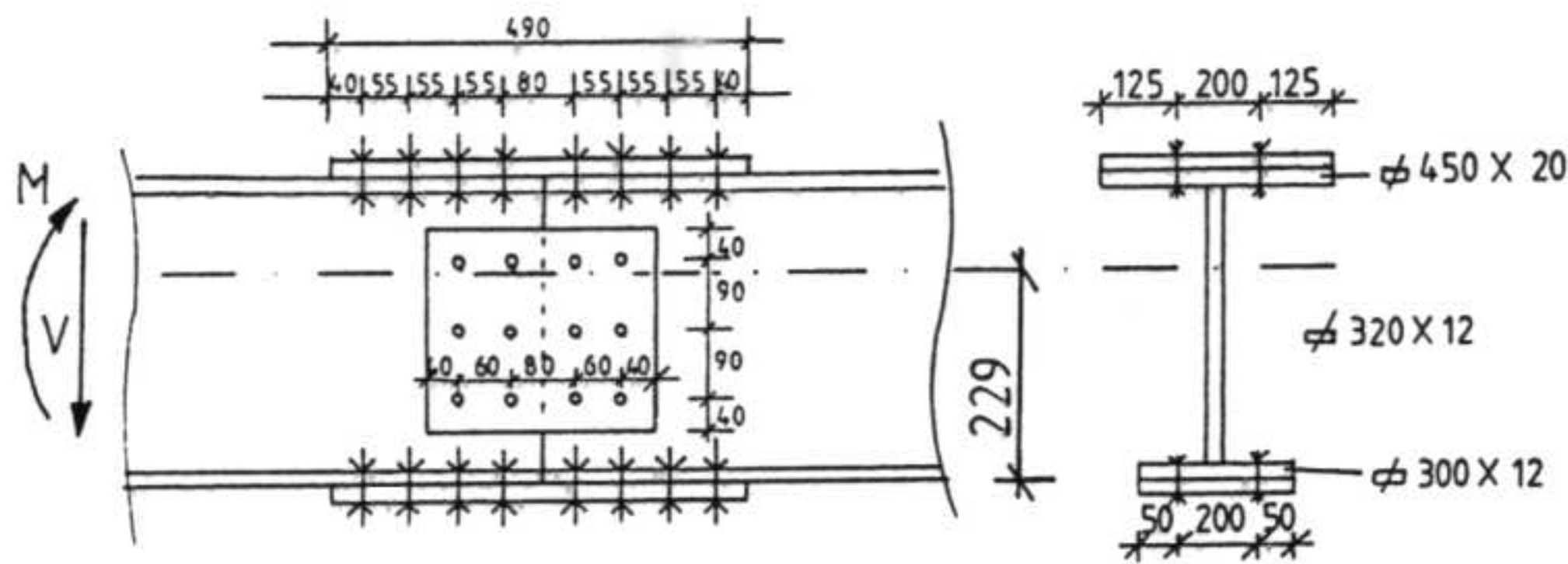
Problem, Sketch, Calculation

Reference

Commentary

Example 3.1.8: Splice of an unsymmetrical I-section

1. System



Member: see sketch
Steel grade: Fe 430
All plates: Steel grade: Fe 430
Bolts: M 16, 8.8 preloaded
→ $d_o = 18$ mm
Surface: class A

2. Loading

Ultimate limit state:
 $V_{Sd} = 237$ kN
 $M_{Sd} = 131$ kNm
Serviceability limit state:
 $V_{Sd} = 158$ kN
 $M_{Sd} = 87,3$ kNm

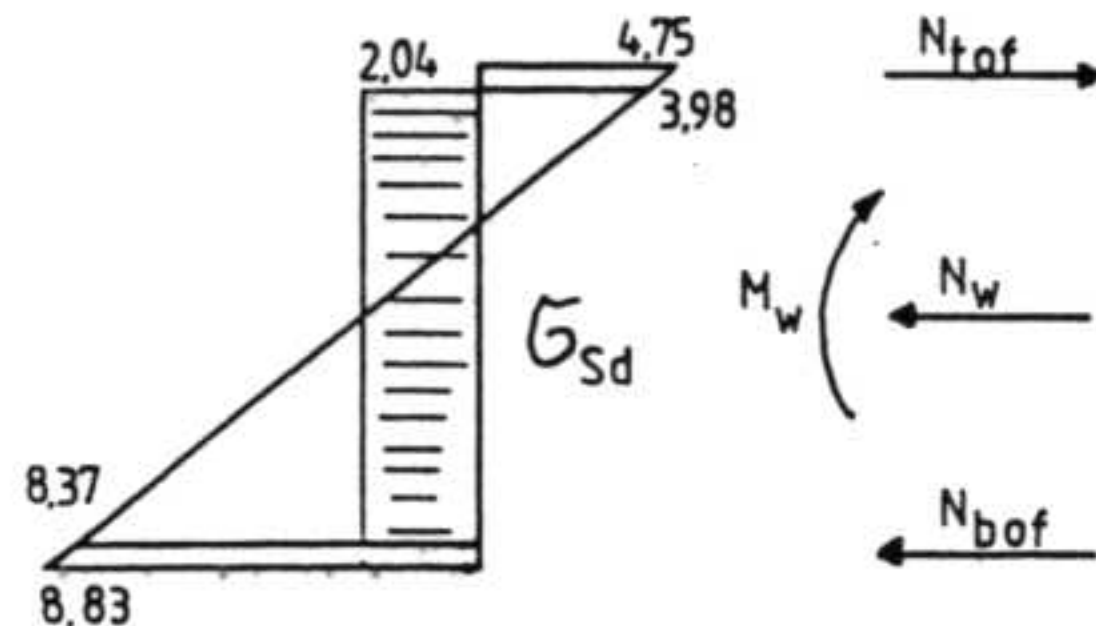
Table 6.7
[7.5.2]

$A = 164,4 \cdot 10^2$ mm²
 $I = 33953 \cdot 10^4$ mm⁴
 $W_{tof} = 2760 \cdot 10^3$ mm³; tof - top flange
 $W_{bof} = 1483 \cdot 10^3$ mm³; bof - bottom flange

flange cover plate (tof): 490 x 450 x 15
flange cover plate (bof): 490 x 300 x 10
web cover plates: 280 x 260 x 10

3. Determination of the resistance of connection

Distribution of action effects on web and flanges



Ultimate limit state:

$N_{tof, Sd} = 392,9$ kN
 $N_{bof, Sd} = 309,6$ kN
 $N_{w, Sd} = 78,3$ kN
 $M_{w, Sd} = 12,3$ kNm
 $V_{w, Sd} = 237$ kN

Serviceability limit state:

$N_{tof, Sd} = 261,9$ kN
 $N_{bof, Sd} = 206,4$ kN
 $N_{w, Sd} = 52,2$ kN
 $M_{w, Sd} = 8,2$ kNm
 $V_{w, Sd} = 158$ kN

3.1 Flange cover plate

3.1.1 Member resistance of the flange cover plate

Tension resistance

gross section: $N_{pl, Rd} = b t \frac{f_y}{\gamma_{Mo}} = 300 \cdot 10 \cdot \frac{275}{1,1} \cdot 10^{-3} = 750,0$ kN > 309,6 kN

net section: $A_{net} = 300 \cdot 10 - 2 \cdot 18 \cdot 10 = 26,4 \cdot 10^2$ mm²

$N_{u, Rd} = 0,9 A_{net} \frac{f_u}{\gamma_{M2}} = 0,9 \cdot 26,4 \cdot 10^2 \cdot \frac{430}{1,25} \cdot 10^{-3} = 817,3$ kN > 309,6 kN

Compression resistance

gross section: $N_{pl, Rd} = b t \frac{f_y}{\gamma_{Mo}} = 450 \cdot 15 \cdot \frac{275}{1,1} \cdot 10^{-3} = 1687,5$ kN > 392,9 kN

5.5.1
[5.4.3]

5.4.2
[5.5.1]

Requirement:
No slip at serviceability limit state

Assumptions:
(1) Shear force is transmitted by the web alone
(2) Due to unsymmetrical section bending moment and normal force exists in web

Problem, Sketch, Calculation	Reference	Commentary
3.1.2 Resistance of the connection Ultimate limit state: Shear resistance of bolts: $F_{v,Rd} = \frac{0,6 f_{ub} A}{\gamma_{Mb}} = \frac{0,6 \cdot 800 \cdot 201}{1,25} \cdot 10^{-3} = 77,2 \text{ kN} > \frac{309,6}{8} = 38,7 \text{ kN}$ Bearing resistance: $e_1/3d_o = 40/(3 \cdot 18) = 0,74$; $p_1/3d_o - 1/4 = 55/(3 \cdot 18) - 1/4 = 0,77$; $\rightarrow \alpha = 0,74$ Flange cover plate: $F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,74 \cdot 430 \cdot 16 \cdot 10}{1,25} \cdot 10^{-3} = 101,8 \text{ kN} > 38,7 \text{ kN}$ Beam flange: Verification not needed because of $t_f > t_p$ Serviceability limit state: Slip resistance: $F_{s,Rd} = \frac{0,7 \cdot \mu \cdot f_{ub} \cdot A_s}{\gamma_{Ms,ser}} = \frac{0,7 \cdot 0,5 \cdot 800 \cdot 157}{1,10} \cdot 10^{-3} = 40,0 \text{ kN} \geq \frac{206,4}{8} = 25,8 \text{ kN}$ $F_{s,Rd} = \frac{17,6 \cdot 2,5}{1,10} = 40,0 \text{ kN} \geq \frac{206,4}{8} = 25,8 \text{ kN}$	6.2.3.2; Table 6.6; [6.5.5; Table 6.5.3] 6.2.3.1 Table 6.7 [6.5.5; Table 6.5.3] 4.3; 6.2.4; Table 4.5 [6.5.8] 6.2.2 [6.5.4]	Assumption: Shear in shank $A = \pi \cdot 16^2 / 4 = 2,01 \cdot 10^2 \text{ mm}^2$ The edge distance e_1 should be taken as the minimum value of both sides when the acting force effects are not parallel to one of the edge distances No slip shall occur at the serviceability limit state Forces acting on the bolt group on each side of the splice. The distribution of internal forces between fasteners need to be proportional to the distance from the centre of rotation in slip-resistant connections $\rightarrow$ polar moment of inertia of the bolt configuration: $I_p = \Sigma y^2 + \Sigma z^2 = 6 \cdot 30^2 + 4 \cdot 90^2 = 378 \cdot 10^2 \text{ mm}^2$ Two web cover plates are used.
3.2 Web cover plate Ultimate limit state: Vertical shear force: $V_{Sd} = 237 \text{ kN}$ Vertical shear force per bolt: $V_{Sd}/n = 237,0/6 = 39,5 \text{ kN}$ Moment due to vertical shear: $M_{Sd} = 237,0 \cdot 70 \cdot 10^{-3} + 12,3 = 28,9 \text{ kNm}$ Horizontal shear force per bolt: $H_{Sd} = 28,9 \cdot 90 \cdot 10^{-3} / 378 \cdot 10^2 + 78,3/6 = 81,9 \text{ kN}$ Resultant shear force at the outer bolt: $F_{Sd} = \sqrt{(39,5^2 + 81,9^2)} = 90,9 \text{ kN}$ Serviceability limit state: Vertical shear force: $V_{Sd} = 158 \text{ kN}$ Vertical shear force per bolt: $V_{b,Sd}/n = 158/6 = 26,3 \text{ kN}$ Torsional moment: $M_{Sd} = 158 \cdot 70 \cdot 10^{-3} + 8,2 = 19,3 \text{ kNm}$ Horizontal shear force per bolt: $H_{Sd} = 19,3 \cdot 90 \cdot 10^{-3} / 378 \cdot 10^2 + 52,2/6 = 54,7 \text{ kN}$ Resultant shear force at the outer bolt: $F_{Sd} = \sqrt{(26,3^2 + 54,7^2)} = 60,7 \text{ kN}$	5.3.3 [5.4.6]	
3.2.1 Member resistance of the web cover plate Shear resistance: $A_{v,net} = 2 \cdot (260 - 3 \cdot 18) \cdot 10 = 41,2 \cdot 10^2 \text{ mm}^2 > 2 \cdot 260 \cdot 10 \cdot 275 / 430 = 33,3 \cdot 10^2 \text{ mm}^2$ $\rightarrow$ no reduction of A_v is required $V_{pl,Rd} = A_v \frac{f_y}{\sqrt{3} \gamma_{Mo}} = 2 \cdot 260 \cdot 10 \cdot \frac{275}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 750,6 \text{ kN} > 237,0 \text{ kN}$ $237,0 \text{ kN} < 0,5 \cdot 750,6 = 375,3 \text{ kN}$ $\rightarrow$ no interaction bending - shear is required		

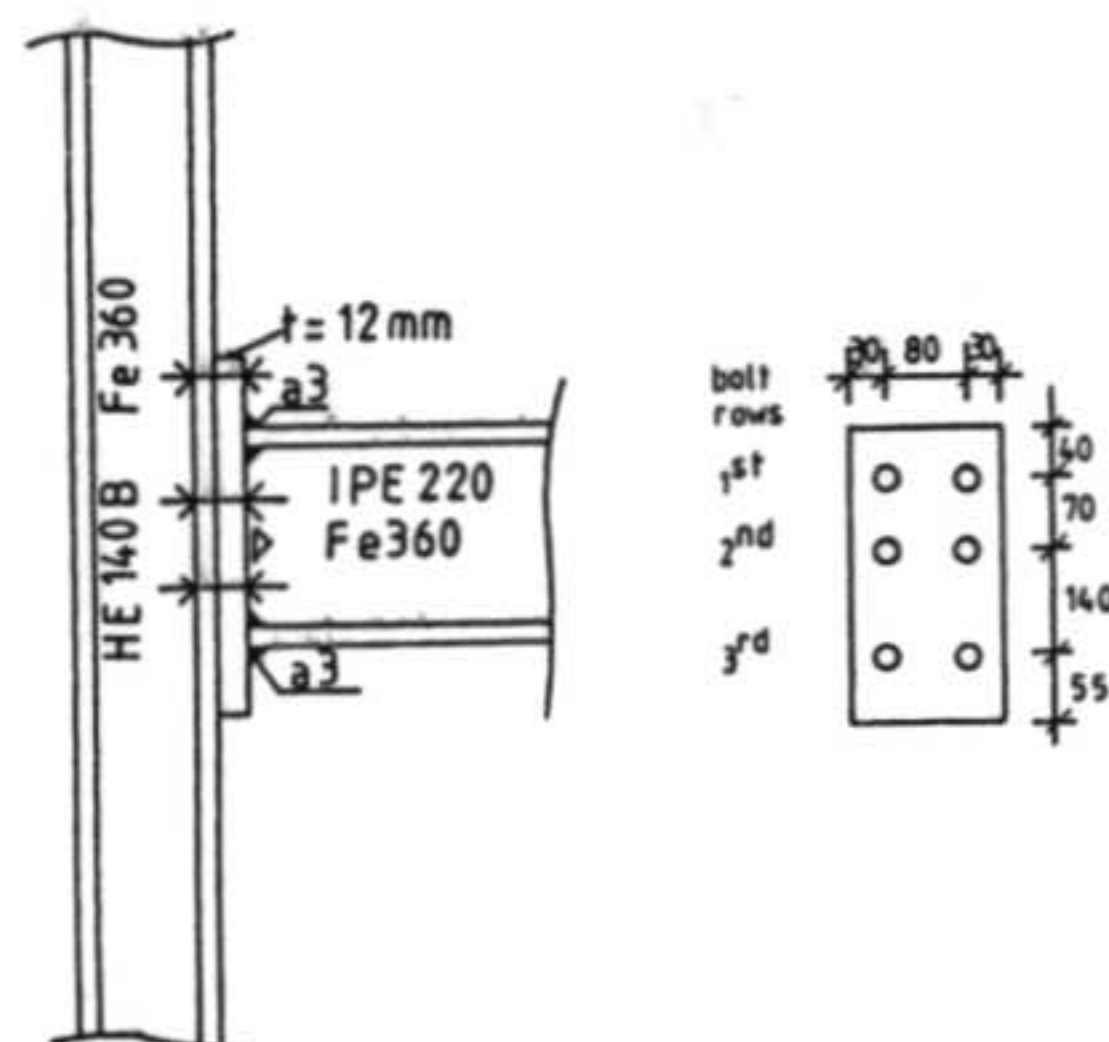
Problem, Sketch, Calculation	Reference	Commentary
3.2.2 Bending resistance of the web cover plate $W_{pl} = 2 \cdot [260^2 \cdot 10/4 + 18 \cdot 10 \cdot 90] = 305,6 \cdot 10^3 \text{ mm}^3$ $M_{pl,Rd} = W_{pl} \frac{f_y}{\gamma_{Mo}} = 305,6 \cdot 10^3 \cdot \frac{275}{1,1} \cdot 10^{-6} = 76,4 \text{ kNm} > 28,9 \text{ kNm}$	5.3.2 [5.4.5]	
3.2.3 Resistance of the connection Ultimate limit state: Shear resistance of bolts: $F_{v,Rd} = m_s \frac{0,6 f_{ub} A}{\gamma_{Mb}} = 2 \frac{0,6 \cdot 800 \cdot 201}{1,25} \cdot 10^{-3} = 154,4 \text{ kN} > 90,9 \text{ kN}$ Bearing resistance: Web cover plate: $e_1/3d_o = 40/(3 \cdot 18) = 0,74$; $p_1/3d_o - 1/4 = 60/(3 \cdot 18) - 1/4 = 0,86$; $\rightarrow \alpha = 0,74$ $F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,74 \cdot 430 \cdot 16 \cdot 10}{1,25} \cdot 10^{-3} = 101,8 \text{ kN} > \frac{90,9}{2} = 45,5 \text{ kN}$ Beam web: $e_1/3d_o = 40/(3 \cdot 18) = 0,74$; $p_1/3d_o - 1/4 = 60/(3 \cdot 18) - 1/4 = 0,86$; $\rightarrow \alpha = 0,74$ $F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 0,74 \cdot 430 \cdot 16 \cdot 12}{1,25} \cdot 10^{-3} = 122,2 \text{ kN} > 90,9 \text{ kN}$	6.2.3.2; Table 6.6; 6.2.3.1 Table 6.7 [6.5.5; Table 6.5.3]	Assumption: Shear in shank. m_s - number of shear planes Here: two shear planes) The edge distance e_1 should be taken as the minimum value of both sides when the acting force effects are not parallel to one of the edge distances The bearing resistance is calculated per member.
Serviceability limit state Slip resistance: $F_{s,Rd} = m_f \frac{0,7 \cdot \mu \cdot f_{ub} \cdot A_s}{\gamma_{Ms,ser}} = 2 \cdot \frac{0,7 \cdot 0,5 \cdot 800 \cdot 157}{1,10} \cdot 10^{-3} = 80,0 \text{ kN} > 60,7 \text{ kN}$ or $F_{s,Rd} = 2 \cdot \frac{17,6 \cdot 2,5}{1,10} = 80,0 \text{ kN} > 60,7 \text{ kN}$	4.3; 6.2.4 [6.5.8]	No slip shall occur at the serviceability limit state m_f = number of friction interfaces Here: Two friction interfaces are available
3.3 Block shear resistance of the beam member $L_1 = a_1 = 90 \text{ mm} > 5d = 5 \cdot 16 = 80 \rightarrow L_1 = 80 \text{ mm}$ $L_2 = (a_2 - kd_{o,1})(f_u/f_y) = (99 - 2,5 \cdot 18)430/275 = 84 \text{ mm}$ $L_3 = L_v + a_1 + a_3 = 180 + 85 + 85 = 350 \text{ mm}$ $< L_v + a_1 + a_3 - nd_{o,v})(f_u/f_y) = (350 - 4 \cdot 18)430/275 = 434,7 \text{ mm}$ $L_{v,eff} = L_v + L_1 + L_2 = 180 + 80 + 84,4 = 344 \text{ mm} < L_3 = 350 \text{ mm}$ $V_{eff,Rd} = L_{v,eff} t \frac{f_y}{\sqrt{3} \gamma_{Mo}} = 344 \cdot 12 \cdot \frac{275}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 595,8 \text{ kN} > 237 \text{ kN}$	5.5.2 [6.5.2.2]	

Problem, Sketch, Calculation

Reference

Commentary

Example 3.1.9: Bolted end plate connection



1. System

Beam: IPE 220, Fe 360
 Column: HE 140 B, Fe 360
 End plate: 305 x 140 x 12, Fe 360
 Bolts: M 16, 8.8

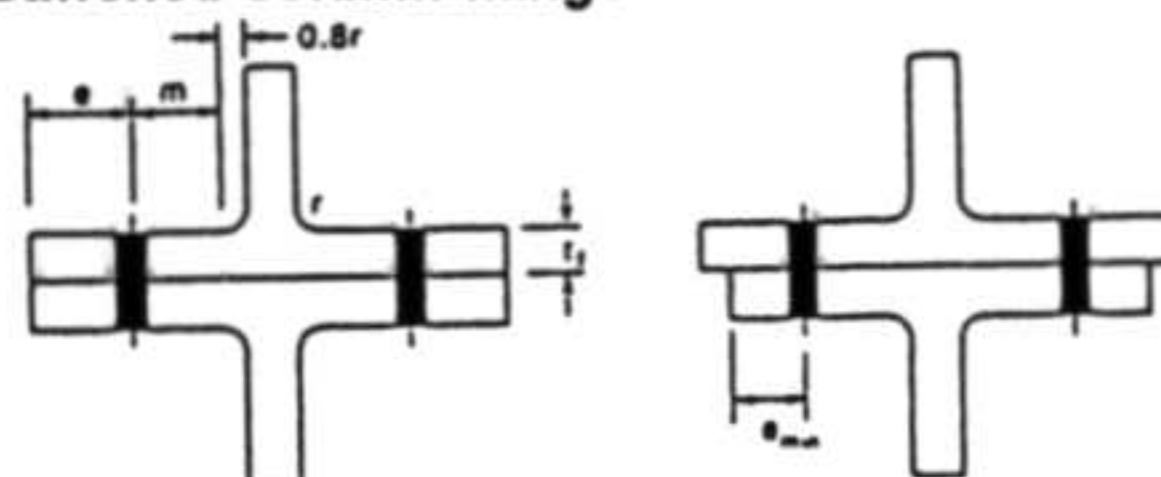
2. Loading

Bending moment: $M_{Sd} = 20,0 \text{ kNm}$
 Shear force: $V_{Sd} = 120 \text{ kN}$

3. Determination of the moment resistance

3.1 Tension zone

3.1.1 Unstiffened column flange

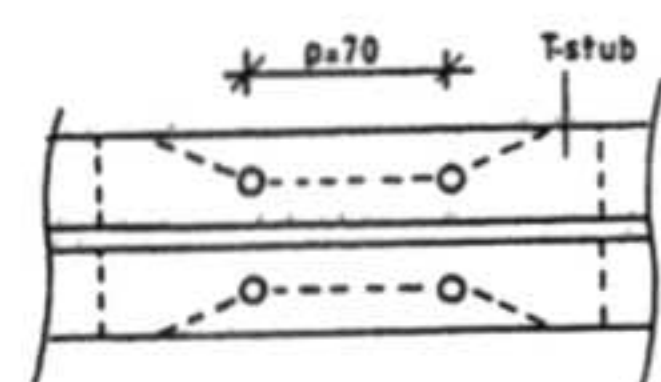


Dimensions of a T-stub

$n = e_{min} = e = 30 \text{ mm}$
 $m = (80 - 7 - 2 \cdot 0,8 \cdot 12) / 2 = 26,9 \text{ mm}$

(1) Determination of the total effective length of the T-stub with 2 bolt rows

Relevant bolt pattern



1st bolt row (end bolt)

$$l_{eff} = 0,5p + 2m + 0,625e$$

2nd bolt row (end bolt)

$$l_{eff} = 0,5p + 2m + 0,625e$$

$l_{eff} = 215,1 \text{ mm}$

$$l_{eff} = 0,5 \cdot 70 + 2 \cdot 26,9 + 0,625 \cdot 30 = 107,55 \text{ mm}$$

$$l_{eff} = 0,5 \cdot 70 + 2 \cdot 26,9 + 0,625 \cdot 30 = 107,55 \text{ mm}$$

Calculation of the plate bending moment of the T-stub

$$M_{pl,Rd} = 0,25 l_{eff} t_f^2 \frac{f_y}{\gamma_{Mo}} k_r ; \text{ assumption: } k_r = 1,0$$

$$= 0,25 \cdot 215,1 \cdot 12^2 \cdot \frac{235}{1,1} \cdot 10^{-6} = 1,65 \text{ kNm}$$

$$t_p = 12 \text{ mm} > 8 \text{ mm} = 0,5d$$

$$\rightarrow B_{t,Rd} = F_{t,Rk} / \gamma_{Mb} = 113,0 / 1,25 = 90,4 \text{ kN}$$

- [Figure J.3.1]

- [J.3.4.1;
Figure J.3.4]

- [J.3.3]

6.2.3.3(2)
Table 6.8
[6.5.5.(4)]

This example illustrates, how a bolted beam-to-column connection is designed by a simplified hand calculation using the T-stub model. When the moment resistance of complex bolted beam-to-column connections should be calculated based on the full procedure given in Annex J it is recommended to develop a software programme.

Simplification:

Calculation of an equivalent T-stub for bolt rows 1 and 2.

The effective length l_{eff} for each row of bolts is based on the yield line patterns and should be taken as the smallest value of the different failure modes. **Here:** The relevant failure mode is the "combined bolt group pattern"

Assumption:

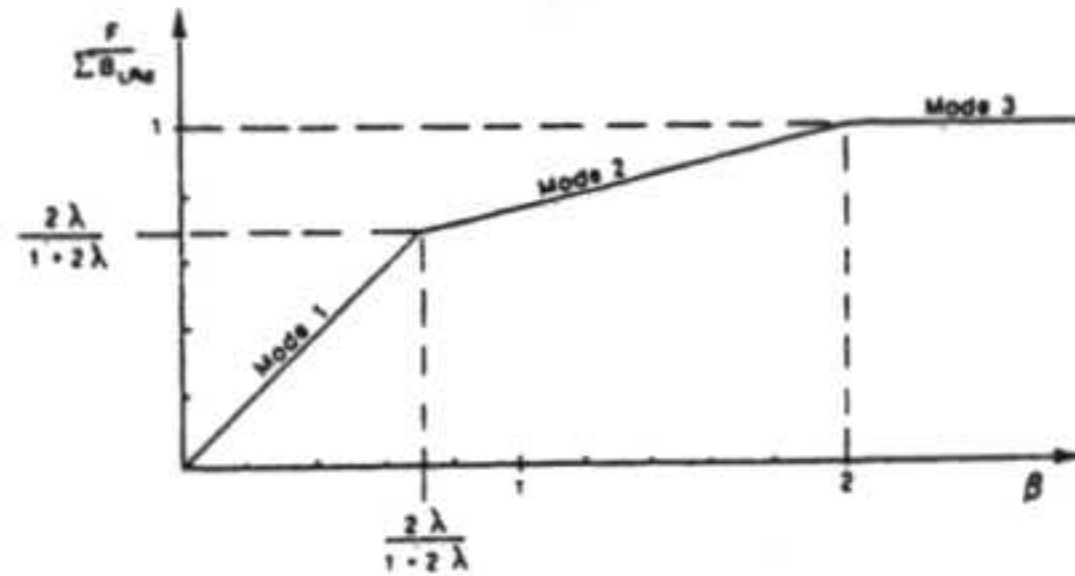
The compressive normal stress $\sigma_{n,Ed}$ in the column flange due to axial force and bending moment does not exceed 180 N/mm^2 at the location of the tension zone.

→ The reduction factor $k_r = 1,0$.

$B_{t,Rd}$ is the design tension resistance of a single bolt plate assembly.

Problem, Sketch, Calculation

Determination of the failure mode of the T-stub



$$\lambda = \frac{n}{m} = \frac{30}{26,9} = 1,115$$

where $n = 50 \text{ mm}$

$$\frac{2\lambda}{1 + 2\lambda} = \frac{2 \cdot 1,115}{1 + 2 \cdot 1,115} = 0,690$$

$$\beta = \frac{4 \cdot M_{pl,Rd}}{m \cdot \sum B_{L,Rd}} = \frac{4 \cdot 1,65}{26,9 \cdot 10^{-3} \cdot 4 \cdot 90,4} = 0,679$$

$$\beta = 0,679 < 0,690 \rightarrow \text{failure mode 1}$$

Design tension resistance of the T-stub

$$F_{t,Rd} = \frac{4 M_{pl,Rd}}{m} = \frac{4 \cdot 1,65}{26,9 \cdot 10^{-3}} = 245,4 \text{ kN}$$

3.1.2 End plate

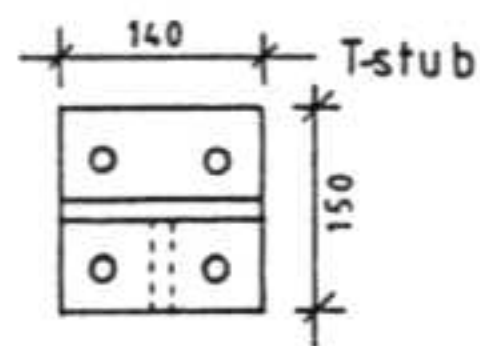
(1) Bolt outside tension flange

Dimensions of a T-stub

$$n = e_{min} = e = 40 \text{ mm}$$

$$m = (70 - 9,2 - 2 \cdot 0,8 \cdot 8 \cdot \sqrt{2}) / 2 = 21,3 \text{ mm}$$

(2) Determination of the total effective length of the T-stub with 2 bolt rows
Relevant bolt pattern



$$\ell_{eff} = b_p$$

$$\ell_{eff} = 140 \text{ mm}$$

Reference

-
[J.3.3;
Figure J.3.3]

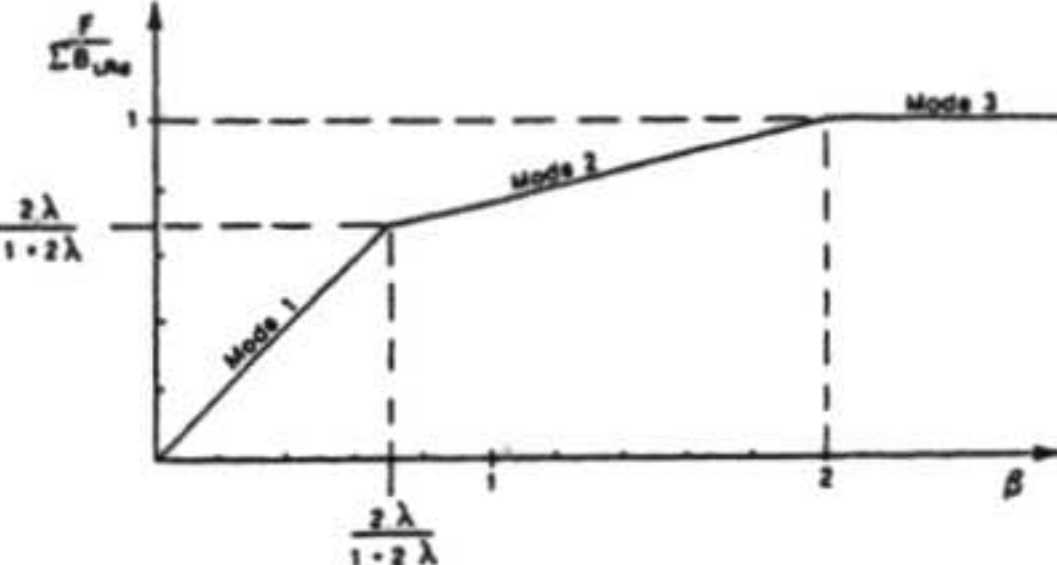
-
[J.3.3;
Figure J.3.2]

-
[J.3.4.4;
Figure J.3.8]

Commentary

The design tension resistance of the T-stub is dependent on the failure mode. The effects of prying forces are taken into account.

The extended end plate is considered as a simple T-stub, see sketch, neglecting the stiffening effect of the beam web.

Problem, Sketch, Calculation	Reference	Commentary
Calculation of the plate bending moment $M_{pl,Rd} = 0,25 t_{eff} t_f^2 \frac{f_y}{\gamma_{Mo}}$ $= 0,25 \cdot 140 \cdot 12^2 \cdot \frac{235}{1,1} \cdot 10^{-8} = 1,08 \text{ kNm}$ $t_p = 12 \text{ mm} > 8 \text{ mm} = 0,5d$ $\rightarrow B_{t,Rd} = F_{t,Rk} / \gamma_{Mb} = 113,0 / 1,25 = 90,4 \text{ kN}$	- [J.3.3] 6.2.3.3.(2); Table 6.8 [6.5.5.(4)]	
Determination of the failure mode of the T-stub  $\lambda = \frac{n}{m} = \frac{40}{21,3} = 1,878$ $\frac{2\lambda}{1+2\lambda} = \frac{2 \cdot 1,878}{1+2 \cdot 1,878} = 0,79$ $\beta = \frac{4 \cdot M_{pl,Rd}}{m_x \cdot \sum B_{t,Rd}}$ $= \frac{4 \cdot 1,08}{21,3 \cdot 10^{-3} \cdot 4 \cdot 90,4} = 0,561$ $\beta = 0,561 < 0,79 \rightarrow \text{failure mode 1}$	- [J.3.3; Figure J.3.3]	
Design tension resistance of the T-stub $F_{t,Rd} = \frac{4 M_{pl,Rd}}{m}$ $= \frac{4 \cdot 1,08}{21,3 \cdot 10^{-3}} = 202,8 \text{ kN}$	- [J.3.3; Figure J.3.2]	The design tension resistance of the T-stub is dependent on the failure mode. The effects of prying forces are taken into account.
3.1.3 Resistance of column web $b_{eff} = 220 \cdot 1,5 = 330 \text{ mm}$ $F_{t,Rd} = t_{wc} b_{eff} \frac{f_{yc}}{\gamma_{Mo}}$ $= 7 \cdot 330 \cdot \frac{235}{1,1} \cdot 10^{-3} = 493,5 \text{ kN}$	- [J.3.4.6]	The effective width b_{eff} of the column web in tension should be taken as equal to the total effective length of the bolt pattern in the tension zone connection.

Problem, Sketch, Calculation	Reference	Commentary
3.2 Resistance of shear zone $d/t = 92/7 = 13,1 < 69 = 69\epsilon \rightarrow$ no shear buckling verification $A_v = 9,8 \cdot 10^2 \text{ mm}^2$ $V_{pl,Rd} = A_v \frac{f_{yc}}{\sqrt{3} \gamma_{Mo}}$ $= 9,8 \cdot 10^2 \cdot \frac{235}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 120,9 \text{ kN}$	5.3.3 [J.3.6]	
3.3 Resistance of compression zone 3.3.1 Crushing resistance $a_p = 3 \text{ mm}$ $b_{eff} = t_{fb} + 2\sqrt{2}a_p + 2t_p + 5(t_{fc} + r_c)$ $= 9,2 + 2\sqrt{2} \cdot 3 + 2 \cdot 12 + 5 \cdot (12 + 12) = 161,7 \text{ mm}$ $F_{t,Rd} = t_{wc} b_{eff} \frac{f_{yc}}{\gamma_{Mo}}$ $= 7 \cdot 161,7 \cdot \frac{235}{1,1} \cdot 10^{-3} = 241,8 \text{ kN}$	- [J.3.5.1.(1)]	For this example it is assumed that $\sigma_{n,Ed} < f_y$. The compression load is introduced by contact $\rightarrow a_p = a_{min} = 3 \text{ mm}$
3.3.2 Buckling resistance $b_{eff} = h_c = 140 \text{ mm}$ $\ell \approx 0,6d = 0,6 \cdot 92 = 55,2 \text{ mm}$ $A = b_{eff} \cdot t_{wc} = 140 \cdot 7 = 9,8 \cdot 10^2 \text{ mm}^2$ $I = b_{eff} t_{wc}^3 / 12 = 140 \cdot 7^3 / 12 = 0,4 \cdot 10^4 \text{ mm}^4$ $i = \sqrt{I/A} = \sqrt{(0,4 \cdot 10^4) / (9,8 \cdot 10^2)} = 2,0 \text{ mm}$ $\bar{\lambda} = \frac{\ell}{i} \frac{1}{\lambda_1}$ $\rightarrow$ using buckling curve c $\rightarrow \chi_c = 0,9542$ $= \frac{55,2}{2,0} \frac{1}{93,9} = 0,29$ $F_{t,Rd} = \chi_c A \frac{f_{yc}}{\gamma_{M1}}$ $= 0,9542 \cdot 9,8 \cdot 10^2 \cdot \frac{235}{1,1} \cdot 10^{-3} = 199,8 \text{ kN}$	5.6.5 [J.3.5.1.(2); 5.7.5] 5.4.2 [5.5.1]	The buckling length of the compression member (column web) is determined from the conditions of lateral and rotational restraint at the flanges at the point of load introduction Non sway mode is assumed $\rightarrow \ell \approx 0,6d$

Problem, Sketch, Calculation	Reference	Commentary
3.4 Moment resistance of the connection Maximum potential resistance of the tension zone, shear zone and compression zone: $\longrightarrow$ shear zone is relevant $\max F_{Rd} = V_{pl,Rd} = 120,9 \text{ kN}$ Distribution of the bolt forces according to the tension resistance of the unstiffened column flange and the end plate The moment resistance of the connection is determined to: $M_{Rd} = \max F_{Rd} h_i = 120,9 \cdot (220 - 9,2) \cdot 10^{-3} = 25,5 \text{ kNm}$ $M_{pl,Rd} = W_{pl,y} f_y / \gamma_{M0} = 285 \cdot 10^3 \cdot 235 \cdot 10^{-6} / 1,1 = 60,9 \text{ kNm}$ $\longrightarrow$ The beam-to-column connection is classified as partial strength [6.4.3.3; 6.9.6.3]	- [J.3.4.5]	
3.5 Shear resistance of the bolted connection Shear resistance of bolts: $F_{v,Rd} = \frac{0,6 f_{ub} A_s}{\gamma_{Mb}} = \frac{0,6 \cdot 800 \cdot 157}{1,25} \cdot 10^{-3} = 60,3 \text{ kN} > \frac{120}{2} = 60,0 \text{ kN}$ or $F_{v,Rd} = \frac{75,4}{1,25} = 60,3 \text{ kN} > \frac{120}{2} = 60,0 \text{ kN}$ Bearing resistance: $e_1/3d_o = 55/(3 \cdot 18) = 1,01$; $p_1/3d_o - 1/4 = 140/(3 \cdot 18) - 1/4 = 2,34$; $\longrightarrow \alpha = 1,0$ $F_{b,Rd} = \frac{2,5 \alpha f_u d t}{\gamma_{Mb}} = \frac{2,5 \cdot 1,0 \cdot 360 \cdot 16 \cdot 12}{1,25} \cdot 10^{-3} = 138,2 \text{ kN} > \frac{120}{2} = 60,0 \text{ kN}$	[6.5.5; Table 6.5.3] 6.2.3.2; Table 6.7 [6.5.5; Table 6.5.3] 6.2.3.1; Table 6.6	The maximum potential design resistance of the column flange is generally not the same as the maximum potential design resistance of the beam end plate. In order to determine the actual design resistance of the tension zone a compatible distribution of bolt-row forces should be obtained. Assumption: Shear in thread Only the lower two bolts are used to transfer the shear force from the beam into the column.
3.6 Determination of the "web weld" resistance $F_{w,Rd} = \frac{f_u}{\sqrt{3} \beta_w \gamma_{Mw}} a \cdot L = \frac{360}{\sqrt{3} \cdot 0,8 \cdot 1,25} \cdot 3 \cdot 177 \cdot 2 \cdot 10^{-3} = 220,7 \text{ kN} > 120 \text{ kN}$ or $F_{w,Rd} = \frac{77,9}{1,25} \cdot \frac{177 \cdot 2}{100} = 220,6 \text{ kN} > 120 \text{ kN}$	- [6.6.5; 6.6.9] 6.3.4.3 Table 6.15	The eccentricity Δe is covered by the design resistances for bolts and need not to be taken into account

3.2 Welded connections

These examples demonstrate the verification of welded connections assuming design values of action effects (N_{sd} , V_{sd} , M_{sd} , etc) which have been calculated by an analysis of the structure and these values already include ψ , γ_F etc.

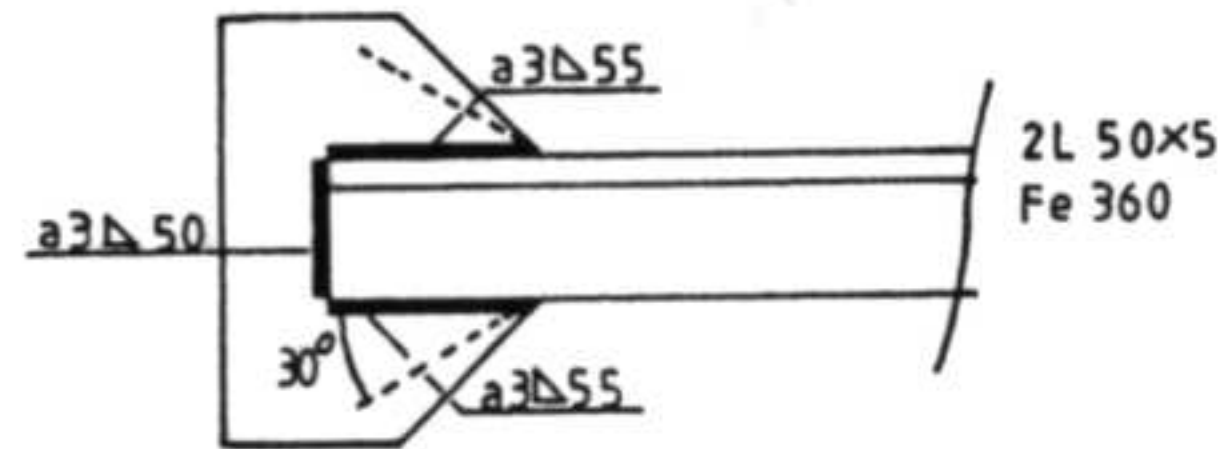
The results presented in the examples are rounded values. For the purpose of easy re-calculation each formula and each check is calculated with the rounded values.

The following examples are included in this chapter:

- | | |
|-----------------------|---|
| Example 3.2.1: | Double angle welded to a gusset plate |
| Example 3.2.2: | Bracket welded on a column |
| Example 3.2.3: | Welded beam to column connection without stiffeners |
| Example 3.2.4: | Welded beam to column connection with stiffeners |
| Example 3.2.5: | Hollow section lattice girder joint |

Problem, Sketch, Calculation

Example 3.2.1: 2 angles welded on a gusset plate



1. System

Gusset plate:	t = 10 mm	Steel grade:	Fe 360
Angle:	2 L 50 x 5	Steel grade:	Fe 360
throat thickness:	a = 3 mm		
weld length:	$\ell = 4 \times 55 + 2 \times 50 = 320 \text{ mm}$		

2. Loading

Tension force: $N_{sd} = 188 \text{ kN}$

3. Determination of the resistance of the connection

3.1 Weld resistance

$$\max \left\{ \begin{array}{l} 40 \text{ mm} \\ 6 \cdot a = 6 \cdot 3 = 18 \text{ mm} \end{array} \right\} < 55 \text{ mm} = \ell < 450 \text{ mm} = 150 \cdot a$$

$$F_{w,Rd} = \frac{f_u}{\sqrt{3} \beta_w \gamma_{Mw}} a \cdot L = \frac{360}{\sqrt{3} \cdot 0,8 \cdot 1,25} \cdot 3 \cdot 320 \cdot 10^{-3} = 199,5 \text{ kN} > 188 \text{ kN}$$

or

$$F_{w,Rd} = \frac{77,9}{1,25} \cdot \frac{320}{100} = 199,4 \text{ kN} > 188 \text{ kN}$$

3.2 Check of Tear-out

$$N_{Rd} = 2 A_t \frac{f_y}{\sqrt{3} \gamma_{M0}} + A_g \frac{f_y}{\gamma_{M0}} = \left(2 \cdot 55 \cdot 10 \cdot \frac{235}{\sqrt{3} \cdot 1,1} + 50 \cdot 10 \cdot \frac{235}{1,1} \right) \cdot 10^{-3} = 242,5 \text{ kN} > 188 \text{ kN}$$

As an alternative the check of Tension resistance of the gusset plate may be done by assuming a load dispersion with an angle of 30°, see sketch.

$$N_{L,Rd} = A \frac{f_y}{\gamma_{M0}} = (2 \cdot 55 \cdot \tan 30^\circ + 50) \cdot 10 \cdot \frac{235}{1,1} \cdot 10^{-3} = 242,5 \text{ kN} > 188 \text{ kN}$$

Reference

-
[6.6.10]

-
[6.6.5.2.(2)]

-
[6.6.5; 6.6.9]

6.3.4.3
Table 6.15

Commentary

In Angles connected by one leg, the welded lap joint end connection may be allowed for by adopting an effective cross-sectional area and then treating as concentrically loaded, as presented in this example. A more accurate method can also be used by considering the eccentricity.

For an equal-leg angle the effective area is taken as equal to the gross section.

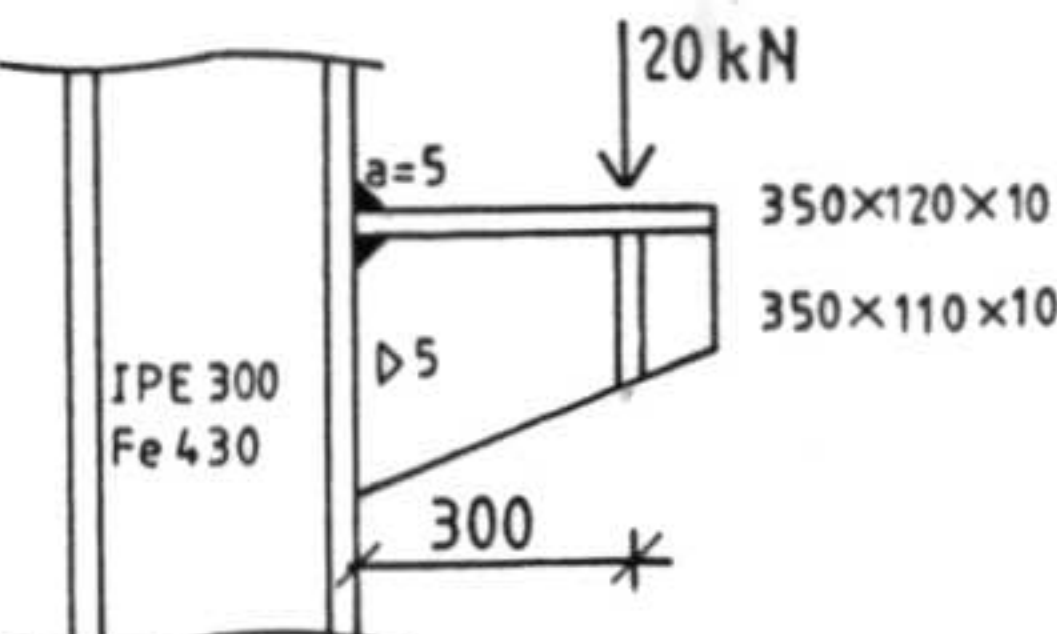
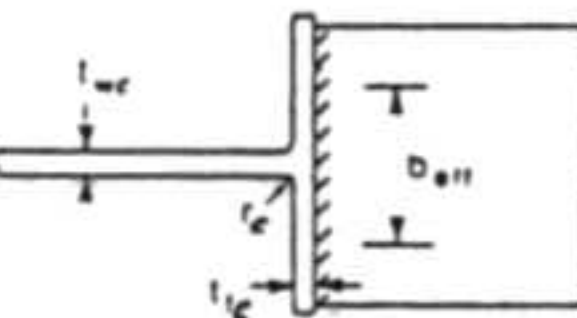
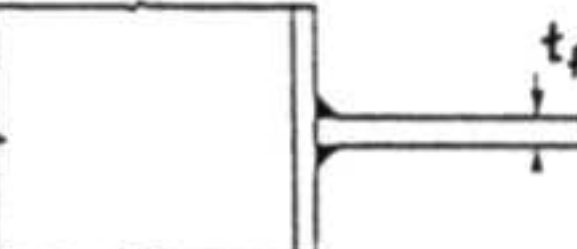
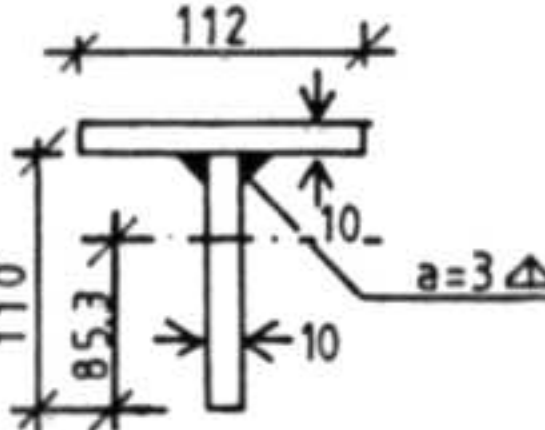
The throat thickness of a fillet weld should not be less than 3 mm

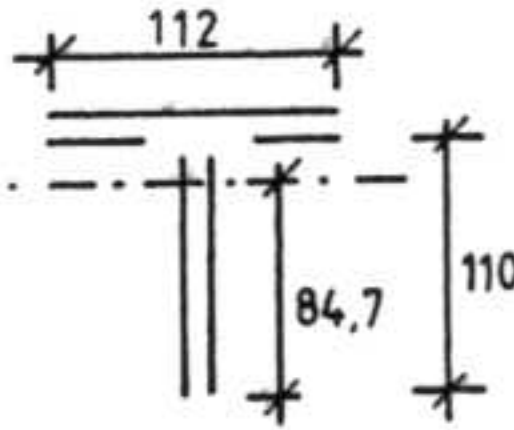
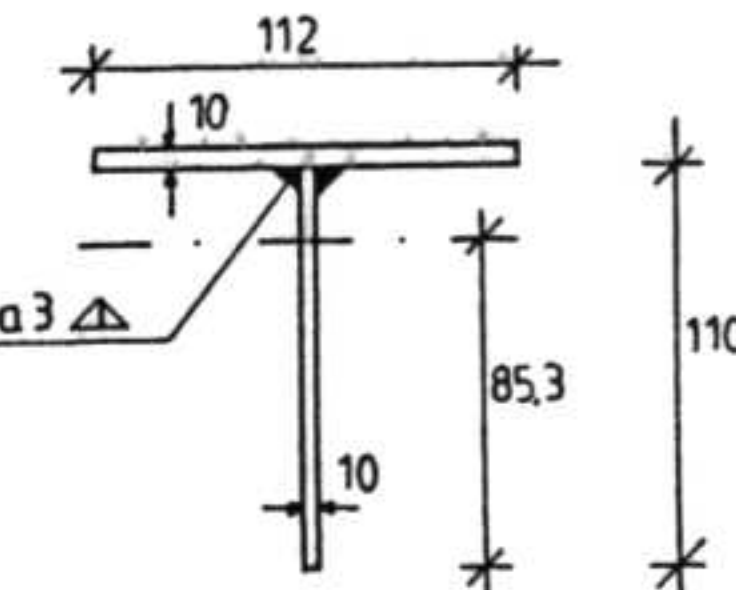
The design force for checking fillet welds should be taken as a resultant of the forces to be transmitted by the weld.

By using the alternative method [Annex M] the resistance of the weld is:

$$F_{w,Rd} = 213,6 \text{ kN}$$

Model used is based on yield strength.

Problem, Sketch, Calculation	Reference	Commentary
Example 3.2.2: Bracket welded on a column 1. System  Column: IPE 300 Steel grade: Fe 430 Bracket: plate 350x120x10 Steel grade: Fe 430 Bracket: plate 350x110x10 Steel grade: Fe 430 Throat thicknesses: Connection: Flange and web: a = 5 mm Section: a = 3 mm 2. Loading Single load: $P = 15,0 \text{ kN} \rightarrow M_{sd} = 0,3 \cdot 15 = 4,5 \text{ kNm}$ 3. Determination of the weld resistance: 3.1 Effective breadth in tension zone:  $b_{eff} = \min \left\{ \begin{aligned} &t_{wc} + 2r_c + 7t_{tc} = 7,1 + 2 \cdot 15 + 7 \cdot 10,7 = 112 \text{ mm} \\ &t_{wc} + 2r_c + 7 \frac{t_{tc}^2}{t_{tb}} \frac{f_{yc}}{f_{yb}} = 7,1 + 2 \cdot 15 + 7 \cdot \frac{10,7^2}{10} = 117 \text{ mm} \end{aligned} \right.$ $b_{eff} = 112 \text{ cm} < 120 \text{ cm} = b_c$ $> 0,7 \cdot 120 = 84 \text{ mm}$ $\rightarrow$ the joint need not be stiffened  3.2 Cross-section verification  $e_w = \frac{112 \cdot 10 \cdot 115 + 110 \cdot 10 \cdot 55}{112 \cdot 10 + 110 \cdot 10} = 85,3 \text{ mm}$ $I = 110^3 \cdot 10 / 12 + 29,7^2 \cdot 112 \cdot 10 + 30,3^2 \cdot 110 \cdot 10 = 310,7 \cdot 10^4 \text{ mm}^4$ $W_u = 310,7 \cdot 10^4 / 85,3 = 36,4 \cdot 10^3 \text{ mm}^3$ $M_{el,Rd} = W_{el} \cdot \frac{f_y}{\gamma_{M0}} = 36,4 \cdot 10^3 \cdot \frac{275}{1,1} \cdot 10^{-6} = 9,1 \text{ kNm} > 4,5 \text{ kNm}$ $V_{pl,Rd} = \frac{A_v \cdot f_y}{\sqrt{3} \gamma_{M0}} = \frac{110 \cdot 10 \cdot 275}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 158,8 \text{ kN} > 15 \text{ kN}$	6.3.6 [6.6.8] 5.3.2 [5.4.5] 5.3.3 [5.4.6]	Elastic stress distribution in cross-section and in the weld is assumed. Load factors are included A reduced effective breadth shall be taken into account both for the parent material and for the welds. Elastic stress distribution in the cross-section is assumed. No interaction bending - shear is required because of $V_{sd} = 15 \text{ kN} < 0,5V_{pl,Rd} = 79,4 \text{ kN}$ (5.3.4 [5.4.7])

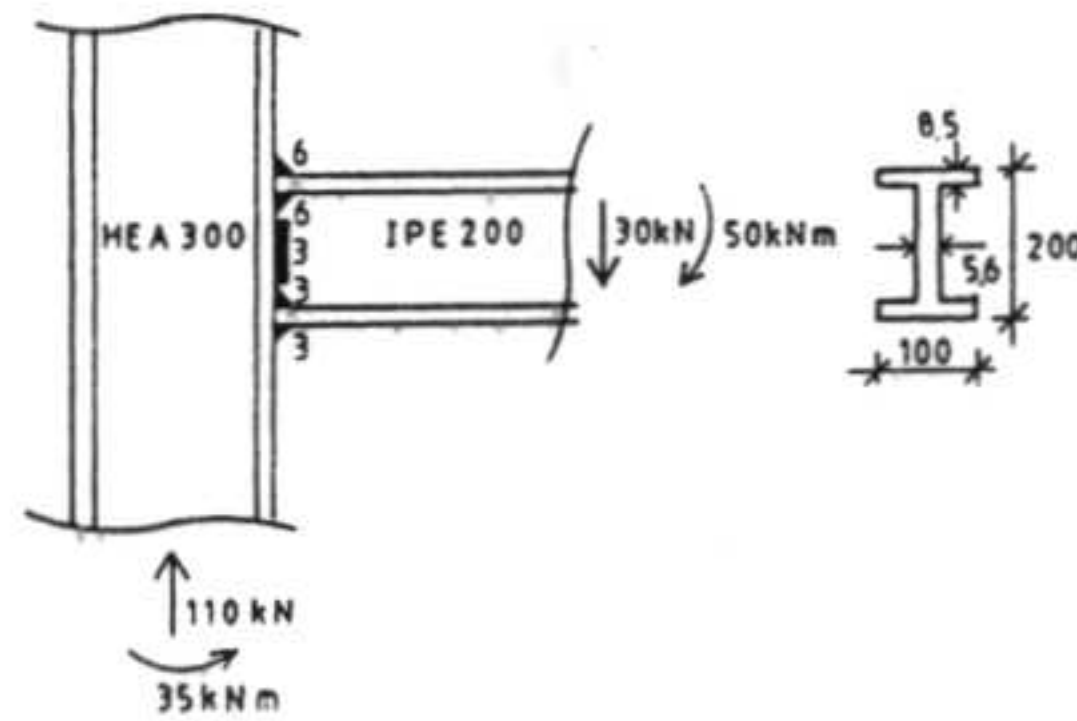
Problem, Sketch, Calculation	Reference	Commentary
3.3 Area and section modulus of the weld  $A_{wf} = 5 \cdot 112 + 5 \cdot (112 - 10) = 10,7 \cdot 10^2 \text{ mm}^2$ $A_{ww} = 2 \cdot 110 \cdot 5 = 11,0 \cdot 10^2 \text{ mm}^2$ $e_w = \frac{112 \cdot 5 \cdot 120 + (112 - 10) \cdot 5 \cdot 110 + 110 \cdot 5 \cdot 2 \cdot 55}{10,7 \cdot 10^2 + 11,0 \cdot 10^2} = 84,7 \text{ mm}$ $I_w = 2 \cdot 110^3 \cdot 5 / 12 + 2 \cdot 29,7^2 \cdot 110 \cdot 5 + 25,3^2 \cdot (112 - 10) \cdot 5 + 35,3^2 \cdot 112 \cdot 5 = 221,6 \cdot 10^4 \text{ mm}^4$ $W_{w,el} = 221,6 \cdot 10^4 / 84,7 = 26,2 \cdot 10^3 \text{ mm}^3$ 3.4 Determination of the stresses in the weld $\tau_{Sd} = 15 \cdot 10^3 / 11,0 \cdot 10^2 = 13,6 \text{ N/mm}^2$ $\sigma_{Sd} = 4,5 \cdot 10^6 / 26,2 \cdot 10^3 = 171,8 \text{ N/mm}^2$ $\sigma_{w,Sd} = \sqrt{(13,6^2 + 171,8^2)} = 172,3 \text{ N/mm}^2$ 3.5 Verification $f_{w,Rd} = \frac{f_u}{\sqrt{3} \gamma_{Mw} \beta_w}$ $= \frac{430}{\sqrt{3} \cdot 0,85 \cdot 1,25} = 233,7 \text{ N/mm}^2 > 172,3 \text{ N/mm}^2$ 3.6 Weld of the section  $S = 112 \cdot 10 \cdot 28,7 = 32,1 \cdot 10^3 \text{ mm}^3$ $\tau_{Sd} = \frac{V_{Sd} \cdot S}{I \cdot 2a} = \frac{15 \cdot 32,1 \cdot 10^3}{310,7 \cdot 10^4 \cdot 2 \cdot 3} \cdot 10^3 = 25,8 \text{ N/mm}^2 < 233,7 \text{ N/mm}^2$	6.3.4.3 [6.6.5.3]	As elastic stress distribution is assumed in the cross-section, it is also assumed for the weld. Assumption: Elastic stress distribution in weld No deformation capacity is necessary for a bracket. → flange weld need not to develop the full design resistance of the beam flange.
		Elastic stress distribution is assumed.

Problem, Sketch, Calculation

Reference

Commentary

Example 3.2.3: Welded beam-to-column connection without stiffeners



1. System

column:	HE 300 A	Steel grade:	Fe 360
beam:	IPE 200	Steel grade:	Fe 360
throat thickness:			
tension flange:	a = 6 mm		
web:	a = 3 mm		
compression flange:	a = 3 mm		

2. Loading

Bending moment beam:	$M_{b,Ed} = 45 \text{ kNm}$
Bending moment column:	$M_{b,Ed} = 35 \text{ kNm}$
Shear force:	$V_{b,Ed} = 30 \text{ kN}$
Axial force:	$N_{c,Ed} = 110 \text{ kN}$

3. Determination of the moment resistance

3.1 Tension zone:

3.1.1 Unstiffened column flange:

$$F_{t,Rd} = (f_{yb} t_{fb} (t_{wc} + 2r_c) + 7 (f_{yc} t_{lc}^2)) / \gamma_{Mo}$$

$$= (235 \cdot 8,5 \cdot (8,5 + 2 \cdot 27) + 7 \cdot (235 \cdot 14^2)) \cdot 10^{-3} / 1,1$$

$$= 406,6 \text{ kN}$$

or

$$F_{t,Rd} \leq f_{yb} t_{fb} (t_{wc} + 2r_c + 7 t_{lc}) / \gamma_{Mo}$$

$$= 235 \cdot 8,5 \cdot (8,5 + 2 \cdot 27 + 7 \cdot 14) \cdot 10^{-3} / 1,1$$

$$= 291,5 \text{ kN}$$

---> $F_{t,Rd} = 291,5 \text{ kN} > 127,1 \text{ kN} = 0,7 \cdot 235 \cdot 8,5 \cdot 100 \cdot 10^{-3} / 1,1 = 0,7 f_{yb} t_{fb} b_{fb} / \gamma_{Mo}$

---> no stiffening required

Verification of the weld in tension zone:

$$L = 100 + 100 - 5,6 - 2 \cdot 12 = 170,4 \text{ mm}$$

$$N_{pl,Rd} = b_{fb} t_{fb} \frac{f_{yb}}{\gamma_{Mo}} = \frac{100 \cdot 8,5 \cdot 235}{1,1} \cdot 10^{-3} = 181,6 \text{ kN}$$

$$F_{w,Rd} = \frac{a L}{\sqrt{3} \beta_w} \frac{f_u}{\gamma_{Mw}} = \frac{6 \cdot 170,4 \cdot 360}{\sqrt{3} \cdot 0,8 \cdot 1,25} \cdot 10^{-3} = 212,5 \text{ kN}$$

3.1.2 Unstiffened column web:

$$F_{t,Rd} = f_{yc} t_{wc} b_{eff} / \gamma_{Mo}$$

$$b_{eff} = t_{fb} + 2\sqrt{2} a_b + 5 (t_{lc} + r_c)$$

$$b_{eff} = 8,5 + 2\sqrt{2} \cdot 6 + 5 (14 + 27) = 230,5 \text{ mm}$$

$$F_{t,Rd} = 235 \cdot 8,5 \cdot 230,5 \cdot 10^{-3} / 1,1 = 418,6 \text{ kN}$$

---> Resistance of the tension zone: $F_{t,Rd} = 291,5 \text{ kN}$

6.3.6
[J.2.3.1.(1)]

[J.2.3.2.(2)]

6.3.6.(5)
[J.2.3.1.(3)]

-
[J.2.3.2]

The welds connecting the beam flange to the column should be designed to develop the full design resistance of the beam flange equal to $f_{yb} \cdot t_{fb} \cdot b_{fb} / \gamma_{Mo}$

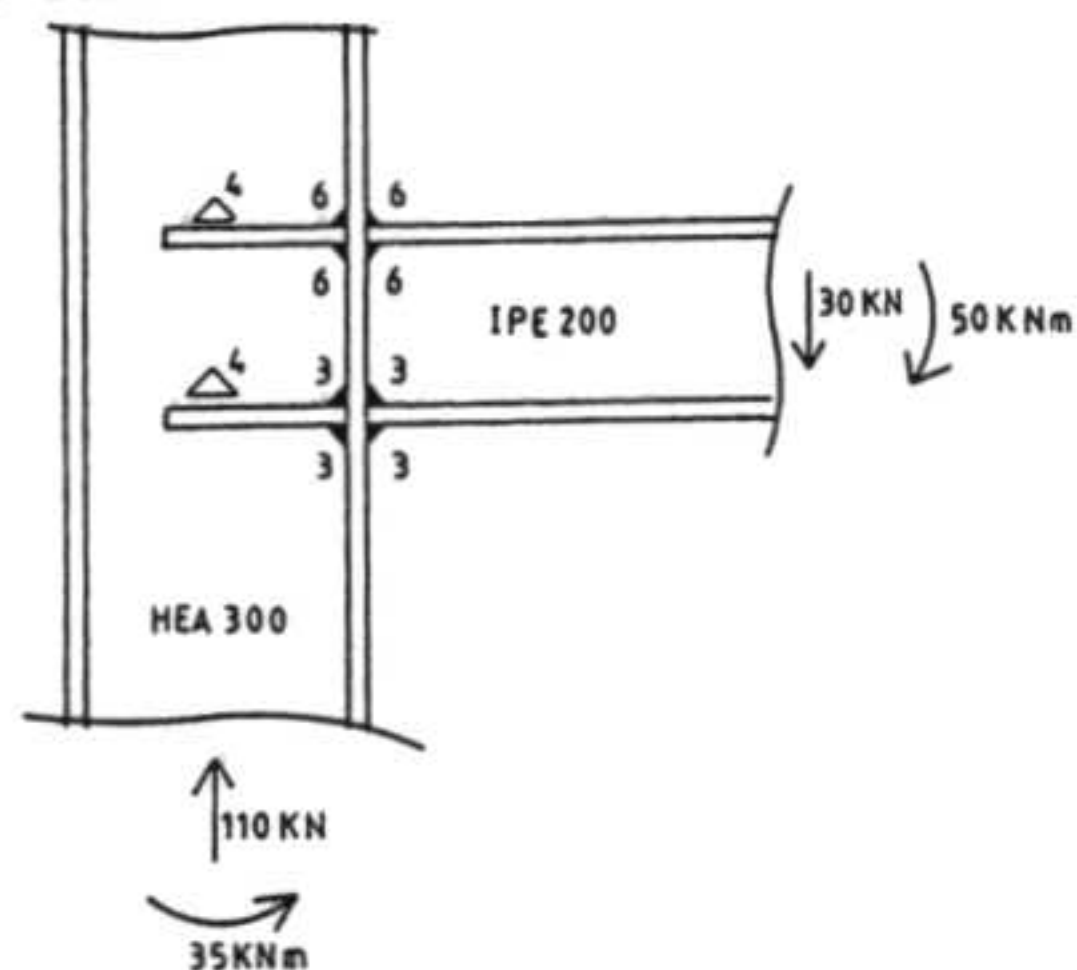
Problem, Sketch, Calculation	Reference	Commentary
3.3 Resistance of the shear zone: $d/t = 208/8,5 = 24,5 < 69 = 69\epsilon \rightarrow$ no shear buckling verification required $V_{Rd} = \frac{f_{yc} A_v}{\sqrt{3} \gamma_{Mo}} = \frac{235 \cdot 24,6 \cdot 10^2}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 303,4 \text{ kN}$ $\rightarrow$ Resistance of the shear zone: $F_{c,Rd} = 303,4 \text{ kN}$ 3.4 Bending moment of the connection: $\rightarrow M_{Rd} = F_{min} (h_b - t_{fb})$ Potential failure modes: Tension zone $F_{t,Rd} = 291,5 \text{ kN}$ Compression zone $F_{c,Rd} = 361,9 \text{ kN}$ Shear zone $F_{v,Rd} = 303,4 \text{ kN}$ $\rightarrow F_{min} = F_{t,Rd} = 291,5 \text{ kN}$ $M_{Rd} = 291,5 \cdot (200 - 8,5) \cdot 10^{-3} = 55,8 \text{ kNm} > 45,0 \text{ kNm}$ 3.5 Cross-section resistance of the beam Shear resistance: $V_{Rd} = \frac{A_v f_y}{\sqrt{3} \gamma_{Mo}} = \frac{11,2 \cdot 10^2 \cdot 235}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 138,1 \text{ kN} > 30 \text{ kN}$ $0,5V_{Rd} = 69,1 \text{ kN} > 30 \text{ kN} = V_{sd} \rightarrow$ no interaction bending - shear is needed Moment resistance: $M_{Rd} = W_{pl} \frac{f_y}{\gamma_{Mo}} = 221 \cdot 10^3 \cdot \frac{235}{1,1} \cdot 10^{-6} = 47,2 \text{ kNm} > 45 \text{ kNm}$ 3.6 Verification of the "web weld" $F_{w,Rd} = \frac{f_u}{\sqrt{3} \beta_w \gamma_{Mw}} a \cdot L = \frac{360}{\sqrt{3} \cdot 0,8 \cdot 1,25} \cdot 3 \cdot 159,2 \cdot 10^{-3} = 198,3 \text{ kN} > 110 \text{ kN}$ or $F_{w,Rd} = \frac{77,9}{1,25} \cdot \frac{2 \cdot 159}{100} = 198,2 \text{ kN} > 110 \text{ kN}$	5.3.3 [J.2.5] 5.3.3 [5.4.6] 5.3.4 [5.4.7] 5.3.2 [5.4.5] [6.6.5.3] 6.3.4.3 Table 6.15	

Problem, Sketch, Calculation

Reference

Commentary

Example 3.2.4: Welded beam-to-column connection with stiffeners



1. System

column:	HE 300 A	Steel grade:	Fe 360
beam:	IPE 200	Steel grade:	Fe 360
throat thickness:			
tension flange:		a =	6 mm
web:		a =	3 mm
compression flange:		a =	3 mm

2. Loading

Bending moment beam:	$M_{b,Sd} = 45 \text{ kNm}$
Bending moment column:	$M_{b,Sd} = 35 \text{ kNm}$
Shear force:	$V_{b,Sd} = 30 \text{ kN}$
Axial force:	$N_{c,Rd} = 110 \text{ kN}$

3. Determination of the moment resistance

3.1 Tension and compression zone:

The design resistance of a stiffened column subject to a transverse tensile or compressive force is at least equal to the design resistance of the beam flange, provided the thickness of the stiffener is not smaller than the flange thickness of the beam. The welds need sufficient deformation capacity.

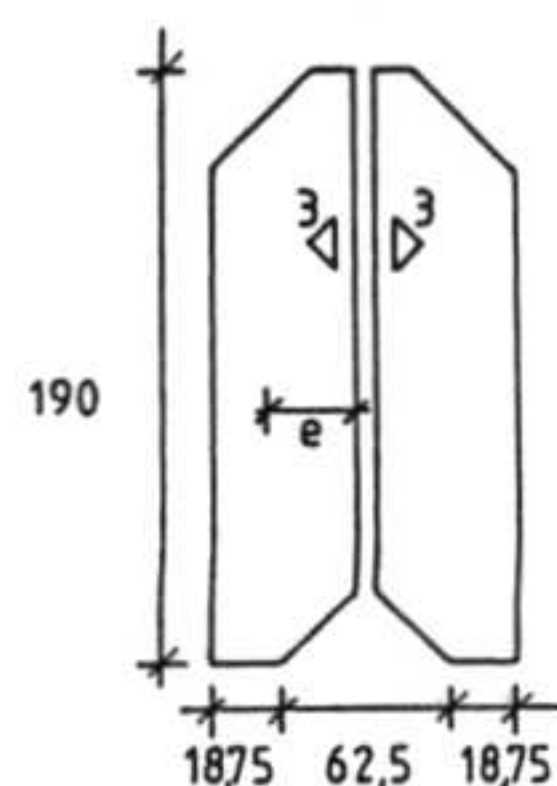
Verification of the weld between the stiffeners and the column flanges in tension zone:

$$L = 100 + 100 - 5,6 - 2 \cdot 12 = 170,4 \text{ mm}$$

$$N_{pl,Rd} = b_{fb} t_{fb} \frac{f_{yb}}{\gamma_{Mo}} = \frac{100 \cdot 8,5 \cdot 235}{1,1} \cdot 10^{-3} = 181,6 \text{ kN}$$

$$F_{w,Rd} = \frac{a L}{\sqrt{3} \beta_w} \frac{f_u}{\gamma_{Mw}} = \frac{6 \cdot 170,4 \cdot 360}{\sqrt{3} \cdot 0,8 \cdot 1,25} \cdot 10^{-3} = 212,5 \text{ kN}$$

Verification of the weld between the stiffeners and the column web



$$f_{w,Rd} = \frac{f_u}{\sqrt{3} \beta_w \gamma_{Mw}} = \frac{360}{\sqrt{3} \cdot 0,8 \cdot 1,25} = 207,8 \text{ N/mm}^2$$

$$\begin{aligned} \max F_{st,Sd} &= f_{yb} t_{fb} b_{fb} / \gamma_{Mo} = 235 \cdot 8,5 \cdot 18,75 \cdot 10^{-3} / 1,1 = 34,0 \text{ kN}; A_w = 2 \cdot 160 \cdot 3 = 9,6 \cdot 10^2 \text{ mm}^2 \\ M_{Sd} &= \max F_{st,Sd} \cdot e = 34,0 \cdot 36,375 \cdot 10^{-3} = 1,24 \text{ kNm}; W_w = 2 \cdot 160^2 \cdot 3 / 6 = 25,6 \cdot 10^3 \text{ mm}^3 \\ \tau_{Sd} &= 34,0 \cdot 10^3 / 9,6 \cdot 10^2 = 35,4 \text{ N/mm}^2 \\ \sigma_{Sd} &= 1,24 \cdot 10^6 / 25,6 \cdot 10^3 = 48,4 \text{ N/mm}^2 \\ \sigma_{w,Sd} &= \sqrt{(35,4^2 + 48,4^2)} = 60,0 \text{ N/mm}^2 < 207,8 \text{ N/mm}^2 = f_{vw,d} \end{aligned}$$

6.3.6
[J.2.3.1.(1)]

[J.2.3.2.(2)]

6.3.6.(5)
[J.2.3.1.(3)]

-
[J.2.3.3]

The welds connecting the beam flange to the column should be designed to develop the full design resistance of the beam flange equal to $f_{yb} \cdot t_{fb} \cdot b_{fb} / \gamma_{Mo}$

st - stiffener

Problem, Sketch, Calculation	Reference	Commentary
3.3 Resistance of the shear zone: $d/t = 208/8,5 = 24,5 < 69 = 69\epsilon \longrightarrow$ no shear buckling verification required $V_{Rd} = \frac{f_{yc} A_v}{\sqrt{3} \gamma_{Mo}} = \frac{235 \cdot 24,6 \cdot 10^2}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 303,4 \text{ kN}$ $\longrightarrow$ Resistance of the shear zone: $F_{c,Rd} = 303,4 \text{ kN}$	5.3.3 [J.2.5]	
3.4 Bending moment of the connection: $\longrightarrow M_{Rd} = F_{min} (h_b - t_{fb})$ Potential failure modes: Shear zone $F_{v,Rd} = 303,4 \text{ kN}$ $\longrightarrow F_{min} = F_{t,Rd} = 303,4 \text{ kN}$ $M_{Rd} = 303,4 \cdot (200 - 8,5) \cdot 10^{-3} = 58,1 \text{ kNm} > 45,0 \text{ kNm}$		
3.5 Cross-section resistance of the beam Shear resistance: $V_{Rd} = \frac{A_v f_y}{\sqrt{3} \gamma_{Mo}} = \frac{11,2 \cdot 10^2 \cdot 235}{\sqrt{3} \cdot 1,1} \cdot 10^{-3} = 138,1 \text{ kN} > 30 \text{ kN}$ $0,5V_{Rd} = 69,1 \text{ kN} > 30 \text{ kN} = V_{sd} \longrightarrow$ no interaction bending - shear is needed Moment resistance: $M_{Rd} = W_{pl} \frac{f_y}{\gamma_{Mo}} = 221 \cdot 10^3 \cdot \frac{235}{1,1} \cdot 10^{-6} = 47,2 \text{ kNm} > 45 \text{ kNm}$ $\longrightarrow$ the beam-to-column connection is classified as full strength [6.4.3.2; 6.9.6.3]	5.3.3 [5.4.6] 5.3.4 [5.4.7] 5.3.2 [5.4.5]	

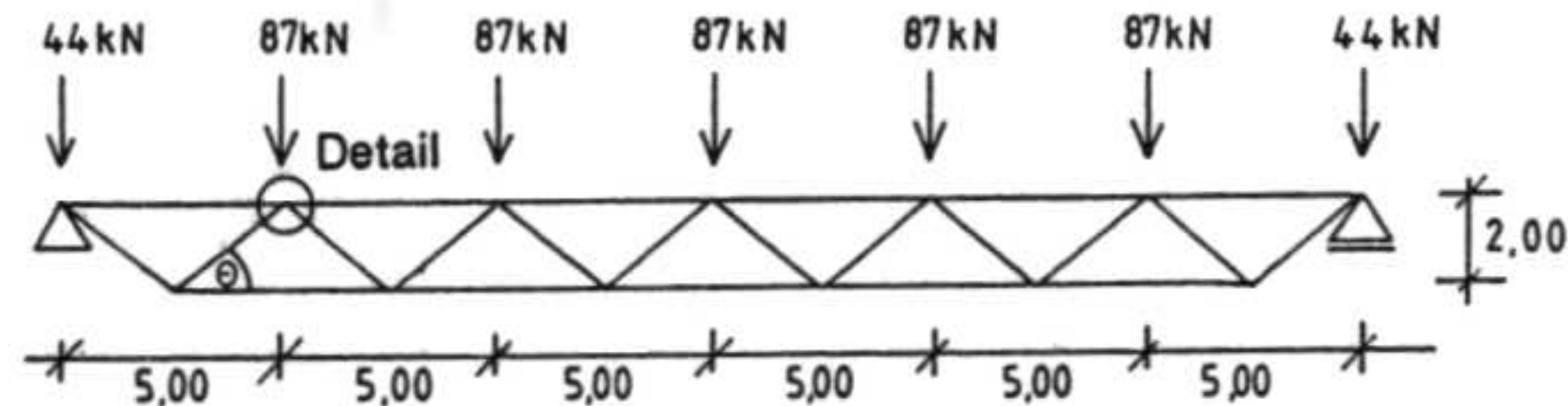
Problem, Sketch, Calculation

Reference

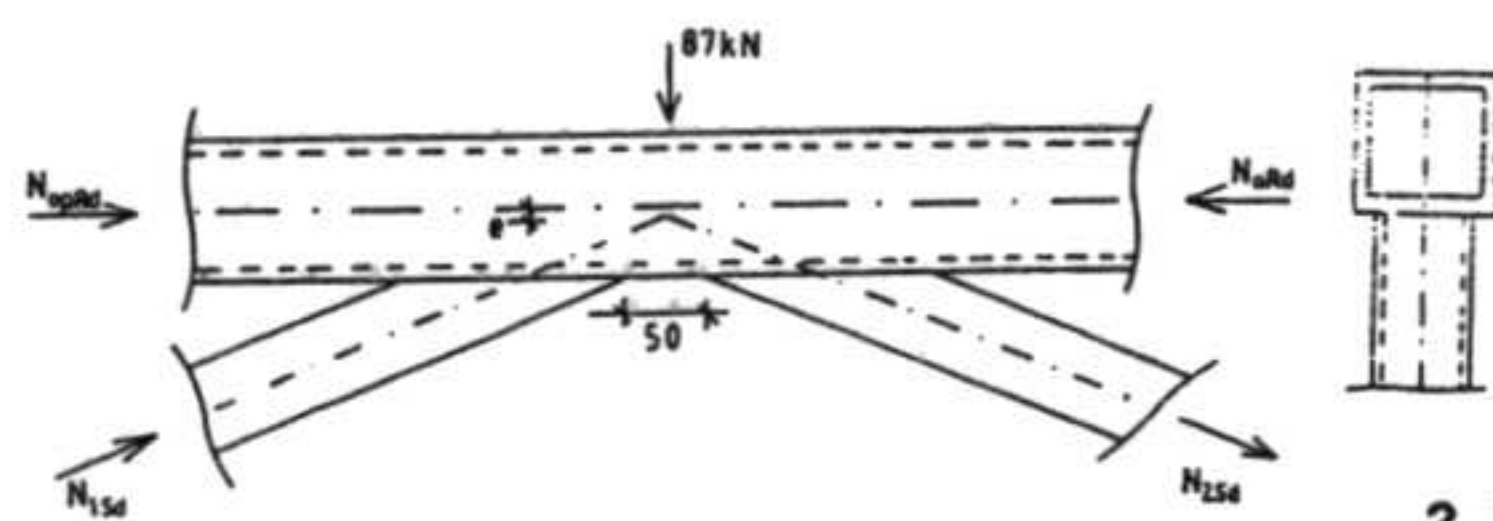
Commentary

Example 3.2.5: Hollow section lattice girder joint

1. System



Detail:



Chord member: RHS 160 x 7.1; hot finished Steel grade Fe 510
 Bracing member: RHS 90 x 5.6; hot finished Steel grade Fe 430
 chosen: gap $g = 50$ mm

2. Loading

$N_{e.Sd} = 880$ kN (Compression force)
 $N_{op.Sd} = 365$ kN (Compression force)
 $N_{1.Sd} = 400$ kN (Compression force)
 $N_{2.Sd} = 260$ kN (Tension force)

3. Determination of the resistance

3.1 Range of validity

$$\beta = b_i/b_o = 90/160 = 0,56 > 0,35 \\ > 0,1 + 0,01 \cdot 160/7.1 = 0,33$$

$$b_i/t_i = 90/5,6 = 16 < 1,25\sqrt{(E/f_{yi})} = 1,25\sqrt{(210000/275)} = 34,5$$

$$15 \leq b_o/t_o = 160/7,1 = 22,5 < 35$$

$$0,6 \leq \frac{b_1 + b_2}{2 b_1} = 1,0 \leq 1,3$$

$$0,5(1 - \beta) = 0,5(1 - 0,56) = 0,22 \leq g/b_o = 50/160 = 0,3125 \leq 1,5(1 - \beta) = 1,5(1 - 0,56) = 0,66 \\ g = 50 \text{ mm} > t_1 + t_2 = 2 \cdot 5,6 = 11,2 \text{ mm}$$

$$\theta_i = 38,7^\circ > 30^\circ$$

[Table K.7.1]

For joints outside the range of validity a more detailed analysis should be made as specified in [6.10.1].

Problem, Sketch, Calculation	Reference	Commentary
3.2 Joint resistance Verification of the eccentricity: $e = \left(\frac{h_1}{2 \sin \theta_1} + \frac{h_2}{2 \sin \theta_2} + g \right) \frac{\sin \theta_1 \sin \theta_2}{\sin(\theta_1 + \theta_2)} - \frac{h_o}{2}$ $= \left(\frac{90}{\sin 38,7} + 50 \right) \frac{\sin^2 38,7}{\sin(38,7 + 38,7)} - \frac{160}{2} = -2,3 \text{ mm}$ $-2,3/160 = -0,01 > -0,55 \rightarrow \text{satisfied}$ Determination of the efficiency: $\eta = \frac{N_{o,Ed} \gamma_{Mo}}{A f_{yo}} = \frac{880 \cdot 1,1 \cdot 10^3}{41,7 \cdot 10^2 \cdot 355} = 0,65$ $k_n = 1,3 - \frac{0,4 \eta}{\beta} = 1,3 - \frac{0,4 \cdot 0,65}{0,56} = 0,84 < 1,0$ $N_{LRd} = 8,9 \frac{f_{yo} t_o^2}{\sin \theta_i} \beta \gamma k_n \frac{1,1}{\gamma_{Mj}}$ $= 8,9 \frac{35,5 \cdot 0,71^2}{\sin 38,66} \frac{90}{160} \sqrt{\frac{16}{2 \cdot 0,71}} 0,84 = 404,4 \text{ kN}$ $N_{i,Ed} < 465,8 \text{ kN} = N_{pl,Rd} = A f_{yi} / \gamma_{Mo}$	- [K.3.(1)] - [K.7.2; Table K.7.2]	
3.3 Weld design $a \geq 0,87 \frac{1,1}{\gamma_{Mj}} \frac{\gamma_{Mw}}{1,25} t_i$ $= 0,87 \cdot \frac{1,1}{1,1} \frac{1,25}{1,25} \cdot 5,6 = 4,9 \text{ mm}$ chosen: a = 5 mm	[K.5.(4)]	If the criterion in [K.5.(4)] is fulfilled the resistance of the weld is equal or higher than the member resistance.

3.3 Pin connections

This example demonstrates the verification of pin connections assuming design values of action effects (N_{sd} , V_{sd} , M_{sd} , etc) which have been calculated by an analysis of the structure and these values already include ψ , γ_F etc.

The results presented in the examples are rounded values. For the purpose of easy re-calculation each formula and each check is calculated with the rounded values.

Example 3.3.1: Pin connection

Problem, Sketch, Calculation	Reference	Commentary
Example 3.3.1: Pin connection 1. System Steel grade: Fe 430 pin: M 36, 8.8 $\rightarrow d_o = 39 \text{ mm}$ $d = 36 \text{ mm}$, $\Delta = 1 \text{ mm}$ $t_1 = 12 \text{ mm}$, $t = 20 \text{ mm}$ 2. Loading $N_{Sd} = 230,0 \text{ kN}$ (compression force) $M_{Sd} = N_{Sd} \cdot (20 + 4 \cdot 1 + 2 \cdot 12) \cdot 10^{-3} / 8 = 1,38 \text{ kNm}$ (bending moment in the bolt) 3. Determination of the pin resistance 3.1 Geometrical conditions for plate by plate thickness $t = 20 \text{ mm}$: $a > \frac{N_{Sd} \gamma_{Mp}}{2 t f_y} + \frac{2 d_o}{3} = \frac{230 \cdot 1,25 \cdot 10^3}{2 \cdot 20 \cdot 275} + \frac{2 \cdot 39}{3} = 52 \text{ mm}$ chosen: $a = 60 \text{ mm}$ $c > \frac{N_{Sd} \gamma_{Mp}}{2 t f_y} + \frac{d_o}{3} = \frac{230 \cdot 1,25 \cdot 10^3}{2 \cdot 20 \cdot 275} + \frac{39}{3} = 39 \text{ mm}$ chosen: $c = 40 \text{ mm}$ 3.2 Shear resistance of the pin $F_{v,Rd} = 0,6 A \frac{f_{up}}{\gamma_{Mp}} = 0,6 \frac{\pi \cdot 36^2}{4} \frac{800}{1,25} \cdot 10^{-3} = 390,9 \text{ kN} > 230,0 \text{ kN}$ 3.3 Bearing resistance of plate and pin $F_{b,Rd} = 1,5 t d \frac{f_y}{\gamma_{Mp}} = 1,5 \cdot 20 \cdot 36 \frac{275}{1,25} \cdot 10^{-3} = 237,6 \text{ kN} > 230,0 \text{ kN}$ 3.4 Bending resistance of pin $M_{Rd} = 0,8 W_{el} \frac{f_{yp}}{\gamma_{Mp}} = 0,8 \cdot \frac{\pi \cdot 36^3}{32} \frac{640}{1,25} \cdot 10^{-6} = 1,88 \text{ kNm} > 1,38 \text{ kNm}$ 3.5 Interaction shear and bending $\left(\frac{M_{Sd}}{M_{Rd}} \right)^2 + \left(\frac{F_{v,Sd}}{F_{v,Rd}} \right)^2 = \left(\frac{1,38}{1,88} \right)^2 + \left(\frac{230,0}{390,9} \right)^2 = 0,89 < 1,0$	- [Figure 6.5.12] - [Table 6.5.6] - [Table 6.5.7] - [Table 6.5.7] - [Table 6.5.7] - [Table 6.5.7]	Such a connection is used if free rotation is required. The bending moment in the bolt is calculated by $M_{Sd} = N_{Sd} \cdot (t + 4\Delta + 2t_1) / 8$ Pin connections should be arranged to avoid eccentricity $W_{el} = \pi d^3 / 32$

Annex A

Values of the reduction factor χ for
the appropriate non-dimensional slenderness $\bar{\lambda}$
of buckling curves a₀, a, b, c, d
are given in the following tables

Reduction factors χ for buckling curve a ₀ ($\alpha = 0,13$)											
$\bar{\lambda}$	0,00	0,01	0,02	0,03	0,04	0,05	0,06	0,07	0,08	0,09	
0,00	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	0,00
0,10	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	0,10
0,20	1,0000	0,9986	0,9973	0,9959	0,9945	0,9931	0,9917	0,9903	0,9889	0,9874	0,20
0,30	0,9859	0,9845	0,9829	0,9814	0,9799	0,9783	0,9767	0,9751	0,9735	0,9718	0,30
0,40	0,9701	0,9684	0,9667	0,9649	0,9631	0,9612	0,9593	0,9574	0,9554	0,9534	0,40
0,50	0,9513	0,9492	0,9470	0,9448	0,9425	0,9402	0,9378	0,9354	0,9328	0,9302	0,50
0,60	0,9276	0,9248	0,9220	0,9191	0,9161	0,9130	0,9099	0,9066	0,9032	0,8997	0,60
0,70	0,8961	0,8924	0,8886	0,8847	0,8806	0,8764	0,8721	0,8676	0,8630	0,8582	0,70
0,80	0,8533	0,8483	0,8431	0,8377	0,8322	0,8266	0,8208	0,8148	0,8087	0,8025	0,80
0,90	0,7961	0,7895	0,7828	0,7760	0,7691	0,7620	0,7549	0,7476	0,7403	0,7329	0,90
1,00	0,7253	0,7178	0,7101	0,7025	0,6948	0,6870	0,6793	0,6715	0,6637	0,6560	1,00
1,10	0,6482	0,6405	0,6329	0,6252	0,6176	0,6101	0,6026	0,5951	0,5877	0,5804	1,10
1,20	0,5732	0,5660	0,5590	0,5520	0,5450	0,5382	0,5314	0,5248	0,5182	0,5117	1,20
1,30	0,5053	0,4990	0,4927	0,4866	0,4806	0,4746	0,4687	0,4629	0,4572	0,4516	1,30
1,40	0,4461	0,4407	0,4353	0,4300	0,4248	0,4197	0,4147	0,4097	0,4049	0,4001	1,40
1,50	0,3953	0,3907	0,3861	0,3816	0,3772	0,3728	0,3685	0,3643	0,3601	0,3560	1,50
1,60	0,3520	0,3480	0,3441	0,3403	0,3365	0,3328	0,3291	0,3255	0,3219	0,3184	1,60
1,70	0,3150	0,3116	0,3083	0,3050	0,3017	0,2985	0,2954	0,2923	0,2892	0,2862	1,70
1,80	0,2833	0,2804	0,2775	0,2746	0,2719	0,2691	0,2664	0,2637	0,2611	0,2585	1,80
1,90	0,2559	0,2534	0,2509	0,2485	0,2461	0,2437	0,2414	0,2390	0,2368	0,2345	1,90
2,00	0,2323	0,2301	0,2280	0,2258	0,2237	0,2217	0,2196	0,2176	0,2156	0,2136	2,00
2,10	0,2117	0,2098	0,2079	0,2061	0,2042	0,2024	0,2006	0,1989	0,1971	0,1954	2,10
2,20	0,1937	0,1920	0,1904	0,1887	0,1871	0,1855	0,1840	0,1824	0,1809	0,1794	2,20
2,30	0,1779	0,1764	0,1749	0,1735	0,1721	0,1707	0,1693	0,1679	0,1665	0,1652	2,30
2,40	0,1639	0,1626	0,1613	0,1600	0,1587	0,1575	0,1563	0,1550	0,1538	0,1526	2,40
2,50	0,1515	0,1503	0,1491	0,1480	0,1469	0,1458	0,1447	0,1436	0,1425	0,1414	2,50
2,60	0,1404	0,1394	0,1383	0,1373	0,1363	0,1353	0,1343	0,1333	0,1324	0,1314	2,60
2,70	0,1305	0,1296	0,1286	0,1277	0,1268	0,1259	0,1250	0,1242	0,1233	0,1224	2,70
2,80	0,1216	0,1207	0,1199	0,1191	0,1183	0,1175	0,1167	0,1159	0,1151	0,1143	2,80
2,90	0,1136	0,1128	0,1120	0,1113	0,1106	0,1098	0,1091	0,1084	0,1077	0,1070	2,90
3,00	0,1063	0,1056	0,1049	0,1043	0,1036	0,1029	0,1023	0,1016	0,1010	0,1003	3,00
3,10	0,0997	0,0991	0,0985	0,0979	0,0972	0,0966	0,0960	0,0955	0,0949	0,0943	3,10
3,20	0,0937	0,0931	0,0926	0,0920	0,0915	0,0909	0,0904	0,0898	0,0893	0,0888	3,20
3,30	0,0882	0,0877	0,0872	0,0867	0,0862	0,0857	0,0852	0,0847	0,0842	0,0837	3,30
3,40	0,0832	0,0828	0,0823	0,0818	0,0814	0,0809	0,0804	0,0800	0,0795	0,0791	3,40
3,50	0,0786	0,0782	0,0778	0,0773	0,0769	0,0765	0,0761	0,0756	0,0752	0,0748	3,50
3,60	0,0744	0,0740	0,0736	0,0732	0,0728	0,0724	0,0720	0,0717	0,0713	0,0709	3,60

Reduction factors χ for buckling curve a ($\alpha = 0,21$)											
$\bar{\lambda}$	0,00	0,01	0,02	0,03	0,04	0,05	0,06	0,07	0,08	0,09	
0,00	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	0,00
0,10	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	0,10
0,20	1,0000	0,9978	0,9956	0,9934	0,9912	0,9889	0,9867	0,9844	0,9821	0,9798	0,20
0,30	0,9775	0,9751	0,9728	0,9704	0,9680	0,9655	0,9630	0,9605	0,9580	0,9554	0,30
0,40	0,9528	0,9501	0,9474	0,9447	0,9419	0,9391	0,9363	0,9333	0,9304	0,9273	0,40
0,50	0,9243	0,9211	0,9179	0,9147	0,9114	0,9080	0,9045	0,9010	0,8974	0,8937	0,50
0,60	0,8900	0,8862	0,8823	0,8783	0,8742	0,8700	0,8657	0,8614	0,8569	0,8524	0,60
0,70	0,8477	0,8430	0,8382	0,8332	0,8282	0,8230	0,8178	0,8124	0,8069	0,8014	0,70
0,80	0,7957	0,7899	0,7841	0,7781	0,7721	0,7659	0,7597	0,7534	0,7470	0,7405	0,80
0,90	0,7339	0,7273	0,7206	0,7139	0,7071	0,7003	0,6934	0,6865	0,6796	0,6726	0,90
1,00	0,6656	0,6586	0,6516	0,6446	0,6376	0,6306	0,6236	0,6167	0,6098	0,6029	1,00
1,10	0,5960	0,5892	0,5824	0,5757	0,5690	0,5623	0,5557	0,5492	0,5427	0,5363	1,10
1,20	0,5300	0,5237	0,5175	0,5114	0,5053	0,4993	0,4934	0,4875	0,4817	0,4760	1,20
1,30	0,4703	0,4648	0,4593	0,4538	0,4485	0,4432	0,4380	0,4329	0,4278	0,4228	1,30
1,40	0,4179	0,4130	0,4083	0,4036	0,3989	0,3943	0,3898	0,3854	0,3810	0,3767	1,40
1,50	0,3724	0,3682	0,3641	0,3601	0,3561	0,3521	0,3482	0,3444	0,3406	0,3369	1,50
1,60	0,3332	0,3296	0,3261	0,3226	0,3191	0,3157	0,3124	0,3091	0,3058	0,3026	1,60
1,70	0,2994	0,2963	0,2933	0,2902	0,2872	0,2843	0,2814	0,2786	0,2757	0,2730	1,70
1,80	0,2702	0,2675	0,2649	0,2623	0,2597	0,2571	0,2546	0,2522	0,2497	0,2473	1,80
1,90	0,2449	0,2426	0,2403	0,2380	0,2358	0,2335	0,2314	0,2292	0,2271	0,2250	1,90
2,00	0,2229	0,2209	0,2188	0,2168	0,2149	0,2129	0,2110	0,2091	0,2073	0,2054	2,00
2,10	0,2036	0,2018	0,2001	0,1983	0,1966	0,1949	0,1932	0,1915	0,1899	0,1883	2,10
2,20	0,1867	0,1851	0,1836	0,1820	0,1805	0,1790	0,1775	0,1760	0,1746	0,1732	2,20
2,30	0,1717	0,1704	0,1690	0,1676	0,1663	0,1649	0,1636	0,1623	0,1610	0,1598	2,30
2,40	0,1585	0,1573	0,1560	0,1548	0,1536	0,1524	0,1513	0,1501	0,1490	0,1478	2,40
2,50	0,1467	0,1456	0,1445	0,1434	0,1424	0,1413	0,1403	0,1392	0,1382	0,1372	2,50
2,60	0,1362	0,1352	0,1342	0,1332	0,1323	0,1313	0,1304	0,1295	0,1285	0,1276	2,60
2,70	0,1267	0,1258	0,1250	0,1241	0,1232	0,1224	0,1215	0,1207	0,1198	0,1190	2,70
2,80	0,1182	0,1174	0,1166	0,1158	0,1150	0,1143	0,1135	0,1128	0,1120	0,1113	2,80
2,90	0,1105	0,1098	0,1091	0,1084	0,1077	0,1070	0,1063	0,1056	0,1049	0,1042	2,90
3,00	0,1036	0,1029	0,1022	0,1016	0,1010	0,1003	0,0997	0,0991	0,0985	0,0978	3,00
3,10	0,0972	0,0966	0,0960	0,0954	0,0949	0,0943	0,0937	0,0931	0,0926	0,0920	3,10
3,20	0,0915	0,0909	0,0904	0,0898	0,0893	0,0888	0,0882	0,0877	0,0872	0,0867	3,20
3,30	0,0862	0,0857	0,0852	0,0847	0,0842	0,0837	0,0832	0,0828	0,0823	0,0818	3,30
3,40	0,0814	0,0809	0,0804	0,0800	0,0795	0,0791	0,0786	0,0782	0,0778	0,0773	3,40
3,50	0,0769	0,0765	0,0761	0,0757	0,0752	0,0748	0,0744	0,0740	0,0736	0,0732	3,50
3,60	0,0728	0,0724	0,0721	0,0717	0,0713	0,0709	0,0705	0,0702	0,0698	0,0694	3,60

Reduction factors χ for buckling curve b ($\alpha = 0,34$)											
$\bar{\lambda}$	0,00	0,01	0,02	0,03	0,04	0,05	0,06	0,07	0,08	0,09	
0,00	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	0,00
0,10	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	0,10
0,20	1,0000	0,9965	0,9929	0,9894	0,9858	0,9822	0,9786	0,9750	0,9714	0,9678	0,20
0,30	0,9641	0,9604	0,9567	0,9530	0,9492	0,9455	0,9417	0,9378	0,9339	0,9300	0,30
0,40	0,9261	0,9221	0,9181	0,9140	0,9099	0,9057	0,9015	0,8973	0,8930	0,8886	0,40
0,50	0,8842	0,8798	0,8752	0,8707	0,8661	0,8614	0,8566	0,8518	0,8470	0,8420	0,50
0,60	0,8371	0,8320	0,8269	0,8217	0,8165	0,8112	0,8058	0,8004	0,7949	0,7893	0,60
0,70	0,7837	0,7780	0,7723	0,7665	0,7606	0,7547	0,7488	0,7428	0,7367	0,7306	0,70
0,80	0,7245	0,7183	0,7120	0,7058	0,6995	0,6931	0,6868	0,6804	0,6740	0,6676	0,80
0,90	0,6612	0,6547	0,6483	0,6419	0,6354	0,6290	0,6226	0,6162	0,6098	0,6034	0,90
1,00	0,5970	0,5907	0,5844	0,5781	0,5719	0,5657	0,5595	0,5534	0,5473	0,5412	1,00
1,10	0,5352	0,5293	0,5234	0,5175	0,5117	0,5060	0,5003	0,4947	0,4891	0,4836	1,10
1,20	0,4781	0,4727	0,4674	0,4621	0,4569	0,4517	0,4466	0,4416	0,4366	0,4317	1,20
1,30	0,4269	0,4221	0,4174	0,4127	0,4081	0,4035	0,3991	0,3946	0,3903	0,3860	1,30
1,40	0,3817	0,3775	0,3734	0,3693	0,3653	0,3613	0,3574	0,3535	0,3497	0,3459	1,40
1,50	0,3422	0,3386	0,3350	0,3314	0,3279	0,3245	0,3211	0,3177	0,3144	0,3111	1,50
1,60	0,3079	0,3047	0,3016	0,2985	0,2955	0,2925	0,2895	0,2866	0,2837	0,2809	1,60
1,70	0,2781	0,2753	0,2726	0,2699	0,2672	0,2646	0,2620	0,2595	0,2570	0,2545	1,70
1,80	0,2521	0,2496	0,2473	0,2449	0,2426	0,2403	0,2381	0,2359	0,2337	0,2315	1,80
1,90	0,2294	0,2272	0,2252	0,2231	0,2211	0,2191	0,2171	0,2152	0,2132	0,2113	1,90
2,00	0,2095	0,2076	0,2058	0,2040	0,2022	0,2004	0,1987	0,1970	0,1953	0,1936	2,00
2,10	0,1920	0,1903	0,1887	0,1871	0,1855	0,1840	0,1825	0,1809	0,1794	0,1780	2,10
2,20	0,1765	0,1751	0,1736	0,1722	0,1708	0,1694	0,1681	0,1667	0,1654	0,1641	2,20
2,30	0,1628	0,1615	0,1602	0,1590	0,1577	0,1565	0,1553	0,1541	0,1529	0,1517	2,30
2,40	0,1506	0,1494	0,1483	0,1472	0,1461	0,1450	0,1439	0,1428	0,1418	0,1407	2,40
2,50	0,1397	0,1387	0,1376	0,1366	0,1356	0,1347	0,1337	0,1327	0,1318	0,1308	2,50
2,60	0,1299	0,1290	0,1281	0,1272	0,1263	0,1254	0,1245	0,1237	0,1228	0,1219	2,60
2,70	0,1211	0,1203	0,1195	0,1186	0,1178	0,1170	0,1162	0,1155	0,1147	0,1139	2,70
2,80	0,1132	0,1124	0,1117	0,1109	0,1102	0,1095	0,1088	0,1081	0,1074	0,1067	2,80
2,90	0,1060	0,1053	0,1046	0,1039	0,1033	0,1026	0,1020	0,1013	0,1007	0,1001	2,90
3,00	0,0994	0,0988	0,0982	0,0976	0,0970	0,0964	0,0958	0,0952	0,0946	0,0940	3,00
3,10	0,0935	0,0929	0,0924	0,0918	0,0912	0,0907	0,0902	0,0896	0,0891	0,0886	3,10
3,20	0,0880	0,0875	0,0870	0,0865	0,0860	0,0855	0,0850	0,0845	0,0840	0,0835	3,20
3,30	0,0831	0,0826	0,0821	0,0816	0,0812	0,0807	0,0803	0,0798	0,0794	0,0789	3,30
3,40	0,0785	0,0781	0,0776	0,0772	0,0768	0,0763	0,0759	0,0755	0,0751	0,0747	3,40
3,50	0,0743	0,0739	0,0735	0,0731	0,0727	0,0723	0,0719	0,0715	0,0712	0,0708	3,50
3,60	0,0704	0,0700	0,0697	0,0693	0,0689	0,0686	0,0682	0,0679	0,0675	0,0672	3,60

Reduction factors χ for buckling curve c ($\alpha = 0,49$)											
$\bar{\lambda}$	0,00	0,01	0,02	0,03	0,04	0,05	0,06	0,07	0,08	0,09	
0,00	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	0,00
0,10	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	0,10
0,20	1,0000	0,9949	0,9898	0,9847	0,9797	0,9746	0,9695	0,9644	0,9593	0,9542	0,20
0,30	0,9491	0,9440	0,9389	0,9338	0,9286	0,9235	0,9183	0,9131	0,9078	0,9026	0,30
0,40	0,8973	0,8920	0,8867	0,8813	0,8760	0,8705	0,8651	0,8596	0,8541	0,8486	0,40
0,50	0,8430	0,8374	0,8317	0,8261	0,8204	0,8146	0,8088	0,8030	0,7972	0,7913	0,50
0,60	0,7854	0,7794	0,7735	0,7675	0,7614	0,7554	0,7493	0,7432	0,7370	0,7309	0,60
0,70	0,7247	0,7185	0,7123	0,7060	0,6998	0,6935	0,6873	0,6810	0,6747	0,6684	0,70
0,80	0,6622	0,6559	0,6496	0,6433	0,6371	0,6308	0,6246	0,6184	0,6122	0,6060	0,80
0,90	0,5998	0,5937	0,5876	0,5815	0,5755	0,5695	0,5635	0,5575	0,5516	0,5458	0,90
1,00	0,5399	0,5342	0,5284	0,5227	0,5171	0,5115	0,5059	0,5004	0,4950	0,4896	1,00
1,10	0,4842	0,4790	0,4737	0,4685	0,4634	0,4583	0,4533	0,4483	0,4434	0,4386	1,10
1,20	0,4338	0,4290	0,4243	0,4197	0,4151	0,4106	0,4061	0,4017	0,3974	0,3931	1,20
1,30	0,3888	0,3846	0,3805	0,3764	0,3724	0,3684	0,3644	0,3606	0,3567	0,3529	1,30
1,40	0,3492	0,3455	0,3419	0,3383	0,3348	0,3313	0,3279	0,3245	0,3211	0,3178	1,40
1,50	0,3145	0,3113	0,3081	0,3050	0,3019	0,2989	0,2959	0,2929	0,2900	0,2871	1,50
1,60	0,2842	0,2814	0,2786	0,2759	0,2732	0,2705	0,2679	0,2653	0,2627	0,2602	1,60
1,70	0,2577	0,2553	0,2528	0,2504	0,2481	0,2457	0,2434	0,2412	0,2389	0,2367	1,70
1,80	0,2345	0,2324	0,2302	0,2281	0,2260	0,2240	0,2220	0,2200	0,2180	0,2161	1,80
1,90	0,2141	0,2122	0,2104	0,2085	0,2067	0,2049	0,2031	0,2013	0,1996	0,1979	1,90
2,00	0,1962	0,1945	0,1929	0,1912	0,1896	0,1880	0,1864	0,1849	0,1833	0,1818	2,00
2,10	0,1803	0,1788	0,1774	0,1759	0,1745	0,1731	0,1717	0,1703	0,1689	0,1676	2,10
2,20	0,1662	0,1649	0,1636	0,1623	0,1611	0,1598	0,1585	0,1573	0,1561	0,1549	2,20
2,30	0,1537	0,1525	0,1514	0,1502	0,1491	0,1480	0,1468	0,1457	0,1446	0,1436	2,30
2,40	0,1425	0,1415	0,1404	0,1394	0,1384	0,1374	0,1364	0,1354	0,1344	0,1334	2,40
2,50	0,1325	0,1315	0,1306	0,1297	0,1287	0,1278	0,1269	0,1260	0,1252	0,1243	2,50
2,60	0,1234	0,1226	0,1217	0,1209	0,1201	0,1193	0,1184	0,1176	0,1168	0,1161	2,60
2,70	0,1153	0,1145	0,1137	0,1130	0,1122	0,1115	0,1108	0,1100	0,1093	0,1086	2,70
2,80	0,1079	0,1072	0,1065	0,1058	0,1051	0,1045	0,1038	0,1031	0,1025	0,1018	2,80
2,90	0,1012	0,1006	0,0999	0,0993	0,0987	0,0981	0,0975	0,0969	0,0963	0,0957	2,90
3,00	0,0951	0,0945	0,0939	0,0934	0,0928	0,0922	0,0917	0,0911	0,0906	0,0901	3,00
3,10	0,0895	0,0890	0,0885	0,0879	0,0874	0,0869	0,0864	0,0859	0,0854	0,0849	3,10
3,20	0,0844	0,0839	0,0835	0,0830	0,0825	0,0820	0,0816	0,0811	0,0806	0,0802	3,20
3,30	0,0797	0,0793	0,0789	0,0784	0,0780	0,0775	0,0771	0,0767	0,0763	0,0759	3,30
3,40	0,0754	0,0750	0,0746	0,0742	0,0738	0,0734	0,0730	0,0726	0,0722	0,0719	3,40
3,50	0,0715	0,0711	0,0707	0,0703	0,0700	0,0696	0,0692	0,0689	0,0685	0,0682	3,50
3,60	0,0678	0,0675	0,0671	0,0668	0,0664	0,0661	0,0657	0,0654	0,0651	0,0647	3,60

Reduction factors χ for buckling curve d ($\alpha = 0,76$)											
$\bar{\lambda}$	0,00	0,01	0,02	0,03	0,04	0,05	0,06	0,07	0,08	0,09	
0,00	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	0,00
0,10	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	1,0000	0,10
0,20	1,0000	0,9921	0,9843	0,9765	0,9688	0,9611	0,9535	0,9459	0,9384	0,9309	0,20
0,30	0,9235	0,9160	0,9086	0,9013	0,8939	0,8866	0,8793	0,8721	0,8648	0,8576	0,30
0,40	0,8504	0,8432	0,8360	0,8289	0,8218	0,8146	0,8075	0,8005	0,7934	0,7864	0,40
0,50	0,7793	0,7723	0,7653	0,7583	0,7514	0,7444	0,7375	0,7306	0,7237	0,7169	0,50
0,60	0,7100	0,7032	0,6964	0,6897	0,6829	0,6762	0,6695	0,6629	0,6563	0,6497	0,60
0,70	0,6431	0,6366	0,6301	0,6237	0,6173	0,6109	0,6046	0,5983	0,5921	0,5859	0,70
0,80	0,5797	0,5736	0,5675	0,5615	0,5556	0,5496	0,5438	0,5379	0,5322	0,5265	0,80
0,90	0,5208	0,5152	0,5096	0,5041	0,4987	0,4933	0,4879	0,4826	0,4774	0,4722	0,90
1,00	0,4671	0,4620	0,4570	0,4521	0,4472	0,4423	0,4375	0,4328	0,4281	0,4235	1,00
1,10	0,4189	0,4144	0,4099	0,4055	0,4012	0,3969	0,3926	0,3884	0,3843	0,3802	1,10
1,20	0,3762	0,3722	0,3683	0,3644	0,3605	0,3568	0,3530	0,3493	0,3457	0,3421	1,20
1,30	0,3385	0,3350	0,3316	0,3282	0,3248	0,3215	0,3182	0,3150	0,3118	0,3086	1,30
1,40	0,3055	0,3024	0,2994	0,2964	0,2935	0,2906	0,2877	0,2849	0,2821	0,2793	1,40
1,50	0,2766	0,2739	0,2712	0,2686	0,2660	0,2635	0,2609	0,2585	0,2560	0,2536	1,50
1,60	0,2512	0,2488	0,2465	0,2442	0,2419	0,2397	0,2375	0,2353	0,2331	0,2310	1,60
1,70	0,2289	0,2268	0,2248	0,2228	0,2208	0,2188	0,2168	0,2149	0,2130	0,2112	1,70
1,80	0,2093	0,2075	0,2057	0,2039	0,2021	0,2004	0,1987	0,1970	0,1953	0,1936	1,80
1,90	0,1920	0,1904	0,1888	0,1872	0,1856	0,1841	0,1826	0,1810	0,1796	0,1781	1,90
2,00	0,1766	0,1752	0,1738	0,1724	0,1710	0,1696	0,1683	0,1669	0,1656	0,1643	2,00
2,10	0,1630	0,1617	0,1604	0,1592	0,1580	0,1567	0,1555	0,1543	0,1532	0,1520	2,10
2,20	0,1508	0,1497	0,1486	0,1474	0,1463	0,1452	0,1442	0,1431	0,1420	0,1410	2,20
2,30	0,1399	0,1389	0,1379	0,1369	0,1359	0,1349	0,1340	0,1330	0,1320	0,1311	2,30
2,40	0,1302	0,1292	0,1283	0,1274	0,1265	0,1257	0,1248	0,1239	0,1231	0,1222	2,40
2,50	0,1214	0,1205	0,1197	0,1189	0,1181	0,1173	0,1165	0,1157	0,1149	0,1142	2,50
2,60	0,1134	0,1127	0,1119	0,1112	0,1104	0,1097	0,1090	0,1083	0,1076	0,1069	2,60
2,70	0,1062	0,1055	0,1048	0,1042	0,1035	0,1029	0,1022	0,1016	0,1009	0,1003	2,70
2,80	0,0997	0,0990	0,0984	0,0978	0,0972	0,0966	0,0960	0,0954	0,0948	0,0943	2,80
2,90	0,0937	0,0931	0,0926	0,0920	0,0914	0,0909	0,0904	0,0898	0,0893	0,0888	2,90
3,00	0,0882	0,0877	0,0872	0,0867	0,0862	0,0857	0,0852	0,0847	0,0842	0,0837	3,00
3,10	0,0832	0,0828	0,0823	0,0818	0,0814	0,0809	0,0804	0,0800	0,0795	0,0791	3,10
3,20	0,0786	0,0782	0,0778	0,0773	0,0769	0,0765	0,0761	0,0757	0,0752	0,0748	3,20
3,30	0,0744	0,0740	0,0736	0,0732	0,0728	0,0724	0,0721	0,0717	0,0713	0,0709	3,30
3,40	0,0705	0,0702	0,0698	0,0694	0,0691	0,0687	0,0683	0,0680	0,0676	0,0673	3,40
3,50	0,0669	0,0666	0,0663	0,0659	0,0656	0,0652	0,0649	0,0646	0,0643	0,0639	3,50
3,60	0,0636	0,0633	0,0630	0,0627	0,0624	0,0620	0,0617	0,0614	0,0611	0,0608	3,60

